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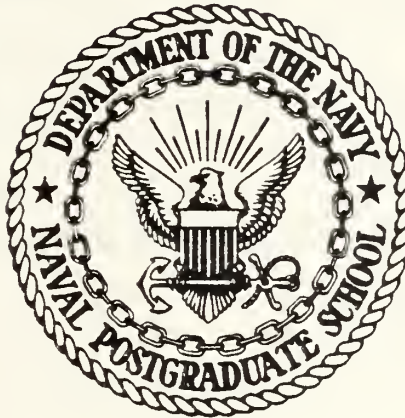






# NAVAL POSTGRADUATE SCHOOL

## Monterey, California



# THESIS

PRELIMINARY PROPELLER SELECTION USING THE  
WAGENINGEN B-SCREW SERIES AND A  
GENERAL PURPOSE NON-LINEAR OPTIMIZER

by

Michael Peter Smith II

June 1983

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This thesis presents the use of a general purpose non-linear optimization program in the preliminary stage of ship design for the selection of a propeller based on methodical series propeller test data. The propeller series utilized is the well-known Wageningen B-Series. Three (3) "Design Cases", representing the thrust, power and matching approaches to powering problems, are formulated as FORTTRAN subprogram analysis codes for solution by



## (20. ABSTRACT Continued)

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- 1) diameter limitation
- 2) cavitation limit on expanded area ratio using Keller's criterion
- 3) strength requirement determined by an empirical relation and by a method developed by Schoenherr with modifications by the author.

Objective functions considered are maximized open water efficiency and minimized propeller blade weight. Optimized solutions to specific problems previously presented by other authors are obtained and results are compared.



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Preliminary Propeller Selection Using the  
Wageningen B-Screw Series and a  
General Purpose Non-Linear Optimizer

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requirements for the degree of

MASTER OF SCIENCE IN MECHANICAL ENGINEERING

from the

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June 1983





### ABSTRACT

This thesis presents the use of a general purpose non-linear optimization program in the preliminary stage of ship design for the selection of a propeller based on methodical series propeller test data. The propeller series utilized is the well-known Wageningen B-Series. Three (3) "Design Cases", representing the thrust, power and matching approaches to powering problems, are formulated as FORTRAN subprogram analysis codes for solution by the synthesis/optimization program COPES/CONMIN. Designer constraints considered are:

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## TABLE OF CONTENTS

I.	INTRODUCTION -----	14
A.	BACKGROUND -----	14
B.	PROBLEM STATEMENT -----	19
C.	SCOPE -----	19
D.	THESIS ORGANIZATION -----	19
II.	OPTIMIZATION -----	22
A.	INTRODUCTION -----	22
B.	DEFINITIONS -----	22
C.	PROBLEM STATEMENT -----	25
D.	COPES/CONMIN -----	28
	1. CONMIN -----	28
	2. COPES -----	30
E.	CONCLUDING NOTE -----	32
III.	POWERING, PERFORMANCE AND PROPELLERS -----	33
A.	INTRODUCTION -----	33
B.	DEFINITIONS -----	33
C.	POWERING CONCEPTS -----	37
	1. Basic Relations -----	37
	2. Approaches to the Powering Problem -----	38
D.	PROPELLER PERFORMANCE CHARACTERISTICS -----	40
E.	THE WAGENINGEN B-SCREW SERIES -----	44
	1. Background -----	44
	2. Series Results -----	45
	3. Limitations on Series Data -----	48
F.	SUMMARY -----	50



IV.	PROPELLER SELECTION--AN OPTIMIZATION PROBLEM ----	52
A.	INTRODUCTION -----	52
B.	DESIGNER'S CONSIDERATIONS -----	52
1.	Propeller Size -----	52
2.	Cavitation -----	53
3.	Strength -----	54
C.	THE DESIGN VECTOR -----	56
1.	Parameters -----	57
2.	Design Variables -----	58
D.	CONSTRAINTS -----	60
E.	OBJECTIVE FUNCTIONS -----	61
F.	PROPELLER SELECTION OPTIMIZATION PROBLEM STATEMENT -----	62
G.	CODING FUNDAMENTALS -----	62
1.	GLOBCM Common Block -----	62
2.	SUBROUTINE ANALIZ -----	62
H.	SUMMARY -----	65
V.	PROPELLER BLADE WEIGHT--AN OBJECTIVE FUNCTION ---	67
A.	INTRODUCTION -----	67
B.	THEORY AND PROCEDURE -----	67
1.	Limits of Integration -----	67
2.	Blade Section Profile -----	69
3.	Blade Section Cross-Sectional Area -----	71
4.	Volume Integration -----	72
5.	Blade Weight -----	72
C.	CODING -----	72
D.	SUMMARY -----	73



VI.	THICKNESS-TO-CHORD RATIO--A DESIGN CONSTRAINT ---	74
A.	INTRODUCTION -----	74
B.	PROPELLER STRENGTH ANALYSIS--A HISTORICAL REVIEW -----	74
C.	SCHOENHERR'S METHOD -----	79
	1. Background -----	79
	2. The Blade Model -----	80
	3. Bending Moments Due to Hydrodynamic Loading -----	82
	4. Force and Bending Moments Due to Centrifugal Loading -----	85
D.	ALGORITHM FOR THE CONSTRAINT -----	97
	1. Theory -----	97
	2. Coding Details -----	99
E.	SUMMARY -----	100
VII.	DESIGN CASE NO. 1--PROGRAMMING AND COMPARISONS --	101
A.	INTRODUCTION -----	101
B.	THRUST APPROACH FORMULATION -----	101
	1. Design Vector $\bar{X}_1$ -----	101
	2. Powering Constraint -----	103
C.	PREVIOUS SOLUTIONS -----	104
D.	SOLUTIONS BY COPES/CONMIN -----	106
	1. Variation 1 -----	107
	2. Variation 2 -----	109
	3. Variation 3 -----	109
	4. Variation 4 -----	111
E.	DISCUSSION -----	111





VIII.	DESIGN CASE NO. 2--PROGRAMMING AND COMPARISONS --	114
A.	INTRODUCTION -----	114
B.	POWER APPROACH FORMULATION -----	114
1.	Design Vector $\overline{X2}$ -----	114
2.	Powering Constraint -----	116
C.	PREVIOUS SOLUTIONS -----	117
D.	SOLUTIONS BY COPEs/CONMIN -----	119
1.	Variation 1 -----	120
2.	Variation 2 -----	122
3.	Variation 3 -----	123
4.	Variation 4 -----	124
E.	DISCUSSION -----	124
IX.	DESIGN CASE NO. 3--PROGRAMMING AND COMPARISONS --	128
A.	INTRODUCTION -----	128
B.	"MATCHING" FORMULATION -----	128
1.	Design Vector $\overline{X3}$ -----	128
2.	Powering Constraint(s) -----	129
C.	PREVIOUS SOLUTIONS -----	131
D.	SOLUTIONS BY COPEs/CONMIN -----	133
1.	Programming Details -----	135
2.	Results -----	135
E.	DISCUSSION -----	136
X.	CONCLUSIONS AND RECOMMENDATIONS -----	139
A.	CONCLUSIONS -----	139
B.	RECOMMENDATIONS -----	141
C.	A FINAL NOTE -----	142



APPENDIX A:	FORTRAN VARIABLE CROSS REFERENCE LIST ----	143
APPENDIX B:	SUBROUTINE LISTINGS -----	145
APPENDIX C:	ANALIZ CODES--DESIGN CASE NO. 1 -----	215
APPENDIX D:	CONTROL CARD IMAGES--DESIGN CASE NO. 1 ---	231
APPENDIX E:	COPEs OUTPUT--DESIGN CASE NO. 1 -----	234
APPENDIX F:	ANALIZ CODES--DESIGN CASE NO. 2 -----	272
APPENDIX G:	CONTROL CARD IMAGES--DESIGN CASE NO. 2 ---	288
APPENDIX H:	COPEs OUTPUT--DESIGN CASE NO. 2 -----	291
APPENDIX I:	ANALIZ CODES--DESIGN CASE NO. 3 -----	330
APPENDIX J:	CONTROL CARD IMAGES--DESIGN CASE NO. 3 ---	334
APPENDIX K:	COPEs OUTPUT--DESIGN CASE NO. 3 -----	336
LIST OF REFERENCES	-----	356
INITIAL DISTRIBUTION LIST	-----	360



## LIST OF TABLES

I.	Summary of the Wageningen B-Screw Series -----	45
II.	Material Identifier Reference -----	58
III.	Global Common (GLOBCM) Catalog -----	63
IV.	Design Case No. 1--Results -----	113
V.	Design Case No. 2--Results -----	127
VI.	Design Case No. 3--Results -----	138



## LIST OF FIGURES

1.1	Traditional Design Spiral -----	15
3.1	Open Water Test Results--B 4-100 Series Propeller -----	42
4.1	Design Vectors $\bar{D}$ and $\bar{X}$ -----	59
5.1	Propeller Blade & Hub--Side View -----	68
5.2	Expanded Cylindrical Blade Section--Profile View -----	70
6.1	Strength Section at $r = r_o$ -----	83
6.2	Bending Moments due to Thrust and Torque -----	86
6.3	Center of gravity "G" and intersection point "N" -----	88
6.4	Internal Loading on a Blade Section at $x = x_o = r_o/R$ -----	89
6.5	Position of "g" of a Blade Section "Slice" at $x = r/R$ -----	93
6.6	Coordinates of centroid "g" of any Blade Section -----	94





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If someone says it's impossible, then, quite obviously, he has never done it.

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## I. INTRODUCTION

### A. BACKGROUND

The ship design process, in its most rudimentary form, has been formulated and tracked by the utilization of the classical design spiral (see Figure 1.1). The design follows a convergent helical path past each major milestone "spoke" until, after numerous iterative cycles, the final configuration is "centered" upon. Whether one attempts to segregate the principal phases of Preliminary, Advanced and Contract Design into separate spirals or combine these phases in series along the entire path to the center, it is not long before the designer's roughed-out sketches give way to serious "number crunching", specifically that of propulsion power estimation.

To estimate the power required to drive the ship through the water at its design speed, a decision must first be made as to what type of propulsor (i.e., propeller, water jet, paddle wheel, etc.) will be used. For the average case, and for the discussion that follows, the marine propeller is chosen to be the propulsion device. Since

a ship propeller may be regarded as a transducer that converts the rotational power transmitted through the shaft into the translational power to propel the ship, [Ref. 18: p. 10]

the selection and design of this device is obviously an important factor in the eventual size (weight and power) of the ship's propulsion plant. While hydrodynamicists provide



# TRADITIONAL DESIGN SPIRAL

LBP,  $L/B$ ,  $B/H$ ,  $C_P$ ,  $C_X$

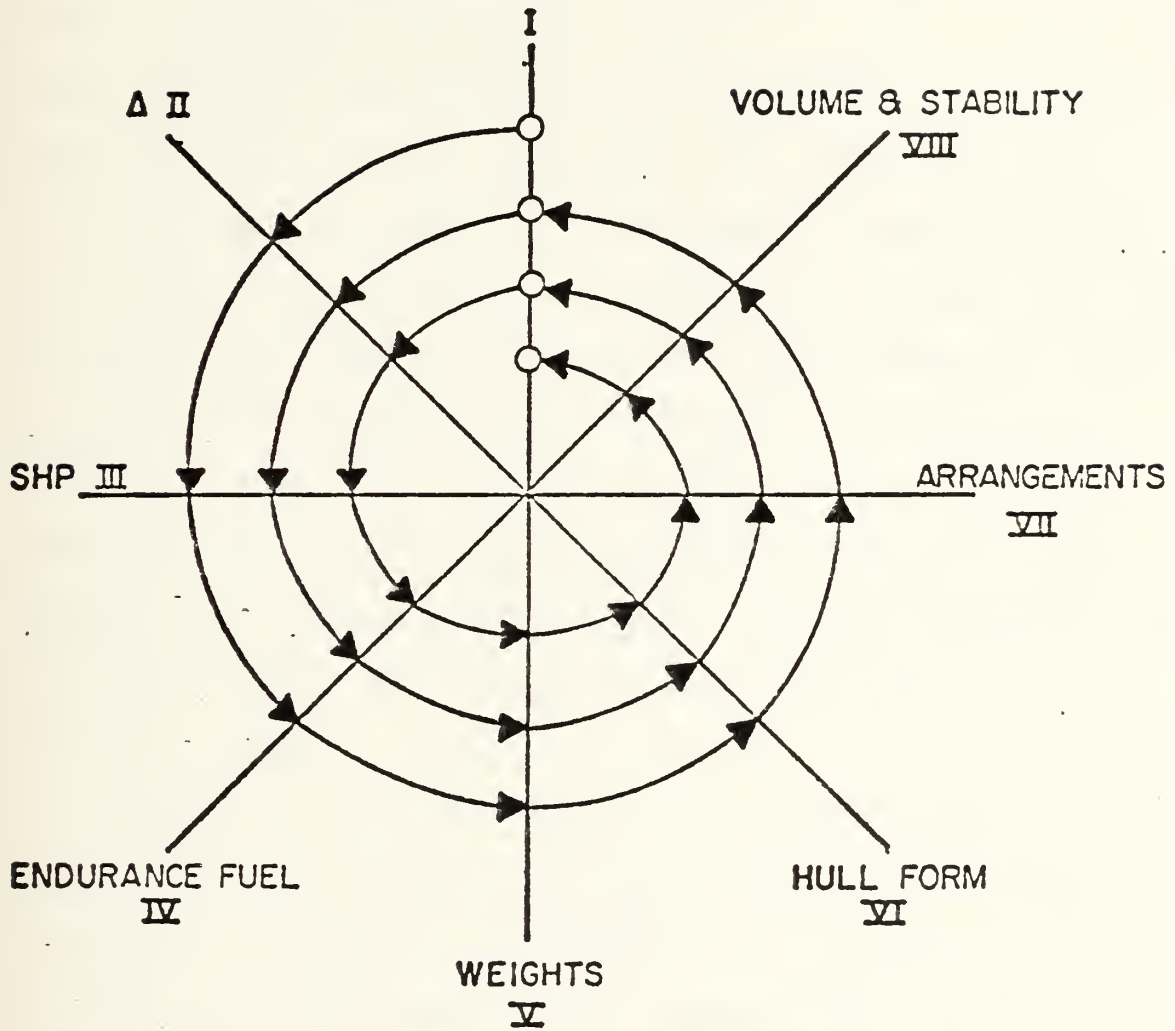


Figure 1.1 Traditional Design Spiral





a myriad of theories and techniques to generate a "custom built" (i.e., wake adapted) propeller for the ship under consideration, their expertise is usually not required in the early stages of preliminary design simply because the design has not been refined enough beyond gross estimates. At this stage, the designer strives to formulate what is possible based on previous experience. For preliminary power estimation, previous propeller designs (i.e., "stock" propellers) and results from methodical series of model propellers are analyzed by the designer in order to select the "best" available propeller under various conditions posed by the problem under consideration. Three examples of typical problems encountered in preliminary ship design are:

- 1) Given the ship's effective horsepower at a specific speed and estimates of hull performance parameters, which propeller, as determined by certain principal characteristics, will require the least amount of delivered power from the propulsion plant?

- 2) Given the delivered power from a specific propulsion system in terms of torque and revolution rate at the propeller/shaft interface and estimates of hull performance parameters, which propeller will generate the largest effective horsepower and speed parameters?

- 3) Given a ship's effective horsepower and speed, various hull performance parameters, and the propulsion plant's delivered power characteristics, which propeller will "match"



these requirements at a minimum amount of weight for a specified material?

(Author's Note: For the sake of brevity, the three selection problems just cited will, henceforth, be referred to as "Design Case No. 1", "Design Case No. 2" and "Design Case No. 3", respectively.)

For this study, the methodical propeller series method is viewed as the designer's choice for preliminary powering analysis. One of the most-widely used methodical series data on model propellers is the Wageningen B-Screw Series. Initially, the results of the series were presented as tabulations of non-dimensional thrust and torque coefficients ( $K_T$  and  $K_Q$  respectively) versus the non-dimensional advance ratio ( $J$ ) for analytical work and as the familiar "Bp- $\delta$ " and "Bu- $\delta$ " diagrams for design purposes. As "trial & error" design methods performed by hand in all engineering disciplines gradually transcended to numerical manipulation by the modern digital computer, the necessity for the adaption of the Series results to a format suitable for use in computer-aided design methods became obvious. This was accomplished through multiple regression analysis of the original open-water test data of the 120 propeller models in the Series and presented in the form of polynomial expressions for " $K_T$ " and " $K_Q$ " [Refs. 1,2].

The adaptation of the Wageningen B-Screw Series polynomials to various types of propeller selection problems formulated



for computer solution has been implemented recently by two authors. Triantafyllou [Ref. 3] and, of late, Markussen [Ref. 4] presented different propeller selection problems and proposed different schemes for computer-aided "optimized" solutions. In short, specific expressions for the constraints imposed and the objective (optimality condition) to be maximized, expressed in terms of a number of design variables and parameters, were developed. Then, each system of equations was solved by a Newton-Raphson method to give a solution set of the design variables which maximized the objective and met all constraints.

Rather than formulating and coding a different optimization scheme each time a propeller selection problem presents a different combination and number of design parameters, variables and constraints, a better approach would involve formulating the problem (constraints and objective function) once in terms of all design parameters and variables and utilizing a general purpose optimization scheme which can handle any combination and number of constraints and design variables. This alternative certainly allows the designer more flexibility in solving his problem. Moreover, it eliminates repetitive coding and debugging associated with the implementation of a computer-sided solution for each particular design problem.



## B. PROBLEM STATEMENT

The problem, then, is that the previously cited computer-aided "optimized" solutions to the propeller selection problem are not broad enough in capability to handle variations in the problem formulation. The objective of this thesis is to apply an available general purpose optimization computer code to the solution of various propeller selection problems encountered in Preliminary Ship Design in order to enhance the flexibility of the selection procedure.

## C. SCOPE

To achieve the stated objective, the general purpose non-linear optimization code CONMIN [Refs. 5,6] together with the engineering synthesis code COPES [Ref. 7] (hereafter referred to collectively as COPES/CONMIN) is utilized in the solution of the three previously cited preliminary design propeller selection problems. Using the Wageningen B-Screw Series propeller characteristics expressed in polynomial expressions of various design variables, three "analysis" codes, required by COPES/CONMIN, are developed in such a way that various combinations of design variables and constraints are used, thereby demonstrating the applicability of the COPES/CONMIN optimization program in the solution of propeller selection problems.

## D. THESIS ORGANIZATION

The remainder of the thesis is organized in the following manner.





Chapter II presents a short description of the optimization problem in general terms and a follow-on discussion of the COPES/CONMIN optimization program and the mathematical techniques employed therein.

Chapter III introduces definitions and concepts applicable to the propeller selection problem. A subsequent discussion on the Wageningen B-Screw Series is followed by final comments on constraints imposed on the propeller selection problem.

Chapter IV presents the formulation of the propeller selection problem as a design optimization problem which can be solved using COPES/CONMIN.

Chapter V discusses the background, formulation and programming utilized in estimating a propeller blade's weight for subsequent consideration as an objective function.

Chapter VI reviews the author's modifications to the propeller strength analysis developed by Schoenherr [Ref. 8] in the early 1960's for the American Bureau of Shipping. A subsequent discussion on the programming details of FORTRAN codes, which are utilized for the determination of adequate propeller blade strength, completes the chapter.

Chapter VII reviews the formulation and programming for the analysis code which is used in solving propeller selection problems represented by Design Case No. 1. Sample solutions are presented and compared to those presented previously by other authors.



Chapters VIII and IX consider Design Case No. 2 and Design Case No. 3 selection problems, respectively, in a similar fashion to Chapter VII.

Chapter X, the final chapter, presents the author's conclusions and recommendations.

As a final note, all computer coding presented in this thesis is done in FORTRAN IV, the language used by COPES/CONMIN. For the reader's convenience, Appendix A provides a cross-reference of the symbols presented throughout the thesis to appropriate FORTRAN variable names appearing in the author's codes.



## II. OPTIMIZATION

### A. INTRODUCTION

The purpose of this chapter is to introduce definitions and concepts used in the formulation and solution of the general optimization problem. Then, a short discussion on the theory and implementation details of COPES/CONMIN is presented.

For further study on the theory and methods of optimization, the reader is directed to the texts by Fox [Ref. 9], Fiacco and McCormick [Ref. 10], and Himmelblau [Ref. 11].

### B. DEFINITIONS

Before discussing the techniques of optimization and their application to engineering problems, some preliminary definitions of basic terminology should be stated. Terms which have relevant significance are:

1) Parameters--The numerical quantities for which values are assigned to produce a design are called parameters. From this, it follows that a design may be specified by a vector  $\bar{D}$  containing "p" components, each of which is associated with a parameter. That is:

$$\bar{D} = \begin{pmatrix} D_1 \\ \vdots \\ D_p \end{pmatrix} \quad (2.1)$$



However, in a design process, the parameters are determined by some logical procedure through analysis of some kind. Some might take on fixed values to become "preassigned" parameters. Interrelationships among other parameters might exist so that only some of the parameters are changed when one design is compared to another. This consequence leads to the definition of "design variable".

2) Design Variables--The parameters for which values are chosen in some fashion to produce a design are called design variables. They represent an ordered collection of components which is a subset of the design vector  $\bar{D}$ . This subset is unique in that its components are "variable", i.e., they may take on different values in the design process. Having "preassigned" or fixed some of the design's parameters and only allowing the remaining "design variables" to change, leads to the conclusion that a design is now uniquely specified by a vector  $\bar{X}$  containing "n" components ( $n \leq p$ ), each of which is associated with a design variable. That is:

$$\bar{X} = \begin{Bmatrix} X_1 \\ \vdots \\ X_n \end{Bmatrix} \quad (2.2)$$

3) Objective Function--The computable function of all or some of the design's preassigned parameters and/or design variables, with respect to which the design is to be optimized, is called the objective function. Single valued in





quantitative terms, the objective function's minimum or maximum value represents the "best" obtainable or "optimized" design. It is expressed as  $F(\bar{D})$  to show its dependence on the design's parameters. But, since a design can be uniquely defined by  $\bar{X}$  alone, then clearly  $F(\bar{X})$  suffices as an expression for the objective function.

4) Constraints--Restrictions on the design which must be satisfied in order to produce an acceptable design are called constraints. A constraint may be classified as a "side" or a "behavior" constraint. A side constraint restricts or bounds the range of the design for reasons other than direct consideration of performance. The side constraint on the "i"th design variable may be expressed as:

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, n \quad (2.3)$$

A constraint derived from those performance or behavior requirements that are explicitly considered is called a behavior constraint. Most often, it appears as a computable functional relation involving the design's parameters, both preassigned and variable alike. The relation may be an inequality so that the "j"th of "m" inequality constraints can be expressed as:

$$G_j(\bar{D}) \leq 0 \quad j = 1, \dots, m \quad (2.4)$$



Alternatively, the relation may be an equality on the "k"th of "ℓ" equality constraints expressed as:

$$H_k(\bar{D}) = 0 \quad k = 1, \dots, \ell \quad (2.5)$$

Of noteworthy importance here is that, as before, if some of the design's parameters are preassigned, then the resulting design is that defined by  $\bar{X}$  which contains only the parameters that can be varied in the design process, i.e., the design variables. Therefore, constraints imposed upon the design may be expressed under one equation as:

$$\begin{aligned} G_j(\bar{X}) &\leq 0 & j &= 1, \dots, m \\ H_k(\bar{X}) &= 0 & k &= 1, \dots, \ell \\ x_i^{\text{lower}} &\leq x_i \leq x_i^{\text{upper}} & i &= 1, \dots, n \end{aligned} \quad (2.6)$$

A final form for constraints is that of the discrete-valued design variable.

5) Feasible Design--A design in which specified constraints are satisfied is called a feasible or "acceptable" design.

6) Infeasible Design--A design in which constraints are violated is called an infeasible or "unacceptable" design.

### C. PROBLEM STATEMENT

If one presupposes that a range of designs exists within a selected design concept, then it follows that different



methodologies also exist by which one may choose the parameters which describe the design. One such method is optimization where parameters are chosen in a way that the design will satisfy all of the limitations and restrictions imposed upon it and will be "best" in some sense. In view of the foregoing definitions, optimization is then a selection method applied to a design problem by which an objective function  $F(\bar{D})$  is minimized to produce an acceptable design which satisfies a certain set of requirements called constraints.

Formulated mathematically, the general, non-linear, constrained optimization problem may be stated under one equation as:

$$\text{Minimize: } F(\bar{D}) = \text{OBJ}$$

$$\text{Subject to: } G_j(\bar{D}) \leq 0 \quad j = 1, \dots, m \quad (2.7)$$

$$H_k(\bar{D}) \leq 0 \quad k = 1, \dots, l$$

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, n$$

Again, as pointed out in the previous section, the design may be uniquely defined by just its design variables as specified by  $\bar{X}$  when some parameters are preassigned. Thus, the general, non-linear, constrained optimization problem can now be stated under one equation as:



$$\text{Minimize: } F(\bar{X}) = \text{OBJ}$$

$$\text{Subject to: } G(\bar{X}) \leq 0 \quad j = 1, \dots, m \quad (2.8)$$

$$H_k(\bar{X}) = 0 \quad k = 1, \dots, \ell$$

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, n$$

Solutions methods for this optimization problem are abundant. Those pertaining to the linear and quadratic optimization problems involving a few design variables are most often presented in graphical or analytic form, although numerical schemes are, by no means, a dormant form. Structural and thermal problem solutions are most prevalent. However, as the optimization problem becomes more complex in terms of non-linear relationships among an increasing number of design variables and of an increased number of design constraints, numerical or mathematical programming techniques dominate the solution methods.

To limit the scope of this discussion, only the numerical techniques relevant to COPES/CONMIN will be considered. For more background on optimization techniques and applications, the reader is directed to a recent paper by Vanderplaats [Ref. 12] which presents a concise, but thorough, qualitative review of optimization. Although this paper deals exclusively with the application of design optimization to structural problems, it also contains a very extensive and current list of references on general techniques and applications of optimization.





#### D. COPES/CONMIN

As previously stated in Chapter I, COPES/CONMIN is the collective acronym for the FORTRAN program utilizing the optimization code CONMIN and the synthesis code COPES. COPES stands for Control Program for Engineering Synthesis; CONMIN is an acronym for CONstrained function MINimization.

##### 1. CONMIN

CONMIN is a FORTRAN program, in subroutine form, which solves the general non-linear constrained optimization problem as stated:

$$\text{Minimize: } F(\bar{X}) = \text{OBJ}$$

$$\text{Subject to: } G_j(\bar{X}) \leq 0 \quad j = 1, \dots, m \quad (2.9)$$

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, n$$

Equation (2.9) applies to the entire statement. Observe that equation (2.9) differs from equation (2.8) in that the equality constraint set, given by  $H_K(\bar{X}) = 0$ , is not specified. This is because the version of COPES/CONMIN used in this study does not consider these types of constraints. However, this will not pose any difficulty in solving the propeller selection problems previously cited.

Again,  $F(\bar{X})$  is the objective function (OBJ). The vector  $\bar{X}$  contains the "n" design variables (NDV).  $G_j(\bar{X})$  are the "m" inequality behavior constraints (NCON) imposed on the optimization problem;  $x_i^{\text{lower}}$  and  $x_i^{\text{upper}}$  are the



respective lower and upper side constraints which bound the "design space" over which  $F(\bar{X})$  and  $G_j(\bar{X})$  are defined. As functional relationships involving  $\bar{X}$ ,  $F(\bar{X})$  and  $G_j(\bar{X})$  may be implicit or explicit, but, in any event, must be continuous and have finite numerical values.

When the inequality condition of equation (2.9) is not satisfied, i.e.,  $G_j(\bar{X}) > 0$  for any constraint, the constraint is said to be violated. If the equality condition is met, i.e.,  $G_j(\bar{X}) = 0$  for any constraint, the constraint is said to be active. And, finally, if the inequality condition is satisfied, i.e.,  $G_j(\bar{X}) < 0$ , for any constraint, that constraint is termed inactive. Any design, defined by  $\bar{X}$ , which satisfies the inequalities of equation (2.9) is designated as a feasible design. Likewise, any one which violates these inequalities is termed an infeasible design. The feasible design with the minimum objective function value, often referred to as the "minimum feasible design", will, therefore, be the optimum design.

During the optimization process, CONMIN employs the Fletcher-Reeves algorithm [Ref. 13] for locally unconstrained problems, and Zoutendijk's method of deasible directions [Refs. 14,15] for locally constrained problems, in a numerical procedure which attempts to minimize the objective function,  $F(\bar{X}) = \text{OBJ}$ , until one or more of the constraints,  $G_j(\bar{X})$ , becomes active. The numerical search procedure begins with an initial  $\bar{X}$  vector which may or may not specify a feasible



design. Modifications are included in CONMIN so that, if the initial design is infeasible, a feasible solution will be obtained with minimal increase in  $F(\bar{X})$ . By iteratively updating the design vector  $\bar{X}$  by the following relation:

$$\bar{X}^{(q+1)} = \bar{X}^{(q)} + \alpha^* \bar{S}^{(q)} \quad (2.10)$$

the optimization process continues by following the constraint boundaries in a direction of search  $\bar{S}$  so that the value of  $F(\bar{X})$  decreases with each iteration  $q$ . The scalar  $\alpha^*$  defines the distance of travel in the direction of search  $\bar{S}$ . The process terminates when a vector  $\bar{X}$  is found such that no further decrease in  $F(\bar{X})$  can be made. The vector  $\bar{X}$  is considered to be optimal and, at least, a local minimum.

CONMIN can be used alone as a subroutine in any FORTRAN program where numerical optimization is desired. However, in order to make the optimization process more "user-friendly", CONMIN has been coupled to COPES in order to simplify its application to various types of problems. Further information on CONMIN can be found in previously cited references [5] and [6].

## 2. COPES

COPES is a FORTRAN program that provides automated design and trade-off capability to the design engineer. It utilizes the optimizer CONMIN to provide the following six specific capabilities:



- 1) simple analysis
- 2) optimization
- 3) sensitivity analysis
- 4) two variable function space analysis
- 5) optimum sensitivity
- 6) optimization using approximation techniques

During the execution of COPES, say for optimization, three principal tasks are performed:

- 1) data management on the design variables and constraints through location assignments in a FORTRAN common block called GLOBCM.
- 2) decision process control on the attainment of an optimal design vector  $\bar{X}$  through multiple calls to the optimizer until a minimum or maximum value of OBJ is achieved and all  $G_j(\bar{X})$  are satisfied.
- 3) evaluation of OBJ and  $G_j(\bar{X})$  at each  $\bar{X}^q$  and  $\bar{X}^{q+1}$  when ICALC = 2 through multiple calls to the user-provided analysis subprogram, SUBROUTINE ANALIZ.

For the application under consideration in this study, only the optimization capability will be used. Therefore, further elaboration on the other capabilities is not warranted.

Reference [7] is the user's manual for COPES/CONMIN. Details on the mechanics of user implementation are presented with subsequent illustration by example. The reader is, therefore, encouraged to familiarize himself with the reference. However, at this point, it is sufficient to be aware of the fact that a user of COPES/CONMIN is required to:





1) provide a FORTRAN subroutine called ANALIZ which performs the input of preassigned parameters, the evaluation of the objective function and constraints during the analysis phase of the optimization search and the output of the results.

2) provide an assembled deck of control cards required by COPES.

#### E. CONCLUDING NOTE

The field of optimization is both extensive and complex and, therefore, the foregoing presentation is, by no means, complete in every detail. However, it is felt that the preceding overview, in conjunction with the cited references, covers the necessary prerequisites that will enable the reader to follow the application of COPES/CONMIN to the various propeller selection problems in the chapters that follow.



### III. POWERING, PERFORMANCE AND PROPELLERS

#### A. INTRODUCTION

The purpose of this chapter is to present an overview of the terminology and concepts that pertain to ship propulsion, propeller selection and the use of model propeller test data. Initially, fundamental definitions used in ship powering problems are presented. This is followed by a discussion of the "classic" types of propeller design/selection problems encountered by the naval architect and marine and naval engineers. Propeller model testing and propeller performance characteristics are reviewed next. The chapter is completed with a discussion of the Wageningen B-Screw Series.

The goal here is brevity. The reader is, therefore, encouraged to investigate the references cited for further details.

#### B. DEFINITIONS

Some fundamental terms associated with most propeller design/selection problems are:

1) Effective Horsepower ( $P_E$ )--power required to tow the "bare" hull (without propeller; rudder and appendage allowance assumed included) that generates a given resistance ( $R_T$ ) at a given speed ( $V$ ). It is determined by:

$$P_E = \frac{R_T V}{550} \quad (3.1)$$



2) Thrust Horsepower ( $P_T$ )--power delivered to water by a propeller developing a thrust force ( $T$ ) and moving at a speed of advance ( $V_A$ ) without the influence of a hull form ahead of it.  $P_T$  is determined by:

$$P_T = \frac{T V_A}{550} \quad (3.2)$$

3) Delivered Horsepower ( $P_D$ )--power delivered by shaft to propeller, normally specified at the outboard side of the stern tube.  $Q_S$  is the torque delivered to the propeller;  $n_p$  is the revolution rate of the shaft and, consequently, the propeller.  $P_D$  is determined by:

$$P_D = \frac{2\pi Q_S n_p}{550} \quad (3.3)$$

4) Shaft Horsepower ( $P_S$ )--power delivered to the inboard side of the stern tube having a transmission efficiency of  $\eta_S$ .  $P_S$  is determined by

$$P_S = \frac{P_D}{\eta_S} \quad (3.4)$$

5) Brake Horsepower ( $P_B$ )--power delivered by the prime mover at connection flange to the power train. While  $P_B$  is normally associated with the prime mover's rated power at this connection (BHP), it can also be specified from the power train/propeller side as:



$$P_B = \frac{P_S}{\eta_B \eta_G} \quad (3.5)$$

where  $\eta_B$  and  $\eta_G$  are, respectively, the bearing system and reduction gear transmission efficiencies.

6) Thrust deduction factor (1-td)--ratio of the tow resistance ( $R_T$ ) to the thrust (T) provided by the propeller. It is determined by

$$(1-td) = \frac{R_T}{T} \quad (3.6)$$

7) Wake Factor (Taylor's) (1-wt)--ratio of the speed of advance ( $V_A$ ) to the ship's speed (V). It is given by:

$$(1-wt) = \frac{V_A}{V} \quad (3.7)$$

8) Advance Ratio (J)--a non-dimensional value, associated with propeller test data presentation (see figure (3.1), given by the following relation:

$$J = \frac{V(1-wt)}{n_p D} = \frac{V_A}{n_p D} \quad (3.8)$$

9) Thrust Coefficient ( $K_T$ )--a non-dimensional value associated with the thrust force (T) developed by a propeller of diameter D which is turning at a rate  $n_p$  and operating in a fluid of density  $\rho$ . It is defined by the following expression:





$$K_T = \frac{T}{\rho n_p^2 D^4} \quad (3.9)$$

10) Torque Coefficient ( $K_Q$ )--a non-dimensional value associated with the torque ( $Q_P$ ) absorbed by a propeller of diameter  $D$  which is turning at a rate  $n_p$  and operating in a fluid of density  $\rho$ . It is defined by the following expression:

$$K_Q = \frac{Q_P}{\rho n_p^2 D^5} \quad (3.10)$$

11) Open Water Efficiency ( $\eta_O$ )--the ratio of  $P_T$  to  $P_D$  for a propeller in open water conditions, with a uniform inflow velocity field at a speed of advance  $V_A$ . It is expressed as:

$$\eta_O = \frac{P_T}{P_D} = \frac{T V_A}{2\pi n_p Q_S} = \frac{J K_T}{2\pi K_Q} \quad (3.11)$$

12) Hull Efficiency ( $\eta_H$ )--a ratio of work done on the ship to that done by the propeller expressed as:

$$\eta_H = \frac{P_E}{P_T} = \frac{R_T V}{T V_A} = \frac{(1-td)}{(1-wt)} \quad (3.12)$$

13) Relative Rotative Efficiency ( $\eta_R$ )--the ratio of the actual, behind-hull efficiency to the open water efficiency. The value of  $\eta_R$  does not, in general, depart from the value



of 1.0. Most often,  $\eta_R$  varies between 0.95 and 1.0 for twin-screw ships and between 1.0 and 1.1 for single screw ships.

Applicable units for the terms in the expressions above are:

- 1) horsepower (hp)-- $P_E, P_T, P_S, P_D$  and  $P_B$
- 2) pounds (lbf)-- $R_T, T$
- 3) feet/second (ft/sec)-- $V, V_A$
- 4) foot-pounds (ft-lbf)-- $Q_S, Q_P$
- 5) feet (ft)-- $D$
- 6) revolutions/second (rps)-- $n_P$
- 7) revolutions/minute (rpm)-- $N_P = n_P/60.0$

The quantities  $T, V_A, D, Q_P$  and  $n_P$  are obtained from the propeller test data results. The quantities  $R_T$  and  $V$  are specified from the design point on the  $R$ - $V$  curve for the hull under study. The quantity  $\rho$  is a property of the fluid in which the hull and propeller operate. And, finally,  $\eta_B, \eta_G$  and  $\eta_S$  are characteristics of the bearing, gear and stern tube systems. In preliminary design studies, nominal values, based on previous designs, are usually assumed unless, of course, these systems have been selected and actual values can be specified.

## C. POWERING CONCEPTS

### 1. Basic Relations

Simply stated, the fundamental powering relationship to be solved in ship propulsion and powering problems is:



$$P_E = \frac{(1-t_d)}{(1-wt)} \cdot \eta_R \eta_O P_D \quad (3.13)$$

Utilizing the definitions just presented, equation (3.13) can be rewritten as:

$$\frac{R_T V}{550} = \frac{(1-t_d)}{(1-wt)} \cdot \eta_R \eta_O \cdot \frac{2\pi Q_S n_P}{550} \quad (3.14)$$

Rearranging terms of equation (3.14) gives:

$$\frac{R_T}{(1-t_d)} \cdot \frac{(1-wt)V}{550} = \eta_R \eta_O \cdot \frac{2\pi Q_S n_P}{550} \quad (3.15)$$

And, finally, when substitutions are made, equation (3.15) becomes:

$$\frac{T V_A}{550} = \frac{T(1-wt)V}{550} = \eta_R \eta_O \cdot \frac{2\pi Q_S n_P}{550} \quad (3.16)$$

Equations (3.14), (3.15), and (3.16) provide the basis for different approaches to the solution of a typical powering problem. More background and information on the definitions and equations presented above may be found in Chapter VI, Sections 10-16 in the text by Comstock [Ref. 16] and O'Brien's book [Ref. 17].

## 2. Approaches to the Powering Problem

From equation (3.16), three types of propeller selection problems are discernible. In the first instance, the propeller thrust  $T$  and the propeller's speed of advance  $V_A$



are taken as known quantities. The fact that  $T$  is known substantiates the "Thrust Approach" nomenclature given to this type of selection problem. In the preliminary (or, in some circles, conceptual) ship design phase, the specification of  $T$  is based upon the requirement imposed by the resistance of the ship ( $R_T$ ) at its design speed ( $V$ ) (or, the effective horsepower ( $P_E$ ) at  $V$ ) and estimates of  $w_t$  and  $t_d$  in the absence of wake surveys and self-propulsion data from model tests. Essentially, the thrust delivered by a selected propeller must provide, at least, the thrust required for the ship hull under study. The objective in the "Thrust Approach" selection problem is to determine, by logical means, the appropriate values of  $Q_S$  and  $n_p$  when the open water efficiency ( $\eta_o$ ) is set by the selected propeller and its performance characteristics.

In the second instance, the delivered torque ( $Q_S$ ) and the propeller shaft speed ( $n_p$ ) are taken to be known. The "Power Approach" nomenclature is given to propeller selection problems of this type because  $P_D$  is known. Here, with the shaft and propeller speeds being equal, the torque absorbed by the propeller ( $Q_p$ ) must be, at least, equivalent to the delivered torque ( $Q_S$ ). The corresponding objective in the "Power Approach" selection problem is to determine, by logical means, the expected ship speed ( $V$ ) (or, the speed of advance ( $V_A$ )) and the associated thrust ( $T$ ) that can be developed when the open water efficiency ( $\eta_o$ ) is, again, set by the selected propeller and its performance characteristics.





The final, and most familiar, types of propeller selection problem occurs when  $T$ ,  $V_A$  or  $V$ ,  $Q_S$  and  $n_p$  are all known. From equation (3.16), the open water efficiency ( $\eta_o$ ) is now established as a requirement to be met. The objective is, simply, to select a propeller whose open water efficiency ( $\eta_o$ ), developed thrust and absorbed torque are equivalent to or "match" the requirements imposed. Obviously, this approach on the selection problem has been designated as a "matching problem".

The reader is directed to the paper by Vassilopoulos [Ref. 18] for further information.

#### D. PROPELLER PERFORMANCE CHARACTERISTICS

Up until the late 1950's, much of the knowledge about the performance of propellers has been gained from experience with models. To study the relationships governing their behavior, a model propeller is built and run in a towing tank without any hull ahead of it. This is done by running the propeller on a long shaft projecting well ahead of a narrow, hydrodynamically shaped pod or "propeller boat" which contains the driving mechanism and recording apparatus and is attached to the towing carriage. The propeller advances into undisturbed fluid (usually water of density  $\rho$  and kinematic viscosity  $\nu$ ) so that the speed of advance ( $V_A$ ) is known and the flow into the "disc" swept by the turning blades is uniform. For the model propeller of diameter ( $D$ ) under test, readings of thrust ( $T$ ), torque ( $Q_S$ ) and shaft revolutions ( $n_p$ )



are recorded over a range of values for speed of advance ( $V_A$ ) in this "open water" condition.

Using the laws of similitude, the collected data is reduced and scaled appropriately into the familiar functional relationships between the advance ratio ( $J$ ) and the non-dimensional coefficients of propeller performance. These coefficients or performance characteristics, defined previously, are:

- 1) Thrust Coefficient ( $K_T$ )
- 2) Torque Coefficient ( $K_Q$ )
- 3) Open Water Efficiency ( $\eta_o$ )

Figure (3.1) graphically depicts the relationship between  $J$  and  $K_T$  and  $K_Q$  derived from test data for a propeller defined by a specific expanded area ratio ( $A_E/A_O$ ), pitch-diameter ratio ( $P/D$ ), number of blades ( $Z$ ) and thickness-to-chord ratio ( $t/c$ ).

Definitions of these terms with graphical illustrations pertaining to various aspects of propeller geometry can be found in Section 15 of references [16], [17] and in van Manen's publication [Ref. 19].

More recently, highly analytical theories (lifting line, modified lifting line, lifting surface, etc.) for use with high-speed digital computers have been formulated and subsequently used in "modeling", in a mathematical sense, the propeller and its behavior in the "wake adapted" (or, behind hull) condition as well as the "open water" condition.



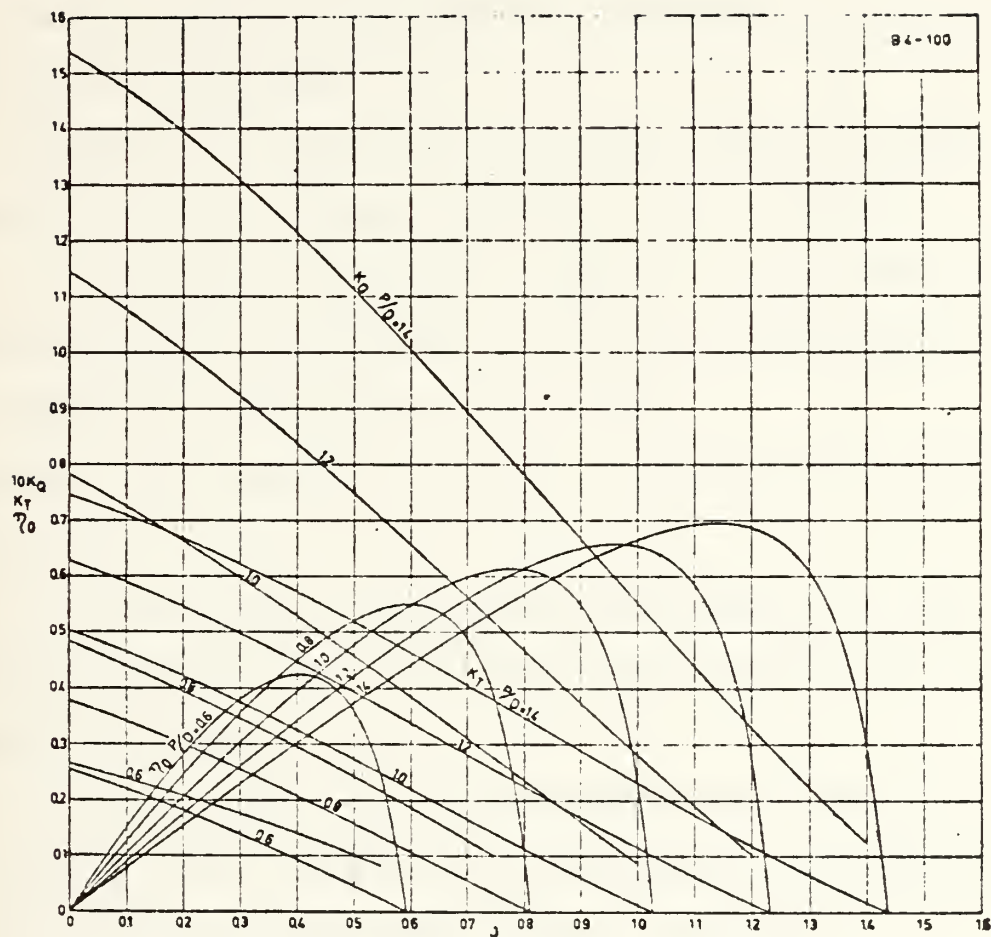


Figure 3.1 Open Water Test Results--B 4-100 Series Propeller



Additional benefits derived from this approach to propeller performance analysis include:

- 1) determination of blade section profiles along the propeller's blade radius ( $R$ ) to achieve uniform lift and internal stress distributions;
- 2) computation of "off-design" performance characteristics in all quadrants;
- 3) subsequent determination of hull surface forces, bearing loads and spindle torques induced by the propeller;
- 4) prediction of steady and unsteady stress distributions in the propeller blade using the finite element method on the blade of the propeller under study.

Obviously, this approach to propeller performance analysis serves to:

- 1) eliminate the time-consuming and expensive model construction and testing of propellers in tow tanks and cavitation tunnels;
- 2) eliminate the "scaling" discrepancies which inhibit the reliability of design charts and model propeller data;
- 3) eliminate those design charts altogether.

As in the case with model experiments, however, the ultimate objective remains the same, i.e., establishing the performance characteristics of the propeller in terms of  $K_T$ ,  $K_Q$  and as functions of  $J$ . Having these relationships enables the ship designer to proceed in solving the power equation (equation (3.13)) through any of the approaches previously discussed.





## E. THE WAGENINGEN B-SCREW SERIES

### 1. Background

The model test data of the Wageningen B-Screw Series have been selected for use in the powering problems to be solved utilizing COPES/CONMIN. The choice was driven by the following considerations:

1) the Series is widely known and, despite its growing obsolescence, is still used in preliminary ship design studies.

2) the availability of previous investigations [Refs. 3,4] which utilized the series, for comparative analysis of optimization results.

3) the applicability of the polynomial expressions for  $K_T$  and  $K_Q$  to computer-aided analysis.

The Series tests were conducted from 1940 through 1960 and, therefore, represent propeller designs (principally naval and merchant applications) and design philosophy of that era.

Specifically, the Series consists of 120 model propellers. As is customary in methodical or systematic model propeller series testing, the number of blades ( $Z$ ), expanded area ratio ( $A_E/A_O$ ) and pitch-diameter ratio ( $P/D$ ) are varied systematically, while the blade outline, the profile of the blade's cross section along the blade radius, blade cross section maximum thickness ( $t$ ), blade section chord length ( $c$ ), diameter ( $D$ ) and propeller hub-to-diameter ratio ( $d/D$ ) were kept constant for given values of  $A_E/A_O$  and  $Z$ .



Table (I) summarizes the variations in  $Z$  and  $A_E/A_0$  for each set of model propellers having pitch-diameter ratios ( $P/D$ ) of 0.4, 0.6, 0.8, 1.0, 1.2 and 1.4.

TABLE I  
Summary of the Wageningen B-Screw Series

Blade number $Z$	Blade area ratio $A_E/A_0$														
2	0.30														
3		0.35			0.50			0.65			0.80				
4			0.40			0.55			0.70			0.85			1.00
5				0.45			0.60			0.75					1.05
6					0.50			0.65			0.80				
7						0.55			0.70			0.85			

## 2. Series Results

The test results of the Wageningen B-Screw Series were originally presented in the form of  $B_p-\delta$ ,  $B_u-\delta$  and  $K_T$ ,  $K_Q$ , and  $-J$  diagrams. As stated in Chapter I, multiple regression analysis was performed (again, [Refs. 1,2]) on the results to produce the polynomial expressions for  $K_T$  and  $K_Q$ . The open water efficiency ( $\eta_0$ ), as a function of  $J$ , follows from equation (3.11). The correction for "scale effects" was achieved by using Lerb's method of equivalent profiles [Ref. 20]. Although Triantafyllou's thesis [Ref. 21] suggests an improved method for scale correction, the results of Reference [2] will be used in this study.



In propeller selection problems which use this Series, values for  $K_T$  and  $K_Q$  are defined as:

$$K_T = f_1(J, P/D, A_E/A_O, Z, t^*/c_{.75R}) \quad (3.17)$$

$$K_Q = f_2(J, P/D, A_E/A_O, Z, t^*/c_{.75R})$$

To compute  $K_T$  and  $K_Q$ , the following equations are used:

$$K_T = K'_T + \Delta K_T \quad (3.18)$$

$$K_Q = K'_Q + \Delta K_Q$$

The polynomial expressions found in Tables (5) and (6) of Reference [2] are then used to evaluate to components  $K'_T$ ,  $K'_Q$ ,  $\Delta K_T$  and  $\Delta K_Q$ .

Table (5) in Reference [2] lists the coefficients used in the polynomial expressions for  $K'_T$  and  $K'_Q$  at an equivalent Reynolds number ( $Rn_{.75R}$ ) of  $2 \times 10^6$ . It is defined as:

$$Rn_{.75R} = \frac{c_{.75R} \sqrt{(V_A)^2 + (0.75\pi n_P D^2)}}{v} \quad (3.19)$$

where:

- $v$  = kinematic viscosity of the fluid ( $\text{ft}^2/\text{sec}$ );
- $c_{.75R}$  = blade section chord length at 3/4 propeller radius ( $.75R$ ) in feet (ft).



To account for "other effects", coefficients  $\Delta K_T$  and  $\Delta K_Q$  are introduced. Table (6) in Reference [2] lists the coefficients used in the polynomial expressions for these coefficients. "Other effects" include the operation of the propeller at an equivalent Reynolds Number different from  $2 \times 10^6$ . Also, variations in other parameters which define the propeller's geometry, specifically  $t/c$  values different from the ones fixed by the Wageningen propellers, are taken into account by corrections to the equivalent Reynolds number. By keeping the blade section's chord length ( $c$ ) at the value of the Wageningen propeller, a change in a blade section's maximum thickness from the standard one defined by the Series ( $t$ ) to one preferred in the selection ( $t^*$ ) produces a new equivalent Reynolds number ( $Rn^*_{.75R}$ ) given by:

$$Rn^*_{.75R} = \exp \left[ 4.6052 + \left\{ \sqrt{\frac{1+2(t/c)_{.75R}}{1+2(t^*/c)_{.75R}}} (\ln Rn_{.75R} - 4.6052) \right\} \right] \quad (3.20)$$

where:

- $Rn^*_{.75R}$  = the new equivalent Reynolds number;
- $(t^*/c)_{.75R}$  = new equivalent  $t/c$  at 3/4 propeller radius;
- $Rn_{.75R}$  = the Reynolds number computed by equation (3.19);
- $(t/c)_{.75R}$  = standard equivalent  $t/c$  at 3/4 propeller radius for Wageningen propellers.





For the Wageningen B-Screw Series, the standard equivalent  $t/c$  is given by:

$$t/c_{.75R} = \frac{(0.0185 - 0.00125Z)Z}{2.073 A_E/A_O} \quad (3.21)$$

Further details on blade section geometry will be addressed in Chapter V. Reference [2] contains background and other information on the equations above.

### 3. Limitations on Series Data

In utilizing the Wageningen B-Screw Series in any propeller selection problem, the following restrictions apply to the Series data:

1) Number of Propeller Blades ( $Z$ )--The Series considers only propellers with numbers of blades as shown in Table (I). Therefore,

$$Z = 2, 3, 4, 5, 6 \text{ or } 7 \quad (3.22)$$

However, the two bladed propeller, i.e.,  $Z = 2$ , is not very common in conventional merchant and naval ship designs and, therefore, is not included in this study.

2) Equivalent Reynolds Number--The Series data, as published, is valid only in the range of equivalent Reynolds numbers given by:

$$2 \times 10^6 \leq Rn_{.75R} \leq 2 \times 10^9 \quad (3.23)$$



If the equivalent  $t/c$  is varied from the standard equivalent value  $(t/c)_{.75R}$ , then the new equivalent Reynolds number  $(Rn^*_{.75R})$ , which results from this variation, must lie within the same limits. That is,

$$2 \times 10^6 \leq Rn^*_{.75R} \leq 2 \times 10^9 \quad (3.24)$$

These limits are appropriate for full-size propellers. For example, given the following:

- a)  $wt = .22$
- b)  $V = 20$  (knots)
- c)  $N_p = 104$  (rpms)
- d)  $D = 25$  (ft)
- e)  $c_{.75R} = 4.0$  (ft)
- f)  $v = 1.2285 \times 10$  (ft<sup>2</sup>/sec)

the value for  $Rn_{.75R}$  is equal to  $3.619 \times 10^7$ .

3) Pitch-diameter Ratio (P/D)--The series data considers only pitch diameter ratios in the range given by:

$$0.4 \leq P/D \leq 1.4 \quad (3.25)$$

4) Advance Ratio (J)--An inspection of the Series results in graphical format shows that J varies over a range given by:

$$0 < J \leq 1.6 \quad (3.26)$$



5) Expanded Area Ratio ( $A_E/A_O$ )--Using Table (I),  $A_E/A_O$  varies over certain ranges depending on Z. This is stated as:

$$0.35 \leq A_E/A_O \leq 0.8 \quad Z = 3 \quad (3.27)$$

$$0.40 \leq A_E/A_O \leq 1.0 \quad Z = 4 \quad (3.28)$$

$$0.45 \leq A_E/A_O \leq 1.05 \quad Z = 5 \quad (3.29)$$

$$0.50 \leq A_E/A_O \leq 0.8 \quad Z = 6 \quad (3.30)$$

$$0.55 \leq A_E/A_O \leq 0.85 \quad Z = 7 \quad (3.31)$$

6) Hub diameter-to-Propeller Diameter Ratio ( $d/D$ )--From Table 37, Section 17 of Reference [16], the Series data requires that:

$$d/D = 0.18 \quad Z = 3,7 \quad (3.32)$$

$$d/D = 0.167 \quad Z = 4,5,6 \quad (3.33)$$

## F. SUMMARY

From the preceding discussions, the following observations can be made:

1) the "Design Cases", defined in Chapter I, are examples of the powering equation solution approaches. That is, Design



Case No. 1 constitutes a "Thrust Approach" problem; Design Case No. 2, a "Power Approach" one; Design Case No. 3, a "Matching" problem.

2) equations (3.17) and (3.8) imply that an optimization solution to the "Design Cases" will involve  $P_E$ ,  $V$ ,  $wt$ ,  $D$ ,  $P/D$ ,  $A_E/A_O$ ,  $(t^*/c)_{.75R}$ ,  $Q_S$  and  $n_p$  as possible design variables.

3) when viewed from the concepts on optimization presented in Chapter II, equations (3.25) through (3.31) constitute side constraints to an optimized solution of a propeller selection problem which uses the Wageningen B-Screw Series. Having noted these points, the propeller selection problem can now be formulated as a general, non-linear, constrained optimization problem.





#### IV. PROPELLER SELECTION--AN OPTIMIZATION PROBLEM

##### A. INTRODUCTION

The purpose of this chapter is to present the formulation of the propeller selection problem as an optimization problem that can be solved using COPES/CONMIN. Three usual restrictions considered by the designer in any propeller selection analysis are stated as constraints. Then, the components of  $\bar{D}$  and  $\bar{X}$  are assembled based on requirements from previously cited relationships. The restrictions considered by the designer and the limitations imposed by the use of the Wageningen B-Screw Series are presented in inequality constraint format. A formal statement of the propeller selection problem as an optimization problem is followed by a review of the GLOBCM common block format and the basic subprograms used in all three versions of SUBROUTINE ANALIZ that pertain to each Design Case.

The FORTRAN subprogram listings are found in Appendix B. Comment cards have been used extensively in the coding development to assist the reader.

##### B. DESIGNER'S CONSIDERATIONS

###### 1. Propeller Size

The first restriction on the selection of any propeller is size. That is, the propeller race in the stern of the hull under consideration will only accommodate a



propeller of some given maximum diameter ( $D_{lim}$ ). As a constraint on a selected propeller of diameter  $D$ , this may be written as:

$$D \leq D_{lim} \quad (4.1)$$

or, alternatively, as:

$$G_9(\bar{X}) = \frac{D}{D_{lim}} - 1 \leq 0 \quad (4.2)$$

## 2. Cavitation

Another item of importance in propeller selection is the cavitation phenomenon. When a propeller of given diameter  $D$  and expanded area ratio  $A_E/A_O$  is operating to produce a thrust  $T$ , the formation and subsequent collapse of water vapor bubbles on the blade surface, i.e., cavitation, is likely to occur if the localized surface pressures, usually on the "back" side of the blade, drop below the pressure at which the fluid would boil ( $p_{watvap}$ ) in the surrounding environment. Avoidance of cavitation can be reasonably assured by selecting a propeller having certain geometric characteristics. A good empirical relationship that establishes these characteristics for propellers typified by the Wageningen B-Screw Series is the Keller Cavitation criterion [Ref. 2: p. 259]. It specifies the minimum required expanded area ratio ( $A_E/A_O \min$ ) to avoid cavitation and is given by:



$$(A_E/A_O)_{\min} = \frac{(1.3 + 0.3Z)}{(p_{\text{atm}} + \rho \cdot acg h_{cl} - p_{\text{watvap}})} \cdot \frac{T}{D^2} + b \quad (4.3)$$

where:

- Z = number of blades;
- T = developed thrust (lbf);
- D = propeller diameter (ft);
- P<sub>atm</sub> = atmospheric pressure (psia);
- P<sub>watvap</sub> = fluid vaporization pressure (psia)
- ρ = fluid density (lbf sec<sup>2</sup>/ft<sup>4</sup>)
- acg = 32.174 (ft/sec<sup>2</sup>)
- h<sub>cl</sub> = depth to shaft centerline (ft)
- b = constant: 0.1 for Z = 2, 0.2 for Z = 1.

As a constraint on the propeller selection, this requirement is written as:

$$(A_E/A_O)_{\min} \leq A_E/A_O \quad (4.4)$$

or

$$G_{10}(\bar{X}) = \frac{(A_E/A_O)_{\min}}{A_E/A_O} - 1 \leq 0 \quad (4.5)$$

### 3. Strength

The final designer's consideration (for this study), included in the selection of a propeller, is that of strength.



Given the propeller's material (promat), selected from Table (II), and the loadings ( $T$  and  $Q_S$ ) imposed, it is important to ensure that the blade's cross sections have proper dimensions (in an ideal sense, maximum blade section thickness ( $t^*$ ) and chord length ( $c$ )) to ensure adequate strength. Since the use of the B-Screw Series requires that the chord length ( $c$ ) vary as a prescribed function of  $D$ ,  $Z$  and  $A_E/A_O$ , as given in Table 1 of Reference [2], the adequacy for strength can be determined by an appropriately selected value for blade section maximum thickness-to-chord ratio ( $t^*/c$ ) alone. So, if  $t_{\min}^*$  is the established minimum blade section maximum thickness, then the strength requirement follows from the constraint given by:

$$t_{\min}^*/c = (t^*/c)_{\min} \leq t^*/c \quad (4.6)$$

The fact that blade section maximum thickness for the B-Screw Series varies linearly with the propeller radius ( $R$ ) allows the strength constraint (equation (4.6)) to be evaluated at one section along the radius. This point is chosen to be at the  $3/4$  radius ( $.75R$ ). Therefore, equation (4.6) becomes:

$$(t^*/c)_{.75R \min} \leq (t^*/c)_{.75R} \quad (4.7)$$

or

$$G_{11}(\bar{X}) = \frac{(t^*/c)_{.75R \min}}{(t^*/c)_{.75}} - 1 \leq 0 \quad (4.8)$$





Reference [2] suggests the following empirical relation for the minimum required equivalent blade section maximum thickness-to-chord ratio  $(t^*/c)_{.75R \min}$ :

$$(t^*/c)_{.75R \min} = \frac{z \left\{ 0.0028 + 0.21 \sqrt[3]{\frac{(2375 - 1125P/D) P_D}{4.123 N_P D^3 (S_C + \frac{D^2 N_P^2}{12.788})}} \right\}}{2.073 A_E/A_O} \quad (4.9)$$

where:

- D = propeller diameter (ft);
- $P_D$  = delivered power (hp);
- $N_P$  = propeller revolution rate (rpm);
- $S_C$  = propeller material allowable stress (psi);
- $P/D$  = pitch-diameter ratio.

However, in Chapter VI of this thesis, an algorithm which employs the Schoenherr formulation [Ref. 8] with some modifications, is presented as an alternative to equation (4.9).

### C. THE DESIGN VECTOR

In view of the preceding presentations on optimization and powering, the design vector  $\bar{D}$  can be assembled for the general propeller selection problem utilizing the B-Screw Series. This vector is composed of preassigned parameters relating to environmental conditions, hull characteristics



and the propeller which are required for various equations and the design variables.

### 1. Parameters

#### a. Environmental

These parameters pertain primarily to the fluid conditions in which the propeller operates and to the atmosphere. Required for various calculations, they are:

- 1) fluid temperature ( $^{\circ}\text{F}$ )--Temp
- 2) fluid density ( $\text{lbf sec}^2/\text{ft}^4$ )-- $\rho$
- 3) fluid viscosity ( $\text{ft}^2/\text{sec}$ )-- $\nu$
- 4) fluid vaporization pressure (psia)-- $p_{\text{watvap}}$
- 5) atmospheric pressure (psia)-- $p_{\text{atm}}$

#### b. Hull Characteristics

These parameters pertain to certain details prescribed for the hull under study in the powering analysis.

They are:

- 1) wake fraction--wt
- 2) thrust deduction--td
- 3) relative rotative efficiency -- $\eta_R$
- 4) number of propellers--noscrw
- 5) shaft centerline depth (ft)-- $h_{\text{cl}}$
- 6) propeller diameter limit (ft)-- $D_{\text{lim}}$

#### c. Propeller

These parameters are specified in view of their discrete-valued nature. They are:

- 1) number of blades--Z
- 2) material--promat



Table (II) lists materials and properties considered in this study. These values are taken from Table (35), Section 15 of Reference [16].

TABLE II  
Material Identifier Reference

promat	Material	Allowable Stress--Sc (psi)	Density--wd (lbf/in <sup>3</sup> )
1	Cast Iron	3600--3950	.260
2	Cast Steel	5915--6265	.289
3	Type 2 Bronze	7200-7585	.305
4	Type 4 Ni-Al Bronze	8910--9430	.278
5	Stainless Steel	5400--5500	.283

## 2. Design Variables

In view of equations (3.13) through (3.17), the design variables common to all selection approaches are:

- 1)  $P_E$
- 2)  $V$
- 3)  $D$
- 4)  $P/D$
- 5)  $A_E/A_O$
- 6)  $(t^*/c) .75R$
- 7)  $N_P$
- 8)  $Q_S$

The vectors  $\bar{D}$  and  $\bar{X}$  are shown schematically in figure (4.1).



$$\bar{X} = \left\{ \begin{array}{l} P_E \\ V \\ D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \\ N_P \\ Q_S \end{array} \right\} \quad \bar{D} = \left\{ \begin{array}{l} \text{Temp} \\ \rho \\ v \\ P_{\text{watvap}} \\ P_{\text{atm}} \\ wt \\ td \\ \eta_R \\ \text{noscw} \\ h_{cl} \\ D_{lim} \\ z \\ \text{promat} \\ P_E \\ V \\ D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \\ N_P \\ Q_S \end{array} \right\}$$

Figure 4.1 Design Vectors  $\bar{D}$  and  $\bar{X}$





#### D. CONSTRAINTS

Besides the constraints imposed by equations (4.2), (4.5) and (4.8), equations (3.25) through (3.31) are rearranged to the format of constraints in equation (2.9). They are listed as follows:

1) Equivalent Reynolds Number--Equation (3.25) becomes:

$$1 \leq \frac{Rn^* \cdot 75R}{2 \times 10^6} \leq 1000 \quad (4.10)$$

Two constraints are derived:

$$G_3(\bar{X}) = 1 - \frac{Rn^* \cdot 75R}{2 \times 10^6} \leq 0 \quad (4.11)$$

$$G_4(\bar{X}) = \frac{Rn^* \cdot 75R}{2 \times 10^6} - 1000 \leq 0 \quad (4.12)$$

2) Expanded Area Ratio--Equations (3.27) through (3.31) become:

$$(A_E/A_O)_{\text{lower}}(Z) \leq A_E/A_O \leq (A_E/A_O)_{\text{upper}}(Z) \quad (4.13)$$

Two constraints are derived:

$$G_5(\bar{X}) = (A_E/A_O)_{\text{lower}}(Z) - A_E/A_O \leq 0 \quad (4.14)$$

$$G_6(\bar{X}) = A_E/A_O - (A_E/A_O)_{\text{upper}}(Z) \leq 0 \quad (4.15)$$



3) Advance Ratio--Equation (3.22) becomes:

$$0 \leq \frac{J}{1.6} \leq 1 \quad (4.16)$$

Two constraints are derived:

$$G_1(\bar{X}) = \frac{-J}{1.6} \leq 0 \quad (4.17)$$

$$G_2(\bar{X}) = \frac{J}{1.6} - 1 \leq 0 \quad (4.18)$$

4) Equivalent Blade Section Maximum Thickness-to-Chord Ratio--Using equation (3.19), boundaries on the range of  $(t^*/c)_{.75R}$  are defined by:

$$\frac{1}{2}(t/c)_{.75R} \leq (t^*/c)_{.75R} \leq 4(t/c)_{.75R} \quad (4.19)$$

Two constraints are derived:

$$G_7(\bar{X}) = \frac{1}{2}(t/c)_{.75R} - (t^*/c)_{.75R} \leq 0 \quad (4.20)$$

$$G_8(\bar{X}) = (t^*/c)_{.75R} - 4(t/c)_{.75R} \leq 0 \quad (4.21)$$

## E. OBJECTIVE FUNCTIONS

Upon consideration of equation (3.13), Design Case No. 1 and Design Case No. 2 require that the open water efficiency  $(\eta_o)$ , given by equation (3.11), be maximized. In terminology related to optimization, this is stated as:



$$OBJ_{1,2} = -\eta_0 \quad (4.22)$$

Design Case No. 3, the "matching" problem, requires that the blade weight (bldwt) be minimized. This is stated as:

$$OBJ_3 = \text{bldwt} \quad (4.23)$$

#### F. PROPELLER SELECTION OPTIMIZATION PROBLEM STATEMENT

As a general, non-linear constrained optimization problem to be solved by COPES/CONMIN, the propeller selection problem for all Design Cases may be stated as one equation given by:

$$\text{Minimize: } F(\bar{X}) = OBJ_{1,2} \text{ or } OBJ_3 \quad (4.24)$$

$$\text{Subject to: } G_j(\bar{X}) \leq 0 \quad j = 1, \dots, 12$$

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, 8$$

The constraint  $G_{12}(\bar{X})$  and the values for  $x_i^{\text{lower}}$  and  $x_i^{\text{upper}}$  will be specified according to each Design Case.

#### G. CODING FUNDAMENTALS

##### 1. GLOBCM Common Block

The GLOBCM common block, required by COPES/CONMIN, is now assembled. Table (III) specifies the assignment locations for the FORTRAN variables which define objective functions, design variables and constraints.

##### 2. SUBROUTINE ANALIZ

While each Design Case uses a different approach, all analyses are very similar. Therefore, each SUBROUTINE



TABLE III

## Global Common (GLOBCM) Catalog

Global Location	FORTTRAN Name	DEFINITION
1	ETAO	$\eta_O$
2	WEIGHT	bldwt
3	AEDVAO	$A_E/A_O$
4	DIA	D
5	N	$N_P = 60 \cdot n_P$
6	PE	$P_E$
7	PDIVD	$P/D$
8	QS	$Q_S$
9	TC75R	$(t^*/c)_{.75R}$
10	V	V (ft/sec)
11	RJCHL	$G_1(\bar{X})$ --eqn (4.17)
12	RJCNU	$G_2(\bar{X})$ --eqn (4.18)
13	R75RCL	$G_3(\bar{X})$ --eqn (4.11)
14	R75RCU	$G_4(\bar{X})$ --eqn (4.12)
15	AEAOCL	$G_5(\bar{X})$ --eqn (4.14)
16	AEAOCU	$G_6(\bar{X})$ --eqn (4.15)
17	TC75CL	$G_7(\bar{X})$ --eqn (4.20)
18	TC75CU	$G_8(\bar{X})$ --eqn (4.21)
19	POWBAL	$G_{12}(\bar{X})$ --eqn (7.10) or (8.11) or (9.4)
20	DIACNU	$G_9(\bar{X})$ --eqn (4.2) or (9.6)
21	AEAOCV	$G_{10}(\bar{X})$ --eqn (4.5)
22	TCSTRS	$G_{11}(\bar{X})$ --eqn (4.8)
23	RJ	J





ANALIZ shares a common structure and other common subroutines which perform calculations required in all cases. Appendices C, F and I contain, respectively, the source listings of SUBROUTINE ANALIZ for Design Case No. 1, Design Case No. 2 and Design Case No. 3.

#### a. Structure

The structure common to all cases follows accordingly:

- 1) all initialization of environmental, hull and propeller parameters is accomplished in the input section (ICALC = 1).
- 2) evaluation of  $K_T$  and  $K_Q$ , all constraints and appropriate objective functions ( $-\eta_o$  or bldwt) are accomplished in the execution section (ICALC = 2).
- 3) output of results for each optimization problem is accomplished in the output section (ICALC = 3).

#### b. Basic Subprograms

The following FORTRAN subprograms are used in all three SUBROUTINE ANALIZ codes:

- 1) SUBROUTINE CH75RA--calculates the equivalent blade section chord length ( $c_{.75R}$ ) for the propeller using Table 1 [Ref. 2, p. 252].
- 2) SUBROUTINE REY75R--calculates the equivalent Reynolds number ( $Rn_{.75R}$ ) using equations (3.19) and (3.20).
- 3) SUBROUTINE COEFS--calculates the thrust and torque coefficients ( $K_T$  and  $K_Q$ ) through sequential calls to SUBROUTINE CALCKT and SUBROUTINE CALCKQ. The polynomial expressions



(Tables (5) and (6), [Ref. 2]) for these coefficients are contained in SUBROUTINE CALCKT and SUBROUTINE CALCKQ respectively.

4) SUBROUTINE OPWEFF--calculates the open water efficiency ( $\eta_o$ ) using equation (3.11).

5) SUBROUTINE JCNA--calculates the constraints on the advance ratio (J) given by equations (4.17) and (4.18).

6) SUBROUTINE REYCNA--calculates the equivalent Reynolds number constraints given by equations (4.11) and (4.12).

7) SUBROUTINE EXTCCN--calculates the constraints on expanded area ratio ( $A_E/A_O$ ) and equivalent blade section maximum thickness-to-chord ratio ( $t^*/c_{.75R}$ ) given by equations (4.14), (4.15), (4.20) and (4.21).

8) SUBROUTINE DICNUA--calculates the constraint on the propeller diameter (D) given by equation (4.2) using the hull parameter on maximum diameter ( $D_{lim}$ ).

9) SUBROUTINE CAVCNA--calculates the constraint for cavitation given by equation (4.5) using equation (4.3).

10) SUBROUTINE STRCNA--calculates the constraint for strength given by equation (4.8) using equation (4.9).

## H. SUMMARY

The propeller selection problem has now been formulated as a constrained optimization problem which can be solved by COPES/CONMIN. Two items remain for discussion before proceeding to specify the final details pertaining to each SUBROUTINE ANALIZ code and to present numerical examples. These items are:



1) the theory and coding relating to the computation of the propeller's blade weight for the evaluation of the objective function in Design Case No. 3 (OBJ<sub>3</sub>).

2) the theory and coding relating to the computation of the minimum required equivalent blade section maximum thickness-to-chord ratio ( $t^*/c_{.75 \text{ min}}$ ) for use in the alternative evaluation of the strength constraint given by equation (4.8).



## V. PROPELLER BLADE WEIGHT--AN OBJECTIVE FUNCTION

### A. INTRODUCTION

In this chapter, the method for the computation of the propeller's blade weight (bldwt), the objective function OBJ, is examined. First, a brief overview on the steps in the computational procedure is presented. The FORTRAN subprogram SUBROUTINE WGTAL developed from the algorithm is then described. Again, Appendix B contains all subprogram listings.

### B. THEORY AND PROCEDURE

Given the material of a propeller, the calculation of the weight of one blade involves nothing more than a volume calculation, a relatively routine task performed by most naval architects/marine engineers. Analogous to the determination of the underwater volume of a ship's hull, the calculation is an integration of blade section profiles' cross-sectional areas over the propeller radius (R).

#### 1. Limits of Integration

Figure (5.1) depicts a side elevation view of a blade and hub, parallel to the propeller shaft axis. The cross-hatched area indicates the trace of the volume to be calculated. In view of equations (3.32) and (3.33), limits of integration are from  $r = .167R$  to  $r = R$  for  $Z = 4, 5, 6$  and  $r = .18R$  to  $r = R$  for  $Z = 3, 7$ . For convenience, a non-dimensional variable "x" will be defined as:





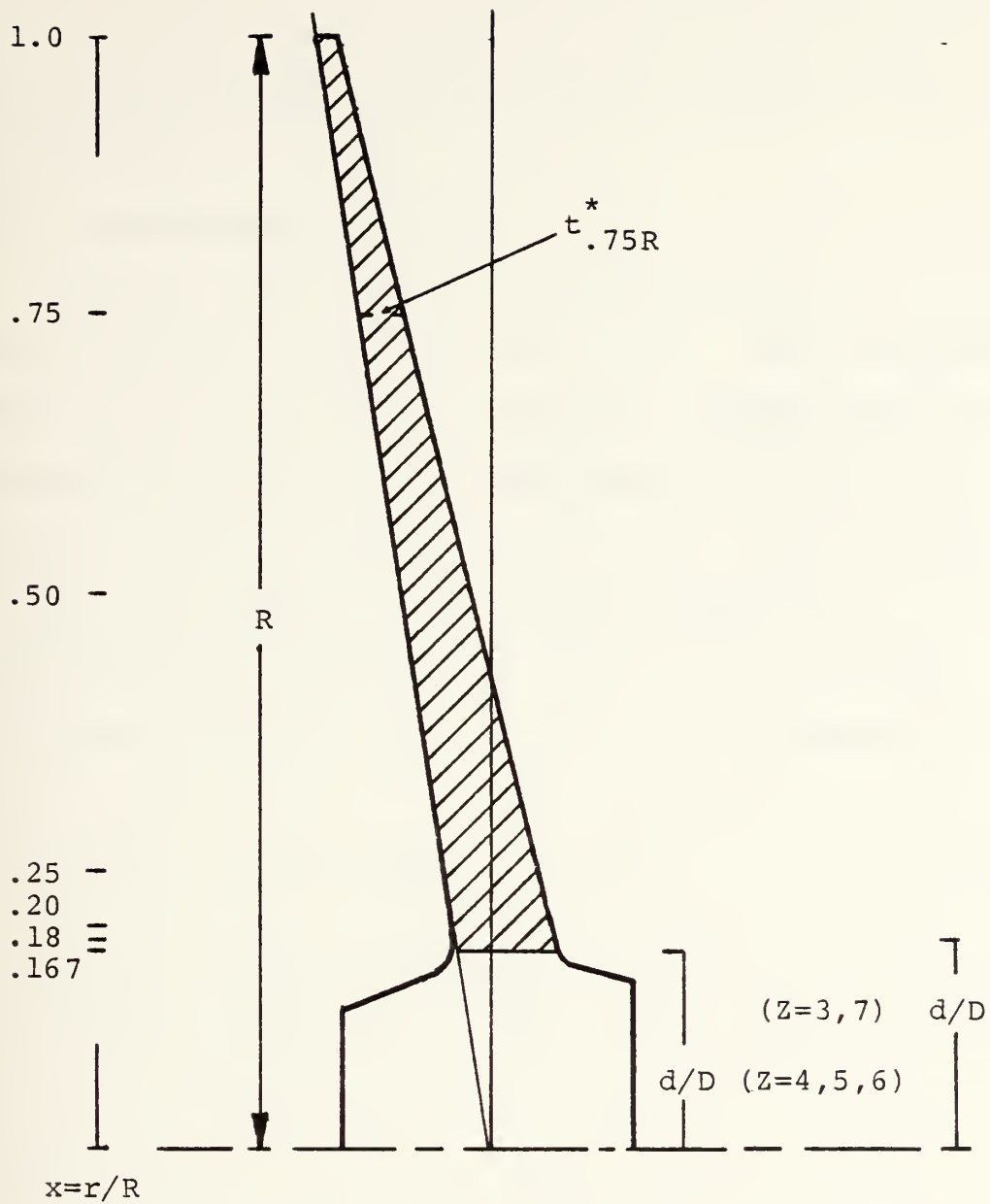


Figure 5.1 Propeller Blade & Hub--Side View



$$x = \frac{r}{R} \quad (5.1)$$

Limits are now expressed as  $x = .167$  or  $.18$  to  $x = 1.0$ .

Quite obviously,  $R = (D/2.0)$ .

## 2. Blade Section Profile

Figure (5.2) depicts an expanded cylindrical blade section in profile view at a given  $r$  or  $x$ . For the Wageningen B-Screw Series, the profile is defined, geometrically, by a succession of vertical ordinates which specify points along the blade section's profile on the "face" ( $y_f$ ) and on the back ( $y_b$ ) with respect to the pitch reference line. At any  $r = xR$ , vertical ordinates for "aft" ( $P < 0$ ) and "fwd" ( $P > 0$ ) portions of the blade section are determined by:

$$y_{fa} = V_1(t^* - t_{te}^*) \quad P \leq 0 \quad (5.2)$$

$$y_{ba} = (V_1 + V_2)(t^* - t_{te}^*) + t_{te}^* \quad P \leq 0 \quad (5.3)$$

and

$$y_{ff} = V_1(t^* - t_{le}^*) \quad P > 0 \quad (5.4)$$

$$y_{bf} = (V_1 + V_2)(t^* - t_{le}^*) + t_{le}^* \quad P > 0 \quad (5.5)$$

where:

$V_1, V_2$  = tabulated values depending on  $x$  and  $P$   
(see Tables (2) and (3), Ref. [2]);



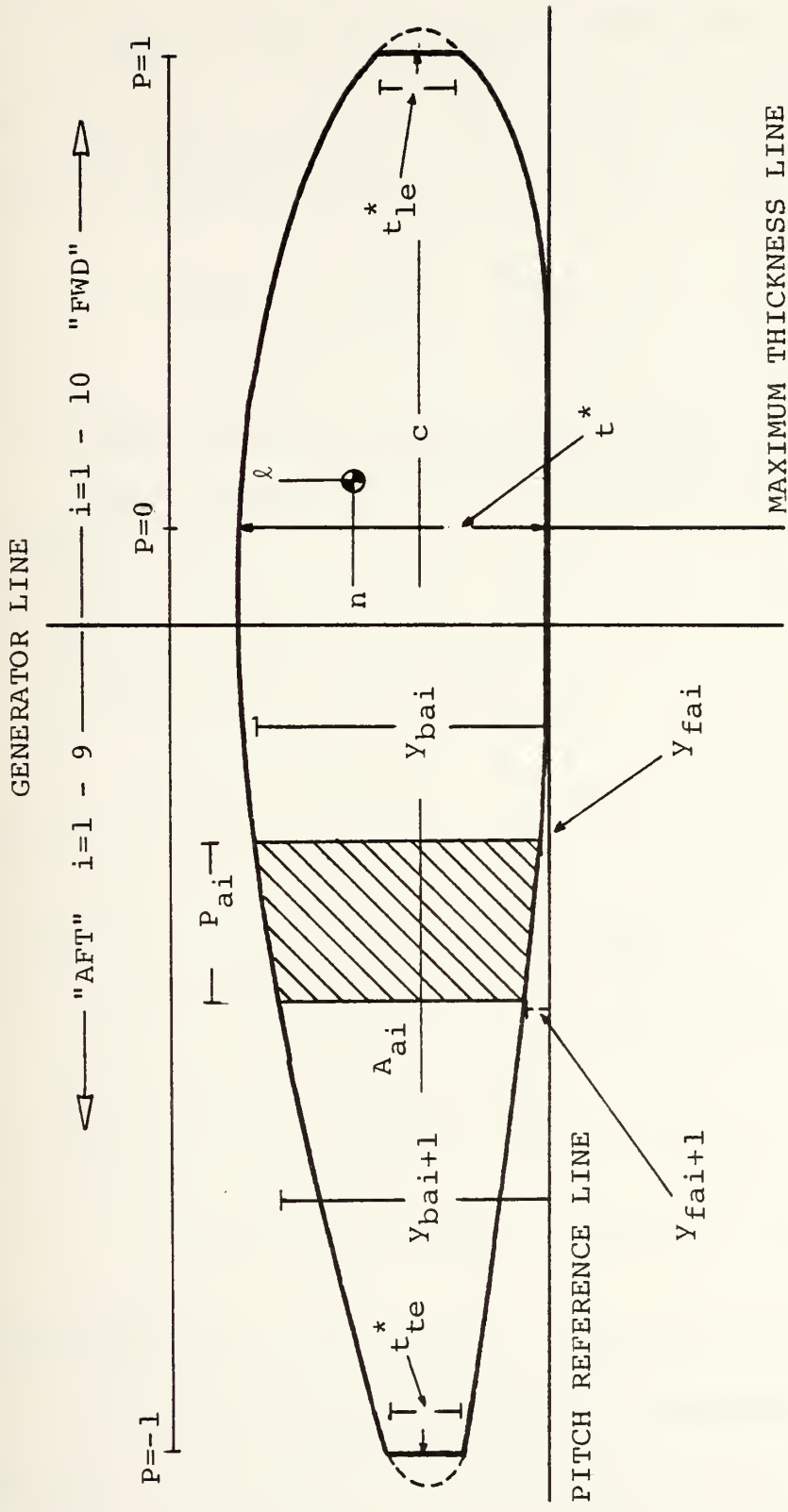


Figure 5.2 Expanded Cylindrical Blade Section--Profile View



$t_{le}^*$  = blade section leading edge thickness (ft);

$t_{te}^*$  = blade section trailing edge thickness (ft).

Units for  $y_{fa}$ ,  $y_{ba}$ ,  $y_{ff}$  and  $y_{bf}$  are feet (ft). For this study, a reasonable assumption is made in that:

$$t_{le}^* = t_{te}^* = \left(\frac{1}{10}\right) t^* \quad (5.6)$$

### 3. Blade Section Cross-Sectional Area

The cross-sectional area at each  $x = r/R$  ( $A(x)$ ) is determined by:

$$A(x) = \sum_{i=1}^9 A_{ai} + \sum_{i=1}^{10} A_{fi} \quad (5.7)$$

where

$$A_{ai} = \Delta P_{ai} \left\{ \frac{h_{ai+1} + h_{ai}}{2} \right\} \quad (5.8)$$

$$h_{ai} = y_{bai} - y_{fai} \quad (5.9)$$

$$h_{ai+1} = y_{bai+1} - y_{fai+1} \quad (5.10)$$

Expressions for  $A_{fi}$ ,  $h_{fi}$  and  $h_{fi+1}$  follow in similar fashion.

Values for  $y_{fai}$  and  $y_{bai}$  are determined at 9 points along the "aft" portion of a given blade section's chord ( $c$ ): values of  $y_{ffi}$  and  $y_{bfi}$  are determined at 10 points along the





"fwd" portion. The values for  $\Delta P_{ai}$  and  $\Delta P_{fi}$  are fractional values of the blade section's chord length (c) at radius  $r = xR$  as determined from Tables (2) and (3) in Reference [2]. The units for  $A(x)$  are square feet ( $\text{ft}^2$ ). Units for  $h_{ai}$ ,  $h_{ai+1}$ ,  $h_{fi}$ ,  $h_{fi+1}$ , c,  $\Delta P_{ai}$  and  $\Delta P_{fi}$  are feet (ft).

#### 4. Volume Integration

The blade volume (bldvol) is finally determined by using Simpson's Rule for integration of  $A(x)$  along the non-dimensional radius  $x$  using appropriate limits.

#### 5. Blade Weight

Once the blade volume (bldvol) is calculated, the weight (bldwt) is determined by:

$$\text{bldwt} = \text{bldvol} \cdot \text{wd} \cdot 1728 \quad (5.11)$$

where:

$\text{bldvol} = \text{volume of one blade (ft}^3\text{)};$

$\text{wd} = \text{material weight density (lbf/in}^3\text{)}.$

Weight Density (wd) depends on blade material (promat).

Table (II) lists appropriate values.

#### C. CODING

SUBROUTINE WGTCAL is the main subprogram for the blade weight calculation. It, in turn, calls the following FORTRAN subprogram for various calculations:



1) SUBROUTINE TDIST--generates, at specified radius values, a distribution of blade section maximum thicknesses ( $t^*$ ).

2) SUBROUTINE BLDPRP--generates, at specified radius values (i.e.,  $r = .167R$  or  $.18R, .2R, .3R, .4R, \dots, .9R, 1.0R$ ), various "blade section properties", one of which is a blade section's cross-sectional area given by equation (5.7). Other properties which are determined (for later use in direct stress computations) include blade section chord lengths and centroids and "critical point" locations as defined in Chapter VI.

3) SUBROUTINE BLDVOL --performs a Simpson's Rule integration for the propeller blade volume (bldvol) using blade section cross-sectional areas generated in SUBROUTINE BLDPRP. The blade weight (bldwt) is computed as a final step in the main subprogram SUBROUTINE WGTCAL.

Examination of the codes in Appendix B reveals extensive use of common blocks for passing data from one subprogram to another. Comment cards provide a full definition of all common blocks as well as a description of the task being performed at various points in a given subprogram.

#### D. SUMMARY

The coding developed for this study is, admittedly, not very compact and efficient. However, the intention has been to write all codes with sufficient documentation in order to facilitate the reader's understanding of the algorithms employed as well as to make the author's debugging work easier.



## VI. THICKNESS-TO-CHORD RATIO--A DESIGN CONSTRAINT

### A. INTRODUCTION

The purpose of this chapter is to examine the development of an algorithm that will be used to determine the minimum required equivalent blade section maximum thickness-to-chord ratio used in equation (4.8). The formulation is based upon the method developed by Dr. Karl E. Schoenherr [Ref. 8] in 1963. After a review of the past and present methods employed in propeller strength analysis is conducted, a description of the Schoenherr model and a list of the assumptions used with that model is presented. Then, a brief restatement of his model's equations which are used in the algorithm is followed by a derivation of the author's modifications to the Schoenherr method. The chapter is completed by conducting a review of the theory and coding employed by the algorithm.

The principal reference which is cited throughout this chapter is, again, Reference [8]. The reader is encouraged to review this reference for further details.

### B. PROPELLER STRENGTH ANALYSIS--A HISTORICAL REVIEW

Marine propeller blades present a special class of structural problem. That problem lies in the difficulty of describing a blade design in simple mathematical terms for subsequent analysis through various means. Until the "finite element method era", analytical methods, including the one



[Ref. 8] adapted for this study, relied heavily on practical experience of the propeller designer and semi-theoretical considerations. Analysis by these methods provided a criterion of stress rather than actual computation of stresses. These methods for predicting blade stresses were developed by using "beam" theory or "shell" theory.

The use of elementary beam theory in propeller strength analysis was first adopted by Taylor [Ref. 22]. He treated a blade as a cantilever beam attached to the propeller hub and loaded by thrust and torque forces distributed linearly over the propeller radius. His approach is often deemed a "modified beam theory" because he chose to calculate the direct stresses using the moment of inertia properties of expanded cylindrical blade sections with neutral axis parallel to the nose-tail (pitch-reference) line or chord of that expanded section. Reasonable estimates of stresses along the blade surface were achieved for the unraked, unskewed and narrow-bladed propellers of his time.

As propellers "modernized" and became skewed and wider with increasing rake (mostly aft), modifications, improvements and alternatives to Taylor's theory were developed. Principally, modifications by Rosingh [Ref. 23] and Hancock [Ref. 24] proposed using moment of inertia properties of a blade section that was normal to the generating line of the axially projected blade outline. Romson [Ref. 25] later improved Taylor's theory for application to wide-bladed





propellers. Morgan [Ref. 26] provided an improved method for calculating the geometric properties of "modern" airfoil-shaped blade sections. Aernoldus and Keyser's [Ref. 27] "quasi-static" modeling of the propeller blade allowed for additional consideration to stresses induced by centrifugal loading of the raked and skewed blade. The beam theory approaches to propeller blade stress analysis culminated, for all practical purposes, with Schoenherr's work [Ref. 8] in 1963.

Alternatives to Taylor's beam theory approach, prior to 1963, consisted of the application of "shell" theory to the propeller blade strength problem. This approach was first proposed by Conn [Ref. 28] and subsequently formulated by Cohen [Ref. 29] who modeled the blade as a helicoidal shell with variable thickness and infinite width. "Shell Theory" was utilized again in experimental studies by Connolly [Ref. 30] who, like his predecessors, was also forced into making an assumption about the behavior of the displacements of the blade sections (i.e., constant displacements normal to the constant pitch blade at each fractional radius distance from the hub) beyond usual assumptions of shell theory. Essentially, his experimental results on one specific propeller contradicted the computational values. Attempts at a generalized numerical solution to Connolly's equations appeared in 1963 [Ref. 31] and 1964 [Ref. 32]. In 1968, Atkinson [Ref. 33], compared Connolly's results with currently



adopted cantilever beam methods and found, based on the inconsistency of results, that it was not possible to recommend one method over the other in the blade strength design procedure; another approach was needed.

Commenting on Atkinson's paper at that time, Sontvedt [Ref. 34] pointed out that, in view of these inconsistencies, the only approach to blade strength analysis which did not require very broad assumptions was the finite element technique. Developments in the method at that time were providing a new and powerful tool for structural analysis. The propeller blade was just another application. Genalis [Ref. 35] developed codes for the determination of displacements and stresses in a blade under hydrodynamic loads using the FEM technique and modeling the blade as a shell, a 3-D element mesh of tetrahedrons and rectangular prisms and, finally, a composite of shell and 3-D elements. As an aside, a finite "difference" solution to Connolly's analytical equations was proposed in 1972 [Ref. 36]. In 1973, Atkinson [Ref. 37] reported the application of both hydrodynamic and centrifugal loads to a blade modeled by a thin-shell triangular mesh and a thick-shell parabolic and cubic curved element mesh. The results of the triangular element were considered unsatisfactory. Another use of the thin-shell triangular element was reported by Sontvedt [Ref. 34] in 1974 using the SESAM-69 code [Ref. 38].

The need to model the blade correctly near the hub, where root stresses are usually critical, necessitated the



consideration of 3-D elements in lieu of thick/thin-shell elements [Refs. 39,40]. The use of the 4- and 10-noded tetrahedral element (i.e., TET-4, TET-10 respectively) to construct a blade mesh was conducted by Beek [Refs. 41,42]. He observed that improved accuracy of stress values, achieved by the use of these meshes, were overshadowed somewhat by the extensive storage capacity required for each analysis. Another "natural" improvement, from a geometrical standpoint, appeared in 1978. A general 3-D curved isoparametric element was incorporated in a computer code [Ref. 43] developed by Ma based on his previous formulation work [Ref. 44].

The finite element approach will continue to grow in use in propeller blade strength analysis with each successive improvement made to the basic elements which are used in the mesh generation of the blade. But, until the computer storage problem is resolved to the point where one mesh generation and subsequent stress analysis of one particular blade becomes a minor processing task, the basic analytical techniques will continue to be a meritable "check" [Ref. 45] in the preliminary (or conceptual) phase of propeller selection/design. In this context, the method formulated by Schoenherr and his colleagues twenty years ago is considered for adoption in determining a minimum required blade section maximum thickness-to-chord ratio.



## C. SCHOENHERR'S METHOD

### 1. Background

Schoenherr's method is applicable to preliminary (or, conceptual) propeller selection problems because it employs an assumed thrust and torque force loading distribution for the propeller blade. This assumption is made at this stage of the ship's design because the exact wake velocity distribution at the propeller race is generally not known.

Also, his method is applicable to propellers represented by the Wageningen B-Screw Series because the blades of these propellers meet Schoenherr's criteria for the blade types covered by his formulation. Specifically, B-Screw Series blades have:

- 1) a small constant rake angle of  $15^\circ$  over the entire blade radius;
- 2) a constant pitch distribution over the propeller radius with the exception of the  $Z = 4$  propeller whose pitch is slightly reduced near the hub;
- 3) mild skew;
- 4) linear distribution of blade section maximum thickness over the radius of the blade;
- 5) aerfoil profile qualities where the nose-tail line or the chord of the blade section is approximately parallel to the pitch reference line.





## 2. The Blade Model

Schoenherr models the propeller blade as a cantilever beam with unsymmetrical and variable area cross sections subjected to loading distributions of hydrodynamic and centrifugal forces. The following additional assumptions apply:

1) Flexure theory applies. This subsequently implies the following: a) plane cross sections remain plane under load, b) Hooke's Law is valid, c) the blade material is homogeneous and isotropic, d) fibers are free to extend and contract independently of adjacent fibers, and e) stresses at a point arising from various forces superimpose.

2) Shearing stresses and their effects are neglected. Only the direct stresses on a strength section are taken into account.

3) The strength sections are taken to be the expanded cylindrical blade sections at various radial locations.

4) The neutral axes of a strength section are straight lines passing through the centroid of the expanded cylindrical blade section and are parallel and normal, respectively, to the pitch reference line, and therefore, the chord, at each blade section.

5) Bending Moments are applied in two planes which are mutually perpendicular to each other. One plane is normal to the pitch-reference line (and chord) of the strength section; the other is parallel.

6) The angle between the principal axes of inertia and the neutral axes is zero.



Using this model and assumptions, Schoenherr applies the following formula for the evaluation of the direct fiber stress ( $[\sigma]_o$ ) in a blade section at a radius  $r = r_o$ :

$$[\sigma]_o = \frac{[M]_{no} u_o}{I_{no}} + \frac{[M]_{\ell o} w_o}{I_{\ell o}} + \frac{[F_c]_o}{A(x_o)} \quad (6.1)$$

where:

$[M]_{no}, [M]_{\ell o}$  = resultant bending moments in planes normal ( $[M]_{no}$ ) and parallel ( $[M]_{\ell o}$ ) to the strength section's chord at  $r = r_o$  (ft-lbf);

$[F_c]_o$  = centrifugal force acting normal to the plane of the strength section at  $r = r_o$  and resulting from the centrifugal acceleration of the remaining blade element mass above that strength section (lbf);

$u_o, w_o$  = coordinates of a point on the strength section's periphery with respect to that section's neutral axes system ( $\ell$ - $n$  system) (ft);

$A(x_o)$  = strength section's cross-sectional area (ft<sup>2</sup>);

$I_{\ell o}$  = moment of inertia of the strength section with respect to the " $\ell$ " axis (ft<sup>4</sup>);

$I_{no}$  = moment of inertia of the strength section with respect to the " $n$ " axis (ft<sup>4</sup>);

$x_o$  = non-dimensional radius given by  $x_o = r_o/R$ .

Since equation (6.1) indicates that the direct fiber stress is greatest at points on the periphery of the strength section, Schoenherr selects to examine four "critical points"



on the periphery where the fiber stress is likely to be a maximum. These points are designated as (1), (2), (3) and (4) on Figure (6.1) and are specified by coordinates  $(u_1, w_1)$ ,  $(u_2, w_2)$ ,  $(u_3, w_3)$  and  $(u_4, w_4)$  respectively in the "l-n" reference system.

The values for  $[M]_{no}$  and  $[M]_{lo}$  are determined by the following relation:

$$[M]_{no} = [M_P]_{no} + [M_{cb}]_{no} \quad (6.2)$$

$$[M]_{lo} = [M_P]_{lo} + [M_{cb}]_{lo} \quad (6.3)$$

where:

$[M_P]_{no}$  = total bending moment due to hydrodynamic loading acting in a plane normal to a strength section's chord at  $r = r_o$  (ftlbft);

$[M_P]_{lo}$  = total bending moment due to hydrodynamic loading acting in a plane parallel to a strength section's chord at  $r = r_o$  (ftlbft);

$[M_{cb}]_{no}$  = total bending moment due to centrifugal loading acting in a plane normal to a strength section's chord at  $r = r_o$  (ftlbft);

$[M_{cb}]_{lo}$  = total bending moment due to centrifugal loading acting in a plane parallel to a strength section's chord at  $r = r_o$  (ftlbft).

### 3. Bending Moments Due to Hydrodynamic Loading

The derivations of  $[M_P]_{no}$  and  $[M_P]_{lo}$  follow directly from Part I of Schoenherr's paper [Ref. 8: p. 83-89] and,





Figure 6.1 Strength Section at  $r = r_o$





therefore, only key equations will be restated. References to equations which contain no decimal point apply to equations as numbered in his paper.

Thrust and torque force are components of the hydrodynamic "lifting" force acting on a blade. Using an assumed non-linear distribution of thrust along the blade radius given by equation (2), Schoenherr derives the following expression for the bending moment due to thrust ( $[M_t]_o$ ) which acts at a blade section located at radius  $r = r_o$ :

$$[M_t]_o = \frac{TR}{Z} \cdot \frac{\phi_2(x_o)}{\phi_1(x_h)} \quad (6.4)$$

where:

- $T$  = propeller thrust (lbf);
- $R$  = propeller radius (ft);
- $Z$  = no. of blades;
- $\phi_2(x_o), \phi_1(x_h)$  = functions of non-dimensional radius  $x$  evaluated at  $x_o = r_o/R$  and  $x_h = 0.2$  and given by equations (4) and (9).

For the bending moment due to torque ( $[M_q]_o$ ) which acts at a blade section located at radius  $r = r_o$ , Schoenherr derives the following:

$$[M_q]_o = \frac{Q_P}{Z} \cdot \frac{\psi_2(x_o)}{\phi_1(x_h)} \quad (6.5)$$

where:



- $Q_P$  = propeller torque (ft-lbf);  
 $Z$  = no. of blades;  
 $\psi_2(x_o), \phi_1(x_h)$  = functions of non-dimensional radius  $x$   
 evaluated at  $x_o = r_o/R$  and  $x_h = 0.2$   
 and given by equations (19) and (4).

Figure (6.2) depicts the component resolution for  $[M_P]_{no}$  and  $[M_P]_{\ell o}$  which results when equations (6.4) and (6.5) are imposed at a strength section at  $r = r_o$  which has a pitch angle  $\beta_o$ . The following relations are derived as equations (42) and (43) in Schoenherr's paper:

$$[M_P]_{no} = [M_t]_o \cos \beta_o + [M_q]_o \sin \beta_o \quad (6.6)$$

$$[M_P]_{\ell o} = [M_t]_o \sin \beta_o - [M_q]_o \cos \beta_o \quad (6.7)$$

#### 4. Force and Bending Moments Due to Centrifugal Loading

The derivation and expressions contained in this section constitute the author's modifications to the formulation in Part II of Schoenherr's paper. In Part II, Schoenherr's derivations for  $[M_{cb}]_{no}$  and  $[M_{cb}]_{\ell o}$  are formulated for computation using a propeller drawing. This follows from the fact that his method, which was funded by the American Bureau of Shipping, was intended to be used as that classification society's "designer's check" on adherence to the Bureau's strength criteria from a propeller blueprint. To evaluate the direct fiber stresses from equation (6.1) for the Wageningen B-Screw Series in accordance with Schoenherr's



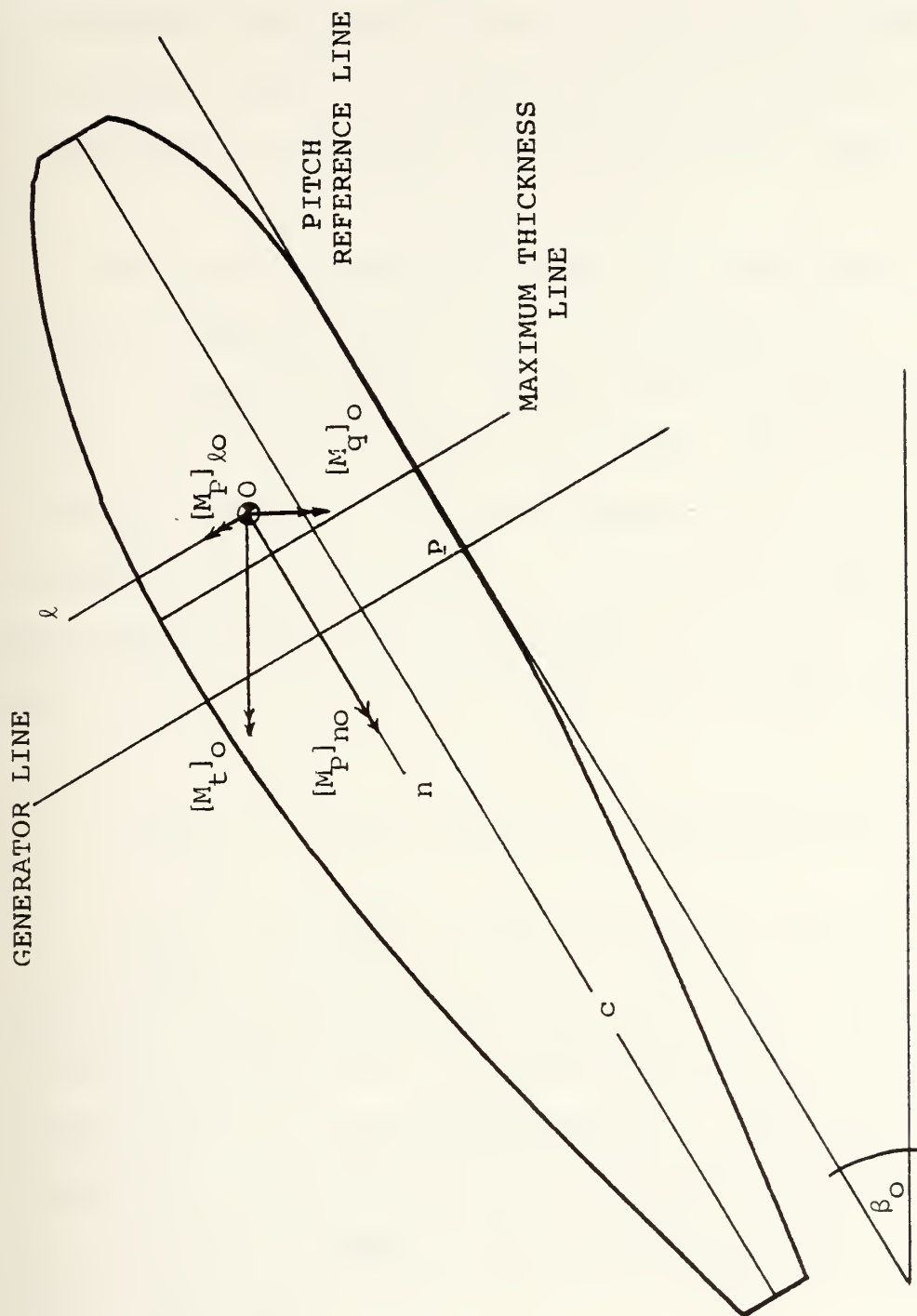


Figure 6.2 Bending Moments due to Thrust and Torque



method,  $[M_{cb}]_{n0}$  and  $[M_{cb}]_{l0}$  must be evaluated from expressions derived from the available information on blade section profiles and other geometric characteristics which are contained in Reference [2] and previously used in Chapter V.

Consider Figure (6.3) where the centrifugal force  $[C_F]_O$  of a blade element above a blade section at  $x = x_O = r_O/R$  acts in a radial direction from the shaft centerline. Its line of action passes through point "N", which is on the same cylindrical surface as the blade section at  $x = x_O = r_O/R$ , and through point "G", which is the center of gravity of the blade element above the blade section at  $x = x_O = r_O/R$ . From the figure, the following expression is derived:

$$[C_F]_O = [C_F]_O \cos \zeta(x_O) + [C_F]_O \sin \zeta(x_O) \quad (6.8)$$

$[C_F]_O$  is shifted to point "N" and is decomposed into components  $[C_F]_O \cos \zeta(x_O)$  and  $[C_F]_O \sin \zeta(x_O)$ . The entire cylindrical surface in which the blade section at  $x = x_O = r_O/R$  and point "N" lie is now expanded into a flat plane for further consideration (see Figure (6.4)). In this configuration,  $[C_F]_O \cos \zeta(x_O)$  is normal to this flat plane while  $[C_F]_O \sin \zeta(x_O)$  lies in this plane.

Let point "O" be the location of the blade section's neutral axes system (i.e., the  $l$ - $n$  system). Then, the forces and moments due to the centrifugal reaction of the blade element above this section act at point "O" and are given by:

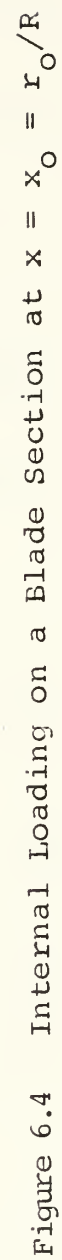






Figure 6.3 Center of gravity "G" and intersection point "N"







$$[F_c]_O = [C_F]_O \cos \zeta(x_O) \quad (6.9)$$

$$[D_c]_O = [C_F]_O \sin \zeta(x_O) \quad (6.10)$$

$$[M_{cb}]_O = [F_c]_O \cdot \overline{NO} \quad (6.11)$$

$$[M_{cw}]_O = [D_c]_O \cdot \overline{LO} \quad (6.12)$$

where:

$[F_c]_O$  = direct force, due to centrifugal action, acting on the blade section located at  $x = x_O = r_O/R$ ;

$[D_c]_O$  = shear force, due to centrifugal action, acting on the blade section located at  $x = x_O = r_O/R$ ;

$[M_{cb}]_O$  = bending moment at point "O" imposed by  $[F_c]_O$  acting through point "N";

$[M_{cw}]_O$  = torsional moment at point "O" imposed by  $[D_c]_O$  acting through point "N".

Since Schoenherr's method does not consider shear forces and their effects,  $[D_c]_O$  and  $[M_{cw}]_O$  will not be considered in this modification. However,  $[F_c]_O$  and  $[M_{cb}]_O$  must now be computed for each blade section along the propeller's radius in order to account for their contributions to equations (6.2), (6.3) and, finally, in equation (6.1).

The computation is derived as follows. Consider Figure (6.4). Again,  $[F_c]_O$  acts through point "N" and is normal (outward) to the plane of the figure.  $[M_{cb}]_O$  is now resolved into components of the  $l$ - $n$  axes system as follows:



$$[M_{cb}]_{no} = [C_F]_o \cos \zeta(x_o) \{p_o \sin \beta_o + q_o \cos \beta_o + Y_{co}\} \quad (6.13)$$

$$[M_{cb}]_{lo} = [C_F]_o \cos \zeta(x_o) \{q_o \sin \beta_o - p_o \cos \beta_o + X_{co}\} \quad (6.14)$$

where:

$\beta_o$  = pitch angle of the blade section at  
 $x = x_o = r_o/R$ ;

$X_{co}$  = distance of the blade sections centroid from  
the generator line (ft);

$Y_{co}$  = distance of the blade section's centroid from  
the pitch reference line (ft);

$q_o$  = distance to point "N" from point "P"  
parallel to the shaft axis at  $x = x_o = r_o/R$   
(ft);

$p_o$  = distance to point "N" from point "P"  
perpendicular to the shaft axis at  
 $x = x_o = r_o/R$  (ft).

The quantity  $\beta_o$  is found by the relation:

$$\tan \beta_o = \frac{1}{\pi} \cdot (P/D)_o \quad (6.15)$$

For the Wageningen B-Screw Series,  $(P/D)_o$  is a constant along R  
except for propellers with  $Z = 4$ .

The quantity  $[C_F]_o$  is computed from the relation:

$$[C_F]_o = 1728 \cdot \frac{wd V_o}{acg} \cdot (2\pi n_P)^2 \cdot R \cdot (\bar{x}_G)_o \quad (6.16)$$

where:





$$\begin{aligned}
V_o &= \text{volume of the blade element above the blade} \\
&\quad \text{section at } x = x_o = r_o/R \text{ (ft}^3\text{)}; \\
acg &= 32.174 \text{ ft/sec}^2; \\
wd &= \text{material weight density (lbf/in}^3\text{)}; \\
n_p &= \text{propeller revolution rate (rps)}; \\
(\bar{x}_G)_o &= \text{non-dimensional radial position of "G"} \\
&\quad \text{for the blade element above the shaft} \\
&\quad \text{axis}; \\
R &= \text{propeller radius (ft)}.
\end{aligned}$$

At this point, only five quantities remain to be determined for the evaluation of the expressions of equations (6.10), (6.13) and (6.14). They are:

- 1)  $\cos \zeta(x_o)$
- 2)  $p_o$
- 3)  $q_o$
- 4)  $(\bar{x}_G)_o$
- 5)  $V_o$

These quantities are determined by integration over the blade element above the blade section, located at  $x = x_o = r_o/R$ , from  $x = x_o$  to  $x = 1.0$ .

The values for  $p_o$  and  $q_o$  will vary with the location of "G" (from Figure (6.3)) which depends on  $x$ . Consider a radially thin slice of the blade element above the blade section, located at  $x = x_o = r_o/R$  (see Figure (6.5)). This thin "slice" is located at a non-dimensional distance  $x$  from the shaft centerline where  $x_o < x < 1.0$ . Figure (6.6) depicts this section expanded onto a plane. Let "g" be the centroid of that "slice". If  $x_g$  is the distance from the generator



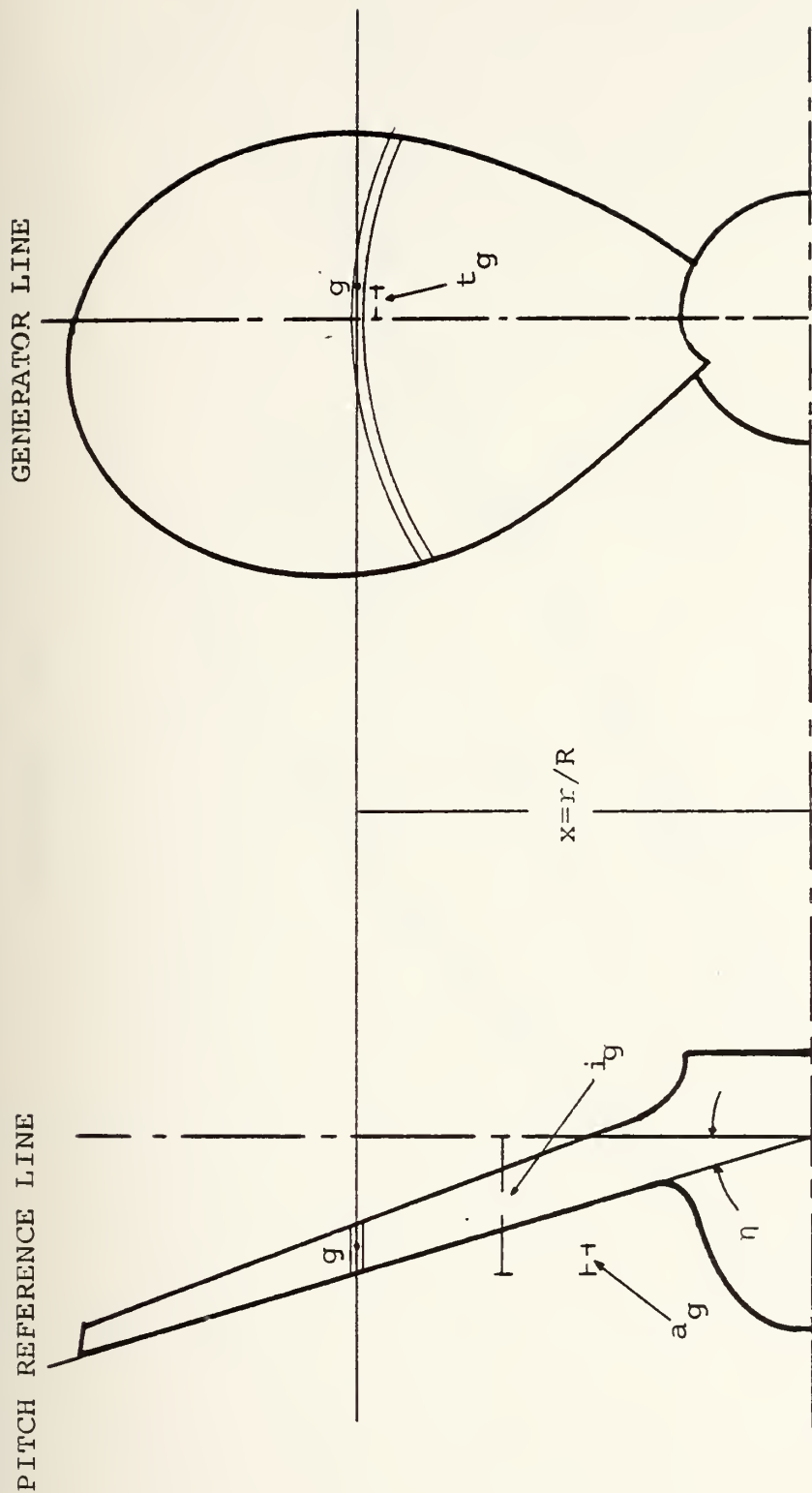


Figure 6.5 Position of "g" of a Blade Section "Slice" at  $x = r/R$



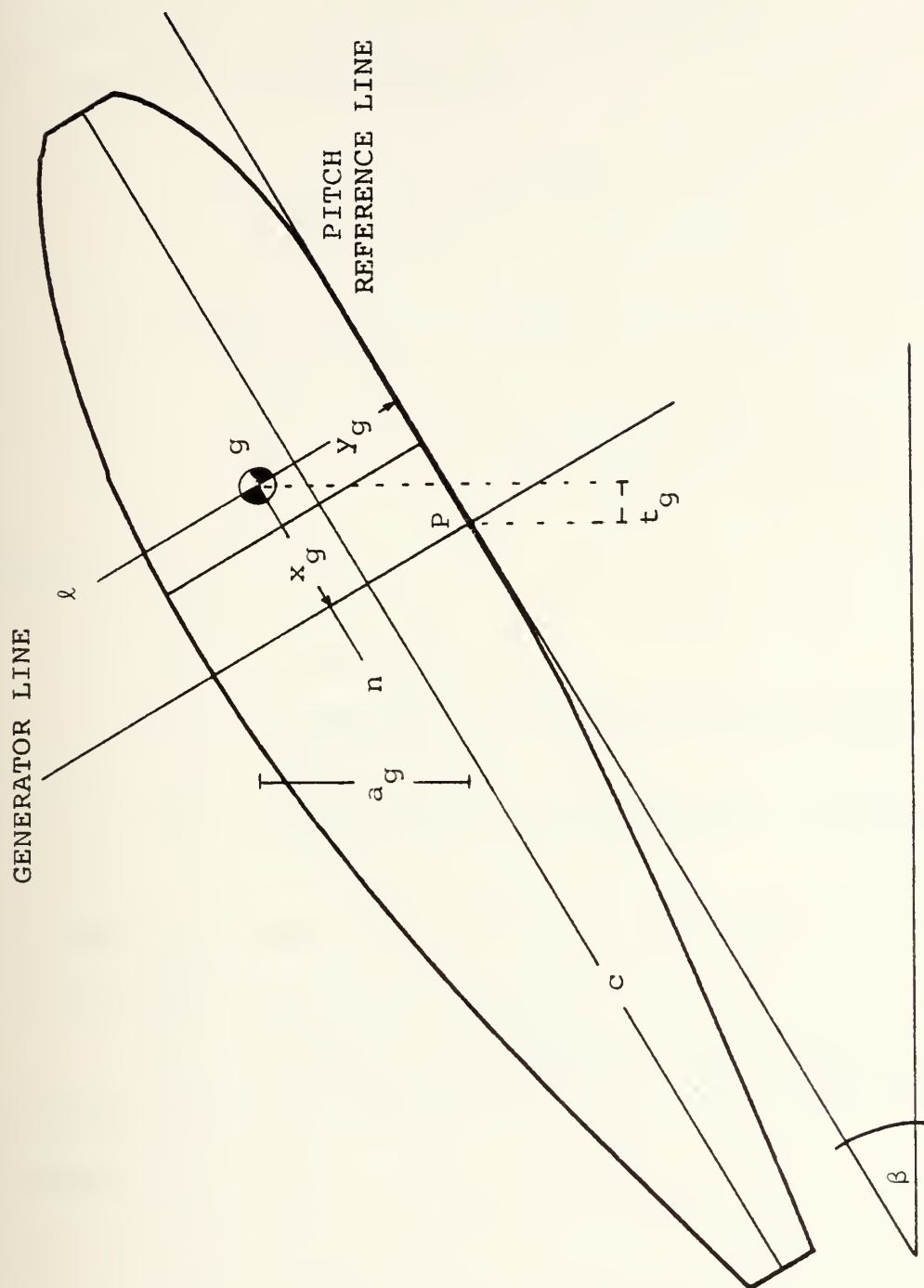


Figure 6.6 Coordinates of centroid "G" of any Blade Section



line to "g" in feet and  $y_g$  is the distance from the pitch reference line in feet, then, from Figures (6.5) and (6.6), the following relations apply:

$$i_g = x R \tan \eta \quad (6.17)$$

$$i_g - a_g = i_g - x_g \sin \beta_g - y_g \cos \beta_g \quad (6.18)$$

$$t_g = x_g \cos \beta_g - y_g \sin \beta_g \quad (6.19)$$

where:

$\eta$  = rake angle at  $x$ ;

$x_g$  = distance of "g" from point "P" parallel to the shaft axis (ft);

$y_g$  = distance of "g" from point "P" perpendicular to the shaft axis (ft).

For the Wageningen B-Screw Series, rake angle  $\eta$  is a constant  $15^\circ$  everywhere along the radius  $R$ .

Now, to compute the volume of the blade element above the blade section located at  $x = x_o = r_o/R$ , the following expression is used:

$$V_o = \int_{x=x_o}^1 RA(x) dx \quad (6.20)$$

To compute the non-dimensional radial position of "G" for the blade element above a blade section located at  $x = x_o = r_o/R$ , the following expression is used:





$$(\bar{x}_G)_O = \frac{\int_{x=x_O}^1 A(x) x dx}{\int_{x=x_O}^1 A(x) dx} \quad (6.21)$$

To compute the tangential position of "G" for the blade element above a blade section located at  $x = x_O = r_O/R$ , the following expression is used:

$$T_O = \frac{\int_{x=x_O}^1 A(x) t_g dx}{\int_{x=x_O}^1 A(x) dx} \quad (6.22)$$

And, finally, to compute the axial position of "G" for the blade element above a blade section located at  $x = x_O = r_O/R$ , the following expression is used:

$$A_O = \frac{\int_{x_O}^1 A(x) (i_g - a_g) dx}{\int_{x_O}^1 A(x) dx} \quad (6.23)$$

Using the values just determined for  $(\bar{x}_G)_O$ ,  $T_O$  and  $A_O$ , the following expressions are used to evaluate  $p_O$  and  $q_O$ :

$$p_O = \frac{x_O}{(\bar{x}_G)_O} T_O \quad (6.24)$$



$$q_o = A_o - x_o \tan \eta \quad (6.25)$$

The expression for  $\cos \zeta(x_o)$  follows:

$$\cos \zeta(x_o) = \frac{p_o}{x_o R} \quad (6.26)$$

The formulation is now complete. Equations (6.10), (6.13) and (6.14) can now be evaluated for any blade section located at  $x = x_o = r_o/R$ . From here, equations (6.2) and (6.3) are evaluated. Finally, using the results from these equations and equation (6.9), equation (6.1) can be evaluated for the four points, specified by coordinates  $(u_1, w_1)$ ,  $(u_2, w_2)$ ,  $(u_3, w_3)$  and  $(u_4, w_4)$ , at any location  $x = x_o = r_o/R$ .

From the development discussed in Chapter V, the values for  $A(x)$ ,  $x_g$ ,  $y_g$ ,  $I_{no}$  and  $I_{\ell o}$  are readily available.

#### D. ALGORITHM FOR THE CONSTRAINT

##### 1. Theory

Schoenherr's formulation with the modifications just derived can be used in determining the minimum required equivalent blade section maximum thickness-to-chord ratio  $((t^*/c)_{.75R \min})$  for use in the constraint  $G_{11}(\bar{X}) \leq 0$  given by equation (4.8). The procedure employed is as follows:

- 1) assume an initial value for  $(t^*/c)_{.75R \min}$  using equation (3.21);
- 2) increase this value by a small amount;



3) using  $(t^*/c)_{.75R \text{ min}}$  obtained from step (2), generate a distribution of minimum required blade section thicknesses  $(t^*_{\text{min}})$  for blade sections at specified points along the propeller radius, say at  $r = .2R, .3R, .4R, .5R, .6R, .7R, .8R$  and  $.9R$ ;

4) determine all blade section properties to include:  
a) cross-sectional area, b) chord length, c) centroid location, d) moments of inertia with respect to the principal axes system (i.e.,  $\ell$ -n system), e) coordinate values for the four critical points defined in the previous section;

5) compute the hydrodynamic bending moment components  $[M_P]_{no}$  and  $[M_P]_{\ell o}$  at radius locations just specified;

6) compute the values of the centrifugal force  $[F_C]_o$  and the bending moment components  $[M_{cb}]_{no}$  and  $[M_{cb}]_{\ell o}$  acting on blade sections at radius locations just specified;

7) calculate the direct fiber stresses at all four critical points for all radius locations specified in step (3);

8) check the following condition on the calculated fiber stress at all four critical points at all specified radius locations using:

$$[\sigma]_o \leq 144 \cdot S_c \quad (6.27)$$

9) if the maximum allowable stress ( $S_c$ ) for the material is exceeded, then return to step (2) and repeat. Otherwise, proceed to next step.



10) since the minimum required equivalent blade section maximum thickness-to-chord ratio assumed in step (2) has produced blade sections of adequate strength, evaluate the constraint given by equation (4.8).

## 2. Coding Details

The algorithm just outlined is incorporated into the main FORTRAN subprogram SUBROUTINE STRCNK. This subprogram, in turn, executes the algorithm through sequential calls to other key FORTRAN subprograms. These subprograms are listed as follows:

1) SUBROUTINE TDIST--accomplishes step (3); generates, at specified radius values, a distribution of minimum required blade section maximum thicknesses ( $t_{\min}^*$ ) for the assumed value of  $(t^*/c)_{.75R \min}$ ;

2) SUBROUTINE BLDPRP--accomplishes step (4); described previously in Chapter V;

3) SUBROUTINE HYDLD--accomplishes step (5); computes the hydrodynamic bending moment components, given by equations (6.6) and (6.7), at specified radius locations;

4) SUBROUTINE CNFGLD--accomplishes step (6); computes the centrifugal force and bending moments, given by equations (6.9), (6.13) and (6.14) respectively, at specified radius locations;

5) SUBROUTINE SIGNDS--accomplishes step (7); computes direct fiber stresses, given by equation (6.1), for all four critical points at every specified radius location.





During the remaining steps of SUBROUTINE STRCNK, the condition on allowable stress, given by equation (6.27), is checked at all critical points of blade sections located at specified radius locations (again,  $r = .2R, .3R, .4R, .5R, .6R, .7R, .8R$  and  $.9R$ ). The final calculation made is that for the constraint given by equation (4.8).

Again, extensive use of common blocks, for passing data from one subprogram to another, is apparent upon examination of the codes just cited. Comment cards are used throughout.

#### E. SUMMARY

The end of this chapter marks the completion of all prerequisite background and formulation discussions on the application of COPES/CONMIN to propeller selection problems involving the Wageningen B-Screw Series. From this point, each specific Design Case can now be solved as an optimization problem.



## VII. DESIGN CASE NO. 1--PROGRAMMING AND COMPARISONS

### A. INTRODUCTION

In this chapter, COPES/CONMIN is used in the solution of propeller selection problems which use the "thrust" approach. First, the thrust approach to the propeller selection problem is formulated. Then, a review of a previous author's solution to this problem is presented. Four variations to this propeller selection problem are solved by COPES/CONMIN. The chapter is completed with a presentation and discussion of the results from the four variations.

### B. THRUST APPROACH FORMULATION

#### 1. Design Vector $\bar{X}$

As previously pointed out at the conclusion of Chapter III, Design Case No. 1 constitutes a propeller selection problem which is solved by the thrust approach. In this approach, the effective horsepower ( $P_E$ ) and the ship's speed ( $V$ ) are specified by the designer. From the viewpoint of optimization, the quantities  $P_E$  and  $V$  become preassigned parameters. This reduces the design vector  $\bar{X}$  (see Figure 4.1) to:

$$\bar{X} = \left\{ \begin{array}{c} D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \\ N_P \\ Q_S \end{array} \right\} \quad (7.1)$$



Having specified  $P_E$  and  $V$ , all of the design variables, as listed in equation (7.1), are not independent. Recalling equations (3.3) and (3.13), the following relationship results:

$$P_E = \frac{(1-t_d)}{(1-wt)} \cdot \eta_R \cdot \eta_o \cdot \frac{2\pi Q_S}{550} \cdot \frac{N_P}{60} \quad (7.2)$$

Rearranging terms, this equation becomes:

$$\eta_o Q_S N_P = \frac{(1-wt)}{(1-t_d)} \cdot \frac{P_E}{\eta_R} \cdot \frac{550 \cdot 60}{2\pi} \quad (7.3)$$

Considering that the open water efficiency ( $\eta_o$ ) is evaluated prior to the computation of ( $\eta_o$ ), or  $OBJ_{1,2}$ , then both  $N_P$  and  $Q_S$  are not independent design variables. One must be selected as the independent design variable. Then, the other variable becomes dependent on the one just selected.

For this study,  $N_P$  is selected as the independent design variable. This choice will reduce the design vector  $\bar{X}$  for propeller selection problems using the thrust approach to the following:

$$\bar{X} = \left\{ \begin{array}{c} D \\ P/D \\ A_E/A_O \\ (t^*/c) \cdot .75R \\ N_P \end{array} \right\} \quad (7.4)$$



Finally, equation (3.8) implies an alternative definition of  $\bar{X}$  as given in equation (7.4). The design vector for Design Case No. 1 propeller selection problems is, therefore, defined as:

$$\bar{X1} = \left\{ \begin{array}{c} D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \\ J \end{array} \right\} \quad (7.5)$$

## 2. Powering Constraint

Having determined the design vector  $\bar{X1}$ , a final restriction to the general propeller selection problem, as stated by equation (4.24), remains for consideration. This restriction constitutes the remaining constraint  $G_{12}(\bar{X})$  mentioned in Chapter IV.

Simply stated, the selected propeller, as defined by  $\bar{X1}$ , must develop enough thrust (T) so that the powering requirement, specified by  $P_E$  and V, is met. Using equation (3.2), the thrust developed by the propeller can be specified in terms of thrust horsepower ( $P_T$ ) as:

$$(P_T)_{dev} = \frac{T V(1-wt)}{550} \quad (7.6)$$

From equation (3.9), it follows that:

$$(P_T)_{dev} = \frac{\rho n_P^2 D^4 K_T}{550} \cdot V(1-wt) \quad (7.7)$$





Using equation (3.12), the developed thrust horsepower can be defined in terms of developed effective horsepower given by:

$$(P_E)_{dev} = \frac{(1-td)}{(1-wt)} \cdot (P_T)_{dev} \quad (7.8)$$

The restriction imposed by the thrust approach method, where  $P_E$  and  $V$  are specified, can now be stated as:

$$P_E \leq (P_E)_{dev} \quad (7.9)$$

Rearranging equation (7.9), the constraint  $G_{12}(\overline{X1})$  follows:

$$G_{12}(\overline{X1}) = 1 - \frac{(P_E)_{dev}}{P_E} \leq 0 \quad (7.10)$$

With the design vector  $\overline{X1}$  and  $G_{12}(\overline{X1})$  defined, the propeller selection problem represented by Design Case No. 1 can be stated under one equation as:

$$\begin{aligned} \text{Maximize:} \quad & F(\overline{X1}) = \text{OBJ}_{1,2} \\ \text{Subject to:} \quad & G_j(\overline{X1}) \leq 0 \quad j = 1, \dots, 12 \\ & x1_i^{\text{lower}} \leq x1_i \leq x1_i^{\text{upper}} \quad i = 1, \dots, 5 \end{aligned} \quad (7.11)$$

### C. PREVIOUS SOLUTIONS

Triantafyllou [Refs. 3,21] considered a propeller selection problem represented by Design Case No. 1. In his example problem, the following parameters were specified:



- 1)  $v = 1.139 \times 10^{-6} \text{ (m}^2\text{/sec)} = 1.22613 \times 10^{-5} \text{ (ft}^2\text{/sec)}$
- 2)  $wt = .22$
- 3)  $td = .19$
- 4)  $\eta_R = 1.025$
- 5)  $noscrw = 1$
- 6)  $Z = 5$
- 7)  $P_E = 18153 \text{ (hp)}$
- 8)  $V = 24 \text{ (knots)}$
- 9)  $D = 22.0 \text{ (ft)}$
- 10)  $A_E/A_O = .85$

The hull under study in his example had the following dimensions:

- 1) Length = 710 (ft)
- 2) Draft = 30 (ft)
- 3) Beam = 100 (ft)

For his analysis, the design vector contained two variables and was specified as:

$$\overline{XT} = \begin{pmatrix} J \\ P/D \end{pmatrix}$$

Using an iterative scheme [Ref. 21: p. 79] to solve two equations in two unknowns, he maximized the open water efficiency ( $\eta_O$ ) to obtain the following results:

$$P/D = 1.1651$$

$$N_P = 104 \text{ (rpm)}$$



$$P_D = 25544 \text{ (hp)}$$

$$\eta_o = .6676$$

For future comparisons, equations (3.8) and (3.3) give:

$$J = .8286$$

$$Q_S = 1290000.0 \text{ (ft-lbf)}$$

Triantafyllou's results are summarized in Table (IV).

#### D. SOLUTIONS BY COPES/CONMIN

The propeller selection problem, as stated by equation (7.11), is now solved by COPES/CONMIN. Four solution variations are considered.

The first and second variations attempt to reproduce the solution given by Triantafyllou. The design vector  $\overline{XT}$  (NDV = 2) is used in both cases. One variation uses SUBROUTINE STRCNA to evaluate the constraint  $G_{12}(\overline{XT})$  given by equation (4.8). The other uses SUBROUTINE STRCNK to determine  $G_{12}(\overline{XT})$ .

The remaining two variations will solve the propeller selection problem using the design vector  $\overline{XI}$  (NDV = 5) defined in equation (7.5). Again, one variation uses SUBROUTINE STRCNA; the other, SUBROUTINE STRCNK.

In all variations, the following parameters are used:

- 1) Temp = 59 (°F)
- 2)  $\rho = 1.9384 \text{ (lbf-sec}^2/\text{ft}^4)$
- 3)  $v = 1.2285 \cdot 10 \text{ (ft}^2/\text{sec)}$



- 4)  $P_{\text{watvap}} = .247$  (psia)
- 5)  $P_{\text{atm}} = 14.7$  (psia)
- 6)  $wt = .22$
- 7)  $td = .19$
- 8)  $\eta_R = 1.025$
- 9)  $\text{noscrrw} = 1$
- 10)  $h_{c_l} = 19.0$  (ft)
- 11)  $D_{\text{lim}} = 22.0$  (ft)
- 12)  $Z = 5$
- 13)  $\text{promat} = 5$  (stainless steel; see Table (II))
- 14)  $P_E = 18153$  (hp)
- 15)  $V = 24.0$  (knots)

All of the above are initialized in the input phase ( $\text{ICALC} = 1$ ) of each SUBROUTINE ANALIZ pertaining to each variation.

The constraint  $G_{12}(\overline{X_I})$  or  $G_{12}(\overline{X_T})$  is evaluated by SUBROUTINE BLPOW1 which appears in the execution section of each SUBROUTINE ANALIZ.

## 1. Variation 1

### a. Programming Details

Since this variation uses the design vector  $\overline{X_T}$ , the following design variables of  $\overline{X_I}$  become parameters and are specified in the input section of SUBROUTINE ANALIZ ( $\text{ICALC} = 1$ ) as:

- 1)  $D = 22.0$  (ft)
- 2)  $A_E/A_O = .85$
- 3)  $(t^*/c)_{.75R} = .0348$  (from equation 3.21).





For constraints, the following are used:

$$G_j(\overline{XT}) \leq 0 \quad j = 1, \dots, 8, 12$$

Only nine of twelve constraints are evaluated (NCON = 9).

Obviously, some of the twelve constraints are redundant since

$D$ ,  $A_E/A_O$ , and  $(t^*/c)_{.75R}$  have been specified.

The upper ( $XT_i^{\text{upper}}$ ) and lower ( $XT_i^{\text{lower}}$ ) limits on the design variables  $J$  and  $P/D$  are set to be:

$$.01 \leq J \leq 1.6$$

$$.4 \leq P/D \leq 1.4$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VLB. The initial value for each design variable ( $XT_i$ ) is also assigned on card image F under the field labeled X. The first list of card images in Appendix D lists all of the COPES control cards used for this variation and variation 2. These cards also specify the locations of the design variables in the common block GLOBCEM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

An examination of SUBROUTINE ANALIZ for this variation, found in Appendix C, shows the calling statement made to SUBROUTINE STRCNA.



## b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed first in Appendix E. Results for this variation of the propeller selection problem are tabulated in Table (IV).

## 2. Variation 2

### a. Programming Details

Everything discussed above for the first variation applies here with one exception. An examination of SUBROUTINE ANALIZ for the second variation, found in Appendix C, shows the calling statement made to SUBROUTINE STRCNK.

## b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed second in Appendix E. Results for this variation of the propeller selection problem are tabulated in Table (IV).

## 3. Variation 3

### a. Programming Details

This variation uses the design vector  $\overline{X1}$ . For constraints, the following are used:

$$G_j(\overline{X1}) \leq 0 \quad j = 1, \dots, 12$$

All twelve constraints are evaluated (NCON = 12).

The upper ( $X1_i^{\text{upper}}$ ) and lower ( $X1_i^{\text{lower}}$ ) limits on the design variables  $D$ ,  $P/D$ ,  $A_E/A_O$ ,  $(t^*/c)_{.75R}$ , and  $J$  are set as:



$$1.0 \leq D \leq 50.0 \quad (\text{ft})$$

$$.4 \leq P/D \leq 1.4$$

$$.2 \leq A_E/A_O \leq 1.1$$

$$.003 \leq (t^*/c)_{.75R} \leq .50$$

$$.01 \leq J \leq 1.6$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VLB. The initial value for each design variable ( $X1_i$ ) is also assigned on card image F under the field labeled X. The second list of card images in Appendix D lists all of the COPES control cards used for this variation and variation 4. These cards also specify the locations of the design variables in the common block GLOBCM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

An examination of SUBROUTINE ANALIZ for this variation, found in Appendix C, shows the calling statement made to SUBROUTINE STRCNA.

#### b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed third in Appendix E. Results for this variation of the propeller selection problem are tabulated in Table (IV).



#### 4. Variation 4

##### a. Programming Details

Everything discussed above for the third variation applies here with one exception. An examination of SUBROUTINE ANALIZ for the fourth variation, found in Appendix C, shows the calling statement made to SUBROUTINE STRCNK instead of SUBROUTINE STRCNA.

##### b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed last in Appendix E. Results for this variation of the propeller selection problem are tabulated in Table (IV).

#### E. DISCUSSION

Overall, the results achieved in all variations compare reasonably well to the solution obtained by Triantafyllou. However, the following points can be made.

Variations 1 and 2 give the same results. This was expected in view of the fact that, even though constraints  $G_9(\overline{XT})$  through  $G_{11}(\overline{XT})$  were evaluated, these constraints were not considered in the optimization search conducted by CONMIN.

In variations 3 and 4, the diameter (D) was driven to the limit ( $D_{lim}$ ). This bears out a fundamental rule in propeller design, i.e., the larger the propeller diameter (D), the greater the open water efficiency ( $\eta_o$ ).

The minimum required equivalent blade section maximum thickness-to-chord ratio ( $(t^*/c)_{.75R_{min}}$ ), computed in





variation 3, is substantially smaller than the one computed for variation 4. As pointed out in Reference [2], the empirical relation, expressed by equation (4.9) and derived from equation (70.x) [Ref. 46: p. 620], does not take into account the effects of centrifugal loading. These effects include, specifically, the direct stresses imposed by the inertia load of the blade and the bending moments which result from rake and skew of the blade. Therefore, the algorithm developed in Chapter VI should, and does, produce a larger value for  $(t^*/c)_{.75R}$ .

A final observation on the results concerns the values of the open water efficiency. The "optimum" open water efficiency ( $\eta_o$ ) achieved by Triantafyllou is lower than those achieved in variations 1 and 2. A possible reason for this might be the neglect of the term " $dRe/dJ$ " in Triantafyllou's formulation of the analytical expressions [Ref. 21: p. 71] that he used in his analysis. The difference in the open water efficiencies subsequently accounts for the differences in the propeller revolution rate ( $N_p$ ) and the delivered torque ( $Q_s$ ) when the relation in equation (3.3) is considered.



TABLE IV

Design Case No. 1--Results

GROUP	ITEM	TRIANTA- FYLLLOU	1	2	3	4
Given	$P_E$	18153.0	18153.0	18153.0	18153.0	18153.0
	V	24.0	24.0	24.0	24.0	24.0
Design Variable Speci- fied	D	22.0	22.0	22.0		
	$A_E/A_O$	.85	.85	.85		
	$(t^*/c)$ .75R		.0348	.0348		
Design Variables	D				21.9991	21.9659
	P/D	1.1651	1.0036	1.0036	.9981	1.0071
	$A_E/A_O$				.8205	.8149
	$(t^*/c)$ .75R				.0330	.0642
	J	.8286	.7371	.7371	.7343	.7394
Maximize	$\eta_O$	.6676	.7091	.7091	.7109	.7066
Restric- tions	$D_{lim}$		(22.0)	(22.0)	22.0	22.0
	$A_E/A_{Omin}$		(0.5258)	(0.5258)	.5269	.5267
	$(t^*/c)$ .75Rmin		(0.21266)	(0.50761)	.021998	.053259
Other	$N_P$	104	116.9	116.9	117.4	116.7
	$Q_S$	1290000	1080451.	1080451	1064574	1084003
	$P_D$	25544.0	24051.3	24051.3	24030.3	24094.8



## VIII. DESIGN CASE NO. 2--PROGRAMMING AND COMPARISONS

### A. INTRODUCTION

In this chapter, COPES/CONMIN is used in the solution of propeller selection problems which use the "power" approach. First, the power approach to the propeller selection problem is formulated. Then, a review of a previous author's solution to this problem is presented. Four variations to this propeller selection problem are solved by COPES/CONMIN. The chapter is completed with a presentation and discussion of the results from the four variations.

### B. POWER APPROACH FORMULATION

#### 1. Design Vector $\bar{X}_2$

As previously pointed out at the conclusion of Chapter III, Design Case No. 2 constitutes a propeller selection problem which is solved by the power approach. In this approach, the delivered torque ( $Q_S$ ) and the propeller revolution rate ( $N_P$ ) are specified by the designer. From the viewpoint of optimization, the quantities  $Q_S$  and  $N_P$  become pre-assigned parameters. This reduces the design vector  $\bar{X}$  (see Figure (4.1)) to:

$$\bar{X} = \left\{ \begin{array}{c} P_E \\ V \\ D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \end{array} \right\} \quad (8.1)$$



Having specified  $Q_S$  and  $N_P$ , all of the design variables, as listed in equation (8.1), are not independent. Recalling equations (3.3), (3.11) and (3.13), the following relationship results:

$$P_E = \frac{(1-t_d)}{(1-wt)} \cdot \eta_R \cdot \frac{J K_T}{2\pi K_Q} \cdot \frac{2\pi Q_S}{550} \cdot \frac{N_P}{60} \quad (8.2)$$

Rearranging terms, this equation becomes:

$$\frac{P_E}{V} = \frac{(1-t_d)}{(1-wt)} \cdot \eta_R \cdot \frac{(1-wt)}{n_P D} \cdot \frac{K_T}{K_Q} \cdot \frac{Q_S}{550} \cdot \frac{N_P}{60} \quad (8.3)$$

Considering the relations for  $K_T$  and  $K_Q$  in equation (3.17), then both  $P_E$  and  $V$  are not independent design variables. One must be selected as independent, while the other becomes dependent on the one selected.

For this study,  $V$  is selected as the independent design variable. This choice will reduce the design vector  $\bar{X}$  for propeller selection problems using the power approach to the following:

$$\bar{X} = \left\{ \begin{array}{c} V \\ D \\ P/D \\ A_E/A_O \\ (t^*/c)_{.75R} \end{array} \right\} \quad (8.4)$$

Finally, equation (3.8) implies an alternative definition of  $\bar{X}$  as given in equation (8.4). The design vector for





Design Case No. 2 propeller selection problems is, therefore, defined as:

$$\overline{X_2} = \left\{ \begin{array}{c} V \\ J \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \end{array} \right\} \quad (8.5)$$

## 2. Powering Constraint

Having determined the design vector  $\overline{X_2}$ , a final restriction to the general propeller selection problem, as stated by equation (4.24), remains for consideration. This restriction constitutes the remaining constraint  $G_{12}(\overline{X})$  mentioned in Chapter IV.

Simply stated, the selected propeller, as defined by  $X_2$ , must absorb at least all of the power delivered to it ( $P_D$ ) which is specified in terms of  $Q_S$  and  $N_P$ . Using equation (3.3), the power absorbed by the propeller can be specified in terms of delivered horsepower ( $P_D$ ) as:

$$(P_D)_{\text{absorb}} = \frac{2\pi Q_P}{550} \cdot \frac{N_P}{60} \quad (8.6)$$

From equation (3.10), it follows that:

$$(P_D)_{\text{absorb}} = \frac{K_Q \rho n_P^2 D^5}{550} \cdot \frac{2\pi N_P}{60} \quad (8.7)$$

But, equation (3.3) also defines the power delivered to the propeller as:



$$P_D = \frac{2\pi Q_S}{550} \cdot \frac{N_P}{60} \quad (8.8)$$

The restriction imposed by the power approach method, where  $Q_S$  and  $N_P$  are specified, can now be stated as:

$$P_D \leq (P_D)_{\text{absorb}} \quad (8.9)$$

Rearranging equation (8.9), the constraint  $G_{12}(\overline{X2})$  follows:

$$G_{12}(\overline{X2}) = 1 - \frac{(P_D)_{\text{absorb}}}{P_D} \leq 0 \quad (8.10)$$

Further simplification of equation (8.10) gives:

$$G_{12}(\overline{X2}) = 1 - \frac{Q_P}{Q_S} \leq 0 \quad (8.11)$$

With the design vector  $\overline{X2}$  and  $G_{12}(\overline{X2})$  defined, the propeller selection problem represented by Design Case No. 2 can be stated under one equation as:

$$\begin{aligned} \text{Minimize: } F(\overline{X2}) &= \text{OBJ}_{1,2} \\ \text{Subject to: } G_j(\overline{X2}) &\leq 0 \quad j = 1, \dots, 12 \\ x2_i^{\text{lower}} &\leq x2_i \leq x2_i^{\text{upper}} \quad i = 1, \dots, 5 \end{aligned} \quad (8.12)$$

### C. PREVIOUS SOLUTIONS

Markussen [Ref. 4] considered a propeller selection problem represented by Design Case No. 2. In his example problem,



the following parameters were specified:

- 1) Temp = 18 (°C) = 64.4 (°F)
- 2)  $P_{\text{watvap}} = 0.0206411$  (bars) = .29943921 (psia)
- 3)  $P_{\text{atm}} = 1.01312856$  (bars) = 14.6974 (psia)
- 4) noscrw = 1
- 5)  $h_{\text{cl}} = 6.7$  (meters) = 21.9827 (ft)
- 6) Z = 6
- 7)  $P_D = 18.9$  (MegaWatts) = 25344.9 (hp)
- 8)  $N_P = 110$  (rpm)
- 9)  $V_A = 15.65$  (knots)

For his analysis, the design vector contained three variables and was specified as:

$$\overline{\text{XM}} = \begin{Bmatrix} J \\ P/D \\ A_E/A_O \end{Bmatrix}$$

A restriction for the minimum required expanded area ratio  $((A_E/A_O)_{\text{min}})$ , given by equation (4.3), was also considered. This imposed a constraint given by equation (4.4).

Using an iterative scheme [Ref. 4: p. 110] to solve three equations in three unknowns, Markussen maximized the open water efficiency ( $\eta_O$ ) to obtain the following results:

$$J = .61095$$

$$P/D = .864380$$

$$A_E/A_O = 36.1861/40.6123 \text{ (m}^2/\text{m}^2\text{)}$$



$$= .891012$$

$$\eta_o = .654391$$

For future comparisons, equations (3.8) and (3.3) give:

$$D = 7.19091 \text{ (meters)} = 23.593375 \text{ (ft)}$$

$$Q_S = 1210130.0 \text{ (ft-lbf)}$$

Markussen's results are summarized in Table (V).

#### D. SOLUTIONS BY COPES/CONMIN

The propeller selection problem, as stated by equation (8.12), is now solved by COPES/CONMIN. Four solution variations are considered.

The first and second variations attempt to reproduce the solution given by Markussen. The design vector  $\overline{XM}$  (NDV = 3) is used in both cases. One variation uses SUBROUTINE STRCNA to evaluate the constraint  $G_{12}(\overline{XM})$  given by equation (4.8). The other uses SUBROUTINE STRCNK to determine  $G_{12}(\overline{XM})$ .

The remaining two variations will solve the propeller selection problem using the design vector  $\overline{X2}$  (NDV = 5) defined in equation (8.5). Again, one variation uses SUBROUTINE STRCNA; the other, SUBROUTINE STRCNK.

In all variations, the following parameters are used:

- 1) Temp = 64.4 (°F)
- 2)  $\rho = 1.9892 \text{ (lbf-sec}^2/\text{ft}^4)$
- 3)  $\nu = 1.1900 \times 10^{-5} \text{ (ft}^2/\text{sec)}$
- 4)  $p_{\text{watvap}} = .2994 \text{ (psia)}$





- 5)  $p_{\text{atm}} = 14.697$  (psia)
- 6)  $wt = .22$
- 7)  $td = .19$
- 8)  $\eta_R = 1.025$
- 9)  $\text{noscw} = 1$
- 10)  $h_{c\ell} = 21.9827$  (ft)
- 11)  $D_{\text{lim}} = 30.0$  (ft)
- 12)  $z = 6$
- 13)  $\text{promat} = 5$  (stainless steel; see Table (II))
- 14)  $Q_S = 1210130$  (ft-lbf)
- 15)  $N_P = 110$  (rpm)

All of the above are initialized in the input phase (ICALC = 1) of each SUBROUTINE ANALIZ pertaining to each variation.

The constraint  $G_{12}(\overline{X2})$  or  $G_{12}(\overline{XM})$  is evaluated by SUBROUTINE BLPOW2 which appears in the execution section of each SUBROUTINE ANALIZ.

# 1. Variation 1

## a. Programming Details

Since this variation uses the design vector  $\overline{XM}$ , the following design variables of  $\overline{X2}$  become parameters and are specified in the input section of SUBROUTINE ANALIZ (ICALC = 1). The ship's speed (V) is specified as:

$$\begin{aligned}
 V &= V_A / (1 - wt) \\
 &= 15.65 / (1 - .22) \\
 &= 20.0641 \text{ (knots)}
 \end{aligned}$$



Markussen elected to use the standard Wageningen blade section maximum thickness-to-chord ratios. Since the equivalent  $t/c$  is given as a function of  $Z$  and  $A_E/A_O$  (see equation (3.21)), then  $(t^*/c)_{.75R}$  is calculated during each analysis ( $ICALC = 2$ ) by the following relation:

$$(t^*/c)_{.75R} = (t/c)_{.75R}$$

For constraints, the following are used:

$$G_j(\overline{XT}) \leq 0 \quad j = 1, \dots, 8, 10, 12$$

Only ten of twelve constraints are considered ( $NCON = 10$ ). Constraints  $G_9(\overline{XM})$  and  $G_{11}(\overline{XM})$  are redundant since no limit on the propeller diameter ( $D_{lim}$ ) appears as a parameter in Markussen's formulation and  $(t^*/c)_{.75R}$  was taken to be the Wageningen standard.

The upper ( $XM_i^{upper}$ ) and lower ( $XM_i^{lower}$ ) limits on the design variables  $J$ ,  $P/D$  and  $A_E/A_O$  are set to be:

$$.01 \leq J \leq 1.1$$

$$.4 \leq P/D \leq 1.4$$

$$.4 \leq A_E/A_O \leq 1.1$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VIB. The initial value for each design variable ( $XM_i$ ) is



also assigned on card image F under the field labeled X. The first list of card images in Appendix G lists all of the COPES control cards used for this variation and variation 2. These cards also specify the locations of the design variables in the common block GLOBCM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

An examination of SUBROUTINE ANALIZ for this variation, found in Appendix F, shows the calling statement made to SUBROUTINE STRCNA.

#### b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed first in Appendix H. Results for this variation of the propeller selection problem are tabulated in Table (V).

### 2. Variation 2

#### a. Programming Details

Everything discussed above for the first variation applies here with one exception. An examination of SUBROUTINE ANALIZ for the second variation, found in Appendix F, shows the calling statement made to SUBROUTINE STRCNK instead of SUBROUTINE STRCNA.

#### b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed second in Appendix H.



Results for this variation of the propeller selection problem are tabulated in Table (V).

### 3. Variation 3

#### a. Programming Details

This variation uses the design vector  $\overline{X2}$ . For constraints, the following are used:

$$G_j(\overline{X2}) \leq 0 \quad j = 1, \dots, 12$$

All twelve constraints are evaluated (NCON = 12).

The upper ( $X2_i^{\text{upper}}$ ) and lower ( $X2_i^{\text{lower}}$ ) limits on the design variables  $V$ ,  $P/D$ ,  $A_E/A_O$ ,  $(t^*/c)_{.75R}$ , and  $J$  are set as:

$$10.0 \leq V \leq 100.0 \quad (\text{ft/sec})$$

$$.4 \leq P/D \leq 1.4$$

$$.4 \leq A_E/A_O \leq 1.1$$

$$.003 \leq (t^*/c)_{.75R} \leq .50$$

$$.01 \leq J \leq 1.1$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VLB. The initial value for each design variable ( $X2_i$ ) is also assigned on card image F under the field labeled X. The second list of card images in Appendix G lists all of the COPES control cards used for this variation and variation 4. These cards also specify the locations of the design variables





in the common block GLOBCM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

An examination of SUBROUTINE ANALIZ for this variation, found in Appendix F, shows the calling statement made to SUBROUTINE STRCNA.

#### b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed third in Appendix H. Results for this variation of the propeller selection problem are tabulated in Table (V).

### 4. Variation 4

#### a. Programming Details

Everything discussed above for the third variation applies here with one exception. An examination of SUBROUTINE ANALIZ for the fourth variation, found in Appendix F, shows the calling statement made to SUBROUTINE STRCNK instead of SUBROUTINE STRCNA.

#### b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed last in Appendix H. Results for this variation of the propeller selection problem are tabulated in Table (V).

## E. DISCUSSION

The results achieved in variations 1 and 2 compare extremely well to the solution obtained by Markussen. As



pointed out in the discussion in Chapter VII, variations 1 and 2 are expected to give the same results for the vector  $\overline{XM}$ . Obviously, the values obtained for  $J$  and  $P/D$ , as well as those for  $D$ , and  $Rn^*_{.75R}$ , are very close to the values generated in Markussen's example. However, the values for  $A_E/A_O$  are somewhat different. It is interesting to note that the value obtained in variations 1 and 2 (and, for that matter, variations 3 and 4) is, essentially, the limiting value for  $A_E/A_O$ , as given in Table (I), for  $Z = 6$ . Markussen's value for  $A_E/A_O$  (i.e., .891012) exceeds the limit (i.e., .80) in this table.

As pointed out at the end of Chapter VII, the minimum required blade section maximum thickness-to-chord ratio  $((t^*/c)_{.75R_{min}})$ , computed in variations 1 and 3, is substantially smaller than the one computed for variations 2 and 4. Again, the same explanation applies here as well.

The results of variations 3 and 4 differ somewhat from Markussen's results. The reason for this is simply that  $V_A$  (or  $V$ ) has not been specified as a parameter. Consequently, a higher value for the advance ratio ( $J$ ), which corresponds to a higher open water efficiency ( $\eta_o$ ), has been found in the optimization search. This result can be interpreted in the following way. Given:

- 1) a six-bladed Wageningen propeller ( $Z = 6$ ) which is made out of stainless steel (promat = 5);
- 2) a power train delivering 25344.9 (hp) at a rate of 110 (rpm);



3) a hull with a wake fraction (wt) equal to .22, a thrust deduction (td) equal to .19 and a shaft centerline depth ( $h_{cl}$ ) of 21.98 (ft), then, the selected propeller, as defined by  $\overline{X2}$ , can drive this hull at a maximum speed of V when the hull has a maximum resistance given by  $P_E$ .



TABLE V

## Design Case No. 2--Results

GROUP	ITEM	MARKUSSEN	VARIATIONS			
			1	2	3	4
Given	$P_D$	25344.9	25344.9	25344.9	25344.9	25344.9
	$Q_S$	1210130	1210130	1210130	1210130	1210130
	$N_P$	110	110	110	110	110
Design Variable Specified	$V_A$	15.65	15.65	15.65		
	$V$		20.0641	20.0641		
Design Variables	$J$	.61095	.6475	.6475	.9927	.8753
	$V$				30.9138	29.5355
	$V_A$				24.1127	23.0377
	$P/D$	.864380	.9036	.9036	1.1986	1.0308
	$A_E/A_O$	.891012	.8018	.8018	.7946	.7986
	$(t^*/c)_{.75R}$		.0397	.0397	.0499	.0638
Maximize	$\eta_O$	.654391	.6660	.6660	.7643	.7330
Restrictions	$D_{lim}$		(30.0)	(30.0)	30.0	30.0
	$A_E/A_{Omin}$	.574729	.5070	.5070	.4622	.4266
	$(t^*/c)_{.75Rmin}$		(.02729)	(.0647)	.02706	.0638
Other	$D$	23.53	22.25	22.25	22.36	24.23
	$P_E$		(14057.3)	(14057.3)	19945.5	20653.7
	$Rn^*_{.75R}$	$6.478 \times 10^7$	$5 \times 10^7$	$5 \times 10^7$	$5 \times 10^7$	$5 \times 10^7$





## IX. DESIGN CASE NO. 3--PROGRAMMING AND COMPARISONS

### A. INTRODUCTION

In this chapter, COPES/CONMIN is used in the solution of a propeller selection problem where "matching" is desired. First, the "matching" approach to the propeller selection problem is formulated. Then, a review of a previous author's solution is presented. One variation to this propeller selection problem is solved by COPES/CONMIN. The chapter is completed with a presentation and discussion of the results.

### B. "MATCHING" FORMULATION

#### 1. Design Vector $\bar{X}$

Design Case No. 3, the final powering problem considered in this study, constitutes a propeller selection problem solved by the "matching" approach. In this approach, the hull's effective horsepower ( $P_E$ ) and speed ( $V$ ), the delivered torque ( $Q_S$ ) and the propeller revolution rate ( $N_p$ ) are specified by the designer. This reduces the design vector  $\bar{X}$  (see Figure (4.1)) to:

$$\bar{X} = \left\{ \begin{array}{c} D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \end{array} \right\} \quad (9.1)$$

For this study, the design vector  $\bar{X}$  is reduced further by eliminating the propeller diameter ( $D$ ) as a design variable.



That is, D will also be specified by the designer so that the design vector for Design Case No. 3 is defined as:

$$\overline{X3} = \left\{ \begin{array}{c} P/D \\ A_E/A_O \\ (t^*/c)_{.75R} \end{array} \right\} \quad (9.2)$$

## 2. Powering Constraint(s)

Having determined the design vector  $\overline{X3}$ , a final restriction to the general propeller selection problem, as stated by equation (4.24), remains for consideration. This restriction constitutes the remaining constraint  $G_{12}(\overline{X})$  mentioned in Chapter IV as well as an additional constraint.

In the "matching" problem, the selected propeller, as defined by  $\overline{X3}$ , must satisfy two conditions. First, it must develop, as a minimum, the effective horsepower ( $P_E$ ) as imposed by the design specification. Citing the formulation previously derived in Chapter VII, this condition can be stated as:

$$P_E \leq (P_E)_{dev} \quad (9.3)$$

The constraint  $G_{12}(\overline{X3})$  follows accordingly as:

$$G_{12}(\overline{X3}) = 1 - \frac{(P_E)_{dev}}{P_E} \leq 0 \quad (9.4)$$

For the second condition, the selected propeller can only absorb, as a maximum, the delivered power ( $P_D$ ) as



specified by the designer. The formulation is the same as that in Chapter VIII except that the inequality signs are reversed. The condition is stated as:

$$(P_D)_{\text{absorb}} \leq P_D \quad (9.5)$$

In defining a constraint  $G_{13}(\overline{X3})$ , another location, say location 24, in the GLOBCM block (see Table (III)) would be assigned. But, considering the fact that constraint  $G_9(\overline{X})$  will not be used because the propeller diameter (D) is specified, there is no reason why  $G_9(\overline{X3})$  cannot be redefined, for this Design Case only, as:

$$G_9(\overline{X3}) = \frac{(P_D)_{\text{absorb}}}{P_D} - 1 \leq 0 \quad (9.6)$$

Further simplification of equation (9.6) gives:

$$G_9(\overline{X3}) = \frac{Q_P}{Q_S} - 1 \leq 0 \quad (9.7)$$

In reality, the constraints just defined should be equality constraints. The word "match" does infer equality in some sense. However, as previously stated in Chapter II, the version of COPES/CONMIN used in this study does not directly handle equality constraints. But, since CONMIN attempts to minimize constraints in the optimization search, it will be assumed that a "match" can be achieved.



With the design vector  $\overline{X3}$  and the constraints  $G_{12}(\overline{X3})$  and  $G_9(\overline{X3})$  defined, the propeller selection problem represented by Design Case No. 3 can now be stated under one equation as:

$$\begin{aligned} \text{Minimize:} \quad & F(\overline{X3}) = \text{OBJ}_3 \\ \text{Subject to:} \quad & G_j(\overline{X3}) \leq 0 \quad j = 1, \dots, 12 \quad (9.8) \\ & x3_i^{\text{lower}} \leq x_i \leq x3_i^{\text{upper}} \quad i = 1, \dots, 3 \end{aligned}$$

### C. PREVIOUS SOLUTIONS

The propeller selection problem considered by Vassilopoulos [Ref. 18] actually represents a propeller "design" problem using the "power" approach. The example problem which he elected to solve is taken from that posed by the International Towing Tank Conference (ITTC) Propeller Committee. This problem is concerned with the determination of propeller thrust (T), diameter (D) and speed of advance ( $V_A$ ) (or ship speed (V)) for a single-screw cargo ship where:

- 1) power available to the propeller (i.e.,  $P_D$ ) is 30,000 (hp)
- 2)  $Z = 6$
- 3)  $N = 105\text{--}110$  (rpm)
- 4)  $D_{\text{lim}} = 23$  (ft)
- 5)  $h_{c\ell} = 19$  (ft)

The variation of ship speed (V) and of hull effective power ( $P_E$ ), thrust deduction factor (td) and the wake fraction (wt) is also given [Ref. 18: p. 20].





The results from Vassilopoulos' propeller design exercise produced a propeller that is "matched" at the following values:

- 1)  $P_E = 21292.6$  (hp)
- 2)  $V = 24.24$  (knots)
- 3)  $Q_S = 1500606.75$  (ft-lbf)
- 4)  $N_P = 105$  (rpm)
- 5)  $P_D = 30000.0$  (hp)

His propeller "design" was based upon the following specified parameters:

- 1) Temp = 59 (°F)
- 2)  $\rho = 1.9905$  (lbf-sec<sup>2</sup>/ft<sup>4</sup>)
- 3)  $p_{\text{watvap}} = .247$  (psia)
- 4) wt = .22
- 5) td = .1725
- 6) noscrw = 1
- 7)  $h_{cl} = 19.0$  (ft)
- 8) Z = 6
- 9) promat = 5 (stainless teel, see Table (II))
- 10) D = 22.0 (ft)
- 11)  $N_P = 105$  (rpm)
- 12)  $P_D = 30000.0$  (hp)

Using an optimization scheme incorporated in his MVAPDP computer program, Vassilopoulos maximized the open water efficiency ( $\eta_o$ ) and designed a propeller with the following characteristics:

- 1) J = .852
- 2)  $K_T = .242$



- 3)  $K_Q = .0478$
- 4)  $\eta_O = .691$
- 5)  $A_E/A_O = .767$
- 6)  $bldwt = 7617.2 \text{ (lbf)}$

By utilizing both the lifting line and lifting surface methods in his design procedure, Vassilopoulos' MVAPDP program evolved a "constant stress" propeller blade. Consequently, the values for  $(t^*/c)$  and  $P/D$  varied non-linearly along the propeller radius ( $R$ ). According to Vassilopoulos, this resulted in a minimum weight propeller. The values for  $P/D$  and  $(t^*/c)$  are listed in Tables (8) and (10) of his paper. From these values,  $(t^*/c)_{.75R}$  is approximately .040.

While the propeller represented by Vassilopoulos' design is different, in many aspects (rake, skew, blade section aerfoil shape, etc.), from the Wageningen B-Screw Series propeller, it does represent a minimum weight propeller that has been "matched" to specific design values. Appropriate results are summarized in Table (VI).

#### D. SOLUTIONS BY COPES/CONMIN

The propeller selection problem, as stated by equation (9.8), is now solved by COPES/CONMIN. One solution variation is considered. The following parameters are used:

- 1)  $Temp = 59 \text{ (}^\circ\text{F)}$
- 2)  $\rho = 1.9905 \text{ (lbf-sec}^2\text{/ft}^4\text{)}$
- 3)  $\nu = 1.2817 \times 10^{-5} \text{ (ft}^2\text{/sec)}$
- 4)  $p_{watvap} = .247 \text{ (psia)}$



- 5)  $p_{atm} = 14.7$  (psia)
- 6)  $wt = .22$
- 7)  $\eta_R = 1.025$
- 8)  $n_{scrw} = 1$
- 9)  $h_{cl} = 19.0$  (ft)
- 10)  $Z = 6$
- 11)  $promat = 5$  (stainless steel, see Table (II))
- 12)  $D = 22.0$  (ft)

Two problems are examined. Problem 1 specifies the following additional parameters:

- 1)  $td = .1725$
- 2)  $P_E = 21292.6$  (hp)
- 3)  $V = 24.24$  (knots)
- 4)  $Q_S = 1500606.75$  (ft-lbf)
- 5)  $N_P = 105$  (rpm)

Problem 2 specifies the same parameters as:

- 1)  $td = .171$
- 2)  $P_E = 17630.0$  (hp)
- 3)  $V = 23.0$  (knots)
- 4)  $Q_S = 1500606.75$  (ft-lbf)
- 5)  $N_P = 105$  (rpm)

All of the above are initialized in the input section (ICALC = 1) of similar versions of SUBROUTINE ANALIZ. Therefore, only one version is included in Appendix I.

The constraints for  $G_9(\overline{X3})$  and  $G_{12}(\overline{X3})$  are evaluated by SUBROUTINE BLPOW3 which appears in the execution section of



SUBROUTINE ANALIZ. Also, note that SUBROUTINE DICNUA has been deleted from the execution section, while SUBROUTINE WGTAL has been added.

### 1. Programming Details

All twelve constraints are evaluated ( $NCON = 12$ ). The upper ( $X3_i^{upper}$ ) and lower ( $X3_i^{lower}$ ) limits on the design variables  $P/D$ ,  $A_E/A_O$  and  $(t^*/c)_{.75R}$  are set to be:

$$.4 \leq P/D \leq 1.4$$

$$.4 \leq A_E/A_O \leq 1.1$$

$$.003 \leq (t^*/c)_{.75R} \leq .50$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VLB. The initial value for each design variable ( $X3_i$ ) is also assigned on card image F under the field labeled X. The list of card images in Appendix J lists all of the COPES control cards used for both problems. These cards also specify the locations of the design variables in the common block GLOBCM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

### 2. Results

The outputs from the optimization/analysis, performed by COPES/CONMIN, are listed in Appendix K. Results of both problems are tabulated in Table (VI).





## E. DISCUSSION

Table (VI) presents the results of problems 1 and 2 along with relevant information from Vassilopoulos' "design". Problem 1 attempted to "match" a Wageningen propeller at the design point found by Vassilopoulos. The first COPES/CONMIN printout in Appendix K indicates that the "match" was achieved at  $P_E$  equal to 21.168.1 (hp) and  $P_D$  equal to 28,150.0 (hp) (or,  $Q_S = 1500607$  (ft-lbf) and  $N_P = 105$  (rpm)). These values are judged to be close enough to the "Given" values in Table (VI).

It is apparent that the Wageningen propeller does not require all of the 30,000 (hp) of delivered horsepower. The propeller characteristics (i.e.,  $J$ ,  $K_T$ ,  $K_Q$  and  $\eta_O$ ) for problem 1 compare very well to Vassilopoulos' values. The expanded area ratios ( $A_E/A_O$ ) are, also, very similar. Of course, the obvious difference is the blade weight (bldwt). The Wageningen propeller blade is over five thousand pounds heavier. Does this make sense for a minimum blade weight?

The answer is yes.

All one has to do is consider the values of  $(t^*/c)$  for problem 1 and Vassilopoulos' design. Vassilopoulos' "constant stress" blade was designed to "absorb" stress up to the allowable design limit of 5,400 (psi) (for stainless steel) all along the entire propeller radius ( $R$ ). Table (12) in Reference [18] gives further details. The Wageningen propeller blade, however, represents an "older" type of blade which was designed with a linear blade section maximum thickness ( $t^*$ )



distribution. Consequently, it was "overdesigned" for strength beyond the  $3/10$ -- $4/10$  radius (i.e.,  $.3R$ -- $.4R$ ) and contains excess material. A heavier blade, therefore, results. Note, also, that the optimizer did not drive the value of  $(t^*/c)_{.75R}$  to the minimum acceptable value,  $(t^*/c)_{.75R \min}$ .

The results of problem 2 show the effect on blade weight (bldwt) for a Wageningen propeller when the hull's powering requirements (i.e.,  $P_E$  at  $V$ ) have been reduced. The weight reduction of 2000 pounds is significant. The complete results are listed in the second COPES/CONMIN printout in Appendix K.



TABLE VI  
Design Case No. 3--Results

GROUP	ITEM	VASSILOPOULOS	PROBLEM	
			1	2
Given	$P_E$	21292.6	21292.6	17630.0
	V	24.24	24.24	23.0
	$Q_S$	1500607	1500607	1500607
	$N_P$	105	105	105
Design Variable Specified	D	22.0	22.0	22.0
Design Variables	P/D	*	1.1813	1.0906
	$A_E/A_O$	.767	.7944	.7742
	$(t^*/c) .75R$	.040	.0794	.0681
Minimize	bldwt	—	12842.6	10464.7
Maximize	$\eta_O$	.691	—	—
Restric- tions	$D_{lim}$	—	—	—
	$A_E/A_{Omin}$	—	.8515	.7722
	$(t^*/c) .75R_{min}$	—	.0691	.0681
Other	J	.852	.8290	.7866
	$K_T$	.242	.2349	.2063
	$K_Q$	.0478	.0448	.0372
	$\eta_O$	—	.6915	.6950
	bldwt	7617.2	—	—

\* P/D varies with R



## X. CONCLUSIONS AND RECOMMENDATIONS

### A. CONCLUSIONS

The general purpose non-linear optimizer/synthesizer COPES/CONMIN has been successfully applied to three typical preliminary ship design propeller selection problems in which the Wageningen B-Screw Series is used. The formulation and programming of each required analysis code (i.e., SUBROUTINE ANALIZ) have been made as general as possible to allow the designer a broad variety of solution options for solving propeller selection problems which can be classified under any of the three Design Cases that were considered. The analysis codes have been "modularized" to the extent that methodical series data from other propeller series, which are available in the polynomial expression format of the B-Screw Series, can be easily adapted for powering analysis utilizing design optimization methods.

Further flexibility in the solution to the propeller selection problem has been achieved by using COPES/CONMIN as the optimizer/synthesizer. The designer has now been afforded the additional capability of specifying the design variables, the objective functions and the constraints of his choice. By solving propeller selection problems in the way presented in this thesis, repetitive problem formulation and coding have been eliminated.

There are other advantages to solving propeller selection problems specifically with COPES/CONMIN which have not been





directly addressed in this study. As stated in Chapter II, COPES/CONMIN is capable of performing optimization analyses, sensitivity studies, optimum sensitivity studies and optimization using approximation techniques. The designer, therefore, can select and perform any of these options, using the same analysis codes which have been presented in this thesis.

While the utilization of a general purpose non-linear optimizer in solving propeller selection problems allows the designer greater flexibility in the selection procedure, there is one important limitation that should be stressed at this point. This concerns the question whether or not the solution vector, determined by the optimizer, is a "global" optimum. As stated in Chapter II, COPES/CONMIN assures that, if a feasible solution vector is found, it is, at least, a "local" minimum (or maximum). This implies that, for two different initial design vectors which are specified in the COPES Control Card deck on card image F, the same optimum solution may not be determined by the optimizer. Both solutions would correspond to minimums (or maximums) of the objective function and are, therefore, correct. But, does one or the other correspond to the minimum (or maximum) of the entire vector design space, i.e., the "global" optimum? For the moment, at least, there is no definitive answer to the question.

Despite this uncertainty, progress in the field of design optimization continues to be made. Current developments [Ref. 47] will soon allow the designer to have a choice in



selecting a specific optimization algorithm from a "library" of proven optimization programs which employ the latest state-of-the-art numerical techniques. Again, using one analysis code, the designer will be able to generate any number of optimized solutions for the problem under study.

## B. RECOMMENDATIONS

For future consideration, it is recommended that the automated design and trade-off capability, provided by a general purpose non-linear optimizer/synthesizer such as COPES/CONMIN, be applied to the more difficult problem of propeller design.

As pointed out in Chapter I, the use of the Wageningen B-Screw Series represents a "selection" procedure rather than a "design" process. Today, analytical propeller design procedures, utilizing lifting line and lifting surface theory, are becoming increasingly popular among propeller designers. The propeller design, which results from the utilization of these analytical methods, is, unquestionably, more efficient than the standard series propeller. However, these methods require consideration of many more design variables in the design process. This appears to be a natural application for the use of a general purpose non-linear optimizer/synthesizer.

Here, an analysis code, much larger than those which have been presented in this study, could be developed which would incorporate the lifting line/lifting surface theory



for the determination of the propeller performance characteristics, the local cavitation numbers and also the calculation of the pressure distributions over the blade. These pressure distributions would be utilized in the strength analysis of the blade. This analysis would utilize the finite element technique on an appropriately generated mesh model of the blade. Having defined the steps for this design procedure in the analysis code, the propeller designer now "couples" his analysis to the optimizer/synthesizer for determination of the optimum design. A massive amount of computer storage would certainly be required, but this concept is feasible and, in the author's view, is worthy of future consideration.

#### C. A FINAL NOTE

In conclusion, this thesis has demonstrated, in effect, another interesting application of the method of design optimization. The author, in no way, wishes to leave the reader with the impression that the techniques of design optimization are the "be all--end all" for engineering analysis. Design optimization techniques are useful and powerful tools that stand to relieve the engineer of the mundane tasks of numerical calculations and subsequent graphic plotting. But, they are just tools. In the final "analysis", good engineering judgment is paramount in their application and use.



# APPENDIX A

## FORTTRAN VARIABLE CROSS REFERENCE LIST

<u>Symbol</u>	<u>Fortran Variable</u>
$A_E/A_O$	AEDVAO
$(A_E/A_O)_{\min}$	AEAOMN
bldwt	WEIGHT
$C_{.75R}$	C75R
D	DIA
$D_{\lim}$	DIALIM
$h_{cl}$	HCL
J	RJ
$K_Q$	KQ
$K_T$	KT
noscrw	NOSCRW
$N_P$	N
$P_E$	PE
$P_D$	PD
P/D	PDIVD
$p_{watvap}$	PWATVA
$p_{atm}$	PATM
promat	PROMAT
$Q_S$	QS
$Rn^*_{.75R}$	R75R
$S_c$	SC
td	TD





# APPENDIX A (CONT.)

<u>Symbol</u>	<u>Fortran Variable</u>	.
Temp	TEMP	
$(t^*/c) .75R$	TC75R	
V (ft/sec)	V	
V (knots)	VK	
wt	WT	
Z	Z	
$\eta_o$	ETAO	
$\eta_R$	ETARR	
$\nu$	WATNU	
$\rho$	WATRO	



## SUBROUTINE LISTINGS

[illegible]







```

5  REAL*4  RJ,C75R,R75R,KT,KQ
1  REAL*4  T00R,T1XR,T20R,T30R,T40R,T50R,T60R,T70R,T75R,T80R,T90R,
1  REAL*4  CR(10),AR(10),PLF(10),PLA(10),IR(10),VLF(10,11),
2  V2F(10,11),VLA(10,10),V2A(10,10),YFACE(11),YBACK(11),
3  FAC(10),Y(11),DELPA(10),AA(10),DLPASM,XA(10),YA(10),SUMAA,
4  SUMAXA,SUMAYA,SUMAX2A,SUMAY2A,HF(11),DELPF(11),AF(11),DLPFSM,
5  XF(11),YF(11),SUMAF,SUMAXF,SUMAYF,SMAX2F,SMAY2F,AREA(10),
1  REAL*4  XMT(10),YPR(10),R1XXNA(10),R1YYNA(10),XCG(10),YCG(10),
1  INTEGER*4  I,IR,IPI
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,EDI,VD,QS,TC75R,V,RJCNL,
1  IRJCNJ,R75RCL,R75RCU,AEAOCL,AEAOCC,TC75CL,TC75CU,PQWBAL,DIACNU,
2  AEACCV,TCSTFS,RJ
COMMON /PAR4M/VK,TD,WI,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1  PROMAT,LIALIM,ETARR,AEADOM,TC75MH,SC
COMMON /THICD/TCOR,T1XR,T20R,T30R,T40R,T50R,T60R,T70R,T75R,T80R,
1  T90R,T1CCR,FAT
COMMON /AREELD/AREA
COMMON /CGX/XCG
COMMON /CGY/YCG
COMMON /CRDINT/CK
COMMON /VAL1/U1
COMMON /VAL2/U2
COMMON /VAL3/U3
COMMON /VAL4/U4
COMMON /VAL1/W1
COMMON /VAL2/W2
COMMON /VAL3/W3
COMMON /VAL4/W4
COMMON /A2MCMX/R1YYNA
COMMON /A2MCMY/R1XXNA

C
C
C  DETERMINE CHORD LENGTHS ALONG BLADE RADII USING TABLE (1), REF 2
IF(.NOT.(Z.EQ.3.0))GO TC 1
CR(1)=(1.5932*AEDVAO*DIA)/Z
GO TO 5
1  CONTINUE
IF(.NOT.(Z.EQ.4.0))GO TO 2
CR(1)=(1.5894*AEDVAO*DIA)/Z
GO TO 5
2  CONTINUE
IF(.NOT.(Z.EQ.5.0))GO TC 3
CR(1)=(1.5894*AEDVAO*DIA)/Z
GO TO 5
3  CONTINUE

```

C  
C  
C





```

IF (.NOT. (Z.EQ.6.0)) GO TO 4
CR(1)=(1.5854*AEDVAU*DIA)/Z
GO TO 5
4 CONTINUE
CR(1)=(1.6180*AEDVAU*DIA)/Z
5 CONTINUE
IF (.NOT. (Z.EQ.3.0)) GO TO 6
CR(2)=(1.6330*AEDVAU*DIA)/Z
CR(3)=(1.8320*AEDVAU*DIA)/Z
CR(4)=(2.0000*AEDVAU*DIA)/Z
CR(5)=(2.1200*AEDVAU*DIA)/Z
CR(6)=(2.1860*AEDVAU*DIA)/Z
CR(7)=(2.1680*AEDVAU*DIA)/Z
CR(8)=(2.1270*AEDVAU*DIA)/Z
CR(9)=(1.6570*AEDVAU*DIA)/Z
CR(10)=(0.0000*AEDVAU*DIA)/Z
GO TO 7
6 CONTINUE
CR(2)=(1.6620*AEDVAU*DIA)/Z
CR(3)=(1.8820*AEDVAU*DIA)/Z
CR(4)=(2.0500*AEDVAU*DIA)/Z
CR(5)=(2.1520*AEDVAU*DIA)/Z
CR(6)=(2.1870*AEDVAU*DIA)/Z
CR(7)=(2.1440*AEDVAU*DIA)/Z
CR(8)=(1.9700*AEDVAU*DIA)/Z
CR(9)=(1.5820*AEDVAU*DIA)/Z
CR(10)=(0.0000*AEDVAU*DIA)/Z
7 CONTINUE

CALCULATE POSITION OF GENERATOR LINE (I.E., AR(I)) AND POSITION OF
MAXIMUM ELASE SECTION THICKNESS (I.E., ER(I)) USING FIGURE (1) AND
TABLE (1), REF 2

IF (.NOT. (Z.EQ.3.0)) GO TO 8
AR(1)=C.617*CR(1)
BR(1)=C.350*CR(1)
GO TO 12
8 CONTINUE
IF (.NOT. (Z.EQ.4.0)) GO TO 9
AR(1)=C.61832*CR(1)
BR(1)=C.350*CR(1)
GO TO 12
9 CONTINUE
IF (.NOT. (Z.EQ.5.0)) GO TO 10
AR(1)=C.61832*CR(1)
BR(1)=C.350*CR(1)
GO TO 12
10 CONTINUE

```

C  
C  
C  
C  
C



```

IF (.NOT. (Z.EQ.6.0)) GO TO 11
  AR(1)=C.61832*CR(1)
  BR(1)=C.350*CR(1)
  GO TO 12
11 CONTINUE
  AR(1)=C.6178*CR(1)
  BR(1)=C.350*CR(1)
  GO TO 13
12 CONTINUE
  IF (.NOT. (Z.EQ.3.0)) GO TO 13
  AR(2)=C.616*CR(2)
  AR(3)=C.611*CR(3)
  AR(4)=C.595*CR(4)
  AR(5)=C.583*CR(5)
  AR(6)=C.558*CR(6)
  AR(7)=C.526*CR(7)
  AR(8)=C.481*CR(8)
  AR(9)=C.400*CR(9)
  AR(10)=0.0
  BR(2)=C.350*CR(2)
  BR(3)=C.350*CR(3)
  BR(4)=C.350*CR(4)
  BR(5)=C.355*CR(5)
  BR(6)=C.385*CR(6)
  BR(7)=C.442*CR(7)
  BR(8)=C.478*CR(8)
  BR(9)=C.500*CR(9)
  BR(10)=0.0
  GO TO 14
13 CONTINUE
  AR(2)=C.617*CR(2)
  AR(3)=C.613*CR(3)
  AR(4)=C.601*CR(4)
  AR(5)=C.586*CR(5)
  AR(6)=C.561*CR(6)
  AR(7)=C.524*CR(7)
  AR(8)=C.463*CR(8)
  AR(9)=C.351*CR(9)
  AR(10)=0.0
  BR(2)=C.350*CR(2)
  BR(3)=C.350*CR(3)
  BR(4)=C.350*CR(4)
  BR(5)=C.355*CR(5)
  BR(6)=C.385*CR(6)
  BR(7)=C.443*CR(7)
  BR(8)=C.475*CR(8)
  BR(9)=C.500*CR(9)
  BR(10)=0.0
  GO TO 14
14 CONTINUE

```

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APP01530
APP01540
APP01550
APP01560
APP01570
APP01580
APP01590
APP02000
APP02010
APP02020
APP02030
APP02040
APP02050
APP02060
APP02070
APP02080
APP02090
APP02100
APP02110
APP02120
APP02130
APP02140
APP02150
APP02160
APP02170
APP02180
APP02190
APP02200
APP02210
APP02220
APP02230
APP02240
APP02250
APP02260
APP02270
APP02280
APP02290
APP02300
APP02310
APP02320
APP02330
APP02340
APP02350
APP02360
APP02370
APP02380
APP02390
APP02400

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 1A(1,1,426)=6.6113  
 1A(1,1,427)=6.6260  
 1A(1,1,428)=6.6413  
 1A(1,1,429)=6.6560  
 1A(1,1,430)=6.6713  
 1A(1,1,431)=6.6860  
 1A(1,1,432)=6.7013  
 1A(1,1,433)=6.7160  
 1A(1,1,434)=6.7313  
 1A(1,1,435)=6.7460  
 1A(1,1,436)=6.7613  
 1A(1,1,437)=6.7760  
 1A(1,1,438)=6.7913  
 1A(1,1,439)=6.8060  
 1A(1,1,440)=6.8213  
 1A(1,1,441)=6.8360  
 1A(1,1,442)=6.8513  
 1A(1,1,443)=6.8660  
 1A(1,1,444)=6.8813  
 1A(1,1,445)=6.8960  
 1A(1,1,446)=6.9113  
 1A(1,1,447)=6.9260  
 1A(1,1,448)=6.9413  
 1A(1,1,449)=6.9560  
 1A(1,1,450)=6.9713  
 1A(1,1,451)=6.9860  
 1A(1,1,452)=7.0013  
 1A(1,1,453)=7.0160  
 1A(1,1,454)=7.0313  
 1A(1,1,455)=7.0460  
 1A(1,1,456)=7.0613  
 1A(1,1,457)=7.0760  
 1A(1,1,458)=7.0913  
 1A(1,1,459)=7.1060  
 1A(1,1,460)=7.1213  
 1A(1,1,461)=7.1360  
 1A(1,1,462)=7.1513  
 1A(1,1,463)=7.1660  
 1A(1,1,464)=7.1813  
 1A(1,1,465)=7.1960  
 1A(1,1,466)=7.2113  
 1A(1,1,467)=7.2260  
 1A(1,1,468)=7.2413  
 1A(1,1,469)=7.2560  
 1A(1,1,470)=7.2713  
 1A(1,1,471)=7.2860  
 1A(1,1,472)=7.3013  
 1A(1,1,473)=7.3160  
 1A(1,1,474)=7.3313  
 1A(1,1,475)=7.3460  
 1A(1,1,476)=7.3613  
 1A(1,1,477)=7.3760  
 1A(1,1,478)=7.3913  
 1A(1,1,479)=7.4060  
 1A(1,1,480)=7.4213  
 1A(1,1,481)=7.4360  
 1A(1,1,482)=7.4513  
 1A(1,1,483)=7.4660  
 1A(1,1,484)=7.4813  
 1A(1,1,485)=7.4960  
 1A(1,1,486)=7.5113  
 1A(1,1,487)=7.5260  
 1A(1,1,488)=7.5413  
 1A(1,1,489)=7.5560  
 1A(1,1,490)=7.5713  
 1A(1,1,491)=7.5860  
 1A(1,1,492)=7.6013  
 1A(1,1,493)=7.6160  
 1A(1,1,494)=7.6313  
 1A(1,1,495)=7.6460  
 1A(1,1,496)=7.6613  
 1A(1,1,497)=7.6760  
 1A(1,1,498)=7.6913  
 1A(1,1,499)=7.7060  
 1A(1,1,500)=7.7213  
 1A(1,1,501)=7.7360  
 1A(1,1,502)=7.7513  
 1A(1,1,503)=7.7660  
 1A(1,1,504)=7.7813  
 1A(1,1,505)=7.7960  
 1A(1,1,506)=7.8113  
 1A(1,1,507)=7.8260  
 1A(1,1,508)=7.8413  
 1A(1,1,509)=7.8560  
 1A(1,1,510)=7.8713  
 1A(1,1,511)=7.8860  
 1A(1,1,512)=7.9013  
 1A(1,1,513)=7.9160  
 1A(1,1,514)=7.9313  
 1A(1,1,515)=7.9460  
 1A(1,1,516)=7.9613  
 1A(1,1,517)=7.9760  
 1A(1,1,518)=7.9913  
 1A(1,1,519)=8.0060  
 1A(1,1,520)=8.0213  
 1A(1,1,521)=8.0360  
 1A(1,1,522)=8.0513  
 1A(1,1,523)=8.0660  
 1A(1,1,524)=8.0813  
 1A(1,1,525)=8.0960  
 1A(1,1,526)=8.1113  
 1A(1,1,527)=8.1260  
 1A(1,1,528)=8.1413  
 1A(1,1,529)=8.1560  
 1A(1,1,530)=8.1713  
 1A(1,1,531)=8.1860  
 1A(1,1,532)=8.2013  
 1A(1,1,533)=8.2160  
 1A(1,1,534)=8.2313  
 1A(1,1,535)=8.2460  
 1A(1,1,536)=8.2613  
 1A(1,1,537)=8.2760  
 1A(1,1,538)=8.2913  
 1A(1,1,539)=8.3060  
 1A(1,1,540)=8.3213  
 1A(1,1,541)=8.3360  
 1A(1,1,542)=8.3513  
 1A(1,1,543)=8.3660  
 1A(1,1,544)=8.3813  
 1A(1,1,545)=8.3960  
 1A(1,1,546)=8.4113  
 1A(1,1,547)=8.4260  
 1A(1,1,548)=8.4413  
 1A(1,1,549)=8.4560  
 1A(1,1,550)=8.4713  
 1A(1,1,551)=8.4860  
 1A(1,1,552)=8.5013  
 1A(1,1,553)=8.5160  
 1A(1,1,554)=8.5313  
 1A(1,1,555)=8.5460  
 1A(1,1,556)=8.5613  
 1A(1,1,557)=8.5760  
 1A(1,1,558)=8.5913  
 1A(1,1,559)=8.6060  
 1A(1,1,560)=8.6213  
 1A(1,1,561)=8.6360  
 1A(1,1,562)=8.6513  
 1A(1,1,563)=8.6660  
 1A(1,1,564)=8.6813  
 1A(1,1,565)=8.6960  
 1A(1,1,566)=8.7113  
 1A(1,1,56





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V1A(1,7)=0.2184
V1A(1,8)=0.2565
V1A(1,9)=0.2764
V1A(1,10)=0.2946
V2F(1,1)=1.0000
V2F(1,2)=0.9754
V2F(1,3)=0.8847
V2F(1,4)=0.8056
V2F(1,5)=0.7167
V2F(1,6)=0.6065
V2F(1,7)=0.4613
V2F(1,8)=0.3751
V2F(1,9)=0.2686
V2F(1,10)=0.1395
V2F(1,11)=0.0000
V2A(1,1)=1.0000
V2A(1,2)=0.9391
V2A(1,3)=0.7868
V2A(1,4)=0.6850
V2A(1,5)=0.5676
V2A(1,6)=0.4372
V2A(1,7)=0.2936
V2A(1,8)=0.1372
V2A(1,9)=0.0576
V2A(1,10)=0.0000

```

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17 CONTINUE

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C
C
C

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... CONTINUE FOR K=.2R

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V1F(2,1)=0.0000
V1F(2,2)=0.0049
V1F(2,3)=0.0304
V1F(2,4)=0.0502
V1F(2,5)=0.0804
V1F(2,6)=0.1180
V1F(2,7)=0.1685
V1F(2,8)=0.2000
V1F(2,9)=0.2353
V1F(2,10)=0.2821
V1F(2,11)=0.3560
V1A(2,1)=0.0000
V1A(2,2)=0.0172
V1A(2,3)=0.0592
V1A(2,4)=0.0880
V1A(2,5)=0.1207
V1A(2,6)=0.1570
V1A(2,7)=0.1967
V1A(2,8)=0.2400

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APP03370
APP03380
APP03390
APP03400
APP03410
APP03420
APP03430
APP03440
APP03450
APP03460
APP03470
APP03480
APP03490
APP03500
APP03510
APP03520
APP03530
APP03540
APP03550
APP03560
APP03570
APP03580
APP03590
APP03600
APP03610
APP03620
APP03630
APP03640
APP03650
APP03660
APP03670
APP03680
APP03690
APP03700
APP03710
APP03720
APP03730
APP03740
APP03750
APP03760
APP03770
APP03780
APP03790
APP03800
APP03810
APP03820
APP03830
APP03840

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```

V1A(2,9)=0.2630
V1A(2,10)=0.2826
V2F(2,1)=1.0000
V2F(2,2)=1.0000
V2F(2,3)=0.3875
V2F(2,4)=0.3170
V2F(2,5)=0.1277
V2F(2,6)=0.6190
V2F(2,7)=0.4777
V2F(2,8)=0.3905
V2F(2,9)=0.2840
V2F(2,10)=0.1560
V2A(2,1)=1.0000
V2A(2,2)=1.0000
V2A(2,3)=0.5446
V2A(2,4)=0.584
V2A(2,5)=0.5995
V2A(2,6)=0.5842
V2A(2,7)=0.5535
V2A(2,8)=0.5060
V2A(2,9)=0.1455
V2A(2,10)=0.0000

```

```

.. CONTINUE FOR R=.3R

```

```

V1F(3,1)=0.0000
V1F(3,2)=0.0027
V1F(3,3)=0.0148
V1F(3,4)=0.0300
V1F(3,5)=0.0503
V1F(3,6)=0.0709
V1F(3,7)=0.1191
V1F(3,8)=0.1445
V1F(3,9)=0.1760
V1F(3,10)=0.2186
V1A(3,1)=0.0000
V1A(3,2)=0.0000
V1A(3,3)=0.0033
V1A(3,4)=0.0202
V1A(3,5)=0.0376
V1A(3,6)=0.0623
V1A(3,7)=0.0943
V1A(3,8)=0.1333
V1A(3,9)=0.1790
V1A(3,10)=0.2040
V2F(3,1)=1.0000

```

CCC

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APP03850
APP03860
APP03870
APP03880
APP03890
APP03900
APP03910
APP03920
APP03930
APP03940
APP03950
APP03960
APP03970
APP03980
APP03990
APP04000
APP04010
APP04020
APP04030
APP04040
APP04050
APP04060
APP04070
APP04080
APP04090
APP04100
APP04110
APP04120
APP04130
APP04140
APP04150
APP04160
APP04170
APP04180
APP04190
APP04200
APP04210
APP04220
APP04230
APP04240
APP04250
APP04260
APP04270
APP04280
APP04290
APP04300
APP04310
APP04320

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V2F(3,2)=C.5750  
V2F(3,3)=C.5920  
V2F(3,4)=C.5315  
V2F(3,5)=C.5520  
V2F(3,6)=C.5505  
V2F(3,7)=C.5130  
V2F(3,8)=C.4265  
V2F(3,9)=C.5197  
V2F(3,10)=C.1890  
V2F(3,11)=C.0000  
V2A(3,1)=C.1.0000  
V2A(3,2)=C.5583  
V2A(3,3)=C.1335  
V2A(3,4)=C.1335  
V2A(3,5)=C.1955  
V2A(3,6)=C.4885  
V2A(3,7)=C.360  
V2A(3,8)=C.1670  
V2A(3,9)=C.1800  
V2A(3,10)=C.0000

C  
C  
C

... CCNTINUE FOR R=.4R

V1F(4,1)=C.000  
V1F(4,2)=C.000  
V1F(4,3)=C.0033  
V1F(4,4)=C.0090  
V1F(4,5)=C.0.189  
V1F(4,6)=C.0.357  
V1F(4,7)=C.0.637  
V1F(4,8)=C.0.833  
V1F(4,9)=C.1088  
V1F(4,10)=C.1467  
V1F(4,11)=C.2181  
V1A(4,1)=C.000  
V1A(4,2)=C.0044  
V1A(4,3)=C.0.116  
V1A(4,4)=C.0.214  
V1A(4,5)=C.0.395  
V1A(4,6)=C.0.630  
V1A(4,7)=C.0.972  
V1A(4,8)=C.1200  
V1A(4,9)=C.1467  
V2F(4,1)=C.000  
V2F(4,2)=C.5725  
V2F(4,3)=C.5933  
V2F(4,4)=C.5345

APP04330  
APP04340  
APP04350  
APP04360  
APP04370  
APP04380  
APP04390  
APP04400  
APP04410  
APP04420  
APP04430  
APP04440  
APP04450  
APP04460  
APP04470  
APP04480  
APP04490  
APP04500  
APP04510  
APP04520  
APP04530  
APP04540  
APP04550  
APP04560  
APP04570  
APP04580  
APP04590  
APP04600  
APP04610  
APP04620  
APP04630  
APP04640  
APP04650  
APP04660  
APP04670  
APP04680  
APP04690  
APP04700  
APP04710  
APP04720  
APP04730  
APP04740  
APP04750  
APP04760  
APP04770  
APP04780  
APP04790  
APP04800



V2 F(4,5)=C.7593  
V2 F(4,6)=C.6590  
V2 F(4,7)=C.5220  
V2 F(4,8)=C.4335  
V2 F(4,9)=C.3235  
V2 F(4,10)=C.1935  
V2 F(4,11)=C.0000  
V2 A(4,1)=C.6645  
V2 A(4,2)=C.5415  
V2 A(4,3)=C.4525  
V2 A(4,4)=C.3553  
V2 A(4,5)=C.2040  
V2 A(4,6)=C.3500  
V2 A(4,7)=C.1810  
V2 A(4,8)=C.0905  
V2 A(4,9)=C.0000  
V2 A(4,10)=C.0000

C  
C  
C

... CCNTINUE FCR R=.5R

V1 F(5,1)=C.0000  
V1 F(5,2)=C.0000  
V1 F(5,3)=C.0000  
V1 F(5,4)=C.0008  
V1 F(5,5)=C.0034  
V1 F(5,6)=C.0085  
V1 F(5,7)=C.0211  
V1 F(5,8)=C.0328  
V1 F(5,9)=C.0500  
V1 F(5,10)=C.0778  
V1 F(5,11)=C.1278  
V1 A(5,1)=C.0000  
V1 A(5,2)=C.0000  
V1 A(5,3)=C.0000  
V1 A(5,4)=C.0012  
V1 A(5,5)=C.0040  
V1 A(5,6)=C.0100  
V1 A(5,7)=C.0190  
V1 A(5,8)=C.0330  
V1 A(5,9)=C.0420  
V1 A(5,10)=C.0522  
V1 A(5,11)=C.0710  
V2 F(5,1)=C.5710  
V2 F(5,2)=C.4880  
V2 F(5,3)=C.4275  
V2 F(5,4)=C.3478  
V2 F(5,5)=C.2430  
V2 F(5,6)=C.1503

APPJ4810  
APPJ4820  
APPJ4830  
APPJ4840  
APPJ4850  
APPJ4860  
APPJ4870  
APPJ4880  
APPJ4890  
APPJ4900  
APPJ4910  
APPJ4920  
APPJ4930  
APPJ4940  
APPJ4950  
APPJ4960  
APPJ4970  
APPJ4980  
APPJ4990  
APPJ5000  
APPJ5010  
APPJ5020  
APPJ5030  
APPJ5040  
APPJ5050  
APPJ5060  
APPJ5070  
APPJ5080  
APPJ5090  
APPJ5100  
APPJ5110  
APPJ5120  
APPJ5130  
APPJ5140  
APPJ5150  
APPJ5160  
APPJ5170  
APPJ5180  
APPJ5190  
APPJ5200  
APPJ5210  
APPJ5220  
APPJ5230  
APPJ5240  
APPJ5250  
APPJ5260  
APPJ5270  
APPJ5280





V2F(5,8)=0.4135  
V2F(5,9)=0.3056  
V2F(5,10)=0.1750  
V2F(5,11)=0.0000  
V2A(5,1)=0.0000  
V2A(5,2)=0.9635  
V2A(5,3)=0.3456  
V2A(5,4)=0.7580  
V2A(5,5)=0.6439  
V2A(5,6)=0.5140  
V2A(5,7)=0.3565  
V2A(5,8)=0.1865  
V2A(5,9)=0.0950  
V2A(5,10)=0.0000

CCC

.. CONTINUE FOR K=.6R

V1F(6,1)=0.0000  
V1F(6,2)=0.0000  
V1F(6,3)=0.0000  
V1F(6,4)=0.0000  
V1F(6,5)=0.0000  
V1F(6,6)=0.0006  
V1F(6,7)=0.0022  
V1F(6,8)=0.0067  
V1F(6,9)=0.0165  
V1F(6,10)=0.0382  
V1A(6,1)=0.0000  
V1A(6,2)=0.0000  
V1A(6,3)=0.0000  
V1A(6,4)=0.0000  
V1A(6,5)=0.0000  
V1A(6,6)=0.0000  
V1A(6,7)=0.0000  
V1A(6,8)=0.0000  
V1A(6,9)=0.0000  
V1A(6,10)=0.0000  
V2F(6,1)=0.0000  
V2F(6,2)=0.3690  
V2F(6,3)=0.3790  
V2F(6,4)=0.3090  
V2F(6,5)=0.2200  
V2F(6,6)=0.4060  
V2F(6,7)=0.4620  
V2F(6,8)=0.3775  
V2F(6,9)=0.2720  
V2F(6,10)=0.1485

APP05290  
APP05300  
APP05310  
APP05320  
APP05330  
APP05340  
APP05350  
APP05360  
APP05370  
APP05380  
APP05390  
APP05400  
APP05410  
APP05420  
APP05430  
APP05440  
APP05450  
APP05460  
APP05470  
APP05480  
APP05490  
APP05500  
APP05510  
APP05520  
APP05530  
APP05540  
APP05550  
APP05560  
APP05570  
APP05580  
APP05590  
APP05600  
APP05610  
APP05620  
APP05630  
APP05640  
APP05650  
APP05660  
APP05670  
APP05680  
APP05690  
APP05700  
APP05710  
APP05720  
APP05730  
APP05740  
APP05750  
APP05760



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V2F(6,11)=0.0000
V2A(6,1)=1.0000
V2A(6,2)=C.5613
V2A(6,3)=C.5426
V2A(6,4)=C.1530
V2A(6,5)=C.6415
V2A(6,6)=C.5110
V2A(6,7)=C.5585
V2A(6,8)=C.1885
V2A(6,9)=C.0965
V2A(6,10)=0.0000
    .. CONTINUE FOR R=.7R
V1F(7,1)=0.0000
V1F(7,2)=0.0000
V1F(7,3)=0.0000
V1F(7,4)=0.0000
V1F(7,5)=C.0000
V1F(7,6)=0.0000
V1F(7,7)=C.0000
V1F(7,8)=0.0000
V1F(7,9)=0.0000
V1F(7,10)=0.0000
V1A(7,1)=0.0000
V1A(7,2)=0.0000
V1A(7,3)=0.0000
V1A(7,4)=0.0000
V1A(7,5)=C.0000
V1A(7,6)=C.0000
V1A(7,7)=C.0000
V1A(7,8)=0.0000
V1A(7,9)=0.0000
V1A(7,10)=0.0000
V2F(7,1)=1.0000
V2F(7,2)=C.5675
V2F(7,3)=C.5660
V2F(7,4)=0.7850
V2F(7,5)=C.6840
V2F(7,6)=0.5615
V2F(7,7)=0.4140
V2F(7,8)=C.3300
V2F(7,9)=0.2337
V2F(7,10)=0.1240
V2A(7,1)=0.0000
V2A(7,2)=1.0000
V2A(7,3)=C.5600

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C  
C  
C

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APP05770
APP05780
APP05790
APP05800
APP05810
APP05820
APP05830
APP05840
APP05850
APP05860
APP05870
APP05880
APP05890
APP05900
APP05910
APP05920
APP05930
APP05940
APP05950
APP05960
APP05970
APP05980
APP05990
APP06000
APP06010
APP06020
APP06030
APP06040
APP06050
APP06060
APP06070
APP06080
APP06090
APP06100
APP06110
APP06120
APP06130
APP06140
APP06150
APP06160
APP06170
APP06180
APP06190
APP06200
APP06210
APP06220
APP06230
APP06240

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V2A(7,3)=C.£400
V2A(7,4)=C.£500
V2A(7,5)=C.£400
V2A(7,6)=C.£100
V2A(7,7)=C.£600
V2A(7,8)=C.£1900
V2A(7,9)=C.£975
V2A(7,10)=C.£0.0000
... CONTINUE FOR R=.8R
V1F(8,1)=0.0000
V1F(8,2)=0.0000
V1F(8,3)=0.0000
V1F(8,4)=0.0000
V1F(8,5)=0.0000
V1F(8,6)=0.0000
V1F(8,7)=0.0000
V1F(8,8)=0.0000
V1F(8,9)=0.0000
V1F(8,10)=0.0000
V1F(8,11)=0.0000
V1A(8,1)=0.0000
V1A(8,2)=0.0000
V1A(8,3)=0.0000
V1A(8,4)=0.0000
V1A(8,5)=0.0000
V1A(8,6)=0.0000
V1A(8,7)=0.0000
V1A(8,8)=0.0000
V1A(8,9)=0.0000
V1A(8,10)=0.0000
V2F(8,1)=1.0000
V2F(8,2)=0.3635
V2F(8,3)=0.3520
V2F(8,4)=0.1635
V2F(8,5)=0.6545
V2F(8,6)=0.3265
V2F(8,7)=0.3765
V2F(8,8)=0.2925
V2F(8,9)=0.2028
V2F(8,10)=0.1050
V2F(8,11)=0.0000
V2A(8,1)=1.0000
V2A(8,2)=0.3600
V2A(8,3)=0.3400
V2A(8,4)=0.1500
V2A(8,5)=0.6400

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C  
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C

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APP06250
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APP06690
APP06700
APP06710
APP06720

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V2A(8,6)=C.5100
V2A(8,7)=C.5600
V2A(8,8)=C.1900
V2A(8,9)=C.5975
V2A(8,10)=0.0000
... CONTINUE FOR R=.9R
V1F(9,1)=0.0000
V1F(9,2)=0.0000
V1F(9,3)=0.0000
V1F(9,4)=0.0000
V1F(9,5)=0.0000
V1F(9,6)=0.0000
V1F(9,7)=0.0000
V1F(9,8)=0.0000
V1F(9,9)=0.0000
V1F(9,10)=0.0000
V1A(9,1)=0.0000
V1A(9,2)=0.0000
V1A(9,3)=0.0000
V1A(9,4)=0.0000
V1A(9,5)=0.0000
V1A(9,6)=0.0000
V1A(9,7)=0.0000
V1A(9,8)=0.0000
V1A(9,9)=0.0000
V1A(9,10)=0.0000
V2F(9,1)=1.0000
V2F(9,2)=0.5600
V2F(9,3)=0.5400
V2F(9,4)=0.1500
V2F(9,5)=0.6400
V2F(9,6)=0.5100
V2F(9,7)=0.5600
V2F(9,8)=0.5775
V2F(9,9)=0.1900
V2F(9,10)=0.0975
V2A(9,1)=1.0000
V2A(9,2)=0.5600
V2A(9,3)=0.5400
V2A(9,4)=0.1500
V2A(9,5)=0.6400
V2A(9,6)=0.5100
V2A(9,7)=0.5600
V2A(9,8)=0.5775
V2A(9,9)=0.1900

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CC

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APP06730
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APP07100
APP07110
APP07120
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APP07170
APP07180
APP07190
APP07200

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V2A(9,5)=0.0975
V2A(9,10)=0.0000
... CONTINUE FOR R=1.0R
V1F(10,1)=0.0000
V1F(10,2)=0.0000
V1F(10,3)=0.0000
V1F(10,4)=0.0000
V1F(10,5)=0.0000
V1F(10,6)=0.0000
V1F(10,7)=0.0000
V1F(10,8)=0.0000
V1F(10,9)=0.0000
V1F(10,10)=0.0000
V1F(10,11)=0.0000
V1A(10,1)=0.0000
V1A(10,2)=0.0000
V1A(10,3)=0.0000
V1A(10,4)=0.0000
V1A(10,5)=0.0000
V1A(10,6)=0.0000
V1A(10,7)=0.0000
V1A(10,8)=0.0000
V1A(10,9)=0.0000
V1A(10,10)=0.0000
V2F(10,1)=0.9600
V2F(10,2)=0.9600
V2F(10,3)=0.8400
V2F(10,4)=0.7500
V2F(10,5)=0.6400
V2F(10,6)=0.5100
V2F(10,7)=0.3600
V2F(10,8)=0.2775
V2F(10,9)=0.1900
V2F(10,10)=0.0975
V2A(10,1)=0.0000
V2A(10,2)=0.9600
V2A(10,3)=0.8400
V2A(10,4)=0.7500
V2A(10,5)=0.6400
V2A(10,6)=0.5100
V2A(10,7)=0.3600
V2A(10,8)=0.1900
V2A(10,9)=0.0975
V2A(10,10)=0.0000

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CC

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APP07660
APP07670
APP07680

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18 CC CONTINUE ((I.EQ.8).OR.(I.EQ.9))GO TO 19
IF(.NOT. DEL PA(I)=(PLA(IR)/10.0)*C.5
19 GC TO 20
CC CONTINUE
20 DEL PA(I)=(PLA(IR)/10.0)*1.0
CC CONTINUE
AA(I)=(0.5*(HA(IP1)+HA(I)))*DELPA(I)
DLPASM=DLPASM+DELPA(I)
XA(I)=DLPASM-(DELPA(I)/2.0)
YA(I)=(Y(IP1)+Y(I))/2.0
SLMAA=SUMAA+AA(I)
SLMAXA=SUMAXA+(AA(I)*XA(I))
SLMAYA=SUMAYA+(AA(I)*YA(I))
SPAX2A=SMAX2A+(AA(I)*((XA(I))**2))+
1 ((1.0/12.0)*((HA(IP1)+HA(IP1))/2.0)*(DELPA(I)**3))
SPAY2A=SMAY2A+(AA(I)*((YA(I))**2))+
1 ((1.0/12.0)*(DELPA(I))*((HA(I)+HA(IP1))/2.0)**3))
21 CONTINUE
CC
CC
CC
CC
22 **CONTINUE WITH "FORWARD" PORTION (P=0 TO P=+1) OF BLADE
SECTION AS DEPICTED IN FIGURE (1), REF 2
DC 25 J=1,10
IF1=I+1
YFACE(I)=(V1F(IR,I))*TR(IR)-(U.1*TR(IR))
YEACK(I)=(V1F(IR,I)+V2F(IR,I))*TR(IR)-(O.1*TR(IR))
1 HF(I)=YBACK(I)-YFACE(I)/2.0
Y(I)=(YBACK(I)+YFACE(I))*TR(IR)-(O.1*TR(IR))
YFACE(IP1)=(V1F(IR,IP1)+V2F(IR,IP1))
YEACK(IP1)=(V1F(IR,IP1)-YFACE(IP1))*TR(IR)-(O.1*TR(IR))
1 HF(IP1)=YBACK(IP1)-YFACE(IP1)/2.0
Y(IP1)=(YBACK(IP1)+YFACE(IP1))*TR(IR)-(O.1*TR(IR))
IF(.NOT. DEL PF(I)=(PLF(IR)/10.0)*2.0
22 GC TO 24
CC CONTINUE
IF(.NOT. ((I.EQ.7).OR.(I.EQ.8).OR.(I.EQ.9).OR.(I.EQ.10))
1 GC TO 23
DEL PF(I)=(PLF(IR)/10.0)*C.5
GC TO 24
CC CONTINUE
DEL PF(I)=(PLF(IR)/10.0)*1.0
23 CC CONTINUE
AF(I)=(0.5*(HF(IP1)+HF(I))*DELPF(I)
24 DLPASM=DLPASM+DELPF(I)
APP08170
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APP08190
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APP08640

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31 IF(.NOT.(DN-LI-AR(IR)))GO TO 31
   XCG(IR)=AR(IR)-DN
   GC TC 33
   CCNTINUE
32 IF(.NOT.(DN-GT-AR(IR)))GO TO 32
   XCG(IR)=- (DN-AR(IR))
   GC TO 33
   CCNTINUE
33 XCG(IR)=0.0
   CCNTINUE
   YCG(IR)=YPRL(IR)

CC DETERMINE "CRITICAL POINTS" OF A BLADE SECTION WITH RESPECT
CC TC NEUTRAL AXES WHERE STRESSES ARE LIKELY TO BE A MAXIMUM
CC
CC ..CALCULATE ORDINATE (U2) AND ABCISSA (W2) OF CRITICAL POINT
CC NO. 2 DEPICTED IN FIGURE (10), REF 8
CC
CC W2(IR)=(AR(IR)-BR(IR))-XCG(IR)
CC U2(IR)=(((VIA(IR,1)+V2A(IR,1))* (TR(IR)-(0.1*TR(IR))))+
CC (0.1*TR(IR)))-YPRL(IR)
CC
CC ..CALCULATE ORDINATE (U4) AND ABCISSA (W4) OF CRITICAL POINT
CC NC. 4 DEPICTED IN FIGURE (10), REF 8
CC
CC W4(IR)=W2(IR)
CC U4(IR)=((VIA(IR,1))* (TR(IR)-(0.1*TR(IR)))-YPRL(IR)
CC
CC ..CALCULATE ORDINATE (U1) AND ABCISSA (W1) OF CRITICAL POINT
CC NC. 1 DEPICTED IN FIGURE (10), REF 8
CC
CC W1(IR)=-((CR(IR)-AK(IR))+XCG(IR))
CC YBK=((VIA(IR,10)+V2A(IR,10))* (TR(IR)-(0.1*TR(IR))))
CC YFC=((VIA(IR,10))* (TR(IR)-(0.1*TR(IR))))
CC IF(.NOT.(YCG(IR).GT.YBK))GC TC 40
   U1(IR)=YFC-YCG(IR)
   GC TC 45
   CCNTINUE
40 IF(.NOT.((YCG(IR).LE.YBK).AND.(YCG(IR).GE.YFC)))GO TO 44
   IF(.NOT.((ABS(YCG(IR)-YFC)).GT.(AES(YBK-YCG(IR))))
   GC TO 41
   U1(IR)=YFC-YCG(IR)
   GC TO 43
   CCNTINUE
41 IF(.NOT.((ABS(YCG(IR)-YFC)).LT.(AES(YBK-YCG(IR))))
   GC TO 42
   U1(IR)=YBK-YCG(IR)
   GC TC 43

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APP05130
APP05140
APP05150
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42 CC CONTINUE
43 CC U1(IR)=0.0
44 CC CONTINUE
45 CC GC TO 45
46 CC CONTINUE
47 CC U1(IR)=YBK-YCG(IR)
48 CC CONTINUE
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SUBROUTINE ELDVCL(VCLBLD)

SUBROUTINE: BLDVOL

DATE OF LAST REVISION: APR 83

INPUT OUTPUT

DIA

PROPELLER DIAMETER (FEET)

Z

NO. OF PROPELLER BLADES

COMMON/AREBLD/

BLADE SECTION AREAS (FEET\*\*2)

VOLBLD

PROPELLER BLADE VOLUME  
(FEET\*\*3)

REAL\*4

ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,

1

RJCNL,RJCNL,R75RCL,R75RCL,AEACCU,TC75CL,TC75CU,

2

PCWBAL,DIACNU,AEACCU,V,TCSTRS,

3

VK,TC,WT,Z,WATRC,WATNU,TEMP,NOSCRW,HCL,PATM,

4

PWATVA,PRGMAT,DIALIM,ETARR,

5

RJ,C75R,R75R,KT,KQ

REAL\*4 AREA(10),WIDTH,VCL1,VOL2,VOLBLD

COMMON

/GLOECM/ETAG,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,

1

RJCNL,R75RCL,R75RCL,AEACCU,TC75CL,TC75CU,PWBAL,DIACNU,

2

AEACCU,TCSTRS,RJ

COMMON

/PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,PWATVA,

1

PRGMAT,DIALIM,ETARR,AEACMN,TC75MN,SC

COMMON

/AREBLD/AREA

DETERMINE BLADE VOLUME BY SIMPSON INTEGRATION SCHEME OF BLADE

CROSS-SECTIONAL AREAS FROM 2/10 RADIUS OUTWARD TO TIP

VOL1=(( (DIA/2.0)\*0.1)/3.0)\*(AREA(2)+(4.0\*AREA(3))+(2.0\*AREA(4))+

1

(4.0\*AREA(5))+(2.0\*AREA(6))+

2

(4.0\*AREA(7))+(2.0\*AREA(8))+

3

(4.0\*AREA(9))+AREA(10))

DETERMINE BLADE VOLUME FROM 2/10 RADIUS INWARD TO R=.18R FOR 3

BLADE PROPELLERS OR R=.167R FOR 4,5,66 BLADE PROPELLERS USING

SIMPLE TRAPEZOIDAL INTEGRATION SCHEME

IF(.NOT.(Z.EQ.3.0))GO TO 1

WIDTH=(DIA/2.0)\*(0.2-0.16)

GO TO 5

1 CONTINUE

IF(.NOT.(Z.EQ.4.0))GO TO 2

GO TO 5

2 CONTINUE

IF(.NOT.(Z.EQ.5.0))GO TO 3

APPLC010  
APPLC020  
APPLC030  
APPLC040  
APPLC050  
APPLC060  
APPLC070  
APPLC080  
APPLC090  
APPLC100  
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APPLC120  
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APPLC360  
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APPLC380  
APPLC390  
APPLC400  
APPLC410  
APPLC420  
APPLC430  
APPLC440  
APPLC450  
APPLC460  
APPLC470  
APPLC480



APP1C490  
 APP1C500  
 APP1C510  
 APP1C520  
 APP1C530  
 APP1C540  
 APP1C550  
 APP1C560  
 APP1C570  
 APP1C580  
 APP1C590  
 APP1C600  
 APP1C610  
 APP1C620  
 APP1C630  
 APP1C640

```

      WIC1F=(DIA/2.0)*(0.2-0.167)
      GO TO 5
3 CONTINUE
      IF(.NOT.(Z.EQ.6.0))GO TO 4
      WIC1F=(DIA/2.0)*(0.2-0.167)
      GO TO 5
4 CONTINUE
      WIC1F=(DIA/2.0)*(0.2-0.18)
5 CONTINUE
      VOL2=((AREA(1)+AREA(2))/2.0)*WIDTH
      CALCULATE PROPELLER BLADE VOLUME
      VOLBLD=VOL1+VOL2
      RETURN
      END
  
```

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SUBROUTINE ELPCW1(KT,SCWBAL)

SUBROUTINE: BLPCW1

INPUT OUTPUT

PE  
KT  
DIA  
N  
WATRO  
V  
WT  
TD  
ETARR

SCWBAL

DATE OF LAST REVISION: FEB 83

DEFINITION

HULL EFFECTIVE HORSEPOWER (HP)  
THRUST COEFFICIENT  
PROPELLER DIAMETER (FT)  
PROPELLER REVOLUTION RATE (RPM)  
WATER DENSITY (LBF-SEC<sup>2</sup>/FT<sup>4</sup>)  
SHIP SPEED (FT/SEC)  
WAKE FRACTION  
THRUST DEDUCTION  
RELATIVE ROTATIVE EFFICIENCY  
CONSTRAINT VARIABLE FOR PRO-  
PELLER-DEVELOPED EFFECTIVE  
HORSEPOWER CONSTRAINT  
(SCWBALCO)

REAL\*4 ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVC,QS,TC75R,V,  
1 RJCNL,RJCNL,R75RCL,R75RCL,AEAOCL,TC75CL,TC75CU,  
2 FOWBAL,DIACNU,AEAOCL,TC75R,RJ,  
3 VK,TC,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,  
4 PWATVA,PRGMAT,DIALIM,ETARR,AEAOCL,TC75MN,SC,  
5 KI,PEDEV,THRST,FT,SCWBAL

COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVC,QS,TC75R,V,RJCNL,  
1 RJCNL,R75RCL,R75RCL,AEAOCL,TC75CL,TC75CU,PWATVA,DIACNU,  
2 AEAOCL,TC75R,RJ  
COMMON /PAR4M/VK,TD,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,  
1 PRGMAT,DIALIM,ETARR,AEAOCL,TC75MN,SC

CALCULATE THRUST DEVELOPED BY EACH PROPELLER

THRST=(KT\*WATRO\*(DIA\*\*4)\*((N/60.0)\*\*2))

DETERMINE HULL EFFECTIVE POWER DEVELOPED BY PROPELLER(S)

PT=(THRST\*((1.0-WT)\*V))/550.0

PEDEV=((((1.0-TC)/(1.0-WT))\*ETARR\*PT)\*NCSCRW)

CALCULATE CONSTRAINT VARIABLE

SCWBAL=1.0-(PEDEV/PE)

RETURN

END

APPLC670  
APPLC680  
APPLC690  
APPLC700  
APPLC710  
APPLC720  
APPLC730  
APPLC740  
APPLC750  
APPLC760  
APPLC770  
APPLC780  
APPLC790  
APPLC800  
APPLC810  
APPLC820  
APPLC830  
APPLC840  
APPLC850  
APPLC860  
APPLC870  
APPLC880  
APPLC890  
APPLC900  
APPLC910  
APPLC920  
APPLC930  
APPLC940  
APPLC950  
APPLC960  
APPLC970  
APPLC980  
APPLC990  
APPL1000  
APPL1010  
APPL1020  
APPL1030  
APPL1040  
APPL1050  
APPL1060  
APPL1070  
APPL1080  
APPL1090  
APPL1100  
APPL1110  
APPL1120  
APPL1130



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SUBROUTINE ELPOW2(KG,SWBAL)
SUBROUTINE: BLPCW2
INPLT      OUTPUT
QS
KQ
DIA
N
WATRO
SOWBAL

DATE OF LAST REVISION: FEB 83
DEFINITION
DELIVERED TORQUE (FT-LBF)
TORQUE COEFFICIENT (FT)
PROPELLER DIAMETER (FT)
PROPELLER REVOLUTION RATE (RPM)
WATER DENSITY (LBF-SEC2/FT4)
CONSTRAINT VARIABLE FOR PRO-
PELLER REQUIRED TORQUE
(SWBALKO)

REAL*4      ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,
1      RJCNU,RJCNU,R75FCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2      PCWBAL,DIAACNU,AEADCV,TCSTKS,
3      VK,TC,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PAIM,
4      PWATVA,PRGMAT,DIALIM,ETARK,
5      KC,QSABD,SOWBAL
COMMON /GLOBEOM/ETAC,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,RJCNU,
1      RJCNU,R75FCL,R75RCU,AEAOCL,AEADCV,TC75CL,TC75CU,PCWBAL,DIAACNU,
2      AEACCV,TCSTKS,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PAIM,PWATVA,
1      PRGMAT,DIALIM,ETARR,AEADCN,TC75MN,SC

CALCULATE THE TORQUE ABSORBED BY PROPELLER
QSAED=(KC*WATRO*(DIA**5)*((N/60.0)**2))
CALCULATE CONSTRAINT VARIABLE
SWBAL=1.0-(QSAED/CS)
RETURN
END

```

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CALCULATE THE POWER ABSORBED BY THE PROPELLER
PDABD=(2.0*FII*QPADBD*N)/33000.0
CALCULATE THE PCWER DELIVERED TO THE PROPELLER
PD=(2.0*FII*QS*N)/33000.0
DETERMINE CONSTRAINT VARIABLES
SOWELP=1.0-(PEDEV/PE)
SOWELC=(PCAED/PC)-1.0
RETURN
END
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APPI2020  
APPI2030  
APPI2040  
APPI2050  
APPI2060  
APPI2070  
APPI2080  
APPI2090  
APPI2100  
APPI2110  
APPI2120  
APPI2130  
APPI2140  
APPI2150  
APPI2160





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SUBROUTINE CALCKQ(J,PUIVD,AEDVAC,Z,REYN,KQ)
SUBROUTINE: CALCKQ      DATE OF LAST REVISION: FEB 83
INPUT      OUTPUT      DEFINITION
AEDVAO      EXPANDED AREA RATIO
PDIVD      PITCH-DIAMETER RATIO
Z      NO. OF BLADES
J      ADVANCE RATIO
REYN      REYNOLDS NO. 3/4 RADIUS
      (CORRECTED FOR 1/4 RADIUS
      THRUST COEFFICIENT
      KQ
REAL*4 J,PDIVD,AEDVAO,Z,REYN,KQ,KQ1,KQ2,KQ3,KQ4,KQ5,KQ6,KQ7,KQ8,
1 KQ9,KQ10,DELKQ,DELKQ1,DELKQ2,DELKQ3,REVFAC
FIRST, CALCULATING "KQ1" THRU "KQ10".....
KQ1=((0.0379368)+((0.00886523)*(J**2))+((-0.032241)*J*PDIVD)+
1 ((-0.0344778)*(PCIVD**2))+((-0.0408811)*PCIVD*AEDVAO)
KQ2=(((-0.108009)*J*PDIVD*AEDVAO)+((-0.0885381)*(J**2)*PDIVD*AEDVAC)+
1 ((-0.188561)*(PDIVD**2)*AEDVAO)+((-0.00370871)*J*Z)+
2 ((-0.00513696)*PCIVD*Z))
KQ3=(((-0.0209449)*J*PDIVD*Z))+((-0.00474319)*(J**2)*PDIVD*Z)+
1 ((-0.00723408)*(J**2)*AEDVAO*Z))+((-0.00438388)*J*PDIVD*AEDVAO*Z)+
2 ((-0.0263403)*(PDIVD**2)*AEDVAO*Z))
KQ4=(((-0.0558082)*(J**3)*AEDVAO)+((-0.0161886)*(PDIVD**3)*AEDVAO)+
1 ((-0.00318086)*J*(PDIVD**3)*AEDVAC)+((-0.015896)*(AEDVAO**2))+
2 ((-0.0471129)*J*(AEDVAO**2)))
KQ5=(((-0.0196283)*(J**3)*(AEDVAO**2))+((-0.0502782)*PDIVD*
1 (AEDVAC**2))+((-0.030055)*(J**3)*PCIVD*(AEDVAO**2))+
2 ((-0.0417122)*(J**2)*(PDIVD**2)*(AEDVAO**2))+
3 ((-0.0397722)*(PCIVD**3)*(AEDVAO**2)))
KQ6=(((-0.00350024)*(PDIVD**6)*(AEDVAC**2))+((-0.0106854)*(J**3)*Z)+
1 ((-0.0110903)*(J**3)*(PDIVD**3)*Z))+((-0.00313912)*(PDIVD**6)*
2 Z))+((-0.0035585)*(J**3)*AEDVAO*Z))
KQ7=(((-0.00142121)*(PDIVD**6)*AEDVAO*Z))+((-0.00383637)*J*
1 (AEDVAC**2)*Z))+((-0.0126803)*(PDIVD**2)*(AEDVAO**2)*Z)+
2 ((-0.00318278)*(J**2)*(PDIVD**3)*(AEDVAC**2)*Z)+
3 ((-0.00334268)*(PCIVD**6)*(AEDVAC**2)*Z))
KQ8=(((-0.00183491)*J*PDIVD*Z))+((-0.000112451)*(J**3)*(PDIVD**2)*
1 Z))+((-0.000297228)*(J**3)*(PCIVD**6)*(Z**2))+
2 ((-0.000269551)*J*AEDVAO*(Z**2))+((-0.00083205)*(J**2)*AEDVAO*
3 (Z**2))
KQ9=(((-0.0015334)*(PDIVD**2)*AEDVAO*(Z**2))+((-0.000302683)*
1 (PCIVD**6)*AEDVAO*(Z**2))+((-0.0001843)*(AEDVAO**2)*(Z**2))+
2 ((-0.000425359)*(PDIVD**3)*(AEDVAC**2)*(Z**2)))+
3 ((-0.000425359)*(PDIVD**3)*(AEDVAC**2)*(Z**2)))+

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APPL12680  
APPL12690  
APPL12700  
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APPL12880  
APPL12890  
APPL12900  
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APPL12920  
APPL12930  
APPL12940

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3      ((.0000869243)*J**3)*(PDIVD**3)*(AEDVAC**2)*(Z**2))
KQ10=(((-.00046559)*(PDIVC**6)*(AEDVAC**2)*(Z**2))+
1      ((.0000554194)*J*(PDIVU**6)*(AEDVAC**2)*(Z**2)))
NEXT,CALCULATE LOGARITHMIC REYNOLDS NUMER FACTOR "REYFAC"
WRITE(6,*)REYNO,REYNO,REYNG
REYFAC=ALOG10(REYNG)-0.301
THEN,CALCULATE "DELKQ1+DELKQ2+DELKQ3".....
DELKQ1=(-.000591412)+((.000696858)*PDIVD)+
1      ((-.000666654)*Z*(PDIVC**6))+((.0160818)*(AEDVAC**2))
DELKQ2=((-000538051)*REYFAC*PDIVD)+((-00059553)*REYFAC*
1      (PDIVD**2))+((.0000782099)*(REYFAC**2)*(PDIVU**2))+
2      ((.000052155)*REYFAC*Z*AEDVAC*(J**2))
DELKQ3=((-0000088528)*(REYFAC**2)*Z*AEDVAC*(PDIVD**J))+
1      ((.000230171)*REYFAC*Z*(PDIVD**6))+((-00000184341)*
2      (REYFAC**2)*Z*(PDIVD**6))+((-000400252)*REYFAC*
3      (AEDVAC**2))+((-000220515)*(REYFAC**2)*(AEDVAC**2))
DELKQ=DELKQ1+DELKQ2+DELKQ3
FINALLY,CALCULATE TRUST COEFFICIENT "KC" WHERE
"KQ=KC1+KQ2+KQ3+KQ4+KQ5+KQ6+KQ7+KQ8+KQ9+KQ10+DELKQ"
KQ=KQ1+KQ2+KQ3+KQ4+KQ5+KQ6+KQ7+KQ8+KQ9+KQ10+DELKQ
RETURN
END

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SUBROUTINE CALCKT(J,PDIVD,AEDVAC,Z,REYNC,KT)
SUBROUTINE: CALCKT      DATE OF LAST REVISION: FEB 83
INPUT      OUTPUT      DEFINITION
AEDVAC     EXPANDED AREA RATIO
PDIVD      PITCH-DIAMETER RATIO
Z          NC. OF BLADES
J          ADVANCE RATIO
REYN       REYNOLDS NO. 3/4 RADIUS
              (CORRECTED FOR T/C 3/4 RADIUS
              THRUST COEFFICIENT)
      KT
REAL*4 J,PDIVD,AEDVAC,Z,REYN,KT,KT1,KT2,KT3,KT4,KT5,KT6,KT7,KT8,
1      DELKT,DELKT1,DELKT2,DELKT3,REYFAC
      FIRST, CALCULATING "KT1" THRU "KT8".....
      KT1=.00860456+((-2045554)*J)+((.166351)*PDIVD)+
1      ((.158114)*(PDIVD**2))+((-147581)*(J**2)*AEDVAC)+
1      ((.014443)*Z)+((-0.0530054)*(J**2)*Z)+((.0143481)*PDIVD**2)
1      KT3=((-.0606826)*J*PDIVC*Z)+((-0.125894)*AEDVAC*Z)+
1      ((.0109689)*J*AEDVAC*Z)+((-0.133698)*(PDIVD**3))+((.00638407)*
2      (PDIVD**6))
1      KT4=((-.00132716)*(J**2)*(PDIVD**6))+((-0.168496)*(J**3)*AEDVAC)+
1      ((-.0507214)*(AEDVAC**2))+((-0.0854555)*(J**2)*(AEDVAC**2))+
2      ((-.0504475)*(J**3)*(AEDVAC**2))
1      KT5=((-.010465)*J*(PDIVC**6)*(AEDVAC**2))+((-0.00841728)*(PDIVD**3)*Z)+
1      ((.0168424)*J*(PDIVC**3)*Z)+((-0.0102296)*(J**3)*(PDIVD**3)*Z)
2      KT6=((-.00317791)*(PDIVD**3)*AEDVAC*Z)+((-0.018604)*J*(AEDVAC**2)*Z)+
1      ((-.00410798)*(PDIVD**2)*(AEDVAC**2)*Z)+((-0.000606848)*(Z**2))
2      +((-0.0045815)*J*(Z**2))
1      KT7=((-.0025583)*(J**2)*(Z**2))+((-0.000560528)*(J**3)*(Z**2))+
1      ((-.00163652)*J*(PDIVC**2)*(Z**2))+((-0.00328787)*J*
2      (PDIVD**6)*Z)+((-0.00116502)*(J**2)*(PDIVD**6)*(Z**2))
1      KT8=((-.000650904)*AEDVAC*(Z**2))+((-0.00421745)*(PDIVD**3)*AEDVAC*
2      (Z**2))+((-0.0000565229)*(J**3)*(PDIVD**6)*AEDVAC*(Z**2))+
1      ((-.00146564)*(PDIVD**3)*(AEDVAC**2)*(Z**2))
      NEXT, CALCULATE LOGARITHMIC REYNOLDS NUMBER FACTOR "REYFAC"
WRITE(6,*)REYNC,REYNC
REYFAC=ALOG10(REYNC)-0.301
      THEN, CALCULATE "DELKT=DELKT1+DELNT2+DELKT3".....

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APPL13450  
 APPL13460  
 APPL13470  
 APPL13480  
 APPL13490  
 APPL13500  
 APPL13510  
 APPL13520  
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 APPL13550  
 APPL13560  
 APPL13570  
 APPL13580  
 APPL13590  
 APPL13600  
 APPL13610

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DELKT1=((.00C353485)+( (-.00333758)*AEDVAC*(J**2)) +
1  ((-.00478125)*AEDVAC*PD1VD**J)
DELKT2=((.00257792)* (REYFAC**2)*AE (CVAQ*(J**2)) +
1  ((.000643152)* (REYFAC*(PD1VD**6)*(J**2)) +
2  ((-.000110636)* (REYFAC**2)*(PD1VD**6)*(J**2)) +
1  DE LKT3=(((-.000276305)* (REYFAC**2)*AE (CVAQ*(J**2)) +
2  ((.000954)*REYFAC*PD1VD**J) +
  DE LKT=DELKT1+DELKT2+DELKT3
  FINALLY,CALCULATE TRUST COEFFICIENT "KT" WHERE
  "KT=KT1+KT2+KT3+KT4+KT5+KT6+KT7+KT8+DELKT".....
  KT=KT1+KT2+KT3+KT4+KT5+KT6+KT7+KT8+DELKT
  RETURN
  END

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SUBROUTINE CALCCS(KC,SCS)
SUBROUTINE: CALCQS
INPUT      OUTPUT
KC
DIA
N
WATRO
SQS
KEAL*4    ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,
1          RJCNU,RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2          FCWBAL,DIACNU,AEAOCL,V,TCSTRS,RJ,
3          VK,TL,WT,Z,WATRO,WATNU,TEMP,NUSCRW,HCL,PATM,
4          FWATVA,PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
5          KC,SCS
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,RJCNU,
1RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2AEACCU,TCSTRS,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NUSCRW,HCL,PATM,PWATVA,
1PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC
CALCULATE DELIVERED TORQUE REQUIRED BY PROPELLER
SQS=(KC*WATFU*(DIA**5)*((N/60.0)**2))
RETURN
END

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APPL13640
APPL13650
APPL13660
APPL13670
APPL13680
APPL13690
APPL13700
APPL13710
APPL13720
APPL13730
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APPL13750
APPL13760
APPL13770
APPL13780
APPL13790
APPL13800
APPL13810
APPL13820
APPL13830
APPL13840
APPL13850
APPL13860
APPL13870
APPL13880
APPL13890
APPL13900
APPL13910
APPL13920

```

SUBROUTINE CALCCS(KC,SCS)

SUBROUTINE: CALCQS

INPUT OUTPUT

KC  
DIA  
N  
WATRO

SQS

KEAL\*4

ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,  
RJCNU,RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,  
FCWBAL,DIACNU,AEAOCL,V,TCSTRS,RJ,  
VK,TL,WT,Z,WATRO,WATNU,TEMP,NUSCRW,HCL,PATM,  
FWATVA,PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,  
KC,SCS  
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,RJCNU,  
1RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,  
2AEACCU,TCSTRS,RJ  
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NUSCRW,HCL,PATM,PWATVA,  
1PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

CALCULATE DELIVERED TORQUE REQUIRED BY PROPELLER

SQS=(KC\*WATFU\*(DIA\*\*5)\*((N/60.0)\*\*2))  
RETURN  
END

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SUBROUTINE CALCPE(KT,SPE)
SUBROUTINE: CALCPE
DATE OF LAST REVISION: FEB 83
DEFINITION
INPUT          OUTPUT
KT
DIA
N
WATRO
TD
WT
V
SPE

THRUST COEFFICIENT (FT)
PROPELLER DIAMETER (FT)
PROPELLER REVOLUTION RATE (RPM)
WATER DENSITY (LBF-SEC2/FT4)
THRUST DEDUCTION
TAYLOR WAKE FRACTION
VELOCITY (FT/SEC)
EFFECTIVE POWER (HP)

REAL*4  ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,
1      RJCNU,RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2      FCWBAL,DIAACNU,AEAOCLV,TCSTKS,RJ,
3      VK,TL,WT,Z,WATRC,WATNU,TEMP,NOSCROW,HCL,PATM,
4      PWATVA,PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
5      KT,SPE,T
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,RJCNU,
1      RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,PWBAL,DIAACNU,
2      AEAOCLV,TCSTKS,RJ
COMMON /PARAM/VK,TC,WT,Z,WATRO,WATNU,TEMP,NOSCROW,HCL,PATM,PWATVA,
1      PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

CALCULATE THRUST DEVELOPED BY PROPELLER
T=(KT*WATRO*(DIA**4)*((N/60.0)**2))

CALCULATE EFFECTIVE HORSEPOWER DEVELOPED BY PROPELLER
SPE=((1.0-TC)/(1.0-WT))*((T*(1.0-WT)*V)/550.0)
RETURN
END

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SUBROUTINE CAVCNA(KT,SEACCV)
SUBROUTINE: CAVCNA
INPUT      OUTPUT
PATM
WATRO
PWATVA
HCL
KT
DIA
N
Z
NOSCRW
AEDVAO
SEACCV

DATE OF LAST REVISION: FEB 83
DEFINITION
ATMOSPHERIC PRESSURE (PSIA)
WATER DENSITY (LBF-SEC2/FT4)
VAPORIZATION PRESSURE FOR WATER
(PSIA)
DEPTH OF PROPELLER SHAFT
CENTERLINE (FT)
THRUST COEFFICIENT
PROPELLER DIAMETER (FT)
PROPELLER REVOLUTION RATE (RPM)
NO. OF BLADES
NO. OF PROPELLERS
PROPELLER EXPANDED AREA RATIO
CONSTRAINT VARIABLE FOR PRO-
PELLER EXPANDED AREA RATIO
CONSTRAINT (SEACCV<0)

REAL*4 ETAO,WEIGHT,AEDVAO,DIA,N,PE,FOIVC,QS,TC75R,V,
1 RJCNU,RJCNU,R75RCL,R75RCU,AEACCU,TC75CL,TC75CU,
2 FCWBAL,DIA,CNU,AEACCV,TCSTKS,RJ
3 VK,ITL,WT,Z,WATRO,WATNU,ETARR,NOSCRW,HCL,PATM,
4 PWATVA,PROMAT,DIALIM,ETARR,AEACMN,TC75MN,SC,
5 KT,PCMN,PV,THRST,SEACMN,SEACCV
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FOIVD,QS,TC75K,V,RJCNU,
1 RJCNU,R75RCL,R75RCU,AEACCU,TC75CL,TC75CU,POWBAL,DIA,CNU,
2 AEACCV,TCSTKS,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,PWATVA,
1 PROMAT,DIALIM,ETARR,AEACMN,TC75MN,SC

CALCULATE DIFFERENCE BETWEEN STATIC PRESSURE AT SHAFT CENTERLINE
& WATER VAPORIZATION PRESSURE
POMNPV=(FATM*144.0)+(WATRO*32.174*(HCL)-(PWATVA*144.0)

DETERMINE MINIMUM REQUIRED EXPANDED AREA RATIO FOR PROPELLER(S)
USING "KELLER" CRITERIA FROM RELATION (13), REF 2
THRST=(KT*WATRO*(DIA**4)*((N/60.0)**2))
SEACMN=((1.3+(0.3*Z))*THRST)/((DIA**2)*POMNPV)

CORRECT ACCORDING TO NUMBER OF PROPELLERS
IF(.NOT.(NOSCRW.EQ.1.0))GO TO 1

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APPI4810
APPI4820
APPI4830
APPI4840
APPI4850
APPI4860
APPI4870
APPI4880
APPI4890
APPI4900
APPI4910
APPI4920
APPI4930
APPI4940
APPI4950
APPI4960

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SEACMN=SEACMN+0.2
GO TO 3
CONTINUE
1 IF(.NOT.(NO$CRW.EQ.2.0))GO TO 2
SEACMN=SEACMN+0.1
GO TO 3
2 CONTINUE
SEACMN=SEACMN+0.0
3 CONTINUE
AEACMN=SEACMN
DETERMINE CONSTRAINT VARIABLE
SEAOCCV=((SEACMN/AEDVAG)-1.0
RETURN
END

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C
C
C

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SUBROUTINE CH75RA(C75R)

SUBROUTINE: CH75RA

DATE OF LAST REVISION: FEB 83

INPLT                    C75R                    OUTPUT

AEDVAO

DIA

Z

DEFINITION

EXPANDED AREA RATIO  
PROPELLER DIAMETER (FEET)

NO. OF ELADES  
CHORD LENGTH AT 3/4 RADIUS  
(FEET)

PDIVD, QS, TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

TC75R, V,  
AEAOCL, AEACCU, TC75CL, TC75CU,

APPL14590  
APPL15000  
APPL15010  
APPL15020  
APPL15030  
APPL15040  
APPL15050  
APPL15060  
APPL15070  
APPL15080  
APPL15090  
APPL15100  
APPL15110  
APPL15120  
APPL15130  
APPL15140  
APPL15150  
APPL15160  
APPL15170  
APPL15180  
APPL15190  
APPL15200  
APPL15210  
APPL15220  
APPL15230  
APPL15240  
APPL15250  
APPL15260  
APPL15270

CALCULATE CHORD LENGTH AT 3/4 RADIUS USING RELATION (17), REF 2

C75R=(2.073\*AEDVAU\*DIA)/Z

RETURN

END

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SUBROUTINE CNFGLD	DATE OF LAST REVISION: APR 83	APPI5300
SUBROUTINE: CNFGLD	DEFINITION	APPI5310
INPUT	PROPELLER DIAMETER (FT)	APPI5320
DIA	PROPELLER REVOLUTION RATE (RPM)	APPI5330
N	PITCH-DIAMETER RATIO	APPI5340
PDIVD	NC. CF PROPELLER BLADES	APPI5350
Z	PROPELLER MATERIAL IDENTIFIER	APPI5360
PRGMAT	1: CAST IRON	APPI5370
	2: CAST STEEL	APPI5380
	3: TYPE 2 BRONZE	APPI5390
	4: TYPE 4 NI-AL BRONZE	APPI5400
	5: STAINLESS STEEL	APPI5410
COMMON /AREELD/ COMMON /CGX/	BLADE SECTION AREAS (FEET**2) LOCATION OF BLADE CROSS-SECTIONAL AREA CENTROID WITH RESPECT TO GENERATOR LINE (FEET)	APPI5420
COMMON /CGY/	LOCATION OF BLADE CROSS-SECTIONAL AREA CENTROID WITH RESPECT TO PITCH-REFERENCE LINE (FEET)	APPI5430
	CENTRIFUGAL FORCE COMPONENTS, ALONG THE PROPELLER RADIUS, PARALLEL TO GENERATOR LINE, WHICH ACT ON A BLADE SECTION AT ITS NEUTRAL AXES ORIGIN (LBF)	APPI5440
COMMON /CFGFD/	CENTRIFUGAL BENDING MOMENT COMPONENTS, ALONG THE PROPELLER RADIUS, PARALLEL TO THE PITCH REFERENCE (CHORD) LINE, WHICH ACT ON A BLADE SECTION AT ITS NEUTRAL AXES ORIGIN (FT-LBF)	APPI5450
COMMON /CMCBN/	CENTRIFUGAL BENDING MOMENT COMPONENTS, ALONG THE PROPELLER RADIUS, PARALLEL TO THE PITCH REFERENCE (CHORD) LINE, WHICH ACT ON A BLADE SECTION AT ITS NEUTRAL AXES ORIGIN (FT-LBF)	APPI5460
COMMON /CMCBL/	CENTRIFUGAL BENDING MOMENT COMPONENTS, ALONG THE PROPELLER RADIUS, NORMAL TO THE PITCH REFERENCE (CHORD) LINE, WHICH ACT ON A BLADE SECTION AT ITS NEUTRAL AXES ORIGIN (FT-LBF)	APPI5470
REAL*4	ETAO, WEIGHT, AEDVAU, DIA, N, PE, PDIVD, QS, TC75R, V,	APPI5480
1	RJCNL, RJCNV, R75FCL, R75RCU, AEAOCL, AEACCU, TC75CL, TC75CU,	APPI5490
2	FCWBL, DIACNU, AEAOCV, TC5TKS,	APPI5500
3	VK, TC, WT, Z, WATRC, WATNU, TEMP, NDCSRW, HCL, PATM,	APPI5510

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4 5 REAL*4 F,WATVA,PRCMAT,DIALIM,ETARR,
REAL*4 RJ,C75R,R75R,KT,KQ
REAL*4 AREA(10),XCG(10),YCG(10),R1XXNA(10),RIYNA(10)
REAL*4 SUMSV, SUMSVR, SUMSVT, SUMSVA
SMALLI(10), SMALLA(10), SMALLT(10), SMAIMA(10),
SINBET(10), CCSBET(10), SUMI(10), SUMA(10),
SUM(10), SUMRC(10), SUMT(10), SUMO(10), SMQO(10)
VGL(10), XBRCCO(10), BIGTO(10), BIGAO(10),
CMCBN(10), CMCBL(10), BIGNO(10), RADIUS
INTEGER*4 IF,IRPI,KCUNT,I
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,RJUNL,
1RJCANU,R75RCL,R75RCL,AEACCL,AEACCU,TC75CL,TC75CU,POWBAL,CIA,CNU,
2AEACCV,TCSTFS,RJ
COMMON /PAR4M/VK,TD,WI,Z,WATRU,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PRCMAT,DIALIM,ETARR,AEACMN,TC75MN,SC
COMMON /AREELD/AREA
COMMON /CGX/XCG
COMMON /CGY/YCG
COMMON /A2MCMX/RIYNA
COMMON /A2MCMY/R1XXNA
COMMON /CFGEMN/CMCBN
COMMON /CFGEML/CMCBL
COMMON /CFGFD/BIGNO
PI=3.141592654

CALCULATE RAKE ANGLE IN RADIAN
ETA=15.C*(PI/180.0)

SPECIFY WEIGHT DENSITY OF MATERIAL SELECTED WHERE...
PRGMAT MATERIAL WEIGHT DENSITY (LBF/IN**3)
1 CAST IRON .260
2 CAST STEEL .284
3 TYPE 2 BRONZE .305
4 TYPE 4 NI-AL BRUNZE .278
5 STAINLESS STEEL .283

IF (PRCMAT.EC.1.C)WD=.260
IF (PRCMAT.EC.2.C)WD=.284
IF (PRCMAT.EC.3.C)WD=.305
IF (PRCMAT.EC.4.C)WD=.278
IF (PRCMAT.EC.5.C)WD=.283

DETERMINE VALUES FCR...
SMALLI(IR) DISTANCE TO THE PITCH REFERENCE LINE, ALONG THE
PROPELLER RADIUS, WITH RESPECT TO A LINE NORMAL

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SMALLA(IR)
TC THE PROPELLER SHAFT AXIS AND PASSING THROUGH
THE INTERSECTION OF THE GENERATOR LINE AND THE
PROPELLER SHAFT AXIS (FEET)
DISTANCE, PLATE PARALLEL TO THE PROPELLER SHAFT AXIS,
OF EACH BLADE SECTION TO THE NEUTRAL AXES ORIGIN, GIVEN
BY COORDINATES (XCG(IR),YCG(IR)), WITH RESPECT
TO THE INTERSECTION OF THE PITCH REFERENCE LINE
AND THE GENERATOR LINE (FEET)
DISTANCE, NORMAL TO THE PROPELLER SHAFT AXIS,
OF EACH BLADE SECTION TO THE NEUTRAL AXES ORIGIN, GIVEN
BY COORDINATES (XCG(IR),YCG(IR)), WITH RESPECT
TO THE INTERSECTION OF THE PITCH REFERENCE LINE
AND THE GENERATOR LINE (FEET)

... FOR EACH BLADE SECTION ALONG THE PROPELLER RADIUS
DO 9 IR=2,10
  IF(.NOT.(Z.EQ.4.0))GO TO 7
  IF(.NOT.(IR.EQ.2))GO TO 2
  RF=0.822
  GC TO 6
  CCNT INUE
  IF(.NOT.(IR.EQ.3))GO TO 3
  RF=0.887
  GC TO 6
  CCNT INUE
  IF(.NOT.(IR.EQ.4))GO TO 4
  RF=0.950
  GC TO 6
  CCNT INUE
  IF(.NOT.(IR.EQ.5))GO TO 5
  RF=0.992
  GC TO 6
  CCNT INUE
  RF=1.00
  CCNT INUE
  DENOM=SQRT(((RF*PDIVD)**2)+(PII**2))
  SINBET(IR)=(RF*PDIVD)/DENOM
  CCSBET(IR)=(PII)/DENOM
  GC TO 8
  CCNT INUE
  DENOM=SQRT((PDIVD**2)+(PII**2))
  SINBET(IR)=(PDIVD)/DENOM
  CCSBET(IR)=(PII)/DENOM
  CCNT INUE
  SMALLI(IR)=(DIA/2.0)*(FLOA(IR)/10.0)*(TAN(ETA))
  SMALLA(IR)=(XCG(IR)*SINBET(IR))+(YCG(IR)*COSBET(IR))
  SMALLT(IR)=(XCG(IR)*CCSBET(IR))-(YCG(IR)*SINBET(IR))

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APPI6260  
APPI6270  
APPI6280  
APPI6290  
APPI6300  
APPI6310  
APPI6320  
APPI6330  
APPI6340  
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APPI6360  
APPI6370  
APPI6380  
APPI6390  
APPI6400  
APPI6410  
APPI6420  
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APPI6490  
APPI6500  
APPI6510  
APPI6520  
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APPI6550  
APPI6560  
APPI6570  
APPI6580  
APPI6590  
APPI6600  
APPI6610  
APPI6620  
APPI6630  
APPI6640  
APPI6650  
APPI6660  
APPI6670  
APPI6680  
APPI6690  
APPI6700  
APPI6710  
APPI6720  
APPI6730





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C          SMAIMA(IR)=SMALLI(IR)-SMALLA(IR)
C          5 CONTINUE
C          CALCULATE TABULAR SUMMATIONS FOR INTEGRATION ALONG THE PROPELLER
C          RADIUS
C          DO 10 IR=2,5
C             IRP1=IR+1
C             SUM(IR)=AREA(IR)+AREA(IRP1)
C             SUMR(IF)=(FLOAT(IR)*AREA(IR))+(FLCAT(IRP1)*AREA(IRP1))
C             SUMT(IF)=(SMALLT(IR)*AREA(IR))+(SMALLT(IRP1)*AREA(IRP1))
C             SUMA(IF)=(SMAIMA(IR)*AREA(IR))+(SMAIMA(IRP1)*AREA(IRP1))
C          10 CONTINUE
C          INTEGRATE ALONG THE PROPELLER RADIUS TO DETERMINE THE FORCE AND
C          BENDING MOMENT COMPONENTS, ACTING ON A BLADE SECTION AT ITS NEUTRAL
C          AXES ORIGIN, WHICH ARE IMPOSED BY CENTRIFUGAL LOADING OF THE BLADE
C          ELEMENT VOLUME ABOVE THE BLADE SECTION UNDER CONSIDERATION
C          DO 12 IR=2,5
C             KCLNT=IR
C             SUMSV=C.0
C             SUMSVR=C.0
C             SUMSVT=C.0
C             SUMSVA=C.0
C             DC 11 J=KCLNT,5
C                SUMSV=SUMSV+SUM(I)
C                SUMSVR=SUMSVR+SUMR(I)
C                SUMSVT=SUMSVT+SUMT(I)
C                SUMSVA=SUMSVA+SUMA(I)
C             11 CONTINUE
C          12 CONTINUE
C          DETERMINE BLADE ELEMENT VOLUME (FT**3) ABOVE THE BLADE
C          SECTION UNDER CONSIDERATION
C          VOLC(IF)=0.5*(CIA/2.0)*(0.1)*(SUMSV)
C          DETERMINE RADIAL FRACTION OF THE BLADE ELEMENT VOLUME'S CG
C          WITH RESPECT TO THE PROPELLER SHAFT AXIS
C          XBRCGO(IR)=(SUMSVR/SUMSV)/10.0
C          DETERMINE DISTANCE (FEET) FROM THE GENERATOR LINE, NORMAL TO
C          THE PROPELLER SHAFT AXIS, OF THE BLADE ELEMENT VOLUME'S CG
C          WITH RESPECT TO THE PROPELLER SHAFT AXIS
C          BIGTO(IR)=(SUMSVT/SUMSV)

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APPI6740
APPI6750
APPI6760
APPI6770
APPI6780
APPI6790
APPI6800
APPI6810
APPI6820
APPI6830
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APPI6870
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APPI6890
APPI6900
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APPI6920
APPI6930
APPI6940
APPI6950
APPI6960
APPI6970
APPI6980
APPI6990
APPI7000
APPI7010
APPI7020
APPI7030
APPI7040
APPI7050
APPI7060
APPI7070
APPI7080
APPI7090
APPI7100
APPI7110
APPI7120
APPI7130
APPI7140
APPI7150
APPI7160
APPI7170
APPI7180
APPI7190
APPI7200
APPI7210

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SUBROUTINE COEFS(SJ,R75R,KT,KQ)
SUBROUTINE: COEFS
INFLT      OUTPUT
AEDVAC
PDI VD
Z
SJ
R75R

      KT
      KQ

REAL*4  ETAO,WEIGHT,AEDVAC,DIA,N,PE,PDI VC,QS,TC75R,V,
1  RJC NL,RJCNU,R75RCL,R75RCU,AEACCU,TC75CL,TC75CU,
2  POWBAL,DIA CNU,AEACCU,TC75CL,RJ,
3  VK,TC,WT,Z,WATRO,WATNU,TEMP,NOS CRW,HCL,PATM,
4  PWATVA,PROMAT,DIALIM,ETARR,AEADMA,TC75MN,SC,
5  R75R,KT,KQ,SJ
COMMON /GLOECM/ETAO,WEIGHT,AEDVAC,DIA,N,PE,PDI VD,QS,TC75R,V,RJC NL,
1RJCNU,R75RCL,R75RCU,AEACCU,TC75CL,TC75CU,POWBAL,DIA CNU,
2AEACCU,TC75R,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NOS CRW,HCL,PATM,PWATVA,
1PROMAT,DIALIM,ETARR,AEADMA,TC75MN,SC

CALCULATE THRUST & TORQUE COEFFICIENTS USING THE WAGENINGEN SERIES
POLYNOMIALS GIVEN IN TABLES (5) AND (6), REF 2

CALL CALCKT(SJ,PDI VC,AEDVAC,Z,R75R,KT)
CALL CALCKQ(SJ,PDI VC,AEDVAC,Z,R75R,KQ)
RETURN
END

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SUBROUTINE DICNUA(SIACNU)
SUBROUTINE: DICNUA
DATE OF LAST REVISION: FEB 83
INPUT          OUTPUT          DEFINITION
DIA
DIALIM
SIACNU
REAL*4  ETAD,WEIGHT,AEDVAU,DIA,N,PE,FDIVL,QS,TC75CL,TC75CU,
1      RJCNU,RJCNU,R75RCL,R75RCU,AEACCU,AEACCU,TC75R,V,
2      FCKBAL,DIAACNU,AEACCU,V,TCSTRS,RJ,
3      VK,TL,WT,Z,WATRC,WATNU,TEMP,NOSCRW,HCL,PATM,
4      PWATVA,PROMAT,DIALIM,ETARR,AEADOMN,TC75MN,SC,
5      SIACNU
COMMON /GLOECM/ETAC,WEIGHT,AEDVAU,DIA,N,PE,FDIVL,QS,TC75R,V,RJCNU,
1      RJCNU,R75RCL,R75RCU,AEACCU,AEACCU,TC75CL,TC75CU,POWBAL,DIAACNU,
2      AEACCU,TCSTRS,RJ
COMMON /PAR4M/VK,TD,WT,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,PWATVA,
1      PROMAT,DIALIM,ETARR,AEADOMN,TC75MN,SC
DETERMINE CONSTRAINT VARIABLE OF PROPELLER DIAMETER'S UPPER BOUND
CONSTRAINT
SIACNU=(DIA/DIALIM)-1.0
RETURN
END

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SUBROUTINE EXTCCN(2, AEDVAO, TC75R, AEAOCU, TC75CL, TC75CU)					APPI8310
SUBROUTINE: EXTCCN	DATE OF LAST REVISION: FEB 83				APPI8320
INPUT	OUTPUT	DEFINITION			APPI8330
2		NC OF BLADES			APPI8340
AEDVAO		EXPANDED AREA RATIO			APPI8350
TC75R	AEAOCU	BLADE THICKNESS-TO-CHORD RATIO			APPI8360
		EXPANDED AREA RATIO (CONSTRAINT)			APPI8370
	TC75CL	VARIABLE (LOWER BOUND) (AEAOCU<0)			APPI8380
		EXPANDED AREA RATIO (CONSTRAINT)			APPI8390
	TC75CU	VARIABLE (UPPER BOUND) (AEAOCU<0)			APPI8400
		BLADE THICKNESS-TO-CHORD RATIO			APPI8410
		CONSTRAINT VARIABLE (LOWER BOUND)			APPI8420
		(TC75CL<0)			APPI8430
		BLADE THICKNESS-TO-CHORD RATIO			APPI8440
		CONSTRAINT VARIABLE (UPPER BOUND)			APPI8450
		(TC75CU<0)			APPI8460
					APPI8470
					APPI8480
					APPI8490
					APPI8500
					APPI8510
					APPI8520
					APPI8530
					APPI8540
					APPI8550
					APPI8560
					APPI8570
					APPI8580
					APPI8590
					APPI8600
					APPI8610
					APPI8620
					APPI8630
					APPI8640
					APPI8650
					APPI8660
					APPI8670
					APPI8680
					APPI8690
					APPI8700
					APPI8710
					APPI8720
					APPI8730
					APPI8740
					APPI8750
					APPI8760
					APPI8770
					APPI8780

```

REAL*4 2, AEDVAO, TC75R, AEAOCU, TC75CL, TC75CU,
1 AEAOCU, AEAOCU, TC75CL, TC75CU

```

```

SET LIMITS ON EXPANDED AREA RATIO BASED ON NUMBER OF BLADES

```

```

IF (.NOT. (2.EQ.3.0)) GO TO 1
  AEAOCU=0.35
  AEAOCU=0.80
  GO TO 6
1 CONTINUE
IF (.NOT. (2.EQ.4.0)) GO TO 2
  AEAOCU=0.40
  AEAOCU=1.00
  GO TO 6
2 CONTINUE
IF (.NOT. (2.EQ.5.0)) GO TO 3
  AEAOCU=0.45
  AEAOCU=1.05
  GO TO 6
3 CONTINUE
IF (.NOT. (2.EQ.6.0)) GO TO 4
  AEAOCU=0.50
  AEAOCU=0.80
  GO TO 6
4 CONTINUE
IF (.NOT. (2.EQ.7.0)) GO TO 5
  AEAOCU=0.55
  AEAOCU=0.85

```



APP18790  
 APP18800  
 APP18810  
 APP18820  
 APP18830  
 APP18840  
 APP18850  
 APP18860  
 APP18870  
 APP18880  
 APP18890  
 APP18900  
 APP18910  
 APP18920  
 APP18930  
 APP18940  
 APP18950  
 APP18960  
 APP18970  
 APP18980

```

5  GO TO 6
   CONTINUE
   AEACCL=0.35
   AEACUP=1.05
6  CONTINUE
   DETERMINE CCNSTRAINT VARIABLES FOR EXPANDED AREA RATIO CCNSTRAINT
   AEACCL=AEADLC-AEDVAC
   AEACCU=AEVAD-AEADUP
   DETERMINE CCNSTRAINT VARIABLES FOR BLADE THICKNESS-TU-CHORD RATIO
   CCNSTRAINT
   TC75LQ=(0.5)*((0.0185-0.00125*Z)*Z)/((2.073*AEADUP))
   TC75UP=(4.0)*((0.0185-0.00125*Z)*Z)/((2.073*AEADLQ))
   TC75CL=TC75LQ-TC75R
   TC75CU=TC75F-TC75UP
   RETURN
   END

```







```

1      (8.0-(8.0*X0)-(3.0*(X0**2))-(2.0*(X0**3)))+
2      (5.0*(X0**4)))*
3      (SQR(1.0-C-X0))
1      GAM2(IF)=(2.0/15.0)*
2      (4.0-(5.0*X0)-(2.0*(X0**2))+(3.0*(X0**3)))*
1      CONTINUE
C      CALCULATE THRUST DEVELOPED BY PROPELLER
C
C      T=(KT*KATRO*(DIA**4)*((N/60.0)**2))
C
C      CALCULATE TORQUE REQUIRED BY PROPELLER
C
C      Q=(KQ*KATRO*(DIA**5)*((N/60.0)**2))
C
C      CALCULATE BENDING MOMENTS DUE TO THRUST ( BMT(IR) ) AND TORQUE
C      ( BMQ(IF) ) ALONG THE PROPELLER RADIUS USING RELATIONS (10) AND
C      (26) RESPECTIVELY, REF 8
C
C      DO 2 IR=2,10
C      BMT(IR)=(T*(DIA/2.0))/Z)*(PHI2(IR)/PHI1(2))
C      BMQ(IR)=(Q/Z)*(GAM2(IR)/PHI1(2))
2      CONTINUE
C
C      RESOLVE CALCULATED BENDING MOMENTS INTO COMPONENTS NORMAL AND
C      PARALLEL TO PITCH REFERENCE (CHORD) LINE FOR EACH BLADE SECTION
C      ALONG THE PROPELLER RADIUS USING RELATIONS (42) AND (43), REF 8
C      REMEMBER TO ACCOUNT FOR PITCH REDUCTION OF FOUR BLADE (Z=4.0)
C      PROPELLERS
C
C      DO 10 IF=2,10
C      IF(.NOT.(Z.EQ.4.0))GO TO 8
C      IF(.NOT.(IR.EQ.2))GO TO 3
C      RF=0.822
C      GO TO 7
C      CONTINUE
C      IF(.NOT.(IR.EQ.3))GO TO 4
C      RF=0.887
C      GO TO 7
C      CONTINUE
C      IF(.NOT.(IR.EQ.4))GO TO 5
C      RF=0.950
C      GO TO 7
C      CONTINUE
C      IF(.NOT.(IR.EQ.5))GO TO 6
C      RF=C.992
C      GO TO 7

```





```

6 CCNT INUE
7 RF=1.00
  CCNT INUE
  DENOM=SQRT(((RF*PDIVD)**2)+((PII**2)))
  HAPN(IR)=(BMT(IR)*((PII/DENOM)))+(
1    (BMC(IR)*((RF*PDIVD)/DENOM)))-
1    HAPL(IR)=(BMT(IR)*((PII/DENOM))
      (BMC(IR)*((PII/DENOM)))
      GC TC 5
8 CCNT INUE
  DENOM=SQRT((PCIVC**2)+((PII**2)))
  HAPN(IR)=(BMT(IR)*((PII/DENOM)))+(
1    (BMQ(IR)*((PCIVD/DENOM)))-
1    HAPL(IR)=(BMT(IR)*((PCIVD/DENOM)))-
      (BMQ(IR)*((PII/DENOM)))
      CCNT INUE
10 CONTINUE
  RETURN
  END

```

```

APP15570
APP15580
APP15590
APP20000
APP20010
APP20020
APP20030
APP20040
APP20050
APP20060
APP20070
APP20080
APP20090
APP20100
APP20110
APP20120
APP20130
APP20140
APP20150

```



```

SUBROUTINE JCNA(RJ,RJCNL,RJCNU)
SUBROUTINE: JCNA
INPLT      OUTPUT
RJ          RJCNL
           RJCNU

REAL*4 RJ,RJCN,RJCNL,RJCNU

DETERMINE CCNSTRANT VARIABLE FOR ADVANCE CCEFFICIENT CONSTRAINT
RJCCN=RJ/1.6
RJCNL=C.C-RJCCN
RJCNU=RJCCN-1.0
RETURN
END

```

CCCCCCCCCCCCCCCC

```

DATE OF LAST REVISION: FEB 83
DEFINITION
ADVANCE COEFFICIENT
CCNSTRANT VARIABLE FOR AD-
VANCE COEFFICIENT (LOWER BOUND
(RJCNL<0)
CCNSTRANT VARIABLE FOR AD-
VANCE COEFFICIENT (UPPER BOUND
(RJCNU)

```

APP2C180  
APP2C190  
APP2C200  
APP2C210  
APP2C220  
APP2C230  
APP2C240  
APP2C250  
APP2C260  
APP2C270  
APP2C280  
APP2C290  
APP2C300  
APP2C310  
APP2C320  
APP2C330  
APP2C340  
APP2C350  
APP2C360  
APP2C370  
APP2C380  
APP2C390



```

SUROUTINE CPWEFF(RJ,KT,KQ,ETAO)
SUBROUTINE: CPWEFF
      INPUT      OUTPUT
      RJ
      KT
      KQ
      ETAO
      REAL*4 FJ,K1,KQ,ETAC,PII
      PII=3.141592654
      CALCULATE GFEN WATER EFFICIENCY
      IF (KT.LE.0.0)KT=0.0
      IF (KQ.LE.0.0)KQ=0.0001
      ETAC=((RJ*KT)/(2.0*PII*KQ)
      RETURN
      END

```

```

C
C
C
C
C
C
C
C
C
C

```

```

DATE OF LAST REVISION: FEB 83
DEFINITION
ADVANCE RATIO
THRUST COEFFICIENT
TORQUE COEFFICIENT
OPEN WATER EFFICIENCY
APP2C420
APP2C430
APP2C440
APP2C450
APP2C460
APP2C470
APP2C480
APP2C490
APP2C500
APP2C510
APP2C520
APP2C530
APP2C540
APP2C550
APP2C560
APP2C570
APP2C580
APP2C590
APP2C600
APP2C610
APP2C620

```



```

SUBROUTINE FDCAL(RCIA)
SUBROUTINE: RDCAL
INPUT
V
N
RJ
WT
RCIA
REAL*4
1
2
3
4
5
COMMON
1RJCNL
2AEACCV
COMMON
1PRCMAT
CALCULATE PROPELLER DIAMETER
RDIA=(V*(1.0-WT))/((N/60.0)*RJ)
RETURN
END

```

DATE OF LAST REVISION: FEB 83

DEFINITION

VELOCITY (FT/SEC)  
 PROPELLER REVOLUTION RATE(RPM)  
 ADVANCE RATIO  
 WAKE FRACTION  
 PROPELLER DIAMETER (FEET)  
 PDIVC, QS, TC75R, V,  
 AEAOCL, AEACCU, TC75CL, TC75CU,  
 TCSTRS, RJ,  
 NOSCRW, HCL, PATM,  
 AEADOMN, TC75MN, SC,  
 DIA, N, PE, FUIVU, QS, TC75R, V, RJCNL,  
 TC75CL, TC75CU, POWBAL, DIACNU,  
 TC75MN, SC,  
 TEMP, NCSCRW, HCL, PATM, PWATVA,

APP20650  
 APP20660  
 APP20670  
 APP20680  
 APP20690  
 APP20700  
 APP20710  
 APP20720  
 APP20730  
 APP20740  
 APP20750  
 APP20760  
 APP20770  
 APP20780  
 APP20790  
 APP20800  
 APP20810  
 APP20820  
 APP20830  
 APP20840  
 APP20850  
 APP20860  
 APP20870  
 APP20880  
 APP20890  
 APP20900  
 APP20910  
 APP20920  
 APP20930  
 APP20940





```

SUBROUTINE FEYCNA(R75R,R75RCL,R75RCU)
SUBROUTINE: REYCNA
INPUT      OUTPUT
R75R
R75R      R75RCL
          R75RCU

REAL*4 F75R,R75RCN,R75RCL,R75RCU
DETERMINE CCNSTRANT VARIABLE FOR CORRECTED REYNOLDS NO. CCNSTRANT
R75RCN=F75R/200000.0
R75RCL=1.0-F75RCN
R75RCU=R75RCN-1000.0
RETURN
END

```

CCCCCCCCCCCCCCCC

DATE OF LAST REVISION: FEB 83

DEFINITION

REYNOLDS NO.  $3/4$  RADIUS  
 (CORRECTED FOR T/C  $3/4$  RADIUS  
 CCNSTRANT VARIABLE FOR KEY-  
 NOLDS NO. (CORRECTED)  
 (LOWER BOUND) (R75RCL<0)  
 CCNSTRANT VARIABLE FOR KEY-  
 NOLDS NO. (CORRECTED)  
 (UPPER BOUND) (R75RCU<0)

APP20970  
 APP20980  
 APP20990  
 APP21000  
 APP21010  
 APP21020  
 APP21030  
 APP21040  
 APP21050  
 APP21060  
 APP21070  
 APP21080  
 APP21090  
 APP21100  
 APP21110  
 APP21120  
 APP21130  
 APP21140  
 APP21150  
 APP21160  
 APP21170  
 APP21180  
 APP21190  
 APP21200



SUBROUTINE FEY75R(C75R,R75R)

SUBROUTINE: REY75R

DATE OF LAST REVISION: FEB 83

INPUT OUTPUT

C75R

CHRD LENGTH AR 3/4 RADIUS

N

PROPELLER REVOLUTIONS (RPM)

DIA

PROPELLER DIAMETER (FEET)

Z

NO. OF BLADES

TC75R

BLADE SECTION THICKNESS-TO-

AEDVAO

CHRD RATIO AT 3/4 RADIUS

WATNU

EXPANDED AREA RATIO

R75R

KINEMATIC VISCOSITY

REAL\*4

ETAO,WEIGHT,AEDVAO,DIA,N,PE,POIVD,QS,TC75R,V,

1

RJCNL,RJCNL,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,

2

FCWBAL,DIACNU,AEAOCLV,TCSTKS,RJ,NCSCRW,HCL,PATM,

3

VK,TC,WI,Z,WATRG,WATNU,TEMP,AEACMN,TC75MN,SC,

4

PWATVA,PRGMAT,DIALIM,ETARR,AEACMN,TC75MN,SC,

5

C75R,PII,VA,R75RUN,TC75WS,NUM,DECOM,R75R

COMMON

/GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,POIVD,QS,TC75R,V,RJCNL,

1

RJCNL,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,

2

AEACCU,TCSTKS,RJ

COMMON

/PARAM/VK,TC,WI,Z,WATRG,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,

1

PRGMAT,DIALIM,ETARR,AEACMN,TC75MN,SC

PII=3.141592654

CALCULATE SPEED OF ADVANCE

VA=V\*(1.0-W1)

CALCULATE UNCORRECTED REYNOLDS NUMBER AT 3/4 PROPELLER RADIUS

USING RELATION (10), REF 2

R75RUN=(1.0/WATNU)\*C75R\*SQR((VA\*\*2)+((.75\*PII\*(N/60.0)\*DIA)\*\*2))

DETERMINE BLADE THICKNESS-TO-CHRD RATIO BASED ON A SPECIFIC "Z"

"AEDVAC" USING RELATION (11), REF 2

TC75WS=((0.0185-(0.00125\*Z))\*Z)/(2.073\*AEDVAO)

DETERMINE REYNOLDS NUMBER AT 3/4 RADIUS, CORRECTED FOR BLADE

THICKNESS-TO-CHRD RATIO "TC75R", USING RELATION (12), REF 2

APP21230  
APP21240  
APP21250  
APP21260  
APP21270  
APP21280  
APP21290  
APP21300  
APP21310  
APP21320  
APP21330  
APP21340  
APP21350  
APP21360  
APP21370  
APP21380  
APP21390  
APP21400  
APP21410  
APP21420  
APP21430  
APP21440  
APP21450  
APP21460  
APP21470  
APP21480  
APP21490  
APP21500  
APP21510  
APP21520  
APP21530  
APP21540  
APP21550  
APP21560  
APP21570  
APP21580  
APP21590  
APP21600  
APP21610  
APP21620  
APP21630  
APP21640  
APP21650  
APP21660  
APP21670  
APP21680  
APP21690  
APP21700



APP21710  
APP21720  
APP21730  
APP21740  
APP21750

```
NUM=1.0+(2.0*TC75WS)  
DENCM=1.0+(2.0*TC75R)  
R75R=EXP(4.6052+((SQRT(NUM/DENUM))* (ALCG(R75RUN)-4.6052)))  
RETURN  
END
```



```

SUBROUTINE FJCAL(J)
SUBROUTINE: RJCAL
INPLT      CLTPUT
V
DIA
N
WT
J
REAL*4  ETAO,WEIGHT,AEDVAD,DIA,N,PE,PDI,VC,QS,TC75R,V,
1  RJCNL,RJCNL,R75RCU,R75FCL,R75RCU,AEACCL,AEACCU,TC75CL,TC75CU,
2  POWBAL,DIACNU,AEACCV,TCSTRS,RJ,CRW,HCL,PATM,
3  VK,TC,WT,Z,WATRO,WATNU,TEMP,NQSCRW,HCL,PATM,SC,
4  PWATVA,PROMAT,DIALIM,ETARK,AEADMA,TC75MN,SC,
5  COMMON /GLOECM/ETAC,WEIGHT,AEDVAD,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1  RJCNL,R75RCU,AEACCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2  AEACCV,TCSTRS,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NQSCRW,HCL,PATM,PWATVA,
1  PROMAT,DIALIM,ETARR,AEACMN,TC75MN,SC
CALCULATE ADVANCE RATIO
J=(V*(1.0-WT))/(DIA*(N/60.0))
RETURN
END

```

APP21780  
 APP21790  
 APP21800  
 APP21810  
 APP21820  
 APP21830  
 APP21840  
 APP21850  
 APP21860  
 APP21870  
 APP21880  
 APP21890  
 APP21900  
 APP21910  
 APP21920  
 APP21930  
 APP21940  
 APP21950  
 APP21960  
 APP21970  
 APP21980  
 APP21990  
 APP22000  
 APP22010  
 APP22020  
 APP22030  
 APP22040  
 APP22050  
 APP22060  
 APP22070

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```

SUBROUTINE FNCAL(RN)
SUBROUTINE: RNCAL
INPUT      OUTPUT      DATE OF LAST REVISION: FEB 83
V          DIA          DEFINITION
RJ          PROPELLER DIAMETER (FEET)
WT          ADVANCE KATIO
           WAKE FRACTION
           PROPELLER REVOLUTION KATE(RPM)
           RN
REAL*4     ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,
1          RJCNU,RJCNU,R75RCL,R75RCU,AEAOCL,TC75CL,TC75CU,
2          POWBAL,DIA,CNU,AEAOCL,TC75CL,TC75CU,POWBAL,DIA,CNU,
3          VK,TC,WT,Z,WATRO,WATNU,TEMP,NCS,CRW,HCL,PATM,
4          PWATVA,PROMAT,DIALIM,ETARK,AEACMN,TC75MN,SC,
5          RN
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,RJCNU,
1RJCNU,R75RCL,R75RCU,AEAOCL,TC75CL,TC75CU,POWBAL,DIA,CNU,
2AEAOCL,TC75CL,TC75CU,POWBAL,DIA,CNU,
3COMMON /PAR/M/VK,TC,WT,Z,WATRO,WATNU,TEMP,NCS,CRW,HCL,PATM,PWATVA,
1PROMAT,DIALIM,ETARK,AEACMN,TC75MN,SC
CALCULATE PROPELLER REVOLUTION KATE
RN=60.C*((V*(1.0-WT))/(DIA*KJ))
RETURN
END

```

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SUBROUTINE SIGNDS	DATE OF LAST REVISION:	APR 83
SUBROUTINE: SIGNDS	DEFINITION	
INPUT		
COMMON /AREELD/ COMMON /VAL11/	BLADE SECTION AREAS (FEET**2) ORIGINATE OF CRITICAL POINT NO. 1 ON BLADE SECTION PERIPHERY WITH RESPECT TO A SYSTEM OF AXES PARALLEL AND NORMAL TO PITCH-REFERENCE LINE WITH ORI- GIN AT THE CENTROID OF THE SECTION (FEET) ABSCISSA OF CRITICAL POINT NC. 1 ON BLADE SECTION PERIPHERY WITH RESPECT TO A SYSTEM OF AXES PARALLEL AND NORMAL TO PITCH-REFERENCE LINE WITH ORI- GIN AT THE CENTROID OF THE SECTION (FEET) ORIGINATE OF CRITICAL POINT NO. 2 ON BLADE SECTION PERIPHERY WITH RESPECT TO A SYSTEM OF AXES PARALLEL AND NORMAL TO PITCH-REFERENCE LINE WITH ORI- GIN AT THE CENTROID OF THE SECTION (FEET) ABSCISSA OF CRITICAL POINT NC. 2 ON BLADE SECTION PERIPHERY WITH RESPECT TO A SYSTEM OF AXES PARALLEL AND NORMAL TO PITCH-REFERENCE LINE WITH ORI- GIN AT THE CENTROID OF THE SECTION (FEET) ORIGINATE OF CRITICAL POINT NO. 3 ON BLADE SECTION PERIPHERY WITH RESPECT TO A SYSTEM OF AXES PARALLEL AND NORMAL TO PITCH-REFERENCE LINE WITH ORI- GIN AT THE CENTROID OF THE SECTION (FEET) ABSCISSA OF CRITICAL POINT NC. 3 ON BLADE SECTION PERIPHERY WITH RESPECT TO A SYSTEM OF AXES PARALLEL AND NORMAL TO PITCH-REFERENCE LINE WITH ORI- GIN AT THE CENTROID OF THE SECTION (FEET)	APP222420 APP222430 APP222440 APP222450 APP222460 APP222470 APP222480 APP222490 APP222500 APP222510 APP222520 APP222530 APP222540 APP222550 APP222560 APP222570 APP222580 APP222590 APP222600 APP222610 APP222620 APP222630 APP222640 APP222650 APP222660 APP222670 APP222680 APP222690 APP222700 APP222710 APP222720 APP222730 APP222740 APP222750 APP222760 APP222770 APP222780 APP222790 APP222800 APP222810 APP222820 APP222830 APP222840 APP222850 APP222860 APP222870 APP222880 APP222890
COMMON /VAL11/		
COMMON /VAL12/		
COMMON /VAL12/		
COMMON /VAL13/		
COMMON /VAL13/		



SECTION (FEET)  
ORIGINATE OF CRITICAL POINT NO.  
4 ON BLADE SECTION PERIPHERY  
WITH RESPECT TO A SYSTEM OF  
AXES PARALLEL AND NORMAL TO  
PITCH-REFERENCE LINE WITH ORI-  
GIN AT THE CENTROID OF THE  
SECTION (FEET)  
ABSCISSA OF CRITICAL POINT NC  
4 ON BLADE SECTION PERIPHERY  
WITH RESPECT TO A SYSTEM OF  
AXES PARALLEL AND NORMAL TO  
PITCH-REFERENCE LINE WITH ORI-  
GIN AT THE CENTROID OF THE  
SECTION (FEET)  
MOMENT OF INERTIA ABOUT NEU-  
TRAL AXIS PARALLEL TO GENERA-  
TOR LINE, PASSING THROUGH  
BLADE CROSS SECTION CENTROID  
(FEET\*4)  
MOMENT OF INERTIA ABOUT NEU-  
TRAL AXIS PARALLEL TO PITCH  
REFERENCE LINE, PASSING THROUGH  
BLADE CROSS SECTION CENTROID  
HYDRODYNAMIC MOMENTS PARALLEL  
TO BLADE SECTION PITCH REFER-  
ENCE (CFORU) LINE (FT-LBF)  
HYDRODYNAMIC MOMENTS NORMAL  
TO BLADE SECTION PITCH REFER-  
ENCE (CFORD) LINE (FT-LBF)  
CENTRIFUGAL FORCE COMPONENTS,  
ALONG THE PROPELLER RADIUS,  
PARALLEL TO GENERATOR LINE,  
WHICH ACT ON A BLADE SECTION  
AT ITS NEUTRAL AXES ORIGIN  
(LBFF)  
CENTRIFUGAL BENDING MOMENT COM-  
PONENTS, ALONG THE PROPELLER  
RADIUS, PARALLEL TO THE PITCH  
REFERENCE (CHORD) LINE, WHICH  
ACT ON A BLADE SECTION AT ITS  
NEUTRAL AXES ORIGIN (FT-LBFF)  
CENTRIFUGAL BENDING MOMENT COM-  
PONENTS, ALONG THE PROPELLER  
RADIUS, NORMAL TO THE PITCH  
REFERENCE (CHORD) LINE, WHICH  
ACT ON A BLADE SECTION AT ITS  
NEUTRAL AXES ORIGIN (FT-LBFF)

APP22500  
APP22510  
APP22520  
APP22530  
APP22540  
APP22550  
APP22560  
APP22570  
APP22580  
APP22590  
APP23000  
APP23010  
APP23020  
APP23030  
APP23040  
APP23050  
APP23060  
APP23070  
APP23080  
APP23090  
APP23100  
APP23110  
APP23120  
APP23130  
APP23140  
APP23150  
APP23160  
APP23170  
APP23180  
APP23190  
APP23200  
APP23210  
APP23220  
APP23230  
APP23240  
APP23250  
APP23260  
APP23270  
APP23280  
APP23290  
APP23300  
APP23310  
APP23320  
APP23330  
APP23340  
APP23350  
APP23360  
APP23370

COMMON /VALCL4/  
  
COMMON /VALV4/  
  
COMMON /A2MCMX/  
  
COMMON /A2MCMY/  
  
COMMON /HYDMOMN/  
  
COMMON /HYDMOML/  
  
COMMON /CFGFD/  
  
COMMON /CMCEN/  
  
COMMON /CMCEL/

CC











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```

SUBROUTINE STRCNA(KC,C75R,SCSTRS)
  INPUT      OUTPUT
  PKCMAT
  DIA
  PDIVD
  KC
  C75R
  NOSCRW
  TC75R

  DEFINITION
  PROPELLER MATERIAL IDENTIFIER
  1: CAST IRON
  2: CAST STEEL
  3: TYPE 2 BRONZE
  4: TYPE 4 NI-AL BRONZE
  5: STAINLESS STEEL
  PROPELLER DIAMETER (FT)
  PROPELLER REVOLUTION RATE (RPM)
  PITCH-DIAMETER RATIO
  TORQUE COEFFICIENT
  CHORD LENGTH 3/4 RADIUS (FT)
  NO. OF PROPELLERS
  BLADE THICKNESS-TU-CHORD RATIO
  3/4 RADIUS
  CONSTRAINT VARIABLE FOR BLADE
  THICKNESS-TU-CHORD RATIO 3/4
  RADIUS CONSTRAINT (TCSTRSCO)

  SCSTRS
  REAL*4  ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,
1  RJCNU,RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2  FCWBAL,DIA,CNU,AEAOCL,V,TCSTRS,RJ,
3  VK,TC,WI,Z,WATRC,WATNU,TEMP,NOSCRW,HCL,PATM,
4  PWATVA,PRI,SC,CABSRB,PABSRB,FACID,FAC2D,FACIN,I75MIN
5  C75R,KQ,PII,SC,CABSRB,PABSRB,FACID,FAC2D,FACIN,I75MIN
COMMON /GLOECM/ETAO,WEACCL,AEACCU,TC75CL,TC75CU,PQWBAL,DIACNJ,
1RJCNU,R75RCL,R75RCU,TCSTRS,RJ
2AEACCU,TCSTRS,RJ
COMMON /PAR/M/VK,TC,WI,Z,WATRC,WATNU,TEMP,NOSCRW,HCL,PATM,PWATVA,
1PRGMAT,DIACIN,ETARR,AEAOCL,TC75MN,SC
PII=3.141592654

  DETERMINE MAXIMUM ALLOWABLE STRESS (PSI) BASED ON NUMBER OF PRO-
PELLERS AND MATERIAL IDENTIFIER
  IF(.NOT.(NOSCRW.EQ.1.0)) GO TO 1
  IF(FRCMAT.EQ.1.0)SC=3600.0
  IF(FRCMAT.EQ.2.0)SC=5915.0
  IF(FRCMAT.EQ.3.0)SC=7200.0
  IF(FRCMAT.EQ.4.0)SC=8910.0
  IF(FRCMAT.EQ.5.0)SC=5400.0
  GO TO 2
1 CONTINUE
  IF(FRCMAT.EQ.1.0)SC=3950.0
  IF(FRCMAT.EQ.2.0)SC=6265.0

```

CCCCCCCCCCCCCCCCCCCC

CCCC



```

IF (PROFAT.EQ.3.0) SC=7585.0
IF (PROFAT.EQ.4.0) SC=9430.0
IF (PROFAT.EQ.5.0) SC=5500.0
2 CONTINUE
      DETERMINE AESORBED POWER FOR EACH PROPELLER
      WABSRB=ABS(KQ*WATRC*(DIA**5)*((N/60.0)**2))
      PABSRB=(2.0*PI*I*QABSRB*N)/55000.0
      CALCULATE MINIMUM REQUIRED BLADE THICKNESS-TO-CHORD RATIO AT 3/4
      BLADE RADIUS USING RELATION (16), REF 2
      FACID=SC+(((DIA*N)**2)/(12.788))
      FAC2D=4.123*N*((DIA**3)
      FACIN=(2375.0-(1125.0*PUIVD))*PABSRB
      T75MIN=(((FACIN/(FAC2D*FACID))**0.33333)*0.21)+0.0028)*DIA
      TC75MN=175MIN/C75R
      DETERMINE CCNSTRANT VARIABLE FOR BLADE THICKNESS-TO-CHORD RATIO
      CCNSTRANT
      SCSTRS=(TC75MN/TC75R)-1.0
      RETURN
      END

```

```

APP24780
APP24790
APP24800
APP24810
APP24820
APP24830
APP24840
APP24850
APP24860
APP24870
APP24880
APP24890
APP24900
APP24910
APP24920
APP24930
APP24940
APP24950
APP24960
APP24970
APP24980
APP24990
APP25000
APP25010
APP25020

```

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SUBROUTINE STRCNK(KQ,KT,C75K,SCSTRS)

SUBROUTINE: STRCNK

DATE OF LAST REVISION: APR 83

INPUT

OUTPUT

DEFINITION

PKCMAT

PROPELLER MATERIAL IDENTIFIER

1: CAST IRON  
2: CAST STEEL  
3: TYPE 2 BRONZE  
4: NI-AL BRONZE  
5: STAINLESS STEEL  
PROPELLER DIAMETER (FT)  
PROPELLER REVOLUTION RATE (RPM)  
PITCH-DIAMETER RATIO  
EXPANDED AREA RATIO  
TORQUE COEFFICIENT  
THROUST COEFFICIENT  
CHORD LENGTH 3/4 RADIUS (FT)  
NO. OF PROPELLERS  
ELADE THICKNESS-TO-CHORD RATIO  
3/4 RADIUS  
CONSTRAINT VARIABLE FOR BLADE  
THICKNESS-TO-CHORD RATIO 3/4  
RADIUS CONSTRAINT (SCSTRS<0)

SCSTRS

REAL\*4

1 ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVC,QS,TC75K,V,  
2 RJCNL,RJCNL,R75RCL,R75RCU,AEAOCL,TC75CL,TC75CU,  
3 FCWBZL,CIAACNU,AEAOCL,TCSTRS,  
4 VK,TC,W,T,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PATM,  
5 FWATVA,PRCMAR,KT,KQ  
1 REAL\*4  
1 FJ,C75R,R75R,KT,KQ  
1 T00R,T1XR,T20R,T30R,T40R,T50R,T60R,T70R,T80R,T90R,  
1 T100R,RAI  
1 CR(10),AR(10),BR(10),PLF(10),PLA(10),TR(10),VIF(10,11),  
1 V2F(10,11),V1A(10,10),V2A(10,10),YFAC(11),YBACK(11),  
1 FA(10),Y(11),DELPA(10),AA(10),DLFASM,XA(10),YA(10),SUMAA,  
1 SUMAXA,SUMAYA,SUMAX2A,SUMAY2A,HF(11),DELPP(11),AF(11),DLPPFSM,  
1 XF(11),YF(11),SUMAF,SUMAXF,SUMAYF,SUMAX2F,SUMAY2F,AREA(10),  
1 XMT(10),YPKL(10),RIXNA(10),RIYNA(10),XCG(10),YCG(10),  
1 U1(10),U2(10),U3(10),U4(10),W1(10),W2(10),W3(10),W4(10),  
1 YBK,YFC  
1 REAL\*4  
1 HMPN(10),HMPPL(10),CMCBN(10),CMCEL(10),BIGNO(10),SIG1(10),  
1 SIG2(10),SIG3(10),SIG4(10),TC75MN,TC75WS,SCSTRS,SC  
1 INTEGER\*4  
1 IF,KOUNT  
1 LOGICAL OKAY1,OKAY2,OKAY3,OKAY4  
1 COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75K,V,RJCNL,  
1 RJCNL,R75RCL,R75RCU,AEAOCL,AEAOCL,TC75CL,TC75CU,PUBAL,CIAACNU,





APP255530  
 APP255540  
 APP255550  
 APP255560  
 APP255570  
 APP255580  
 APP255590  
 APP255600  
 APP255610  
 APP255620  
 APP255630  
 APP255640  
 APP255650  
 APP255660  
 APP255670  
 APP255680  
 APP255690  
 APP255700  
 APP255710  
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 APP255890  
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 APP255920  
 APP255930  
 APP255940  
 APP255950  
 APP255960  
 APP255970  
 APP255980  
 APP255990  
 APP26000

2AEACCV,7CSTFS,RJ  
 COMMON /FARM/VK,TD,WT,2,WATRQ,WATNL,TEMP,NCSCRW,HCL,PATM,PWATVA,  
 1PROMAT,LIALIM,ETARR,AEACMN,IC75MN,SC  
 COMMON /IHC/TOR,ILXR,T20R,T30R,T40R,T50R,T60R,T70R,T75R,T80R,  
 1T90R,T1CCR,FAT  
 COMMON /ARELD/AREA  
 COMMON /CGX/XCG  
 COMMON /CGY/YCG  
 COMMON /CRDINT/CR  
 COMMON /VAL1/U1  
 COMMON /VAL2/U2  
 COMMON /VAL3/U3  
 COMMON /VAL4/U4  
 COMMON /VAL1/W1  
 COMMON /VAL2/W2  
 COMMON /VAL3/W3  
 COMMON /VAL4/W4  
 COMMON /A2MCX/R1YYNA  
 COMMON /A2MCY/R1XXNA  
 COMMON /HYMCML/HMPL  
 COMMON /CFGEMN/CMCBN  
 COMMON /CFGEML/CMCBL  
 COMMON /CFGFL/BIGNO  
 COMMON /STRES1/SIG1  
 COMMON /STRES2/SIG2  
 COMMON /STRES3/SIG3  
 COMMON /STRES4/SIG4

C  
 C  
 C  
 C  
 DETERMINE MAXIMUM ALLOWABLE STRESS (PSI) BASED UPON THE NUMBER OF  
 PROPELLERS AND TYPE OF MATERIAL

IF(.NOT.(NOSCRW.EQ.1.0)) GO TO 1  
 IF(PROMAT.EQ.1.0)SC=3600.0  
 IF(PROMAT.EQ.2.0)SC=5515.0  
 IF(PROMAT.EQ.3.0)SC=7200.0  
 IF(PROMAT.EQ.4.0)SC=8510.0  
 IF(PROMAT.EQ.5.0)SC=5400.0  
 GO TO 2  
 1 CONTINUE  
 IF(PROMAT.EQ.1.0)SC=3550.0  
 IF(PROMAT.EQ.2.0)SC=6265.0  
 IF(PROMAT.EQ.3.0)SC=7585.0  
 IF(PROMAT.EQ.4.0)SC=9430.0  
 IF(PROMAT.EQ.5.0)SC=5500.0  
 2 CONTINUE

C  
 C  
 INITIALIZE MINIMUM REQUIRED BLADE THICKNESS-TC-CHORD RATIO AT 3/4







```

DC 8 IF=2,S
IF(.NOT.(ABS(SIG1(IR)).GE.(SC#144.0)))GO TO 4
      CKA Y1=.FALSE.
CCNT INUE
IF(.NOT.(ABS(SIG2(IR)).GE.(SC#144.0)))GO TO 5
      CKA Y2=.FALSE.
CCNT INUE
IF(.NOT.(ABS(SIG3(IR)).GE.(SC#144.0)))GO TO 6
      CKA Y3=.FALSE.
CCNT INUE
IF(.NOT.(ABS(SIG4(IR)).GE.(SC#144.0)))GO TO 7
      CKA Y4=.FALSE.
CCNT INUE
      CCNT INLE
      IF(.NOT.((OKAY1).AND.(CKAY2).AND.(OKAY3).AND.(OKAY4)))GC TO 3
      IF(.NOT.((CKAY1).AND.(CKAY2).AND.(CKAY3).AND.(CKAY4)).OR.
1    (KCUNT.EQ.100))GC TO 3
      HAVING DETERMINED MINIMUM REQUIRED ELASE THICKNESS-TO-CHORD
RATIO AT 3/4 PROPELLER RADIUS, CALCULATE CONSTRAINT VARIABLE
SCSTRS=(TC75MN/TC75R)-1.0
RETURN
END

```









APP27230  
APP27240  
APP27250  
APP27260  
APP27270  
APP27280  
APP27290  
APP27300  
APP27310  
APP27320  
APP27330  
APP27340  
APP27350  
APP27360  
APP27370  
APP27380  
APP27390  
APP27400  
APP27410  
APP27420  
APP27430  
APP27440  
APP27450  
APP27460

```

IF(.NOT.(Z.EQ.5.0))GO TO 3
  TCCR=0.040*DIA
  T1XR=RAT*DIA*(0.054646-(0.004165*Z))
  GO TO 5
3 CONTINUE
IF(.NOT.(Z.EQ.6))GO TO 4
  TCCR=0.035*DIA
  T1XR=RAT*DIA*(0.054646-(0.004165*Z))
  GO TO 5
4 CONTINUE
  TCCR=0.035*DIA
  T1XR=RAT*DIA*(0.05384-(0.0041*Z))
  GO TO 5
5 CONTINUE
  T20R=RAT*CL1*(0.0526-(0.0040*Z))
  T30R=RAT*CL1A*(0.0464-(0.0035*Z))
  T40R=RAT*CL1A*(0.0402-(0.0030*Z))
  T50R=RAT*CL1A*(0.0340-(0.0025*Z))
  T60R=RAT*CL1A*(0.0278-(0.0020*Z))
  T70R=RAT*CL1A*(0.0216-(0.0015*Z))
  T80R=RAT*CL1A*(0.0154-(0.0010*Z))
  T90R=RAT*CL1A*(0.0092-(0.0005*Z))
  T100R=RAT*CL1A*(0.0030-(0.0000*Z))
  RETLNRN
END

```



SUBROUTINE WGTAL(C75R)

SUBROUTINE: WGTAL

DATE OF LAST REVISION: MAR 83

INPUT OUTPUT

AEDVAO  
C75R

DIA  
PRCMAT

TC75R

Z

WEIGHT

REAL\*4

1 ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVC,QS,TC75R,V,  
2 RJCNL,RJCNL,R75RCL,R75RCL,AEAOCL,AEAOCL,TC75CL,TC75CU,  
3 FCWBL,DIA,AEAOCL,AEAOCL,TCSTFS,  
4 VK,TL,W,T,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,  
5 FWAIVA,PRCMAT,DIALIN,ETARR,  
1 TCOR,T1XR,T2OR,T3OR,T4OR,T5OR,T6CR,T7OR,T75R,T8OR,T9OR,  
1 TICOR,RAT  
1 CR(10),AR(10),BR(10),PLF(10),PLA(10),TR(10),VIF(10,11),  
2 V2F(10,11),VIA(10,10),V2A(10,10),YFACE(11),YBACK(11),  
3 FA(10),Y(11),DELPA(10),AA(10),DLFASM,XA(10),YA(10),SUMAA,  
4 SUMAXA,SUMAYA,SUMAX2A,SUMAY2A,HF(11),DELPF(11),AF(11),DLPFSM,  
5 XF(11),YF(11),SUMAF,SUMAXF,SMAY2F,SMAY2F,AREA(10),  
1 XMT(10),YPR(10),RIXXNA(10),RIYXNA(10),XCG(10),YCG(10),  
2 U1(10),U2(10),U3(10),U4(10),W1(10),W2(10),W3(10),W4(10),  
3 YBK,YFC  
1 WCL,VCLBLD  
1 COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVC,QS,TC75R,V,RJCNL,  
2 RJCNL,R75RCL,R75RCL,AEAOCL,AEAOCL,TC75CL,TC75CU,PWMBAL,DIA,AEAOCL,  
3 AEAOCL,TCSTFS,RJ  
1 COMMON /PAR/WM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,  
2 PRCMAT,DIALIN,ETARR,AEADMN,TC75MN,SC  
1 COMMON /THICC/TOUR,T1XR,T2OR,T3OR,T4OR,T5OR,T6OR,T7OR,T75R,T8OR,  
2 T9OR,T1CCR,FAT  
1 COMMON /AREELD/AREA  
1 COMMON /CGX/XCG  
1 COMMON /CGY/YCG

APP27490  
APP27500  
APP27510  
APP27520  
APP27530  
APP27540  
APP27550  
APP27560  
APP27570  
APP27580  
APP27590  
APP27600  
APP27610  
APP27620  
APP27630  
APP27640  
APP27650  
APP27660  
APP27670  
APP27680  
APP27690  
APP27700  
APP27710  
APP27720  
APP27730  
APP27740  
APP27750  
APP27760  
APP27770  
APP27780  
APP27790  
APP27800  
APP27810  
APP27820  
APP27830  
APP27840  
APP27850  
APP27860  
APP27870  
APP27880  
APP27890  
APP27900  
APP27910  
APP27920  
APP27930  
APP27940  
APP27950  
APP27960







# APPENDIX C

## ANALIZ CODES--DESIGN CASE NO. 1

```

SUBROUTINE ANALIZ(ICALC)
  INTEGER*4 ICALC
  REAL*4
    1 ETAO,WEIGHT,AEDVAO,DIA,N,PE,POIVC,QS,TC75R,V,
    2 RJCNU,RJCNU,R75FCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
    3 FOWBAL,DIACNU,AEAOCL,V,ICSTRS,RJ,
    4 VK,TC,WI,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PATM,
    5 PWATVA,PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
    6 C75R,R75R,KT,KC,PD
  COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FOIVD,QS,TC75R,V,RJCNU,
  1 RJCNU,R75FCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,PWMBAL,DIACNU,
  2 AEACCU,ICSTRS,RJ
  COMMON /PAR1M/VK,TC,WI,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
  1 PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

  THIS SUBROUTINE, COUPLED WITH COPEX/CONMIN, CONSTITUTES ANALYSIS
  METHOD FOR "DESIGN CASE 1" PROPELLER SELECTION PROBLEMS

  INPUT-INITIALIZATION PHASE

  PI1=3.14159264
  IF(.NOT.(ICALC.EQ.1))GO TO 1

  SET "DESIGN CASE 1" PARAMETERS

  ENVIRONMENTAL
    TEMP=55.0
    WATRO=1.9384
    WATNU=.0000122850
    PATM=14.7
    PWATVA=.247

  PROPELLER PARAMETERS
    Z=5.0
    PROMAT=5.0

  HULL PARAMETERS
    WT=C.22
    TC=C.15
    ETARR=1.025
    NCSCRW=1.0
    HCL=19.0
    DIALIM=22.0

  SET DESIGN VARIABLES HELD FIXED FOR "DESIGN CASE 1"

```





APP00C490  
APP00C500  
APP00C510  
APP00C520  
APP00C530  
APP00C540  
APP00C550  
APP00C560  
APP00C570  
APP00C580  
APP00C590  
APP00C600  
APP00C610  
APP00C620  
APP00C630  
APP00C640  
APP00C650  
APP00C660  
APP00C670  
APP00C680  
APP00C690  
APP00C700  
APP00C710  
APP00C720  
APP00C730  
APP00C740  
APP00C750  
APP00C760  
APP00C770  
APP00C780  
APP00C790  
APP00C800  
APP00C810  
APP00C820  
APP00C830  
APP00C840  
APP00C850  
APP00C860  
APP00C870  
APP00C880  
APP00C890  
APP00C900  
APP00C910  
APP00C920  
APP00C930  
APP00C940  
APP00C950  
APP00C960

```

PE=18153.0
VK=24.C
V=1.68E*VK
AECVAD=0.85
DIA=22.0
TC75R=((0.0185-U.C0125*Z)*Z)/(2.073*AECVAD)

CC
CC
CC
C
C
C

      END OF INPUT-INITIALIZATION PHASE

      GO TO 3

      EXECUTION PHASE

1 CONTINUE
  IF(.NOT.(ICALC.EQ.2))GC TO 2
    CALL RNCAL(N)
    CALL C75RA(C75R)
    CALL REY75R(C75R,R75R)
    CALL CEFSA(RJ,KI,KQ)
    CALL OFWEFF(RJ,KI,KQ,ETAU)
    CALL CALCS(KC,QS)
    CALL JCNA(RJ,RJCNL,RJCNL)
    CALL REYCNAL(R75R,R75RCL,R75RCL)
    CALL EXTCCN(Z,AECVAD,TC75R,AECCL,AECU,TC75CL,TC75CU)
    CALL BLPOW(KI,PUWBAL)
    CALL DICNUA(DIACNU)
    CALL CAVCNA(KI,AECV)
    CALL STRCNA(KC,C75R,TCSTRS)

    CC
    CC
    CC

      END OF EXECUTION PHASE

      GO TO 3

2 CONTINUE

      OUTPUT-RESULT PHASE

      PD=(2.C*PII*QS*N)/33000.0
      WRITE(6,9000)
      WRITE(6,9001)
      WRITE(6,9002) TEMP,WATRO,WAINU,PATN,PWATVA
      WRITE(6,9003) WT,TD,ETAKR,NOSCRW,HCL,DIALIM
      IF(.NOT.(PRCMAT.EQ.1.C))GC TO 81
        WRITE(6,9005)SC
      GC TO 86
      CCNTINLE
      IF(.NOT.(PRCMAT.EQ.2.0))GC TO 82
        WRITE(6,9006)SC

      81

```



APP0C970  
 APP0C980  
 APP0C990  
 APP01000  
 APP01010  
 APP01020  
 APP01030  
 APP01040  
 APP01050  
 APP01060  
 APP01070  
 APP01080  
 APP01090  
 APP01100  
 APP01110  
 APP01120  
 APP01130  
 APP01140  
 APP01150  
 APP01160  
 APP01170  
 APP01180  
 APP01190  
 APP01200  
 APP01210  
 APP01220  
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 APP01240  
 APP01250  
 APP01260  
 APP01270  
 APP01280  
 APP01290  
 APP01300  
 APP01310  
 APP01320  
 APP01330  
 APP01340  
 APP01350  
 APP01360  
 APP01370  
 APP01380  
 APP01390  
 APP01400  
 APP01410  
 APP01420  
 APP01430  
 APP01440

```

82      GC TC E6
      CCNTINLE
      IF(.NOT.(PROMAT.EQ.3.0))GO TO 83
      WRITE(6,9007)SC

83      GC TC E6
      CCNTINLE
      IF(.NOT.(PROMAT.EQ.4.0))GO TO 84
      WRITE(6,9008)SC

84      GC TC E6
      CCNTINLE
      IF(.NOT.(PROMAT.EQ.5.0))GO TO 85
      WRITE(6,9009)SC

85      GC TC E6
      CCNTINLE
      WRITE(6,9010)

86      CCNTINLE
      WRITE(6,9011)PE,V,N,QS,PD,KJ,KT,KC,ETAC,R75R,DIA,PDIVD,
      AED,VAO,TC75R

1      WRITE(6,9012)DIALIM,AEAUMN,TC75MN

3 CONTINUE
  RETURN

C MISCELLANEOUS FORMAT STATEMENTS
C
9000 FORMAT(1X,OPTIMIZATION RESULTS ----- DESIGN CASE NO.1,/,
1X,DESIGN VARIABLES SPECIFIED: SUBROUTINE "STRCNAME",/,
9001 FORMAT(1X,ENVIRONMENTAL PARAMETERS: PE,V,AED,VAO,DIA,TC75R,/,
9002 FORMAT(1X,4,/,
1X,25X,DENSITY (LBF-IN/FT3),12X,=,F10.4,/,
1X,25X,VISCOSITY (FT2/SEC),15X,=,E16.9,/,
1X,25X,ATMOSPHERIC PRESSURE (PSIA),7X,=,F10.4,/,
1X,25X,WATER VAPORIZATION PRESSURE (PSIA)=,F10.4,/,
9003 FORMAT(1X,FULL PARAMETERS:,13X,WAKE FUNCTION,21X,=,F10.4,/,
1X,25X,THICKNESS DEDUCTION EFFICIENCY,6X,=,F10.4,/,
1X,25X,RELATIVE ROTATIVE EFFICIENCY,6X,=,F10.1,/,
1X,25X,NUMBER OF PROPELLERS,14X,=,F10.1,/,
1X,25X,DEPTH TO SHAFT CENTERLINE (FT),4X,=,F10.4,/,
1X,25X,DIAMETER LIMIT (FT),15X,=,F10.4,/,
9004 FORMAT(1X,FROPELLER PARAMETERS:,8X,NUMBER OF BLADES,16X,=,
1X,25X,1),
9005 FORMAT(1X,25X,MATERIAL TYPE,21X,= CAST IRON,/,
1X,25X,ALLCABLE STRESS (PSI),12X,=,F10.1,/,
9006 FORMAT(1X,25X,MATERIAL TYPE,21X,= CAST STEEL,/,
1X,25X,ALLCABLE STRESS (PSI),12X,=,F10.1,/,
9007 FORMAT(1X,25X,MATERIAL TYPE,21X,= BRONZE,/,
1X,25X,ALLCABLE STRESS (PSI),12X,=,F10.1,/,
9008 FORMAT(1X,25X,MATERIAL TYPE,21X,= NI-AL BRONZE,/,

```



```

1X,25X, 'ALL QWABLE STRESS (PSI)', 12X, '=', F10.4, /,
1X,25X, 'MATERIAL TYPE', 21X, '=', STAINLESS STEEL, /,
1X,25X, 'MATERIAL TYPE', 21X, '=', 12X, CCNSIDERED, /,
1X,25X, 'SELECTI (FT/SEC)', 25X, '=', PE (FT/SEC), 27X, '=', F10.1, /,
1X,25X, 'V (RPM)', 23X, '=', F10.4, /,
1X,25X, 'N S (FT-LBF)', 23X, '=', F12.1, /,
1X,25X, 'QPD (HP)', 23X, '=', F10.2, /,
1X,25X, 'J', 23X, '=', F10.4, /,
1X,25X, 'KT', 23X, '=', F10.4, /,
1X,25X, 'KQ', 23X, '=', F10.4, /,
1X,25X, 'ETAC', 23X, '=', F10.4, /,
1X,25X, 'REY 75R', 23X, '=', F10.4, /,
1X,25X, 'DIA (FT)', 23X, '=', F10.4, /,
1X,25X, 'P/D', 23X, '=', F10.4, /,
1X,25X, 'AE/AO', 23X, '=', F10.4, /,
1X,25X, 'T/C', 23X, '=', F10.4, /,
1X,25X, 'CON STRA INT VALUES: ', 11X, 'MAX DIA (FT)', 22X, '=', F10.4, /,
1X,25X, 'MIN AE/AO', 25X, '=', F10.4, /,
1X,25X, 'MIN T/C', 25X, '=', F10.4, /,
9005 1 FORMAT(
9010 1 FORMAT(
9011 2
3
4
4
4
4
4
4
4
4
5
6
7
9
8
9012 1 FORMAT(
2
END

```



```

SUBROUTINE ANALIZ(ICALC)
INTEGER*4 ICALC
REAL*4
1  RJCNU, RJCNU, R75FCL, R75RCU, AEAQCL, AEAQCL, TC75CL, TC75CU,
2  FCBAL, DIACNU, AEAQCV, TCSTRS, RJ,
3  VK, TC, WT, Z, WATRU, WATNU, TEMP, NCSCRW, HCL, PATM,
4  PWATVA, PROMAT, DIALIM, ETARR, AEAOMN, TC75MN, SC,
5  C75R, R75R, KT, KC, PD
COMMON /GLOECM/ETAG, WEIGHT, AEDVAG, DIA, N, PE, FDI, VC, QS, TC75R, V, RJCNU,
1RJCNU, R75FCL, R75RCU, AEAQCL, AEAQCV, TC75CL, TC75CU, POWBAL, DIACNU,
2AEAQCV, TCSTRS, RJ
COMMON /PARAM/VK, TD, WT, Z, WATRU, WATNU, TEMP, NCSCRW, HCL, PATM, PWATVA,
1PRGMAT, DIALIM, ETARR, AEAOMN, TC75MN, SC

THIS SUBROUTINE, COUPLED WITH CUPEX/CONMIN, CONSTITUTES ANALYSIS
METHOD FOR "DESIGN CASE 1" PROPELLER SELECTION PROBLEMS

INPLT-INITIALIZATION PHASE

PII=3.14159264
IF(.NOT.(ICALC.EQ.1))GC TO 1

SET "DESIGN CASE 1" PARAMETERS
ENVIRONMENTAL

TEMP=55.0
WATRU=1.9384
WATNU=.0000122850
PATM=14.7
PWATVA=.247

PROPELLER PARAMETERS

Z=5.0
PRGMAT=5.0

HULL PARAMETERS

WT=C.22
TD=C.15
ETARR=1.025
NCSCRW=1.0
HCL=19.0
DIALIM=22.0

SET DESIGN VARIABLES FELD FIXED FOR "DESIGN CASE 1"

```





```

PE=18153.0
VK=24.0
V=1.68E*VK
AEDVAD=0.85
DIA=22.0
TC75R=((0.0185-0.00125*Z)*Z)/(2.073*AEDVAC)

CC
CC
CC
C
C
C
END OF INPLT-INITIALIZATION PHASE

GO TO 3

EXECUTION PHASE

1 CONTINUE
IF(.NOT.(ICALC.EQ.2))GC TO 2
CALL RNCAL(N)
CALL CF75RA(C75R)
CALL REY75R(C75R,R75R)
CALL CCEFFSA(RJ,R75R,KT,KQ)
CALL CFWEFF(RJ,KT,KQ,ETAD)
CALL CALCCS(KC,QS)
CALL JCNA(RJ,RJCNL,RJCNU)
CALL REYCN(R75R,R75RCL,R75RCU)
CALL EXTCCN(Z,AEDVAD,TC75R,AEACCL,AEAGCU,TC75CL,TC75CU)
CALL BLPOW1(KT,POWBAL)
CALL DICNUA(DIACNU)
CALL CAVCNA(KT,AEAGCV)
CALL STRCNK(KC,KT,C75R,TCSTKS)

CC
CC
CC
END OF EXECUTION PHASE

GO TO 3

2 CONTINUE

OUTPUT-RESULT PHASE

PC=(2.0*PII*QS*N)/33000.0
WRITE(6,9000)
WRITE(6,9001)
WRITE(6,9002) TEMP,WATRU,WATNU,PATM,PWATVA
WRITE(6,9003) WT,TD,ETARR,NUSCRW,HCL,DIALIM
IF(.NOT.(PRCMAT.EQ.1.0))GO TO 81
WRITE(6,9005)SC
GC TO 86
CCATINCE
IF(.NOT.(PRGMAT.EQ.2.0))GC TO 82
WRITE(6,9006)SC

81

```

```

APP02170
APP02180
APP02190
APP02200
APP02210
APP02220
APP02230
APP02240
APP02250
APP02260
APP02270
APP02280
APP02290
APP02300
APP02310
APP02320
APP02330
APP02340
APP02350
APP02360
APP02370
APP02380
APP02390
APP02400
APP02410
APP02420
APP02430
APP02440
APP02450
APP02460
APP02470
APP02480
APP02490
APP02500
APP02510
APP02520
APP02530
APP02540
APP02550
APP02560
APP02570
APP02580
APP02590
APP02600
APP02610
APP02620
APP02630
APP02640

```



```

82      GC TC E6
      CCNTINLE
      IF(.NOT.(PRCMAT.EQ.3.0))GC TO E3
      WRITE(6,SC07)SC

83      GC TC E6
      CCNTINLE
      IF(.NOT.(PRCMAT.EQ.4.0))GC TO E4
      WRITE(6,SC08)SC

84      GC TC E6
      CCNTINLE
      IF(.NOT.(PRCMAT.EQ.5.0))GC TO E5
      WRITE(6,SC09)SC

85      GC TC E6
      CCNTINLE
      WRITE(6,SC10)

86      CCNTINLE
      WRITE(6,9011)PE,V,N,QS,PD,RJ,KI,KC,ETAC,R75R,DIA,PCIVD,
      AEDVAC,TC75R
1      WRITE(6,9012)DIALIM,AEADMAN,TC75MN
3 CONTINUE
      RETURN

C MISCELLANEOUS FORMAT STATEMENTS
C
9000 FORMAT('1','OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1',/,
1      '1X',DESIGN VARIABLES SPECIFIED: PE,V,AEDVAC,DIA,TC75R',/,
9001 FORMAT('1X',ENVIRONMENTAL PARAMETERS:
1      '1X,4',/,
1      '1X,25X',DENSITY (LBF-SEC2/FT4)',12X,'=',F10.4,/,
2      '1X,25X',VISCOSITY (FT2/SEC)',15X,'=',E16.9,/,
3      '1X,25X',ATMOSPHERIC PRESSURE (PSIA)',7X,'=',F10.4,/,
4      '1X,25X',WATER VAPORIZATION PRESSURE (PSIA)',7X,'=',F10.4,/,
9003 FORMAT('1X',FULL PARAMETERS:
1      '1X,25X',WAKE FRACTION',9X,'=',F10.4,/,
2      '1X,25X',THRUST DEDUCTION EFFICIENCY',6X,'=',F10.4,/,
3      '1X,25X',RELATIVE ROTATIVE EFFICIENCY',6X,'=',F10.4,/,
4      '1X,25X',NUMBER OF PROPELLERS',14X,'=',F10.1,/,
5      '1X,25X',DEPTH TO SHAFT CENTERLINE (FT)',4X,'=',F10.4,/,
9004 FORMAT('1X',DIAMETER LIMIT (FT)',15X,'=',F10.4,/,
1      '1X,25X',PROPELLER PARAMETERS:
1      '1X,25X',NUMBER OF BLADES',18X,'=',
9005 FORMAT('1X,10.1)',MATERIAL TYPE',21X,'= CAST IRON',/,
1      '1X,25X',ALLOWABLE STRESS (PSI)',12X,'=',F10.1,/,
9006 FORMAT('1X,25X',MATERIAL TYPE',21X,'= CAST STEEL',/,
1      '1X,25X',ALLOWABLE STRESS (PSI)',12X,'=',F10.1,/,
9007 FORMAT('1X,25X',MATERIAL TYPE',21X,'= BRONZE',/,
1      '1X,25X',ALLOWABLE STRESS (PSI)',12X,'=',F10.1,/,
9008 FORMAT('1X,25X',MATERIAL TYPE',21X,'= NI-AL BRONZE',/,

```

```

APP02650
APP02660
APP02670
APP02680
APP02690
APP02700
APP02710
APP02720
APP02730
APP02740
APP02750
APP02760
APP02770
APP02780
APP02790
APP02800
APP02810
APP02820
APP02830
APP02840
APP02850
APP02860
APP02870
APP02880
APP02890
APP02900
APP02910
APP02920
APP02930
APP02940
APP02950
APP02960
APP02970
APP02980
APP02990
APP03000
APP03010
APP03020
APP03030
APP03040
APP03050
APP03060
APP03070
APP03080
APP03090
APP03100
APP03110
APP03120

```



```

1 9009 1 FORMAT(1X,2SX,'ALL CRAWABLE STRESS (PSI)',12X,'=',F10.1,/,
1 9010 1 FORMAT(1X,2SX,'MATERIAL TYPE',2SX,'S (PSI)',12X,'STAINLESS STEEL',/,
1 9011 1 FORMAT(1X,2SX,'MATERIAL TYPE',2SX,'S (PSI)',12X,'STAINLESS STEEL',/,
2 1X,2SX,'SELECTION VALUES:',23X,'=',F10.1,/,
3 1X,2SX,'V (FT/SEC)',23X,'=',F10.4,/,
4 1X,2SX,'N (RPM)',23X,'=',F10.4,/,
4 1X,2SX,'CS (FT-LBF)',23X,'=',F12.1,/,
4 1X,2SX,'PD (HP)',23X,'=',F10.2,/,
4 1X,2SX,'J',23X,'=',F10.4,/,
4 1X,2SX,'KT',23X,'=',F10.4,/,
4 1X,2SX,'KQ',23X,'=',F10.4,/,
4 1X,2SX,'ETAC',23X,'=',F10.4,/,
5 1X,2SX,'REV75K',23X,'=',F10.4,/,
6 1X,2SX,'DIA (FT)',23X,'=',F10.4,/,
7 1X,2SX,'P/D',23X,'=',F10.4,/,
9 1X,2SX,'AE/AC',23X,'=',F10.4,/,
8 1X,2SX,'CONSTRA INT T/C',23X,'=',F10.4,/,
1 9012 1 FORMAT(1X,2SX,'VALUES:',11X,'MAX DIA (FT)',22X,'=',F10.4,/,
2 1X,2SX,'MIN AE/AC',25X,'=',F10.4,/,
2 1X,2SX,'MIN T/C',22X,'=',F10.6)
2 END

```

```

APP03130
APP03140
APP03150
APP03160
APP03170
APP03180
APP03190
APP03200
APP03210
APP03220
APP03230
APP03240
APP03250
APP03260
APP03270
APP03280
APP03290
APP03300
APP03310
APP03320
APP03330
APP03340

```



```

SUBROUTINE ANALIZ(ICALC)
INTEGER*4 ICALC
REAL*4
1  ETAG,WEIGHT,AEDVAO,CIA,N,PE,FDIVC,QS,TC75R,V,
2  RJCNU,RJCNU,R75RCL,R75RCL,AEACCL,AEACCU,TC75CL,TC75CU,
3  FQWBL,DIAACNU,AEACCU,TCSTRS,RJ,
4  VK,TC,WT,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PAIM,
5  PWATVA,PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
COMMON /GLOECM/ETAG,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,KJCNU,
1RJCNU,R75RCL,R75RCU,AEACCL,AEACCU,TC75CL,TC75CU,POWBAL,DIAACNU,
2AEACCU,TCSTRS,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PAIM,PWATVA,
1PRCMAT,DIALIM,ETARR,AEACMN,TC75MN,SC

THIS SUBROUTINE, COUPLED WITH COPE/CUNMIN, CONSTITUTES ANALYSIS
METHOD FOR "DESIGN CASE 1" PROPELLER SELECTION PROBLEMS

INPUT-INITIALIZATION PHASE

PII=3.14159264
IF(.NOT.(ICALC.EQ.1))GO TO 1

SET "DESIGN CASE 1" PARAMETERS

ENVIRONMENTAL

TEMP=59.0
WATRO=1.9384
WATNU=.000012285
PAIM=14.7
PWATVA=.247

PROPELLER PARAMETERS

Z=5.0
PRCMAT=5.0

HULL PARAMETERS

WT=C.22
TC=C.15
ETARR=1.025
NCSCRW=1.0
HCL=19.0
DIALIM=22.0

SET DESIGN VARIABLES HELD FIXED FOR "DESIGN CASE 1"

```

CC  
CC  
CC  
CC  
CC

CC  
CC  
CC  
CC  
CC

CC  
CC  
CC

CC  
CC  
CC

CC  
CC  
CC













```

9010 FORMAT(1X,25X,MATERIAL TYPE,21X,PE,12X,PE,NCT CCNSIDERED,/)
9011 FORMAT(1X,25X,ELECTION VALUES,23X,PE,10.4,/,
1X,25X,V,23X,PE,10.4,/,
1X,25X,ACS,23X,PE,10.4,/,
1X,25X,CPD,23X,PE,10.4,/,
1X,25X,J,23X,PE,10.4,/,
1X,25X,KT,23X,PE,10.4,/,
1X,25X,KQ,23X,PE,10.4,/,
1X,25X,ETA,23X,PE,10.4,/,
1X,25X,REV,23X,PE,10.4,/,
1X,25X,DIA,23X,PE,10.4,/,
1X,25X,P/D,23X,PE,10.4,/,
1X,25X,AE/AC,23X,PE,10.4,/,
1X,25X,T/C,23X,PE,10.4,/,
9012 FORMAT(1X,CONSTRAINT VALUES,11X,MAX DIA (FT),22X,PE,10.4,/,
1X,25X,MIN AE/AC,25X,PE,10.4,/,
1X,25X,MIN T/C,25X,PE,10.4,/)
END

```

```

APP04810
APP04820
APP04830
APP04840
APP04850
APP04860
APP04870
APP04880
APP04890
APP04900
APP04910
APP04920
APP04930
APP04940
APP04950
APP04960
APP04970
APP04980
APP04990

```



```

SUBROUTINE ANALIZ(ICALC)
  INTEGER*4 ICALC
  REAL*4  ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,
1  FJCNL,RJCNL,R75RCL,R75RCU,AEAGCL,AEACCU,TC75CL,TC75CU,
2  FGWBAL,DIA,AEACCU,AEAGCL,TCSTRS,RJ,
3  VK,TC,WT,Z,WATRG,WATNU,TEMP,NCS,CRW,HCL,PATM,
4  FWATVA,PRCMAT,DIALIM,ETARR,AEACMN,TC75MN,SC,
5  C15R,R75R,KT,KC,PD
  COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1  RJCNL,R75RCL,R75RCU,AEAGCL,AEACCU,TC75CL,TC75CU,PWBAL,DIA,
2  AEACCU,TCSTRS,RJ
  COMMON /PARAM/VK,TC,WT,Z,WATRG,WATNU,TEMP,NCS,CRW,HCL,PATM,
1  PRCMAT,DIALIM,ETARR,AEACMN,TC75MN,SC

  THIS SUBROUTINE, COUPLED WITH COPEX/CONMIN, CONSTITUTES ANALYSIS
  METHOD FOR "DESIGN CASE 1" PROPELLER SELECTION PROBLEMS

  INPT-INITIALIZATION PHASE

  PII=3.14159264
  IF(.NOT.(ICALC.EQ.1))GO TO 1

  SET "DESIGN CASE 1" PARAMETERS

  ENVIRONMENTAL

  TEMP=55.0
  WATRG=1.9384
  WATNU=.00001285
  PATM=14.7
  PWATVA=.247

  PROPELLER PARAMETERS

  Z=5.0
  PRCMAT=5.0

  HULL PARAMETERS

  WT=C.25
  TC=C.15
  ETARR=1.025
  NCS,CRW=1.0
  HCL=19.0
  DIALIM=22.0

  SET DESIGN VARIABLES HELD FIXED FOR "DESIGN CASE 1"

```

CC  
CC  
CC  
CC  
CC

CC  
CC  
CC  
CC  
CC

CC  
CC  
CC

CC  
CC  
CC

CC  
CC  
CC





PE=181.53.0  
VK=24.C  
V=1.68E\*VK

CC  
CC  
CC

END OF INPUT-INITIALIZATION PHASE

GO TO 3

C  
C  
C

EXECUTION PHASE

```

1 CONTINUE
IF(.NOT.(ICALC.EQ.2))GC TO 2
  CALL RNCAL(N)
  CALL CF75RA(C75R)
  CALL REY75R(C75R,R75R)
  CALL CCEFFSA(RJ,R75R,KT,KQ)
  CALL OPWEFF(RJ,KT,KQ,ETA0)
  CALL CALCCS(KQ,QS)
  CALL JCNA(RJ,RJCNL,KJCNU)
  CALL REYCNNA(R75R,R75RCL,R75RCL)
  CALL EXTCCN(Z,AEDVA0,TC75R,AEAGCL,AEAGCU,TC75CL,TC75CL)
  CALL BLPOWI(KI,PCWBAL)
  CALL DICNUA(DIACNU)
  CALL CAVCNNA(KI,AEACCV)
  CALL STRCNK(KQ,KT,C75R,TCSTRS)

```

CC  
CC  
CC

END OF EXECUTION PHASE

GO TO 3

C  
C  
C

OUTPUT-RESULT PHASE

```

PC=(2.C*PII*QS*N)/33000.0
WRITE(6,9000)
WRITE(6,9001)
WRITE(6,9002) TEMP,WATRO,WATNU,PATM,PWATVA
WRITE(6,9003) WT,TD,ETAR,NUSCRW,HCL,DIALIM
IF(.NOT.(PRCMAT.EC.1.0))GC TO 81
  WRITE(6,9005) SC
  GC TO 86
CONTINUE
IF(.NOT.(PRMAT.EC.2.0))GC TO 82
  WRITE(6,9006) SC
  GC TO 86
CONTINUE
IF(.NOT.(PRCMAT.EC.3.0))GC TO 83

```

81

82

APP05500  
APP05510  
APP05520  
APP05530  
APP05540  
APP05550  
APP05560  
APP05570  
APP05580  
APP05590  
APP05600  
APP05610  
APP05620  
APP05630  
APP05640  
APP05650  
APP05660  
APP05670  
APP05680  
APP05690  
APP05700  
APP05710  
APP05720  
APP05730  
APP05740  
APP05750  
APP05760  
APP05770  
APP05780  
APP05790  
APP05800  
APP05810  
APP05820  
APP05830  
APP05840  
APP05850  
APP05860  
APP05870  
APP05880  
APP05890  
APP05900  
APP05910  
APP05920  
APP05930  
APP05940  
APP05950  
APP05960  
APP05970











# APPENDIX D

## CONTROL CARD IMAGES--DESIGN CASE NO. 1

\$A	TITLE	E-SERIES	PROPELLER	CPTIMIZATION	IPNPUT	IPDBG
\$B	WAGENINGEN	NDV	NSV	N2VAR	0	NACMX1
\$C	NCALC	ITMAX	ICNDR	NSCAL	LINCBJ	15
\$D1	IPRINT	ICOO	CT	CTMIN	CTL	THETA
\$D2	FDCH	FDCHM	ALPFA	ABGBJI		
\$E	C.OO	C.OO				
\$F	CELFUN	CABFUN				
\$G	NDVTOT	ICBJ	SGNCP			
\$H	VLB	VLB	1.0	SCAL		
\$I1	0.01	1.8	0.1	1.0		
\$I2	0.4	1.4	0.5	1.0		
-1.0	NDSSGN	IDSGN	AMULT			
-1.0	1	23	1.0			
-1.0	2	7				
-1.0	NCON	JCON	LCCN	SCAL2		
-1.0	1	SCAL1	BU	1.0		
-1.0	11	1.0	0.0	1.0		
-1.0	12	1.0	0.0	1.0		
-1.0	13	1.0	0.0	1.0		
-1.0	14	1.0	0.0	1.0		
-1.0	15	1.0	0.0	1.0		
-1.0	16	1.0	0.0	1.0		
-1.0	17	1.0	0.0	1.0		
-1.0	18	1.0	0.0	1.0		
-1.0	19	1.0	0.0	1.0		
-1.0	16	1.0	0.0	1.0		
\$V	END					





\$A	TITLE	E--SERIES	PROPELLER	CPTIMIZATION	IPNPUT	IPD66
\$B	WAGENINGEN	NDV	NSV	N2VAR	LINUBJ	NACMXI
\$C	IPRINT	ITPAX	ICNDR	NSCAL	CTL	THETA
\$D1	FDCF	FDCHM	CT	CTMIN		
\$D2	CELFUN	COCOL	ALPHA	ABCBJI		
\$E	NDVTGT	ICBJ	SENCPT			
\$F	VLE	VUB	1.0 X	SCAL		
	0.2	1.1	0.30	1.0		
	1.0	5C.0	30.0	10.0		
	0.01	1.6	0.1	1.0		
	0.4	1.4	0.5	1.0		
\$G	NDSSGN	DSGN	0.0300	0.010		
	1	3	AMULT			
	2	4	1.0			
	3	23	1.0			
	4	7	1.0			
	5	9	1.0			
\$H	NCON	JCON	LCCN	SCAL2		
\$I1	ICCN	SCAL1	BU	1.0		
\$I2	11	11	0.0	1.0		
-1.0	+11	12	0.0	1.0		
-1.0	+12	13	0.0	1.0		
-1.0	+13	14	0.0	1.0		
-1.0	+14	15	0.0	1.0		
-1.0	+15	16	0.0	1.0		
-1.0	+16	17	0.0	1.0		
-1.0	+17	18	0.0	1.0		
-1.0	+18	19	0.0	1.0		
-1.0	+19	20	0.0	1.0		
-1.0	+20	16	0.0	1.0		



-1.0	21	0.0	1.0
-1.0	+19	0.0	1.0
\$V	+22		
END	1.0		



# APPENDIX E

## COPEs OUTPUT--DESIGN CASE NO. 1

```

CCCCCCC  CCCCCC  P P P P P P P  E E E E E  S S S S S
C C C C C  C C C C C  U U U U U  E E E E E  S S S S S
C C C C C  C C C C C  U U U U U  E E E E E  S S S S S
C C C C C  C C C C C  U U U U U  E E E E E  S S S S S
C C C C C  C C C C C  U U U U U  E E E E E  S S S S S

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C C N T R C L  P R O G R A M
      F O R
E N G I N E E R I N G  S Y N T H E S I S

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T I T L E
H A G E N I N C E N  B - S E R I E S  P R O P E L L E R  O P T I M I Z A T I O N

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CARD IMAGES OF CONTROL DATA

CARD	IMAGE
11	1A TITLE
21	2A RAGNIN GEN B-SERIES PROPELLER OPTIMIZATION
31	3E KCALC 2 NSV
41	4C IFRINT 2 ICDIK NSCAL IIRM
51	5I ITHAX 1 FOCUM CT
61	6I ITHAX 1 FOCUM CT
71	7C 0.0001 DABFUN ALPHAX ABCRJI
81	8C 0.0001 DABFUN ALPHAX ABCRJI
91	9C 0.0001 DABFUN ALPHAX ABCRJI
101	10E NEVTOT 100J SGNCTI
111	11E NEVTOT 100J SGNCTI
121	12E NEVTOT 100J SGNCTI
131	13E NEVTOT 100J SGNCTI
141	14E NEVTOT 100J SGNCTI
151	15E NEVTOT 100J SGNCTI
161	16E NEVTOT 100J SGNCTI
171	17E NEVTOT 100J SGNCTI
181	18E NEVTOT 100J SGNCTI
191	19E NEVTOT 100J SGNCTI
201	20E NEVTOT 100J SGNCTI
211	21E NEVTOT 100J SGNCTI
221	22E NEVTOT 100J SGNCTI
231	23E NEVTOT 100J SGNCTI
241	24E NEVTOT 100J SGNCTI
251	25E NEVTOT 100J SGNCTI
261	26E NEVTOT 100J SGNCTI
271	27E NEVTOT 100J SGNCTI
281	28E NEVTOT 100J SGNCTI
291	29E NEVTOT 100J SGNCTI
301	30E NEVTOT 100J SGNCTI
311	31E NEVTOT 100J SGNCTI
321	32E NEVTOT 100J SGNCTI
331	33E NEVTOT 100J SGNCTI
341	34E NEVTOT 100J SGNCTI
351	35E NEVTOT 100J SGNCTI
361	36E NEVTOT 100J SGNCTI
371	37E NEVTOT 100J SGNCTI
381	38E NEVTOT 100J SGNCTI
391	39E NEVTOT 100J SGNCTI
401	40E NEVTOT 100J SGNCTI
411	41E NEVTOT 100J SGNCTI
421	42E NEVTOT 100J SGNCTI
431	43E NEVTOT 100J SGNCTI
441	44E NEVTOT 100J SGNCTI
451	45E NEVTOT 100J SGNCTI
461	46E NEVTOT 100J SGNCTI
471	47E NEVTOT 100J SGNCTI
481	48E NEVTOT 100J SGNCTI
491	49E NEVTOT 100J SGNCTI
501	50E NEVTOT 100J SGNCTI
511	51E NEVTOT 100J SGNCTI
521	52E NEVTOT 100J SGNCTI
531	53E NEVTOT 100J SGNCTI
541	54E NEVTOT 100J SGNCTI
551	55E NEVTOT 100J SGNCTI
561	56E NEVTOT 100J SGNCTI
571	57E NEVTOT 100J SGNCTI
581	58E NEVTOT 100J SGNCTI
591	59E NEVTOT 100J SGNCTI
601	60E NEVTOT 100J SGNCTI
611	61E NEVTOT 100J SGNCTI
621	62E NEVTOT 100J SGNCTI
631	63E NEVTOT 100J SGNCTI
641	64E NEVTOT 100J SGNCTI
651	65E NEVTOT 100J SGNCTI
661	66E NEVTOT 100J SGNCTI
671	67E NEVTOT 100J SGNCTI
681	68E NEVTOT 100J SGNCTI
691	69E NEVTOT 100J SGNCTI
701	70E NEVTOT 100J SGNCTI
711	71E NEVTOT 100J SGNCTI
721	72E NEVTOT 100J SGNCTI
731	73E NEVTOT 100J SGNCTI
741	74E NEVTOT 100J SGNCTI
751	75E NEVTOT 100J SGNCTI
761	76E NEVTOT 100J SGNCTI
771	77E NEVTOT 100J SGNCTI
781	78E NEVTOT 100J SGNCTI
791	79E NEVTOT 100J SGNCTI
801	80E NEVTOT 100J SGNCTI
811	81E NEVTOT 100J SGNCTI
821	82E NEVTOT 100J SGNCTI
831	83E NEVTOT 100J SGNCTI
841	84E NEVTOT 100J SGNCTI
851	85E NEVTOT 100J SGNCTI
861	86E NEVTOT 100J SGNCTI
871	87E NEVTOT 100J SGNCTI
881	88E NEVTOT 100J SGNCTI
891	89E NEVTOT 100J SGNCTI
901	90E NEVTOT 100J SGNCTI
911	91E NEVTOT 100J SGNCTI
921	92E NEVTOT 100J SGNCTI
931	93E NEVTOT 100J SGNCTI
941	94E NEVTOT 100J SGNCTI
951	95E NEVTOT 100J SGNCTI
961	96E NEVTOT 100J SGNCTI
971	97E NEVTOT 100J SGNCTI
981	98E NEVTOT 100J SGNCTI
991	99E NEVTOT 100J SGNCTI
1001	100E NEVTOT 100J SGNCTI





TITLE: WAGENINGEN B-SERIES PROPELLER OPTIMIZATION

CONTRMUL PARAMEters: NCALC = 3 000000 0  
 C-LOCAL DESIGN VARIABLES, NSV =  
 C-CONSTRAINTS, NSC =  
 NUMBER OF CONSTRAINTS, NSV =  
 NUMBER OF FLUXIONS IN TWO-SPACE, NSVPRX =  
 NUMBER OF APPROXIMATING VAR, IPBPRX =  
 INPUT ALPHAT CODE, IPBPRX =

CALCULATION CONTRMUL, NCALC  
 VALUE  
 1 SINGLE ANALYSIS  
 2 OPTIMIZATION  
 3 SENSITIVITY  
 4 TWO-VARIABLE FUNCTIONAL SPACE  
 5 APPROXIMATE OPTIMIZATION

# • • OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE  
 MULTIPLIER NEGATIVE INDICATES MINIMIZATION = 0.1000E+01  
 CONTRMUL PARAMEters (IF ZERO, CONTRMUL DEFAULT WILL OVER-RIDE)

IPRINT	ITNKA	ICNCLM	NSCAL	ITM	LINOBJ	NACHAI	NFOG
1	0	0	-1	0	0	15	0
FOCHM	FOCHM	FOCHM	FOCHM	FOCHM	FOCHM	FOCHM	FOCHM
0.1000E-03	0.1000E-03	0.1000E-03	0.1000E-03	0.1000E-03	0.1000E-03	0.1000E-03	0.1000E-03
CTI	CTI	CTI	CTI	CTI	CTI	CTI	CTI
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
DELFUN	DELFUN	DELFUN	DELFUN	DELFUN	DELFUN	DELFUN	DELFUN
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

DESIGN VARIABLE INFORMATION  
 NON-ZERO INITIAL VALUE WILL OVER-RIDE MCODE INPUT  
 C. V. LOWER BOUND UPPER BOUND INITIAL VALUE SCALE  
 NO. 1 0.1000E+01 0.1000E+01 0.1000E+01 0.1000E+01  
 2 0.1000E+01 0.1000E+01 0.1000E+01 0.1000E+01

DESIGN VARIABLES GLOBAL MULTIPLYING  
 ID 0. V. VAR. NG. 0.1000E+01  
 1 1 1 1  
 2 2 2 2

# CONSTRAINT INFORMATION

HERE	AF	CONSTRAINT	SET	GLOBAL	LINEAR	LOWER BOUND	UPPER BOUND	NORMALIZATION FACTOR	UPPER BOUND	NORMALIZATION
1	1	1	1	1	1	0.000E+00	0.000E+00	0.000E+00	0.000E+00	0.000E+00
2	2	2	2	2	2	0.000E+00	0.000E+00	0.000E+00	0.000E+00	0.000E+00
3	3	3	3	3	3	0.000E+00	0.000E+00	0.000E+00	0.000E+00	0.000E+00
4	4	4	4	4	4	0.000E+00	0.000E+00	0.000E+00	0.000E+00	0.000E+00
5	5	5	5	5	5	0.000E+00	0.000E+00	0.000E+00	0.000E+00	0.000E+00
6	6	6	6	6	6	0.000E+00	0.000E+00	0.000E+00	0.000E+00	0.000E+00
7	7	7	7	7	7	0.000E+00	0.000E+00	0.000E+00	0.000E+00	0.000E+00
8	8	8	8	8	8	0.000E+00	0.000E+00	0.000E+00	0.000E+00	0.000E+00
9	9	9	9	9	9	0.000E+00	0.000E+00	0.000E+00	0.000E+00	0.000E+00

TOTAL ALPHAM OF CONSTRAINED PARAMETERS = 9

# • • ESTIMATED DATA STORAGE REQUIREMENTS







```

*****
*               *
*   C C P M I N   *
*   F O R T R A N   *
*   P R O G R A M   *
*   F O R   *
*   C O N S T R A I N E C   *
*   F U N C T I O N   *
*   M I N I M I Z A T I O N   *
*               *
*****

```

INITIAL FUNCTION INFORMATION

CBJ = -C.152694E+00

DECISION VARIABLES (X-VECTOR)

1) C.1000E+00 0.5000E+00

CONSTRAINT VALUES (C-VECTOR)

1) -C.5250E+01 -C.9375E+00 -C.2278E+00 -C.2371E+02 -C.1648E+03 -C.4000E+00 -C.2000E+00



```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.7C9073E+0C
DECISION VARIABLES IX-VECTOR
|| C.73712E+00 0.10032E+01
CONSTRAINT VALUES IG-VECTOR I
|| -C.46070E+00 -0.53935E+00 -0.32395E+02 -0.96660E+03 -0.40600E+00 -0.20000E+00
|| -C.26691E-01 -0.22788E+00 0.12219E-04
THERE ARE 1 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(11-0BJ11-11/0BJ1111 LESS THAN DOLFUN FOR 3 ITERATIONS
ABS(0BJ11-0BJ11-111 LESS THAN DOLFUN FOR 3 ITERATIONS
NUMBER OF ITERATIONS = 14
OBJECTIVE FUNCTION WAS EVALUATED 69 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 69 TIMES

```





## OPTIMIZATION RESULTS

```
OBJECTIVE FUNCTION      1
GLOBAL LOCATION         FUNCTION VALUE 0.70907E+00
```

DESIGN VARIABLES

D. V.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
10	23	C:1000E-01	0.73712E+00	0.1600E+01
11	7	C:1000E+00	0.10035E+01	0.1400E+01

## DESIGN CONSTRAINTS

GLOBAL VALUE NO.	LOWER BOUND	VALUE	UPPER BOUND
0	0	0	0
1	0	0	0
2	0	0	0
3	0	0	0
4	0	0	0
5	0	0	0
6	0	0	0
7	0	0	0
8	0	0	0
9	0	0	0
10	0	0	0
11	0	0	0
12	0	0	0
13	0	0	0
14	0	0	0
15	0	0	0
16	0	0	0
17	0	0	0
18	0	0	0
19	0	0	0
20	0	0	0
21	0	0	0
22	0	0	0
23	0	0	0
24	0	0	0
25	0	0	0
26	0	0	0
27	0	0	0
28	0	0	0
29	0	0	0
30	0	0	0
31	0	0	0
32	0	0	0
33	0	0	0
34	0	0	0
35	0	0	0
36	0	0	0
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85	0	0	0
86	0	0	0
87	0	0	0
88	0	0	0
89	0	0	0
90	0	0	0
91	0	0	0
92	0	0	0
93	0	0	0
94	0	0	0
95	0	0	0
96	0	0	0
97	0	0	0
98	0	0	0
99	0	0	0



```

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
SUBROUTINE "STRNAM"

DESIGN VARIABLES SPECIFIED:  PE,VAELVAC,DIA,ATC,SK
ENVIRONMENTAL PARAMETERS:
TEMP (DEG F) = 55.0000
DENSITY (LBF/IN3) = 1.5384
VISCOSITY (LBF/IN2-SEC/FT2) = 0.122900
WATER VAPOR PRESSURE (PSIA) = 0.122900
WATER VAPORIZATION PRESSURE (PSIA) = 0.2470

FULL PARAMETERS:
WAKE FRACTION = 0.2200
THRUST REDUCTION COEFFICIENT = 0.0500
NOISE COEFFICIENT = 1.0000
NOISE COEFFICIENT = 15.0000
DEPT OF SHAFT CENTERLINE (FT) = 22.0000
DIAMETER LIMIT (IN) =

PROPELLER PARAMETERS:
NUMBER OF BLADES = 5
ALLOWABLE STRESS (PSI) = STAINLESS STEEL
= 50000

SELECTION VALUES:
PE (HP) = 18133.0
V (FT/SEC) = 42.5120
N (RPM) = 112.912
PO (HP) = 248943.0
JT = 0.371
KJ = 0.172
ETA = 0.0285
DIAM (FT) = 0.0000
P/C (FT) = 2.0000
AE/AL = 0.0500
T/C .75F = 0.0348

CONSTRAINT VALUES:
MAX CIA (FT) = 25.0000
MIN AE/AL = 0.01266
MIN T/C .75F =

```



PROGRAM CALLS TO ANALYZE  
ICALL CALLS  
1 2  
2 3



```

CCCCC  UUUUU  PPPPP  EEEEE  SSSSS
C      U      P      E      S
C      U      P      E      S
C      U      P      E      S
CCCCC  UUUUU  PPPPP  EEEEE  SSSSS

```

# CENTRAL PROGRAM FOR ENGINEING SYNTHESIS

TITLE  
MAGNETIC D-SERIES PROPELLER OPTIMIZATION









TITLE  
MACMINCEN B-SERIES PROPELLER OPTIMIZATION

CONTROL CARPENTER  
CALCULATED CONSTRAINTS  
NUMBER OF DESIGN VARIABLES, NDCG = 3  
NUMBER OF SENSITIVITY VARIABLES, NSV = 0  
NUMBER OF CONSTRAINTS, NCON = 0  
NUMBER OF FUNCTIONAL IN-TWO-SPACE, NZVAX = 0  
NUMBER OF APPROXIMATION IN-TWO-SPACE, NZAPRX = 0  
INPUT APPROXIMATION PRINT CODE, IPAPRI = 0  
DEBUG PRINT CODE, IPDBG = 0

CALCULATION CONTROL, NDCG  
VALUE  
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OPTIMIZATION RESULTS ----- DESIGN CASE NO.1
SUBROUTINE "STRCKM"

DESIGN VARIABLES SPECIFIED:  PEIN,AE,CVAC,DIA,IC,ISR
ENVIRONMENTAL PARAMETERS:
    TEMP 1016. FT
    DENSITY 14.7 LB/FT-SEC2/FT4
    VISCOSITY 1.2E-5 SEC
    ATMOSPHERIC PRESSURE 14.7 PSIA
    WATER VAPORIZATION PRESSURE 14.7 PSIA
    FULL PARAMETERS:
        WAKE FRACTION
        THRUST COEFFICIENT
        RELATIVE ROTATIVE EFFICIENCY
        NUMBER OF PROPELLERS
        DEPTH OF SHAF CENTERLINE (FT)
        DIAPHRAGM LIFT (FT)
    PROPELLER PARAMETERS:
        NUMBER OF BLADES
        MATERIAL TYPE
        ALLOWABLE STRESS (PSI)
    SELECTION VALUES:
        PE (HP)
        VE (FT/SEC)
        N (RPM)
        US (FT-LBF)
        PD (HP)
        XT
        KI
        KE
        REYN
        DIA (FT)
        P/C
        AE/C
        T/C .156
    CONSTRAINT VALUES:
        MAX DIA (FT)
        MIN AE/40
        MIN T/C .156

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INITIAL FUNCTION INFORMATION
GBJ = -C.152694E+0C
DECISION VARIABLES (X-VECTOR)
1) C.16000E+00 C.50000E+00
CONSTRAINT VALUES (G-VECTOR)
1) -C.26691E-01 -0.22788E+00 -0.23735E+02 -0.76465E+03 -C.40000E+00 -0.20000E+00

```





```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.7C9073E+00
DECISION VARIABLES (X-VECTOR)
  C.73112E+00  0.10034E+01
CONSTRAINT VALUES (G-VECTOR)
  -C.44070E+00 -0.53934E+00 -0.32335E+02 -0.96660E+03 -C.40C00E+00 -0.20000E+00
  -C.21691E-01 -0.22788E+00  0.12279E-04
THERE ARE 1 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
  5
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
  ABS(1-CBJ1)-11/OBJ11) LESS THAN DELFUN FOR 3 ITERATIONS
  ABS(1CBJ1)-OBJ11) LESS THAN DABFUN FOR 3 ITERATIONS
NUMBER OF ITERATIONS = 14
OBJECTIVE FUNCTION WAS EVALUATED 69 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 69 TIMES

```



# OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 6.70907E+00  
GLOBAL TOL 1E-01

## DESIGN VARIABLES

ID	LC V. NO.	GLOBAL VAR NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	1	0.000E+00	0.7332E+01	0.1E+01
2	2	2	0.4000E+00	0.1000E+01	0.1E+01

## DESIGN CONSTRAINTS

ID	GLOBAL VAR NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	0.000E+00	0.000E+00	0.0
2	2	0.000E+00	0.000E+00	0.0
3	3	0.000E+00	0.000E+00	0.0
4	4	0.000E+00	0.000E+00	0.0
5	5	0.000E+00	0.000E+00	0.0
6	6	0.000E+00	0.000E+00	0.0
7	7	0.000E+00	0.000E+00	0.0
8	8	0.000E+00	0.000E+00	0.0



OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1  
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED: P=V,AE,VAC,DIA,T,C,TSR

ENVIRONMENTAL PARAMETERS:

TEMP (DEG F)	=	59.0000
DENSITY (LB-SEC/FT <sup>3</sup> )	=	0.12849997E-04
VISCOSITY (FT <sup>2</sup> /SEC)	=	14.7000
ATMOSPHERIC PRESSURE (PSIA)	=	14.7000
WATER VAPORIZATION PRESSURE (PSIA)	=	0.2470

PULL PARAMETERS:

WAVE FRACTION	=	0.2200
THROTTLE COEFFICIENT FRACTION	=	0.1800
RELATIVE ROTATIVE EFFICIENCY	=	1.0350
NUMBER OF PROPELLERS	=	1.0
DEPTH OF SHAFT CENTERLINE (FT)	=	15.0000
DIAPHYS (LIFT/FT)	=	22.0000

PROPELLER PARAMETERS:

NUMBER OF BLADES	=	2.0
MATERIAL TYPE	=	STAINLESS STEEL
ALLOWABLE STRESS (PSI)	=	5400.0

SELECTION VALUES:

PE (FT/SEC)	=	1413.70
N (RPM)	=	14.0129
QS (FT-LBF)	=	1160.51.0
PO (HP)	=	24051.27
J	=	0.7371
K	=	0.1622
AE/VAC	=	0.7691
TSR	=	0.7408
DIA (FT)	=	21.0000
P/D	=	1.0036
AE/AC	=	0.8500
T/C .754	=	0.0348

CONSTRAINT VALUES:

MAX DIA (FT)	=	21.0000
MIN AE/PO	=	0.5558
MIN T/C .754	=	0.050761



PROGRAM CALLS TO ANALYZE  
CALC CALLS  
1 1  
2 2  
3 3





C O N T R O L   P R O G R A M  
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E N G I N E E R I N G   S Y N T H E S I S

T I T L E

W A G E N I N G E N B - S E R I E S   P R O P E L L E R   O P T I M I Z A T I O N







TITLE: WAGENINCA B-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:  
CALCULATION CONTROL: NCALL = 3  
NUMBER OF GLOBAL DESIGN VARIABLES: NSV = 3  
NUMBER OF SENSITIVITY VARIABLES: NSV = 0  
NUMBER OF FUNCTIONS IN TWO-STEP: NSVAR = 0  
NUMBER OF FUNCTIONS IN ONE-STEP: NSVAR = 0  
NUMBER OF CALCULATION VARIABLES: NSCALC = 0  
NUMBER OF PRINTING VARIABLES: NSPRINT = 0  
DEBUG PRINT CODE: 10080

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TOTAL NUMBER OF CONSTRAINED PARAMETERS = 14

\* \* ESTIMATED DATA STORAGE REQUIREMENTS

INPUT	REAL	AVAILABLE	INPUT	INTEGER	AVAILABLE
87	110	10000	57	124	1000





## OPTIMIZATION RESULTS -----

## DESIGN VARIABLES SPECIFIED: P.E.V.

## ENVIRONMENTAL PARAMETERS: TEMP

TEMP (DEG F)	=	56.0000
DENSITY (LBF-SEC <sup>2</sup> /FT <sup>4</sup> )	=	1.9384
VISCOSITY (FT <sup>2</sup> /SEC)	=	0.124989
ATMOSPHERIC PRESSURE (PSIA)	=	14.7000
WATER VAPORIZATION PRESSURE (PSIA)	=	0.2470

**FULL PARAMETERS:**

WAKE FRACTION	0.2200
THRUST COEFFICIENT	0.1900
RELATIVE ROTATIVE EFFICIENCY	1.0250
NUMBER OF PROPELLERS	1.0
DEPTH (FATHOMS)	15.0000
DIAPYCNIC LIMIT (FT)	22.0000

### PROPELLER PARAMETERS:

NUMBER OF BLADES	5.0
MATERIAL TYPE	STAINLESS
ALLOWABLE STRESS (PSI)	5400.0

SELECTION VALUES:

PE	IFP)	16153.0
V	(FT/SEC)	40.5120
N	(RPM)	631.9868
QS	(FT-LBF)	83126272.
PO	(HP)	0.00000000

**CONSTRAINT VALUES:**

MAX	CIA IFTI	=	26.0000
MIN	AE/AG	=	31.1615
MIN	I/C .15R	=	0.657532



```

.....
C C A M I N
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F O R T R A N   P R O G R A M   F O R
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C O N S T R A I N E D   F U N C T I O N   M I N I M I Z A T I O N
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# INITIAL FUNCTION INFORMATION

```

CBJ = -C.1E+09E+00
DECIMAL VARIABLES (X-VECTOR)
11 C.30000E+00 C.30000E+02 0.10000E+00 0.50000E+00 C.30000E-01
CONSTRAINT VALUES (C-VECTOR)
11 -C.15000E-01 -0.93750E+00 -0.10070E+03 -0.66820E+03 C.15000E+00
11 -C.15000E-01 -C.23264E+00 -0.10910E+03 0.36364E+00 C.10287E+03 0.95106E+00

```



```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.710857E+0C
DECISICA VARIABLES IX-VECTORI
11 C.62045E+00 0.21995E+C2 0.73433E+00 C.99815E+00 0.33004E-01
CONSTRAINT VALUES (C-VECTOR)
11 -C.41895E+00 -0.54105E+00 -0.38016E+02 -0.96096E+03 -0.37089E+00 -0.36921E+00
7 -0.18935E-01 -0.22863E+00 -0.18354E-C2 -0.31233E-C4 -C.35786E+00 -0.33350E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5 10
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
AR5ICBJ11-OBJ61-111 LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 83
OBJECTIVE FUNCTION WAS EVALUATED 548 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 548 TIMES

```



# OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.71086E+00

## DESIGN VARIABLES

ID	C. V. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	3	0.2000E+00	0.2049E+00	0.1000E+01
2	2	2	0.1000E+01	0.1999E+02	0.5000E+02
3	3	1	0.0000E+00	0.1433E+00	0.5000E+00
4	4	4	0.0000E+00	0.3300E-01	0.5000E+00

## DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	0.1000E+00	0.00	0.0
2	2	0.1000E+00	0.00	0.0
3	3	0.1000E+00	0.00	0.0
4	4	0.1000E+00	0.00	0.0
5	5	0.1000E+00	0.00	0.0
6	6	0.1000E+00	0.00	0.0
7	7	0.1000E+00	0.00	0.0
8	8	0.1000E+00	0.00	0.0
9	9	0.1000E+00	0.00	0.0
10	10	0.1000E+00	0.00	0.0
11	11	0.1000E+00	0.00	0.0
12	12	0.1000E+00	0.00	0.0





OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1  
SUBROUTINE "STRONA"

DESIGN VARIABLES SPECIFIED:

ENVIRONMENTAL PARAMETERS:  
PE, V  
TEMP (DEG F) = 55.0000  
DENSITY (LBS/SEC/FT4) = 1.93E-04  
VIScosity (FT2/SEC) = 0.122E-04  
ATMOSPHERIC PRESSURE (PSIA) = 14.7000  
WATER VAPORIZATION PRESSURE (PSIA) = 0.2470

HULL PARAMETERS:

WAKE FRACTION = 0.2200  
YAW FRACTION = 0.1900  
RELATIVE PROPELLER EFFICIENCY = 1.0290  
NUMBER OF PROPELLERS = 1.0  
DEPTH TO SHAFT CENTERLINE (FT) = 15.0000  
DIAMETER LIMIT (FT) = 22.0000

PROPELLER PARAMETERS:

NUMBER OF BLADES = 5.0  
MATERIAL TYPE = STAINLESS STEEL  
ALLOWABLE STRESS (PSI) = 2400.0

SELECTION VALUES:

PE (HP) = 1E+33.0  
V (FT/SEC) = 1.93E-04  
OS (FT/HP) = 1.02E-04  
PD (HP) = 2400.00  
J = 0.7343  
KI = 0.1713  
KQ = 0.0182  
KAC = 0.0182  
REYN = 0.18E+08  
RDTA (FT) = 21.9991  
P/C = 0.9981  
AE/AD = 0.8205  
T/C = 0.0330

CONSTRAINT VALUES:

MAX CIA (FT) = 22.0000  
MIN AE/AD = 0.5529  
MIN T/C = 0.01698



PROGRAM CALLS TO ANALYZ  
ICALL CALLS  
3 2  
5 2



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CCCCC  UUUUU  P P P P P  E E E E E  S S S S S
C C C  U U U  P P P P P  E E E E E  S S S S S
C C C  U U U  P P P P P  E E E E E  S S S S S
C C C C C  U U U U U  P P P P P  E E E E E  S S S S S

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C O N T R O L   P R O G R A M  
 F O R  
 E N G I N E E R I N G   S Y N T H E S I S

T I T L E  
 WAGENINGEN B-SERIES PROPELLER OPTIMIZATION









## 264



TOTAL NUMBER OF UNSOLVED PARAMETERS = 12

\* \* ESTIMATED DATA STORAGE REQUIREMENTS

INPUT	EXECUTION	AVAILABLE	INPUT	EXECUTION	AVAILABLE
87	210	10000	57	124	1000
	REAL			INTEGER	



```

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:  P.E.V
ENVIRONMENTAL PARAMETERS:
TEMP (DEG F) = 55.0000
DENSITY (LBF-SEC2/FT4) = 0.12849997E-04
VISCOSITY (LBF2/SEC) = 14.7000
ATMOSPHERIC PRESSURE (PSIA) = 14.7000
WATER VAPORIZATION PRESSURE (PSIA) = 6.2470

FULL PARAMETERS:
WAKE FRACTION = 6.2200
THRUST COEFFICIENT FRACTION = 6.1500
RELATIVE ROTATIONAL EFFICIENCY = 6.0250
NUMBER OF PROPELLERS = 1.0
DEPTH TO SHAFT CENTERLINE (FT) = 15.0000
DIAPETER LIMIT (FT) = 22.0000

PROPELLER PARAMETERS:
NUMBER OF BLADES = 5.0
MATERIAL TYPE = STAINLESS STEEL
ALLOWABLE STRESS (PSI) = 5400.0

SELECTION VALUES:
PE (HP/SEC) = 16153.0
N (RPM) = 4015120
QS (FT-LBF) = 6319868
PD (HP) = 8312672.0
JT = *****
KT = 6.1000
KV = 6.0770
KVAC = 6.0770
REYN75R = 6.1865
DIA (FT) = 0.71809
P/D = 36.0000
AE/AC = 6.5000
T/C .75R = 6.3000
T/C .75R = 6.0300

CONSTRAINT VALUES:
MAX CIA (FT) = 24.0000
MIN AE/DO = 31.1615
MIN T/C .75R = 0.198487

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.....
C C A M I N
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FORTRAN PROGRAM FOR
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CONSTRAINED FUNCTION MINIMIZATION
.....

```

INITIAL FUNCTION INFORMATION

```

OBJ = -C.166489E+00
DECISION VARIABLES (X-VECTOR)
11 C.30000E+00 0.30000E+02 0.10000E+00 0.50000E+00 C.30000E-01
CONSTRAINT VALUES (C-VECTOR)
71 -C.67500E-01 -0.93250E+00 -0.30074E+03 -0.96824E+03 C.15000E+00 -0.75000E+00
-C.15990E-01 -0.23264E+00 -0.10917E+03 0.36584E+03 C.10287E+03 0.56182E+01

```





```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.7C657E+0C
DECISICA VARIABLES (X-VARIABLE)
11 C.81488E+00 0.21968E+02 0.73936E+00 C.10071E+01 C.64152E-01
CONSTRAINT VALUES (C-VECTOR)
11 -C.46210E+00 -0.5375CE+00 -0.25388E+02 -0.97361E+03 -0.36588E+00 -0.25516E+00
11 -C.56082E-01 -0.15848E+00 0.17331E-02 -0.15509E-02 -0.35365E+00 -0.16980E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
1C
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(11-COBJ1)-OBJ(11) LESS THAN DLFUN FOR 30 ITERATIONS
ABS(1CBJ11)-OBJ(11) LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 73
OBJECTIVE FUNCTION WAS EVALUATED 491 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 491 TIMES

```



# OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.70658E+00  
GLOBAL LOCATION

DESIGN VARIABLES				
ID	G. V. NO.	GLOBAL VAR. NO.	LOWER BOUND	UPPER BOUND
1	1	1	0.2000E+00	0.2000E+01
2	2	2	0.0000E+00	0.2000E+02
3	3	3	0.0000E+00	0.2000E+03
4	4	4	0.0000E+00	0.2000E+04
5	5	5	0.0000E+00	0.2000E+05

## DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	0.0000E+00	0.0000E+00	0.0000E+00
2	2	0.0000E+00	0.0000E+00	0.0000E+00
3	3	0.0000E+00	0.0000E+00	0.0000E+00
4	4	0.0000E+00	0.0000E+00	0.0000E+00
5	5	0.0000E+00	0.0000E+00	0.0000E+00
6	6	0.0000E+00	0.0000E+00	0.0000E+00
7	7	0.0000E+00	0.0000E+00	0.0000E+00
8	8	0.0000E+00	0.0000E+00	0.0000E+00
9	9	0.0000E+00	0.0000E+00	0.0000E+00
10	10	0.0000E+00	0.0000E+00	0.0000E+00
11	11	0.0000E+00	0.0000E+00	0.0000E+00
12	12	0.0000E+00	0.0000E+00	0.0000E+00



```

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
                               SUBROUTINE "STRCKK"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
PE,V
TEMP (DEG F) 55.0000
DENSITY (LBF-SEC2/FT4) 1.9384
VISCOSITY (LBF2/SEC2) 0.12844997E-04
ATMOSPHERIC PRESSURE (PSIA) 14.7000
WATER VAPORIZATION PRESSURE (PSIA) 0.2470

FULL PARAMETERS:
WAKE FRACTION 0.2200
THRUST REDUCTION FRACTION 0.1900
RELATIVE ROTATIVE EFFICIENCY 1.0250
NUMBER OF PROPELLERS 1.0
DEPTH (C SHAFT CENTERLINE (FT)) 15.0000
DIAPETER (IN) 22.0000

PROPELLER PARAMETERS:
NUMBER OF BLADES 5.0
MATERIAL TYPE STAINLESS STEEL
ALLOWABLE STRESS (PSI) 5400.0

SELECTION VALUES:
PE (LBF/SEC) 14193.0
V (FT/SEC) 40.5120
N (RPM) 116.7419
QS (FT-LBF) 1684003.0
PO (HP) 24094.79
J 0.7394
KT 0.1174
KQ 0.0289
KVA 0.7066
REYNOLDS DIA (FT) 0.56+08
P/O 21.9659
AE/AC 1.0071
T/C .756 0.8149
0.0842

CONSTRAINT VALUES:
MAX CIA (FT) 22.0000
MIN AE/AC 0.5267
MIN T/C .756 0.53259

```



PROGRAM CALLS TO ANALYZ	
LOCAL	CALLS
1	451
2	2
3	





# APPENDIX F

## ANALIZ CODES--DESIGN CASE NO. 2

```

SUBROUTINE ANALIZ(ICALC)
INTEGER*4 ICALC
REAL*4  ETAO,WEIGHT,AEDVAO,DIA,N,PE,POIVL,QS,TC75R,V,
1      RJC�L,RJCNL,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2      FCWBAL,DIA,CNU,AEADCV,TCSTRS,RJ,
3      VK,TC,WT,Z,WATRC,WATNU,TEMP,NOSCRW,HCL,PAIM,
4      PWATVA,PRCMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
5      C75R,R75R,KT,KC,PD,VAK
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1RJCNL,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,PWBAL,DIA,CNU,
2AEACCV,TCSTRS,RJ
COMMON /PARAM/VK,TE,WT,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PAIM,PWATVA,
1PRCMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

THIS SUBROUTINE, COUPLED WITH COPEX/CONMIN, CONSTITUTES ANALYSIS
METHOD FOR "DESIGN CASE 2" PROPELLER SELECTION PROBLEMS

INPLT-INITIALIZATION PHASE

PII=3.14159264
IF(.NOT.(ICALC.EQ.1))GC TO 1

SET "DESIGN CASE 2" PARAMETERS

ENVIRONMENTAL

TEMP=64.4
WATRO=1.9852
WATNU=.000011500
PAIM=12.657
PWATVA=.295435291

PRCPELLER PARAMETERS

Z=6.0
PRCMAT=5.0

HULL PARAMETERS

WT=C.22
TC=C.15
ETARR=1.025
NOSCRW=1.0
HCL=21.9827
DIALIM=30.0

SET DESIGN VARIABLES FELD FIXED FOR "DESIGN CASE 2"

```



```

CC      QS=121C129.835
CC      N=110.C
CC      VK=20.C641C256
CC      V=1.68E#VK
CC
CC      END OF INPUT-INITIALIZATION PHASE
CC
CC      GO TO 3
CC
CC      EXECUTION PHASE
CC
1 CONTINUE
  IF(.NOT.(ICALC.EQ.2))GC TO 2
  TC75R=((O.C185-O.C0125*Z)*Z)/(2.073*AEDVAG)
  CALL RCCAL(DIA)
  CALL CF75RA(C75R,R75R)
  CALL REY75R(C75R,R75R)
  CALL CCEESA(RJ,R75R,KI,KQ)
  CALL OPWEFF(RJ,KI,KQ,ETAD)
  CALL CALCPK(KI,PE)
  CALL JCNA(RJ,RJCNL,RJCNJ)
  CALL REYCNA(R75R,R75RCL,R75RCU)
  CALL EXTCCN(Z,AEDVAG,TC75R,AEADCL,AEADCU,TC75CL,TC75CU)
  CALL BLPOW2(KC,PC,BAL)
  CALL DICNUA(DIACNU)
  CALL C7VCNA(KI,AEACCV)
  CALL STRCNA(KQ,C75R,TCSTRS)
CC
CC      END OF EXECUTION PHASE
CC
CC      GO TO 3
CC
CC      OUTPUT-RESULT PHASE
CC
  VK=V/1.688
  VAK=(1.0-KI)*VK
  PC=(2.C*PII*QS*N)/33000.0
  WRITE(6,9000)
  WRITE(6,9001)TEMP,WATRO,WATNU,PATM,PWATVA
  WRITE(6,9003)WT,TD,ETARR,NOSCRW,HCL,DIALIM
  WRITE(6,9004)Z
  IF(.NOT.(PROMAT.EQ.1.0))GO TO 81
    WRITE(6,9005)SC
  GC TO 86
CC      CONTINUE
  IF(.NOT.(PROMAT.EQ.2.0))GC TO 82
CC
81

```















```

SUBROUTINE ANALIZ(ICALC)
INTEGER*4 ICALC
REAL*4 ETAG,WEIGHT,AEDVAD,DIA,N,PE,PDIVC,QS,TC75R,V,
1 RJC�L,RJCNU,R75FCL,R75RCU,AEADCL,AEACCU,TC75CL,TC75CU,
2 FCKBAL,DIACNU,AEACCV,TCSTRS,RJ,NCSCRW,HCL,PATM,
3 VK,TC,WI,Z,WATRO,WATNU,TEMP,AEADCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
4 PWATVA,PRUMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
5 C75R,R75R,KT,KQ,PD,VAK
COMMON /GLOECM/ETAC,WEIGHT,AEDVAD,DIA,N,PE,PDIVC,QS,TC75R,V,RJC�L,
1 RJCNU,R75RCL,R75RCU,AEADCL,AEACCV,TCSTRS,RJ
2 AEACCV,TCSTRS,RJ
COMMON /PARAM/VK,TD,WI,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1 PRUMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

THIS SUBROUTINE, COUPLED WITH COPEX/CONMIN, CONSTITUTES ANALYSIS
METHOD FOR "DESIGN CASE 2" PROPELLER SELECTION PROBLEMS

INPT-INITIALIZATION PHASE

PII=3.14159264
IF(.NOT.(ICALC.EQ.1))GO TO 1

SET "DESIGN CASE 2" PARAMETERS

ENVIRONMENTAL

TEMP=64.4
WATRO=1.9852
WATNU=.000011500
PATM=14.697
PWATVA=.299439291

PROPELLER PARAMETERS

Z=6.0
PRCMAT=5.0

HULL PARAMETERS

WT=C.22
TC=C.15
ETARR=1.025
NCSCRW=1.0
HCL=21.9827
DIALIM=30.0

SET DESIGN VARIABLES FELD FIXED FOR "DESIGN CASE 2"

```

CC  
CC  
CC  
CC  
CC  
CC

CC  
CC  
CC  
CC  
CC  
CC

CC  
CC  
CC

CC  
CC  
CC

CC  
CC  
CC



```

CC      QS=121(129.835
CC      N=110.C
CC      VK=20.(6410256
CC      V=1.68E*VK
CC
CC      END OF INPUT-INITIALIZATION PHASE
CC
CC      GO TO 3
CC
CC      EXECUTION PHASE
CC
1 CONTINUE
  IF(.NOT.(ICALC.EQ.2))GO TO 2
  TC75R=((0.0185-0.C0125*Z)*Z)/(2.073*AELVAO)
  CALL RCCAL(DIA)
  CALL CF75RA(C75R)
  CALL REY75R(C75R,R75R)
  CALL CCEFFSA(RJ,R75R,KT,KQ)
  CALL QWEEFF(RJ,KT,KQ,EIAD)
  CALL CALCPE(KT,PE)
  CALL JCNA(RJ,RJCNL,RJCNU)
  CALL REYCNA(R75R,R75RCL,R75KCU)
  CALL EXTCCN(Z,AELVAO,TC75R,AEACCL,AEADCU,TC75CL,TC75CU)
  CALL BLPOW2(KQ,POMBAL)
  CALL DICNUA(DIACNU)
  CALL CAVCNA(KT,AEADCV)
  CALL STRCNK(KQ,KT,C75R,TCSTKS)
CC
CC      END OF EXECUTION PHASE
CC
CC      GO TO 3
CC
2 CONTINUE
  OUTPUT-RESULT PHASE
  VK=V/1.688
  VAK=(1.0-WI)*VK
  PC=(2.C*PII*QS*N)/33000.0
  WRITE(6,90C0)
  WRITE(6,9001) TEMP,WATRO,WATRU,PATM,PWATVA
  WRITE(6,9002) WT,TD,ETARR,NOSCRW,HCL,DIALIM
  WRITE(6,9003) Z
  IF(.NOT.(PRMAT.EQ.1.0))GO TO 81
  WRITE(6,9004) SC
  GC TC 86
  CC CONTINUE
  IF(.NOT.(PRMAT.EQ.2.0))GO TO 82
  81

```

```

APP02200
APP02210
APP02220
APP02230
APP02240
APP02250
APP02260
APP02270
APP02280
APP02290
APP02300
APP02310
APP02320
APP02330
APP02340
APP02350
APP02360
APP02370
APP02380
APP02390
APP02400
APP02410
APP02420
APP02430
APP02440
APP02450
APP02460
APP02470
APP02480
APP02490
APP02500
APP02510
APP02520
APP02530
APP02540
APP02550
APP02560
APP02570
APP02580
APP02590
APP02600
APP02610
APP02620
APP02630
APP02640
APP02650
APP02660
APP02670

```













```

SUBROUTINE ANALIZ(ICALC)
INTEGER*4 ICALC
REAL*4
1  ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVC,QS,TC75R,V,
2  RJCNU,RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
3  FGWBAL,DIACNU,AEACUV,TCSTRS,RJ,
4  VK,TL,WI,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PATM,
5  PWATVA,PRGMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVC,QS,TC75R,V,RJCNU,
1RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2AEACCV,TCSTRS,RJ
COMMON /PARAM/VK,TD,WI,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PRGMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

THIS SUBROUTINE, COUPLED WITH COPEX/CONMIN, CONSTITUTES ANALYSIS
METHOD FOR "DESIGN CASE 2" PROPELLER SELECTION PROBLEMS

INPUT-INITIALIZATION PHASE

PI=3.14159264
IF(.NOT.(ICALC.EQ.1))GO TO 1

SET "DESIGN CASE 2" PARAMETERS

ENVIRONMENTAL

TEMP=64.4
WATRO=1.9892
WATNU=.000011500
PATM=14.697
PWATVA=.255435291

PROPELLER PARAMETERS

Z=6.0
PRGMAT=5.0

HULL PARAMETERS

WT=0.22
TC=0.15
ETARR=1.025
NCSCRW=1.0
HCL=21.9827
DIALIM=30.0

SET DESIGN VARIABLES HELD FIXED FOR "DESIGN CASE 2"

```



```

CC      QS=121(125.835
CC      N=110.C
CC
C      ENL OF INPUT-INITIALIZATION PHASE
C
C      GO TO 3
C
C      EXECUTION PHASE
C
1 CONTINUE
  IF(.NOT.(ICALC.EQ.2))GC TO 2
  CALL RCCAL(DIA)
  CALL CF75RA(C75R)
  CALL REY75R(C75R,R75K)
  CALL CCEESA(RJ,R75R,KT,KQ)
  CALL OFWEFF(RJ,KT,KQ,ETAU)
  CALL CALCPE(KT,PE)
  CALL JCNA(RJ,RJCNL,RJCNU)
  CALL REYCNA(R75R,R75RCL,R75RCU)
  CALL EXTCCN(Z,AEDVAG,TC75R,AEACCL,AEADCU,TC75CL,TC75CU)
  CALL BLPOW2(KC,POWBAL)
  CALL DICNUA(DIACNU)
  CALL CAVCNA(KT,AEADCV)
  CALL STRCNA(KC,C75R,TCSTRS)
  ENL GF EXECUTION PHASE
CC
CC
CC
C      GO TO 3
C
2 CONTINUE
  OUTPUT-RESULT PHASE
  VK=V/1.688
  VAK=(1.0-WI)*VK
  PC=(2.C#PI*QS*N)/33000.0
  WRITE(6,9000)
  WRITE(6,9001)
  WRITE(6,9002)TEMP,WATRO,WATNU,PATM,PWATVA
  WRITE(6,9003)WT,ID,ETARR,NOSCRW,HCL,DIALIM
  WRITE(6,9004)Z
  IF(.NOT.(PROMAT.EQ.1.0))GC TO 81
  WRITE(6,9005)SC
  GC TO 86
  CCATINLE
  IF(.NOT.(PROMAT.EQ.2.0))GC TO 82
  WRITE(6,9006)SC
  GC TO 86
  CCATINLE
81
82

```



```

83      IF(.NOT.(PROMAT.EQ.3.0))GO TO 83
        WRITE(6,9007)SC
        GC TC 86
        CCNTINLE
84      IF(.NOT.(PROMAT.EQ.4.0))GO TO 84
        WRITE(6,9008)SC
        GC TC 86
        CCNTINLE
85      IF(.NOT.(PROMAT.EQ.5.0))GO TC 85
        WRITE(6,9009)SC
        GC TC 86
        CCNTINLE
86      CCNTINLE
        WRITE(6,9010)
        WRITE(6,9011)PE,V,VK,VAK,N,GS,PD,RJ,KT,KQ,ETAO,R75R,DIA,
        PCIVC,AECVAO,IC75R
        WRITE(6,9012)DIALIM,AEAOMN,IC75MN
3      CONTINUE
      RETURN
C
C      MISCELLANEOUS FORMAT STATEMENTS
C
9000  FORMAT(1,'OPTIMIZATION RESULTS -----
1X,
1X,DESIGN VARIABLES SPECIFIED:
9001  FORMAT(1X,ENVIRONMENTAL PARAMETERS:
9002  FORMAT(1X,
1X,4,/,
1X,25X,DENSITY (LBF-SEC2/FT4),12X,=,F10.4,/,
1X,25X,VISCOSITY (FT2/SEC),15X,=,E16.9,/,
1X,25X,ATMOSPHERIC PRESSURE (PSIA),7X,=,F10.4,/,
1X,25X,ATMOSPHERIC PRESSURE FRACTION (PSIA)=,F10.4,/,
1X,25X,WATER VAPORIZATION PRESSURE FRACTION,21X,=,F10.4,/,
1X,25X,WATER VAPORIZATION PRESSURE FRACTION,9X,=,F10.4,/,
9003  FORMAT(1X,FULL THRUST DEDUCTION EFFICIENCY,6X,=,F10.4,/,
1X,25X,RELATIVE ROTATIVE EFFICIENCY,14X,=,F10.1,/,
1X,25X,RELATIVE ROTATIVE EFFICIENCY,14X,=,F10.1,/,
1X,25X,DEPTH TO SHAFT CENTERLINE (FT),4X,=,F10.4,/,
1X,25X,DIAMETER LIMIT (FT),15X,=,F10.4,/,
9004  FORMAT(1X,PROPELLER PARAMETERS:,8X,NUMBER OF BLADES,18X,=,
1X,F10.1),
9005  FORMAT(1X,25X,MATERIAL TYPE,21X,=,CAST IRON,/,
1X,25X,ALLCABLE STRESS (PSI),12X,=,F10.1,/,
9006  FORMAT(1X,25X,MATERIAL TYPE,21X,=,CAST STEEL,/,
1X,25X,ALLCABLE STRESS (PSI),12X,=,F10.1,/,
9007  FORMAT(1X,25X,MATERIAL TYPE,21X,=,BRONZE,/,
1X,25X,ALLCABLE STRESS (PSI),12X,=,F10.1,/,
9008  FORMAT(1X,25X,MATERIAL TYPE,21X,=,N1-AL BRONZE,/,
1X,25X,ALLCABLE STRESS (PSI),12X,=,F10.1,/,
9009  FORMAT(1X,25X,MATERIAL TYPE,21X,=,STAINLESS STEEL,/,

```

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APP04390
APP04400
APP04410
APP04420
APP04430
APP04440
APP04450
APP04460
APP04470
APP04480
APP04490
APP04500
APP04510
APP04520
APP04530
APP04540
APP04550
APP04560
APP04570
APP04580
APP04590
APP04600
APP04610
APP04620
APP04630
APP04640
APP04650
APP04660
APP04670
APP04680
APP04690
APP04700
APP04710
APP04720
APP04730
APP04740
APP04750
APP04760
APP04770
APP04780
APP04790
APP04800
APP04810
APP04820
APP04830
APP04840
APP04850
APP04860

```









```

SUBROUTINE ANALIZ(ICALC)
INTEGER*4 ICALC
REAL*4 ETAO,WEIGHT,AEDVAO,DIA,N,PE,PD1VC,QS,TC75R,V,
1 RJCNL,RJCNL,R75RCL,R75RCU,AEACCL,AEACCU,TC75CL,TC75CU,
2 FCWBFL,DIA,NU,AEACCV,TCSTRS,RJ,
3 VK,TC,WT,Z,WATRC,WATNU,TEMP,NO$CRW,HCL,PAIM,
4 PWATVA,PRCMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
5 C75R,R75R,KT,KC,PD,VAK
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1 RJCNL,R75RCL,R75RCU,AEACCL,AEACCU,TC75CL,TC75CU,PWBAL,DIA,NU,
2 AEACCV,TCSTRS,KJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NC$CRW,HCL,PAIM,PWATVA,
1 PRCMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

THIS SUBROUTINE, COUPLED WITH COPEX/COMIN, CONSTITUTES ANALYSIS
METHOD FOR "DESIGN CASE 2" PROPELLER SELECTION PROBLEMS

INPUT-INITIALIZATION PHASE

PII=3.14159264
IF(.NOT.(ICALC.EQ.1))GO TO 1

SET "DESIGN CASE 2" PARAMETERS

ENVIRONMENTAL

TEMP=64.4
WATRO=1.9852
WATNU=.00011900
PATN=14.697
PWATVA=.295435291

PRCFELLER PARAMETERS

Z=6.0
PRCMAT=5.0

HULL PARAMETERS

WT=C.22
TC=C.15
ETARR=1.025
NC$CRW=1.0
HCL=21.9827
DIALIM=30.0

SET DESIGN VARIABLES HELD FIXED FOR "DESIGN CASE 2"

```

CC  
CC  
CC  
CC

CC  
CC  
CC  
CC

CC  
CC  
CC

CC  
CC  
CC

CC  
CC  
CC



APP05590  
 APP05600  
 APP05610  
 APP05620  
 APP05630  
 APP05640  
 APP05650  
 APP05660  
 APP05670  
 APP05680  
 APP05690  
 APP05700  
 APP05710  
 APP05720  
 APP05730  
 APP05740  
 APP05750  
 APP05760  
 APP05770  
 APP05780  
 APP05790  
 APP05800  
 APP05810  
 APP05820  
 APP05830  
 APP05840  
 APP05850  
 APP05860  
 APP05870  
 APP05880  
 APP05890  
 APP05900  
 APP05910  
 APP05920  
 APP05930  
 APP05940  
 APP05950  
 APP05960  
 APP05970  
 APP05980  
 APP05990  
 APP06000  
 APP06010  
 APP06020  
 APP06030  
 APP06040  
 APP06050  
 APP06060

```

    QS=121C129.835
    N=110.C
    ENC OF INPUT-INITIALIZATION PHASE

    GO TO 3
    EXECUTION PHASE

    1 CONTINUE
    IF(.NOT.(ICALC.EQ.2))GC TO 2
    CALL RICAL(DIAL)
    CALL CF75RA(C75R)
    CALL REY75R(C75R,R75R)
    CALL CCEFFSA(RJ,R75R,KI,KQ)
    CALL OFWEFF(RJ,KT,KQ,ETAO)
    CALL CALCOPE(KT,PE)
    CALL JCNA(RJ,RJCNL,RJCNL)
    CALL REYCNA(R75R,R75RCL,R75RCU)
    CALL EXTCCN(Z,AEDVAO,TC75R,AEACCL,AEADCU,TC75CL,TC75CU)
    CALL BLPOW2(KQ,POWBAL)
    CALL DIGNUA(DIACNU)
    CALL CAVCNA(KT,AEADCV)
    CALL STRCNK(KC,KT,C75R,TCSTRS)
    ENC OF EXECUTION PHASE

    GO TO 3
    2 CONTINUE
    OUTPUT-RESULT PHASE

    VK=V/1.688
    VAK=(1.0-WT)*VK
    PC=(2.C*PI)*QS*N/35000.0
    WRITE(6,9000)
    WRITE(6,9001) TEMP,WATRU,WATNU,PAIM,PWATVA
    WRITE(6,9002) WT,TD,ETARR,NUSCRW,HCL,DIALIM
    WRITE(6,9004) Z
    IF(.NOT.(PROMAT.EQ.1.0))GC TO 81
    WRITE(6,9005)SC
    GC TO 86
    CCNTINLE
    IF(.NOT.(PROMAT.EQ.2.0))GC TO 82
    GC TO 86
    CCNTINLE
  
```

CC  
CC  
CC

C  
C  
C

CC  
CC  
CC

C  
C  
C

81

82







```

1 1X,25X,ALL QWABLE STRESS (PSI),12X,=,F10.1,/,
9010 1X,25X,MATERIAL TYPE,12X,=,ACT CCNSIDEREC,/,
9011 1X,25X,SELECTION VALUES,12X,=,F10.1,/,
1 1X,25X,V (FT/SEC),23X,=,F10.4,/,
1X,25X,V (KNCITS),23X,=,F10.4,/,
1X,25X,V (KNOTS),23X,=,F10.4,/,
1X,25X,N (RPM),23X,=,F10.4,/,
1X,25X,N (CS (FT-LBF),23X,=,F12.1,/,
1X,25X,N (PD (HP),23X,=,F10.2,/,
1X,25X,N (KT),23X,=,F10.4,/,
1X,25X,N (KQ),23X,=,F10.4,/,
1X,25X,N (ETA C),23X,=,F10.4,/,
1X,25X,N (REV 75R),23X,=,F10.1,/,
1X,25X,N (C/D),23X,=,F10.4,/,
1X,25X,N (AE/AG),23X,=,F10.4,/,
1X,25X,N (T/C),23X,=,F10.4,/,
9012 1X,25X,CONSTRAINT VALUES,11X,MAX DIA (FT),22X,=,F10.4,/,
1X,25X,MIN AE/AG,25X,=,F10.4,/,
1X,25X,MIN T/C,75R,2X,=,F10.6)
END

```





# APPENDIX G

## CONTROL CARD IMAGES--DESIGN CASE NO. 2

\$A	TITLE	B-SERIES	PROPELLER	CPTIMIZATION	NXAPRX	IPNPUT	IPDBG
\$B	WAGENINGEN	NDV	NSV	N2VAR		0	NACMX1
\$C	IPRINT	ITMAX	ICNDR	NSCAL	ITRM	LINUBJ	15
\$D1	FDCH	ICOO	CT	CTMIN	3C	CTLMIN	THETA
\$D2	DELFUN	OCOC	ALPHA	ABGBJI	CTL		
\$E	NDVTOT	ICBJ	SGNCPT				
\$F	VLB	VUB	1.0	SCAL			
	0.4	1.1	0.40	1.0			
	0.01	1.1	0.1	1.0			
\$G	NDSSGN	1.4	1.0	1.0			
	1	IDSGN	AMULT				
	2	3	1.0				
	3	23	1.0				
\$H	NCUNS	7					
\$I1	10	JCON	LCON	SCAL2			
\$I2	ICON	SCAL1	BU	1.0			
-1.0	11	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
-1.0	+16	1.0	0.0	1.0			
\$V	+16	1.0	0.0	1.0			
END							



\$A	TITLE	GEN	B-SERIES	PROPELLER	OPTIMIZATION	IPNPUT	IPDBG
\$B	WAGENING	NALC	ADV	NSV	N2VAR	LINOBJ	NACMX1
\$C	IPRINT	2	5	ICNDR	NSCAL	CTLMIN	15
\$D1	FDCH	1	ITMAX	CT	CTMIN		THETA
\$D2	DELFUN	1	1COO	ALPHAX	ABCBJI		
\$E	ADVTOI	5	FDCHM	SGNCP			
\$F	VLB	5	0.001	1.0			
			0.01	0.40	SCAL		
			0.4	0.1	1.0		
			0.4	1.0	1.0		
			10.0	50.0	10.0		
\$G	NDSSGN	1	100.0	0.0500	0.01		
		2	100.0	AMULT			
		3	100.0	1.0			
		4	100.0	1.0			
		5	100.0	1.0			
\$H	NCON	12	100.0	1.0			
\$I1	ICCN	12	100.0	1.0			
\$I2	ICCN	12	100.0	1.0			
-1.0		11	100.0	1.0			
-1.0		12	100.0	1.0			
-1.0		13	100.0	1.0			
-1.0		14	100.0	1.0			
-1.0		15	100.0	1.0			
-1.0		16	100.0	1.0			
-1.0		17	100.0	1.0			
-1.0		18	100.0	1.0			
-1.0		19	100.0	1.0			
-1.0		20	100.0	1.0			
-1.0		21	100.0	1.0			



-1.0	21	0.0	1.0
-1.0	+16	0.0	1.0
4V	22	0.0	1.0
END	+16		



# APPENDIX H

## COPEs OUTPUT--DESIGN CASE NO. 2

```

CCCCC 000000 PPPPP EEEEE SSSSS
C      0 0 0 P P P P
C      0 0 0 P P P P
C      0 0 0 P P P P
CCCCC 000000 PPPPP EEEEE SSSSS

```

```

C C N T R C L P R O G R A M
F O R
E N G I N E E R I N G S Y N T H E S I S

```

```

T I T L E
W A G E N I N G E N B - S E R I E S P R O P E L L E R O P T I M I Z A T I O N

```









# TITLE: WAGGONRULN D-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:  
CALCULATION CONTROL: NCALC = 3  
NUMBER OF GLOBAL OPTIMUM VARIABLES: NOV = 3  
NUMBER OF SENSITIVITY VARIABLES: NSV = 0  
NUMBER OF APPROXIMATING VARIABLES: NVARP = 0  
NUMBER OF APPROXIMATING VARIABLES: NVARP = 0  
INPUT INTERUPTION CODE: IFLB = 0  
DEBUG PRINT CODE:

CALCULATION CONTROL: NCALC  
VALUE  
PRINTING  
SINGLE ANALYSIS  
INITIALIZATION  
SENSITIVITY  
PRO-VARIABLES: FUNCTION SPACE  
APPROXIMATE OPTIMIZATION

## \* \* OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE  
MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = 0.1000E+01

CONTROL PARAMETERS IF ZERO, COMMON DEFAULT WILL OVER-RIDE

PRINT LIMIT: LNCAL 30 LNCBU 15 NFEQ 0  
FOLH 0.1000E-03 FOLHM 0.1000E-02 CLIMIN 0.0  
CL 0.0 CLHIA 0.0  
DELFUN 0.0 UAFUN 0.0 ALPHAX 0.0  
ADUBUJ 0.0

LESLIM VARIABLE INFORMATION  
NON-ZERO INITIAL VALUE: UPPER OVER-RIDE INITIAL SCALE  
NO. 1 0.1000E+01 0.1000E+01 0.1000E+01  
2 0.1000E+01 0.1000E+01 0.1000E+01  
3 0.1000E+01 0.1000E+01 0.1000E+01

DESIGN VARIABLES  
L.V. GLOBAL MULTIPLYING  
ID L.V. VAR. NO. FACTOR  
1 1 1 0.1000E+01  
2 2 0.1000E+01  
3 3 0.1000E+01

## CONSTRAINT INFORMATION

HERE AFF JC	GLOBAL LINEAR	UPPER	LOWER	NORMALIZATION	UPPER	LOWER	NORMALIZATION
ID	VAR.	1	2	10	10	10	10
1	1	1	1	0.1000E+01	0.0	0.0	0.1000E+01
2	2	1	1	0.1000E+01	0.0	0.0	0.1000E+01
3	3	1	1	0.1000E+01	0.0	0.0	0.1000E+01
4	4	1	1	0.1000E+01	0.0	0.0	0.1000E+01
5	5	1	1	0.1000E+01	0.0	0.0	0.1000E+01
6	6	1	1	0.1000E+01	0.0	0.0	0.1000E+01
7	7	1	1	0.1000E+01	0.0	0.0	0.1000E+01
8	8	1	1	0.1000E+01	0.0	0.0	0.1000E+01
9	9	1	1	0.1000E+01	0.0	0.0	0.1000E+01
10	10	1	1	0.1000E+01	0.0	0.0	0.1000E+01

TOTAL NUMBER OF CONSTRAINED PARAMETERS = 10



INPUT	REAL	AVAILABLE	INPUT	INJECTION	AVAILABLE
67	490	10000	43	104	1000



```

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 2
SUBROUTINE "STRUNA"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
QS,N,V,IC75R
TEMP (DEG F) = 64.4000
DENSITY (LBF-SEC2/FT4) = 0.119892
VISCOSITY (LBF-SEC/IN2) = 0.189997E-04
WATER VAPORIZATION PRESSURE (PSIA) = 1.2234
WATER VAPORIZATION PRESSURE (PSIA) = 0.2234

FULL PARAMETERS:
WAKE FRACTION = 0.2200
THRU-SECTION FRACTION = 0.1900
RELATIVE WAKE EFFICIENCY = 1.0150
DEPT. OF SHOT CENTERLINE (FT) = 21.9827
DIAPETER LIMIT (FT) = 36.0000

PROPELLER PARAMETERS:
NUMBER OF BLADES = 6
MATERIAL TYPE = STAINLESS STEEL
ALLOWABLE STRESS (PSI) = 3400.0

SELECTION VALUES:
PE (FT/SEC) = 5152.440.0
V (KNOTS) = 33.8682
VA (KNOTS) = 20.0641
QS (FT/HP) = 11.6500
PD (HP) = 1101000.0
JT = 2344.30
KQAC = 0.1000
REF5R = 0.4001
DFA (FT) = 0.0529
P/C = 0.1162
AE/AC = 14.1617
T/C .75R = 14.1617
AE/AC = 1.0000
T/C .75R = 1.4000
AE/AC = 0.4000
T/C .75R = 0.0796

CONSTRAINT VALUES:
MAX CIA (FT) = 36.0000
MIN AE/AC = 0.4798
MIN T/C .75R = 0.062833

```





```

* * * * *
* * * * * C C P M I N * * * * *
* * * * * F O R T R A N   P R O G R A M   F O R * * * * *
* * * * * C O N S T R A I N E D   F U N C T I O N   M I N I M I Z A T I O N * * * * *
* * * * *

```

INITIAL FUNCTION INFORMATION

```

OBJ = -C.118204E+00
DECISION VARIABLES (X-VECTOR)
11 C.40000E+00 0.10000E+00 0.10000E+01
CONSTRAINT VALUES (G-VECTOR)
11 -C.82500E-01 -0.97750E+00 -0.52020E+03 -0.47874E+03 C.10000E+00 -0.40000E+00
71 -C.58696E-01 -0.47751E-01 -0.99936E+00 0.12210E+03

```



```

FINAL OPTIMIZATION INFORMATION
CBJ = -C.665992E+00
DECISION VARIABLES (X-VECTOR)
11 C.8177E+00 0.84752E+00 0.50364E+00
CONSTRAINT VALUES (G-VECTOR)
11 -C.46470E+00 -0.59836E+00 -0.24820E+02 -0.97418E+03 -C.30178E+00 0.17773E-02
11 -C.15810E-01 -0.87842E-01 -0.23307E-02 -0.36712E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
6
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(OBJ1-OBJ11) LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 45
OBJECTIVE FUNCTION WAS EVALUATED 230 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 230 TIMES

```



```

OPTIMIZATION RESULTS

OBJECTIVE FUNCTION      1      FUNCTION VALUE  0.66599E+00
GLOBAL LOCATION

DESIGN VARIABLES
  D, V,      GLOBAL      LOWER      VALUE      UPPER
  NC,      VAR, NG,      BOUND      BOUND      BOUND
  1      1      23      0.4000E+00      0.8017E+00      0.1000E+01
  2      2      23      0.1000E+01      0.8432E+00      0.1000E+01
  3      3      23      0.4000E+00      0.5036E+00      0.1000E+01

DESIGN CONSTRAINTS
  GLOBAL      LOWER      VALUE      UPPER
  VAR, NO.      BOUND      BOUND      BOUND
  10      1      0000E+00      0.0000E+00      0.0000E+00
  20      2      0000E+00      0.0000E+00      0.0000E+00
  30      3      0000E+00      0.0000E+00      0.0000E+00
  40      4      0000E+00      0.0000E+00      0.0000E+00
  50      5      0000E+00      0.0000E+00      0.0000E+00
  60      6      0000E+00      0.0000E+00      0.0000E+00
  70      7      0000E+00      0.0000E+00      0.0000E+00
  80      8      0000E+00      0.0000E+00      0.0000E+00
  90      9      0000E+00      0.0000E+00      0.0000E+00
  100     10     0000E+00      0.0000E+00      0.0000E+00

```



OPTIMIZATION RESULTS ----- DESIGN CASE NO. 2  
SUBROUTINE "STRUNA"

DESIGN VARIABLES SPECIFIED:

ENVIRONMENTAL PARAMETERS:  
QS,N,V,IC75H  
TEMP (DEG F)  
DENSITY (LBF/SEC<sup>3</sup>)  
VISCOSITY (LBF/SEC<sup>2</sup>)  
WATER VAPORIZATION PRESSURE (PSIA)  
WATER VAPORIZATION PRESSURE (PSIA)

= 64.4000  
= 0.189992  
= 0.1899997E-04  
= 1.0E-04  
= 0.2994

HULL PARAMETERS:

WAKE FRACTION  
THRUST COEFFICIENT  
REQUIRED PROPELLER EFFICIENCY  
REQUIRED PROPELLER CENTERLINE IFT  
DIAMETER LIMIT (FT)

= 0.2200  
= 0.1900  
= 1.0E-04  
= 21.9427  
= 36.0000

PROPELLER PARAMETERS:

NUMBER OF BLADES  
MATERIAL TYPE  
ALLOWABLE STRESS (PSI)

= 6  
= STAINLESS STEEL  
= 2400.0

SELECTION VALUES:

PE (FT)  
V (FT/SEC)  
VA (KNOTS)  
VA (KNOTS)  
NS (LBF/IN)  
PO (HP)

= 14057.3  
= 33.8682  
= 20.0641  
= 11.0E-04  
= 11.0E-04  
= 2344.00.0

JT  
KT  
KG/C  
RE/C  
DIA (FT)  
P/C  
AE/AC  
T/C .75R

= 0.9775  
= 0.1719  
= 0.0266  
= 0.0266  
= 21.5553  
= 0.9036  
= 0.8018  
= 0.0397

CONSTRAINT VALUES:

MAX CIA (FT)  
MIN AE/AC  
MIN T/C .75R

= 36.0000  
= 0.5070  
= 0.02297





PROGRAM CALLS TO ANALYZ  
CALL CALLS  
1  
2  
3







CARD IMAGES OF CONTROL DATA

CARD	IMAGE	PROPPELLER OPTIMIZATION									
		1	2	3	4	5	6	7	8	9	10
1	1	1	1	1	1	1	1	1	1	1	1
2	2	1	1	1	1	1	1	1	1	1	1
3	3	1	1	1	1	1	1	1	1	1	1
4	4	1	1	1	1	1	1	1	1	1	1
5	5	1	1	1	1	1	1	1	1	1	1
6	6	1	1	1	1	1	1	1	1	1	1
7	7	1	1	1	1	1	1	1	1	1	1
8	8	1	1	1	1	1	1	1	1	1	1
9	9	1	1	1	1	1	1	1	1	1	1
10	10	1	1	1	1	1	1	1	1	1	1
11	11	1	1	1	1	1	1	1	1	1	1
12	12	1	1	1	1	1	1	1	1	1	1
13	13	1	1	1	1	1	1	1	1	1	1
14	14	1	1	1	1	1	1	1	1	1	1
15	15	1	1	1	1	1	1	1	1	1	1
16	16	1	1	1	1	1	1	1	1	1	1
17	17	1	1	1	1	1	1	1	1	1	1
18	18	1	1	1	1	1	1	1	1	1	1
19	19	1	1	1	1	1	1	1	1	1	1
20	20	1	1	1	1	1	1	1	1	1	1
21	21	1	1	1	1	1	1	1	1	1	1
22	22	1	1	1	1	1	1	1	1	1	1
23	23	1	1	1	1	1	1	1	1	1	1
24	24	1	1	1	1	1	1	1	1	1	1
25	25	1	1	1	1	1	1	1	1	1	1
26	26	1	1	1	1	1	1	1	1	1	1
27	27	1	1	1	1	1	1	1	1	1	1
28	28	1	1	1	1	1	1	1	1	1	1
29	29	1	1	1	1	1	1	1	1	1	1
30	30	1	1	1	1	1	1	1	1	1	1
31	31	1	1	1	1	1	1	1	1	1	1
32	32	1	1	1	1	1	1	1	1	1	1
33	33	1	1	1	1	1	1	1	1	1	1
34	34	1	1	1	1	1	1	1	1	1	1
35	35	1	1	1	1	1	1	1	1	1	1
36	36	1	1	1	1	1	1	1	1	1	1
37	37	1	1	1	1	1	1	1	1	1	1
38	38	1	1	1	1	1	1	1	1	1	1
39	39	1	1	1	1	1	1	1	1	1	1
40	40	1	1	1	1	1	1	1	1	1	1
41	41	1	1	1	1	1	1	1	1	1	1
42	42	1	1	1	1	1	1	1	1	1	1
43	43	1	1	1	1	1	1	1	1	1	1
44	44	1	1	1	1	1	1	1	1	1	1
45	45	1	1	1	1	1	1	1	1	1	1
46	46	1	1	1	1	1	1	1	1	1	1
47	47	1	1	1	1	1	1	1	1	1	1
48	48	1	1	1	1	1	1	1	1	1	1
49	49	1	1	1	1	1	1	1	1	1	1
50	50	1	1	1	1	1	1	1	1	1	1
51	51	1	1	1	1	1	1	1	1	1	1
52	52	1	1	1	1	1	1	1	1	1	1
53	53	1	1	1	1	1	1	1	1	1	1
54	54	1	1	1	1	1	1	1	1	1	1
55	55	1	1	1	1	1	1	1	1	1	1
56	56	1	1	1	1	1	1	1	1	1	1
57	57	1	1	1	1	1	1	1	1	1	1
58	58	1	1	1	1	1	1	1	1	1	1
59	59	1	1	1	1	1	1	1	1	1	1
60	60	1	1	1	1	1	1	1	1	1	1
61	61	1	1	1	1	1	1	1	1	1	1
62	62	1	1	1	1	1	1	1	1	1	1
63	63	1	1	1	1	1	1	1	1	1	1
64	64	1	1	1	1	1	1	1	1	1	1
65	65	1	1	1	1	1	1	1	1	1	1
66	66	1	1	1	1	1	1	1	1	1	1
67	67	1	1	1	1	1	1	1	1	1	1
68	68	1	1	1	1	1	1	1	1	1	1
69	69	1	1	1	1	1	1	1	1	1	1
70	70	1	1	1	1	1	1	1	1	1	1
71	71	1	1	1	1	1	1	1	1	1	1
72	72	1	1	1	1	1	1	1	1	1	1
73	73	1	1	1	1	1	1	1	1	1	1
74	74	1	1	1	1	1	1	1	1	1	1
75	75	1	1	1	1	1	1	1	1	1	1
76	76	1	1	1	1	1	1	1	1	1	1
77	77	1	1	1	1	1	1	1	1	1	1
78	78	1	1	1	1	1	1	1	1	1	1
79	79	1	1	1	1	1	1	1	1	1	1
80	80	1	1	1	1	1	1	1	1	1	1
81	81	1	1	1	1	1	1	1	1	1	1
82	82	1	1	1	1	1	1	1	1	1	1
83	83	1	1	1	1	1	1	1	1	1	1
84	84	1	1	1	1	1	1	1	1	1	1
85	85	1	1	1	1	1	1	1	1	1	1
86	86	1	1	1	1	1	1	1	1	1	1
87	87	1	1	1	1	1	1	1	1	1	1
88	88	1	1	1	1	1	1	1	1	1	1
89	89	1	1	1	1	1	1	1	1	1	1
90	90	1	1	1	1	1	1	1	1	1	1
91	91	1	1	1	1	1	1	1	1	1	1
92	92	1	1	1	1	1	1	1	1	1	1
93	93	1	1	1	1	1	1	1	1	1	1
94	94	1	1	1	1	1	1	1	1	1	1
95	95	1	1	1	1	1	1	1	1	1	1
96	96	1	1	1	1	1	1	1	1	1	1
97	97	1	1	1	1	1	1	1	1	1	1
98	98	1	1	1	1	1	1	1	1	1	1
99	99	1	1	1	1	1	1	1	1	1	1
100	100	1	1	1	1	1	1	1	1	1	1



# TITLE1 WAGENINGEN B-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:  
CALCULATION CONTROL: NCALC = 3  
NUMBER OF GLOBAL DESIGN VARIABLES, NDV = 3  
NUMBER OF PENALTY FUNCTIONS, NPF = 0  
NUMBER OF PENALTY FUNCTIONS IN TROUBLE, NXPFX = 0  
NUMBER OF APPROXIMATE VARS, NPAVAR = 0  
INPUT INFORMATION PRINT CODE, IPOBC = 0  
DEBUG PRINT CODE.

CALCULATION CONTROL: NCALC  
VALUE  
1  
SINGLE ANALYSIS  
2  
ITERATION  
3  
FUNCTION SPACE  
4  
MINIMUM SENSITIVITY  
5  
APPROXIMATE OPTIMIZATION  
6

## • • OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE  
MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = 0.1000E+01

CONSTRAINT PARAMETERS (IF ZERO, CONSTRAINT WILL OVER-RIDE)

PRINT ITHAX IGENR NSCAL ITRM LINDBJ NACHAT NPDG  
1 1000 0 -1 30 0 15 0

FOCUS FOCUS CT CTMIN  
0.1000E-03 0.1000E-03 -0.1000E-02 0.0

CTL CTMIN THETA PHI  
0.0 0.0 0.0 0.0

DELFUN DELFUN ALPHAX ABUSJ  
0.0 0.0 0.0 0.0

DESIGN VARIABLE INFORMATION

NON-ZERO INITIAL VALUE WILL OVER-RIDE HIGHER INPUT

NO. V. LOWER UPPER INITIAL

1 0.1000E+00 0.1000E+01 0.1000E+00 0.1000E+01

2 0.1000E+00 0.1000E+01 0.1000E+00 0.1000E+01

3 0.1000E+00 0.1000E+01 0.1000E+00 0.1000E+01

DESIGN VARIABLES

NO. V. GLOBAL MULTIPLYING

1 0.1000E+00 0.1000E+01

2 0.1000E+00 0.1000E+01

3 0.1000E+00 0.1000E+01

CONSTRAINT INFORMATION

THERE ARE 10 CONSTRAINT SETS

NO. V. GLOBAL LINEAR

1 0.1000E+00 0.1000E+01

2 0.1000E+00 0.1000E+01

3 0.1000E+00 0.1000E+01

4 0.1000E+00 0.1000E+01

5 0.1000E+00 0.1000E+01

6 0.1000E+00 0.1000E+01

7 0.1000E+00 0.1000E+01

8 0.1000E+00 0.1000E+01

9 0.1000E+00 0.1000E+01

10 0.1000E+00 0.1000E+01

TOTAL NUMBER OF CONSTRAINED PARAMETERS = 10

NORMALIZATION  
FACTOR  
1 0.1000E+01  
2 0.1000E+01  
3 0.1000E+01  
4 0.1000E+01  
5 0.1000E+01  
6 0.1000E+01  
7 0.1000E+01  
8 0.1000E+01  
9 0.1000E+01  
10 0.1000E+01

LOWER  
BOUND  
1 0.1000E+00  
2 0.1000E+00  
3 0.1000E+00  
4 0.1000E+00  
5 0.1000E+00  
6 0.1000E+00  
7 0.1000E+00  
8 0.1000E+00  
9 0.1000E+00  
10 0.1000E+00

NORMALIZATION  
FACTOR  
1 0.1000E+01  
2 0.1000E+01  
3 0.1000E+01  
4 0.1000E+01  
5 0.1000E+01  
6 0.1000E+01  
7 0.1000E+01  
8 0.1000E+01  
9 0.1000E+01  
10 0.1000E+01

LOWER  
BOUND  
1 0.1000E+00  
2 0.1000E+00  
3 0.1000E+00  
4 0.1000E+00  
5 0.1000E+00  
6 0.1000E+00  
7 0.1000E+00  
8 0.1000E+00  
9 0.1000E+00  
10 0.1000E+00





INPUT	REAL	AVAILABLE	INPUT	INTELEX	AVAILABLE
67	EXECUTION	10000	43	EXCLUSION	1000
	490			104	



```

OPTIMIZATION RESULTS -----
DESIGN CASE NO. 2
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
    QSN,V,1075K
    TIME IDIG FJ
    ORCAIY ILWF-SEC2/FT4)
    VISCOSITY (FT2/SEC)
    ATMOSPHERIC PRESSURE (PSIA)
    WATER VAPORIZATION PRESSURE (PSIA)
    = 64.4030
    = 0.11859997E-04
    = 14.6970
    = 6.2994

FULL PARAMETERS:
    MAKE FRACTION
    THRUST (REDUCTION FRACTION)
    RELATIVE ROTATIVE EFFICIENCY
    NUMBER OF PROPELLERS
    DEPTH (SHAFT CENTERLINE IFT)
    DIAPETER LIMIT (FT)
    = 6.2000
    = 6.1900
    = 1.0250
    = 1.0
    = 21.9827
    = 30.0000

PROPELLER PARAMETERS:
    NUMBER OF BLADES
    MATERIAL TYPE
    ALLOWABLE STRESS (PSI)
    = 6.0
    = STAINLESS STEEL
    = 5400.0

SELECTION VALUES:
    PE (HP)
    V (FT/SEC)
    VA (KNOTS)
    NS (RPM)
    QS (FT-LBF)
    PD (HP)
    J
    KT
    KQ
    ETAC
    KEV75K
    PDA (FT)
    PDA
    AEPAC
    T/C .75K
    = 5752440.0
    = 26.0641
    = 11.6500
    = 115.0000
    = 25344.90
    = 6.1000
    = 6.4001
    = 6.0539
    = 6.1182
    = 0.1E+10
    = 144.0330
    = 1.0000
    = 6.4000
    = 0.0196

CONSTRAINT VALUES:
    MAX LIA IFT)
    MIN T/C .75K
    = 36.0000
    = 0.179593

```



```

* * * * *
* * * * * C L P M I N * * * * *
* * * * * FORTRAN PROGRAM FOR * * * * *
* * * * * CONSTRAINED FUNCTION MINIMIZATION * * * * *
* * * * *
INITIAL FUNCTION INFORMATION
CBJ = -C.118204E+00
DECISION VARIABLES (X-VECTOR)
11 C.42000E+00 0.10000E+00 0.10000E+01
CONSTRAINT VALUES (C-VECTOR)
11 -C.62500E-01 -0.53750E+00 -0.52020E+03 -0.47875E+03
11 -0.55690E-01 -0.47750E-01 -0.55950E+00 0.12270E+03
0.10000E+00 -0.40000E+00

```



```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.665992E+00
DECISION VARIABLES (X-VECTOR)
11 C.8C177E+00 0.64752E+00 0.50364E+00
CONSTRAINT VALUES (G-VECTOR)
11 -C.441E+00 -0.5926E+00 -0.24820E+03 -0.97419E+03 -0.30178E+00 0.17773E-02
11 -C.15810E-01 -0.87622E-01 -0.24301E-02 -0.36772E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5
6
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(OBJ)-OBJ11-111 LESS THAN DAEFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 45
OBJECTIVE FUNCTION WAS EVALUATED 230 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 230 TIMES

```





# OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.66599E+00  
GLOBAL LOCATION

DESIGN VARIABLES			
ID	D. V. NO.	GLOBAL VAR. NO.	UPPER BOUND
1	1	C.1000E+00	0.80177E+00
2	2	C.1000E-01	0.64152E+00
3	3	C.1000E+00	0.90364E+00

DESIGN CONSTRAINTS			
ID	GLOBAL VAR. NO.	LOWER BOUND	UPPER BOUND
1	1	-0.1000E+16	0.0
2	2	-0.1000E+16	0.0
3	3	-0.1000E+16	0.0
4	4	-0.1000E+16	0.0
5	5	-0.1000E+16	0.0
6	6	-0.1000E+16	0.0
7	7	-0.1000E+16	0.0
8	8	-0.1000E+16	0.0
9	9	-0.1000E+16	0.0
10	10	-0.1000E+16	0.0



OPTIMIZATION RESULTS ----- DESIGN CASE NO. 2  
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:

ENVIRONMENTAL PARAMETERS: US, P, V, C, T, S, R  
TEMP IDIG, E, F, SEC2/FT4 = 64.4000  
DEAKTIDIG, E, F, SEC2/FT4 = 0.11899997E-04  
VISCOUSITY (F72/SEC) = 14.6970  
ATMOSPHERIC PRESSURE (PSIA) = 14.6970  
WATER VAPORIZATION PRESSURE (PSIA) = 14.6970

HULL PARAMETERS:

WAKE FRACTION = 0.2200  
THRUST REDUCTION FRACTION = 0.2200  
RELATIVE ROTATIVE EFFICIENCY = 1.0250  
NUMBER OF PROPELLERS = 1.0  
DEPTH OF SHAFT CENTERLINE (FT) = 21.9827  
DIAPETER LIMIT (FT) = 30.0000

PROPELLER PARAMETERS:

NUMBER OF BLADES = 6  
MATERIAL TYPE = STAINLESS STEEL  
ALLOWABLE STRESS (PSI) = 24000.0

SELECTION VALUES:

VE (FT/SEC) = 3087.3  
V (KNOTS) = 3087.3  
VA (KNOTS) = 3087.3  
N (RPM) = 115.0000  
QS (FT-LBF) = 2310133.0  
PO (HP) = 2310133.0  
KT = 0.6415  
KQ = 0.1719  
ETA = 0.0266  
REVS/R = 0.6660  
DIA (FT) = 0.54608  
P/D = 27.2533  
T/C = 0.0338  
T/C = 0.0338

CONSTRAINT VALUES:

MAX LIA (FT) = 30.0000  
MIN DIA (FT) = 0.5070  
MIN T/C = 0.034705



PROGRAM CALLS TO ANALYZ  
 ICALL 2  
 CALLS 2



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T I T L E  
 N A G E N I N G B - S E R I E S P R O P E L L E R O P T I M I Z A T I O N









TITLE:  
WAGENJAGEN B-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS  
CALCULATION CONTROL NCALC = 3  
NUMBER OF GLOBAL DESIGN VARIABLES, NODV = 3  
NUMBER OF SENSITIVITY VARIABLES, NSDV = 0  
NUMBER OF CONSTRAINTS, NCONS = 0  
NUMBER OF EQUATIONS, NREQS = 0  
INPUT INITIATION POINT CODE, NIPRPT = 0  
DEBUG PRINT CODE, IPRDBG = 0

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TOTAL ASPECT OF CORRELATING PARAMETERS = 12

\* • ESTIMATED DATA STORAGE REQUIREMENTS

INPUT	87	EXECUTION	1000	AVAILABLE	1000
INPUT	57	EXECUTION	122	AVAILABLE	1000



```

OPTIMIZATION RESULTS -----
DESIGN CASE NO. 2
SUBROUTINE "STRONA"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
US-N
TEMP (DEG F) SEC2/FT41 = 64.4000
DENSITY (LBS/FT3) SEC2/FT41 = 0.49897
VISCOSITY (CENTIPOISE) SEC2/FT41 = 0.169839
ATMOSPHERIC PRESSURE (PSIA) = 14.6970
WATER VAPORIZATION PRESSURE (PSIA) = 0.2994

FULL PARAMETERS:
WAKE FRACTION = 0.2000
RELATIVE EQUATION FRACTION = 0.1900
RELATIVE ROTATIVE EFFICIENCY = 1.0250
NUMBER OF PROPELLERS = 1.0
DEPTH TO SHAFT CENTERLINE (FT) = 21.9827
DIAPETER LIMIT (FT) = 30.0000

PROPELLER PARAMETERS:
NUMBER OF BLADES = 6.0
MATERIAL TYPE = STAINLESS STEEL
ALLOWABLE STRESS (PSI) = 24000.0

SELECTION VALUES:
PR (FT/SEC) = *****
V (FT/SEC) = 30.0000
VA (KNOTS) = 34.9042
N (RPM) = 1100.0000
US (FT-LBF) = 110133.0
PD (HP) = 25344.50
KT = 0.1000
KVAC = 0.002
REYN = 0.038
DIA (FT) = 0.4810
P/C = 21.7272
AEZAC = 1.0000
I/C = 0.0000

CONSTRAINT VALUES:
MAX DIA (FT) = 30.0000
MIN AEZAC = 101.8935
MIN I/C = 0.02794

```





```

* * * * *
* * * * * C L A M I N * * * * *
* * * * * FURIRAN PROGRAM FOR * * * * *
* * * * * CONSTRAINED FUNCTION MINIMIZATION * * * * *
* * * * *

```

INITIAL FUNCTION INFORMATION

```

OBJ = -C.11842E+0C
DECISION VARIABLES (X-VECTOR)
1) C.40000E+00 0.10000E+00 0.10000E+01 C.50000E+02 0.50000E-01
CONSTRAINT VALUES (G-VECTOR)
1) -C.6200E+01 -0.9750E+00 -0.1780E+04 0.7812E+03 6.10000E+00 -0.40000E+00
-C.3010E-01 -0.7735E-01 -0.5999E+00 0.6090E+01 0.2601E+03 0.2558E+00

```



```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.77049E+0C
LEJSION VARIABLES IX-VECTORI
11 C.75462E+00 0.99270E+00 0.11980E+01 0.52182E+02 C.45512E-01
CONSTRAINT VALUES IC-VECTORI
11 -C.62044E+00 -0.37954E+00 -0.33017E+05 -0.97599E+03 -C.89402E+00 -0.52495E-02
11 -C.30014E-01 -0.77435E-01 -0.36015E-05 -0.25451E+00 -0.41823E+00 -0.45777E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
6
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(OBJ1-OBJ11) LESS THAN DABFUN FOR 30 ITERATIONS
ABS(OBJ11-OBJ11-11) LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 80
OBJECTIVE FUNCTION WAS EVALUATED 544 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 544 TIMES

```



# OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.75724E+00  
GLOBAL LOCATION

## DESIGN VARIABLES

ID	C. V. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	23	0.4000E+00	0.4462E+00	0.1000E+01
2	2	17	0.4000E+00	0.9272E+00	0.1000E+01
3	3	19	0.4000E+00	0.1589E+01	0.4000E+01
4	4	19	0.4000E+00	0.2591E+01	0.1000E+01
5	5	19	0.4000E+00	0.2591E+01	0.1000E+01

## DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	0.1000E+00	0.4462E+00	0.0
2	2	0.1000E+00	0.9272E+00	0.0
3	3	0.1000E+00	0.1589E+01	0.0
4	4	0.1000E+00	0.2591E+01	0.0
5	5	0.1000E+00	0.2591E+01	0.0
6	6	0.1000E+00	0.2591E+01	0.0
7	7	0.1000E+00	0.2591E+01	0.0
8	8	0.1000E+00	0.2591E+01	0.0
9	9	0.1000E+00	0.2591E+01	0.0
10	10	0.1000E+00	0.2591E+01	0.0
11	11	0.1000E+00	0.2591E+01	0.0
12	12	0.1000E+00	0.2591E+01	0.0



```

OPTIMIZATION RESULTS -----
DESIGN CASE INV. 2
SUBROUTINE "STRONA"

US.A
TEXT (DEG-FT-LBF-SEC/FT^4)      = 64.4033
DENSITY (LBF/SEC^3)              = 0.1189999E-04
VISCOSITY (FT^2/SEC)            = 14.6970
ATMOSPHERIC PRESSURE (PSIA)     = 0.2594
WATER VAPORIZATION PRESSURE (PSIA) =

FULL PARAMETERS:
WAKE FRACTION                    = 0.4200
THRU-SECTION FRACTION           = 0.1900
RELATIVE ROTATIVE EFFICIENCY    = 1.0250
NUMBER OF PROPELLERS            = 1.0
DEPTH TO SHAFT CENTERLINE (FT) = 21.9827
DIAPYLET CHAM (FT)              = 30.0000

PROPELLER PARAMETERS:
NUMBER OF BLADES                 = 6.0
MATERIAL TYPE                    = STAINLESS STEEL
ALLOWABLE STRESS (PSI)          = 5400.0

SELECTION VALUES:
PE (HP/SEC)                      = 15915.5
V (KNOTS)                        = 30.3127
VA (KNOTS)                       = 24.1127
N (RPM)                           = 110.0000
QS (FT-LBF)                      = 110130.0
PD (HP)                           = 2344.90

J                                  = 0.927
KC                                  = 0.1552
ETAL                               = 0.0325
REV7-8                             = 0.7576
DIA (FT)                           = 9.5609
KCAL                                = 27.645
KCAL                                = 1.386
TTC .75F                           = 0.6699

CONSTRAINT VALUES:
MAX DIA (FT)                      = 30.0000
MIN REV7-8                         = 0.1622
MIN TTC .75F                       = 0.621082

```





PROGRAM CALLS TO ANALYZE  
LOCAL CALLS  
1 2  
2 3  
3 4  
4 5



```

CCCCC  GGGGG  PPPPP  EEEEE  SSSSS
C      G      P      E      S
C      G      P      E      S
C      G      P      E      S
CCCCC  GGGGG  PPPPP  EEEEE  SSSSS

```

C C N T R O L   P R O G R A M  
                   F O R  
 E N G I N E E R I N G   S Y N T H E S I S

T I T L E  
 WAGENINGEN B-SERIES PROPELLER OPTIMIZATION



# CARL IMAGES OF CONTROL DATA

CARL		IMAGE		PROPYLEN OPTIMIZATION										IPROG	
1A	TITLE	B-SERIES	NSV	ICNDIR	CT	ALPHA	ABOBI	NSCAL	CTMIN	ABOBI	NSCAL	CTMIN	ABOBI	IPROG	THETA
11	1A	1A	1A	1A	1A	1A	1A	1A	1A	1A	1A	1A	1A	1A	1A
12	1B	1B	1B	1B	1B	1B	1B	1B	1B	1B	1B	1B	1B	1B	1B
13	1C	1C	1C	1C	1C	1C	1C	1C	1C	1C	1C	1C	1C	1C	1C
14	1D	1D	1D	1D	1D	1D	1D	1D	1D	1D	1D	1D	1D	1D	1D
15	1E	1E	1E	1E	1E	1E	1E	1E	1E	1E	1E	1E	1E	1E	1E
16	1F	1F	1F	1F	1F	1F	1F	1F	1F	1F	1F	1F	1F	1F	1F
17	1G	1G	1G	1G	1G	1G	1G	1G	1G	1G	1G	1G	1G	1G	1G
18	1H	1H	1H	1H	1H	1H	1H	1H	1H	1H	1H	1H	1H	1H	1H
19	1I	1I	1I	1I	1I	1I	1I	1I	1I	1I	1I	1I	1I	1I	1I
20	1J	1J	1J	1J	1J	1J	1J	1J	1J	1J	1J	1J	1J	1J	1J
21	1K	1K	1K	1K	1K	1K	1K	1K	1K	1K	1K	1K	1K	1K	1K
22	1L	1L	1L	1L	1L	1L	1L	1L	1L	1L	1L	1L	1L	1L	1L
23	1M	1M	1M	1M	1M	1M	1M	1M	1M	1M	1M	1M	1M	1M	1M
24	1N	1N	1N	1N	1N	1N	1N	1N	1N	1N	1N	1N	1N	1N	1N
25	1O	1O	1O	1O	1O	1O	1O	1O	1O	1O	1O	1O	1O	1O	1O
26	1P	1P	1P	1P	1P	1P	1P	1P	1P	1P	1P	1P	1P	1P	1P
27	1Q	1Q	1Q	1Q	1Q	1Q	1Q	1Q	1Q	1Q	1Q	1Q	1Q	1Q	1Q
28	1R	1R	1R	1R	1R	1R	1R	1R	1R	1R	1R	1R	1R	1R	1R
29	1S	1S	1S	1S	1S	1S	1S	1S	1S	1S	1S	1S	1S	1S	1S
30	1T	1T	1T	1T	1T	1T	1T	1T	1T	1T	1T	1T	1T	1T	1T
31	1U	1U	1U	1U	1U	1U	1U	1U	1U	1U	1U	1U	1U	1U	1U
32	1V	1V	1V	1V	1V	1V	1V	1V	1V	1V	1V	1V	1V	1V	1V
33	1W	1W	1W	1W	1W	1W	1W	1W	1W	1W	1W	1W	1W	1W	1W
34	1X	1X	1X	1X	1X	1X	1X	1X	1X	1X	1X	1X	1X	1X	1X
35	1Y	1Y	1Y	1Y	1Y	1Y	1Y	1Y	1Y	1Y	1Y	1Y	1Y	1Y	1Y
36	1Z	1Z	1Z	1Z	1Z	1Z	1Z	1Z	1Z	1Z	1Z	1Z	1Z	1Z	1Z
37	2A	2A	2A	2A	2A	2A	2A	2A	2A	2A	2A	2A	2A	2A	2A
38	2B	2B	2B	2B	2B	2B	2B	2B	2B	2B	2B	2B	2B	2B	2B
39	2C	2C	2C	2C	2C	2C	2C	2C	2C	2C	2C	2C	2C	2C	2C
40	2D	2D	2D	2D	2D	2D	2D	2D	2D	2D	2D	2D	2D	2D	2D
41	2E	2E	2E	2E	2E	2E	2E	2E	2E	2E	2E	2E	2E	2E	2E
42	2F	2F	2F	2F	2F	2F	2F	2F	2F	2F	2F	2F	2F	2F	2F
43	2G	2G	2G	2G	2G	2G	2G	2G	2G	2G	2G	2G	2G	2G	2G
44	2H	2H	2H	2H	2H	2H	2H	2H	2H	2H	2H	2H	2H	2H	2H
45	2I	2I	2I	2I	2I	2I	2I	2I	2I	2I	2I	2I	2I	2I	2I
46	2J	2J	2J	2J	2J	2J	2J	2J	2J	2J	2J	2J	2J	2J	2J
47	2K	2K	2K	2K	2K	2K	2K	2K	2K	2K	2K	2K	2K	2K	2K
48	2L	2L	2L	2L	2L	2L	2L	2L	2L	2L	2L	2L	2L	2L	2L
49	2M	2M	2M	2M	2M	2M	2M	2M	2M	2M	2M	2M	2M	2M	2M
50	2N	2N	2N	2N	2N	2N	2N	2N	2N	2N	2N	2N	2N	2N	2N
51	2O	2O	2O	2O	2O	2O	2O	2O	2O	2O	2O	2O	2O	2O	2O
52	2P	2P	2P	2P	2P	2P	2P	2P	2P	2P	2P	2P	2P	2P	2P
53	2Q	2Q	2Q	2Q	2Q	2Q	2Q	2Q	2Q	2Q	2Q	2Q	2Q	2Q	2Q
54	2R	2R	2R	2R	2R	2R	2R	2R	2R	2R	2R	2R	2R	2R	2R
55	2S	2S	2S	2S	2S	2S	2S	2S	2S	2S	2S	2S	2S	2S	2S
56	2T	2T	2T	2T	2T	2T	2T	2T	2T	2T	2T	2T	2T	2T	2T
57	2U	2U	2U	2U	2U	2U	2U	2U	2U	2U	2U	2U	2U	2U	2U
58	2V	2V	2V	2V	2V	2V	2V	2V	2V	2V	2V	2V	2V	2V	2V
59	2W	2W	2W	2W	2W	2W	2W	2W	2W	2W	2W	2W	2W	2W	2W
60	2X	2X	2X	2X	2X	2X	2X	2X	2X	2X	2X	2X	2X	2X	2X
61	2Y	2Y	2Y	2Y	2Y	2Y	2Y	2Y	2Y	2Y	2Y	2Y	2Y	2Y	2Y
62	2Z	2Z	2Z	2Z	2Z	2Z	2Z	2Z	2Z	2Z	2Z	2Z	2Z	2Z	2Z
63	3A	3A	3A	3A	3A	3A	3A	3A	3A	3A	3A	3A	3A	3A	3A
64	3B	3B	3B	3B	3B	3B	3B	3B	3B	3B	3B	3B	3B	3B	3B
65	3C	3C	3C	3C	3C	3C	3C	3C	3C	3C	3C	3C	3C	3C	3C
66	3D	3D	3D	3D	3D	3D	3D	3D	3D	3D	3D	3D	3D	3D	3D
67	3E	3E	3E	3E	3E	3E	3E	3E	3E	3E	3E	3E	3E	3E	3E
68	3F	3F	3F	3F	3F	3F	3F	3F	3F	3F	3F	3F	3F	3F	3F
69	3G	3G	3G	3G	3G	3G	3G	3G	3G	3G	3G	3G	3G	3G	3G
70	3H	3H	3H	3H	3H	3H	3H	3H	3H	3H	3H	3H	3H	3H	3H
71	3I	3I	3I	3I	3I	3I	3I	3I	3I	3I	3I	3I	3I	3I	3I
72	3J	3J	3J	3J	3J	3J	3J	3J	3J	3J	3J	3J	3J	3J	3J
73	3K	3K	3K	3K	3K	3K	3K	3K	3K	3K	3K	3K	3K	3K	3K
74	3L	3L	3L	3L	3L	3L	3L	3L	3L	3L	3L	3L	3L	3L	3L
75	3M	3M	3M	3M	3M	3M	3M	3M	3M	3M	3M	3M	3M	3M	3M
76	3N	3N	3N	3N	3N	3N	3N	3N	3N	3N	3N	3N	3N	3N	3N
77	3O	3O	3O	3O	3O	3O	3O	3O	3O	3O	3O	3O	3O	3O	3O
78	3P	3P	3P	3P	3P	3P	3P	3P	3P	3P	3P	3P	3P	3P	3P
79	3Q	3Q	3Q	3Q	3Q	3Q	3Q	3Q	3Q	3Q	3Q	3Q	3Q	3Q	3Q
80	3R	3R	3R	3R	3R	3R	3R	3R	3R	3R	3R	3R	3R	3R	3R
81	3S	3S	3S	3S	3S	3S	3S	3S	3S	3S	3S	3S	3S	3S	3S
82	3T	3T	3T	3T	3T	3T	3T	3T	3T	3T	3T	3T	3T	3T	3T
83	3U	3U	3U	3U	3U	3U	3U	3U	3U	3U	3U	3U	3U	3U	3U
84	3V	3V	3V	3V	3V	3V	3V	3V	3V	3V	3V	3V	3V	3V	3V
85	3W	3W	3W	3W	3W	3W	3W	3W	3W	3W	3W	3W	3W	3W	3W
86	3X	3X	3X	3X	3X	3X	3X	3X	3X	3X	3X	3X	3X	3X	3X
87	3Y	3Y	3Y	3Y	3Y	3Y	3Y	3Y	3Y	3Y	3Y	3Y	3Y	3Y	3Y
88	3Z	3Z	3Z	3Z	3Z	3Z	3Z	3Z	3Z	3Z	3Z	3Z	3Z	3Z	3Z
89	4A	4A	4A	4A	4A	4A	4A	4A	4A	4A	4A	4A	4A	4A	4A
90	4B	4B	4B	4B	4B	4B	4B	4B	4B	4B	4B	4B	4B	4B	4B
91	4C	4C	4C	4C	4C	4C	4C	4C	4C	4C	4C	4C	4C	4C	4C
92	4D	4D	4D	4D	4D	4D	4D	4D	4D	4D	4D	4D	4D	4D	4D
93	4E	4E	4E	4E	4E	4E	4E	4E	4E	4E	4E	4E	4E	4E	4E
94	4F	4F	4F	4F	4F	4F	4F	4F	4F	4F	4F	4F	4F	4F	4F
95	4G	4G	4G	4G	4G	4G	4G	4G	4G	4G	4G	4G	4G	4G	4G
96	4H	4H	4H	4H	4H	4H	4H	4H	4H	4H	4H	4H	4H	4H	4H
97	4I	4I	4I	4I	4I	4I	4I	4I	4I	4I	4I	4I	4I	4I	4I
98	4J	4J	4J	4J	4J	4J	4J	4J	4J	4J	4J	4J	4J	4J	4J
99	4K	4K	4K	4K	4K	4K	4K	4K	4K	4K	4K	4K	4K	4K	4K
100	4L	4L	4L	4L	4L	4L	4L	4L	4L	4L	4L	4L	4L	4L	4L









```

OPTIMIZATION RESULTS -----
DESIGN CASE NO. 2
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
      Q5,A
      TEMP (LOG FL) = 64.8000
      DENSITY (LBF-SEC/FT3) = 0.189989E-04
      VISCOSITY (LBF-SEC/FT2) = 1.9892
      ATMOSPHERIC PRESSURE (PSIA) = 14.6970
      WATER VAPORIZATION PRESSURE (PSIA) = 2.2994

FULL PARAMETERS:
      MAKE FRACTION = 0.2200
      THRUST REDUCTION FRACTION = 0.1900
      RELATIVE ROTATIVE EFFICIENCY = 1.0250
      NUMBER OF PROPELLERS = 1.020
      DIAPHRAGM SPART CENTERLINE (FT) = 30.9827
      DIAPHRAGM LIP (FT) = 30.0000

PROPELLER PARAMETERS:
      NUMBER OF BLADES = 6.0
      MATERIAL TYPE = STAINLESS STEEL
      ALLOWABLE STRESS (PSI) = 5400.0

SELECTION VALUES:
      PE (LBF/SEC) = *****
      V (KNOTS) = 30.0000
      VA (KNOTS) = 30.0208
      N (RPM) = 23.1042
      PO (HP) = 10.0000
      J = 1.0133.0
      KT = 2544.90
      KO = 0.1000
      ETAC = 0.4002
      REY50K = 0.0538
      DIAL (FT) = 0.1184
      PDCAC = 0.4570
      AEZAC = 21.4570
      T/C = 0.0002
      T/C = 0.0000
      T/C = 0.0500

CONSTRAINT VALUES:
      MAX DIA (FT) = 30.0000
      MIN T/C = 0.1184
      MIN T/C = 0.1753

```



```

*****
*               *
*   C L A M I N   *
*               *
*   FURTHAN PROGRAM FOR   *
*               *
*   CONSTRAINED FUNCTION MINIMIZATION   *
*               *
*****

```

INITIAL FUNCTION INFORMATION

```

OBJ = -C.118442E+0C
DECISION VARIABLES IX=VECTOR
11 C.46038E+00 0.10000E+00 0.10000E+01 0.50000E+02 0.50000E+03
CONSTRAINT VALUES IG=VECTOR
11 -C.62500E-01 -0.93750E+00 -0.47803E+04 0.78120E+03 0.10000E+00
11 -C.36101E-01 -0.77352E-01 -0.59995E+00 0.60909E+01 0.26601E+03 0.25919E+01

```



```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.722977E+0C
DECISION VARIABLES (X-VECTOR)
11 C.75858E+00 0.81525E+00 0.10308E+01 0.49656E+02 0.63838E-01
CONSTRAINT VALUES (G-VECTOR)
11 -C.54704E+00 -0.45594E+00 -0.22531E+02 -0.97655E+03 -0.29459E+00 -0.14130E-03
71 -C.43940E-01 -0.63513E-01 -0.68188E-01 -0.19217E+00 -0.46186E+00 0.43178E-03
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
6
12
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(CBJ(1)-OBJ(1-1)) LESS THAN DABUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 35
OBJECTIVE FUNCTION WAS EVALUATED 209 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 209 TIMES

```



# OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.73297E+00  
GLOBAL LOCATION

## DESIGN VARIABLES

GLOBAL VAR. NO.	U.C. NO.	LOWER BOUND	VALUE	UPPER BOUND
10	1	0.4000E+00	0.73297E+00	0.1000E+01
11	2	0.4000E+00	0.73297E+00	0.1000E+01
12	3	0.4000E+00	0.73297E+00	0.1000E+01
13	4	0.4000E+00	0.73297E+00	0.1000E+01
14	5	0.4000E+00	0.73297E+00	0.1000E+01

## DESIGN CONSTRAINTS

GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
10	0.4000E+00	0.73297E+00	0.1000E+01
11	0.4000E+00	0.73297E+00	0.1000E+01
12	0.4000E+00	0.73297E+00	0.1000E+01
13	0.4000E+00	0.73297E+00	0.1000E+01
14	0.4000E+00	0.73297E+00	0.1000E+01
15	0.4000E+00	0.73297E+00	0.1000E+01
16	0.4000E+00	0.73297E+00	0.1000E+01
17	0.4000E+00	0.73297E+00	0.1000E+01
18	0.4000E+00	0.73297E+00	0.1000E+01
19	0.4000E+00	0.73297E+00	0.1000E+01
20	0.4000E+00	0.73297E+00	0.1000E+01
21	0.4000E+00	0.73297E+00	0.1000E+01
22	0.4000E+00	0.73297E+00	0.1000E+01





```

OPTIMIZATION RESULTS -----
DESIGN CASE NO. 2
SUBROUTINE "STRCKM"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
QS,N
TIME (SEC) 64.3302
VELOCITY (FT/SEC) 189997E-04
ATMOSPHERIC PRESSURE (PSIA) 0.146973
WATER VAPORIZATION PRESSURE (PSIA) 0.5994

HULL PARAMETERS:
WAVE FRACTION FRACTION 0.1200
RELATIVE ROTATIVE EFFICIENCY 1.0250
NUMBER OF PROPELLERS 1.0
DEPTH TO SHAFT CENTERLINE (FT) 21.9827
DIAPETER LIMIT (FT) 30.0000

PROPELLER PARAMETERS:
NUMBER OF BLADES 6.0
MATERIAL TYPE STAINLESS STEEL
ALLOWABLE STRESS (PSI) 5400.0

SELECTION VALUES:
PE (FT/SEC) 2663.7
V (FT/SEC) 45.8560
VA (KNOTS) 33.5372
N (RPM) 110.0000
PS (FT-LBF) 110130.0
PO (HP) 25344.50
J 0.8753
KO 0.0232
KYAC 0.7330
REY75R 0.5E+08
DIA (FT) 24.2346
P/C 1.0308
AGE/AD 0.7598
I/C .75F 0.0038

CONSTRAINT VALUES:
MAX DIA (FT) 30.0000
MIN REY75R 0.4266
MIN I/C .75F 0.0038668

```



PROGRAM CALLS TO ANALYZ  
ICALL CALLS  
1 210  
2 2  
3



## ANALIZ CODES--DESIGN CASE NO. 3

330



```

VK=24.24
V=1.68E*VK
N=1C5.C
QS=150C606.75
DIA=22.0
CC
CC
CC
      ENC CF INPUT-INITIALIZATION PHASE
      GO TO 3
      EXECUTION PHASE
      1 CONTINUE
      IF(.NOT.(ICALC.EQ.2))GO TO 2
      CALL RJCAL(RJ)
      CALL CF75RA(C75R)
      CALL REY75R(C75R,R75R)
      CALL CCEFSARJ,R75R,KT,KQ)
      CALL OFWEFF(RJ,KT,KQ,ETA0)
      CALL JCNA(RJ,RJCNL,RJCNU)
      CALL REYCNAR75R,R75RCL,R75RCL)
      CALL EXTCN(Z,AEDVAO,TC75R,AEACCL,AEACCU,TC75CL,TC75CL)
      CALL BLPOWE(KT,KQ,POWBAL,DIACNU)
      CALL CAVCNA(KT,AEACCV)
      CALL STRCNK(KC,KT,C75R,TCSTRS)
      CALL WGTAL(C75R)
      CC
      CC
      CC
      END OF EXECUTION PHASE
      GO TO 3
      2 CONTINUE
      OUTPUT-RESULT PHASE
      PC=(2.C*PII*QS*N)/33000.0
      TCEV=(KT*WATRC*(DIA**4)*((N/60.0)**2))
      PTCEV=(TDEV*(1.0-WT)*V)/550.C
      PECEV=((1.0-TD)/(1.0-WT))*ETARR*FTDEV)*NOSCRW)
      QREQ=(KQ*WATRC*(DIA**5)*((N/60.0)**2))
      PCREQ=(2.0*PII*QREQ*N)/33000.0
      ETACSP=((1.0-WT)/(1.0-TD))*(((PE/NOSCRW)*33000.0)/ETARR))
      1
      WRITE(6,9000)
      WRITE(6,9001)
      WRITE(6,9002) TEMP,WATRO,WATNU,FATM,PWATVA
      WRITE(6,9003) WT,TD,ETARR,NOSCRW,HCL,DIALIM
      WRITE(6,9004) Z
      IF(.NOT.(PROMAT.EQ.1.0))GO TO 81
      APP0C490
      APP0C500
      APP0C510
      APP0C520
      APP0C530
      APP0C540
      APP0C550
      APP0C560
      APP0C570
      APP0C580
      APP0C590
      APP0C600
      APP0C610
      APP0C620
      APP0C630
      APP0C640
      APP0C650
      APP0C660
      APP0C670
      APP0C680
      APP0C690
      APP0C700
      APP0C710
      APP0C720
      APP0C730
      APP0C740
      APP0C750
      APP0C760
      APP0C770
      APP0C780
      APP0C790
      APP0C800
      APP0C810
      APP0C820
      APP0C830
      APP0C840
      APP0C850
      APP0C860
      APP0C870
      APP0C880
      APP0C890
      APP0C900
      APP0C910
      APP0C920
      APP0C930
      APP0C940
      APP0C950
      APP0C960

```

















\$V  
END





## APPENDIX K

COPEs OUTPUT--DESIGN CASE NO. 3

C	C	C	C	C	C	C	C
C	C	C	C	C	C	C	C
C	C	C	C	C	C	C	C
P	P	P	P	P	P	P	P
E	E	E	E	E	E	E	E
S	S	S	S	S	S	S	S
S	S	S	S	S	S	S	S

# CONTROL PROGRAM FOR ENCIPHERING SYNTHESIS

WAGeningen B-SERIES PROPELLER OPTIMIZATION  
TITLE



## CARD PAGES OF CONTINUAL DATA

## CASE

# IMAGE

1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100	101	102	103	104	105	106	107	108	109	110	111	112	113	114	115	116	117	118	119	120	121	122	123	124	125	126	127	128	129	130	131	132	133	134	135	136	137	138	139	140	141	142	143	144	145	146	147	148	149	150	151	152	153	154	155	156	157	158	159	160	161	162	163	164	165	166	167	168	169	170	171	172	173	174	175	176	177	178	179	180	181	182	183	184	185	186	187	188	189	190	191	192	193	194	195	196	197	198	199	200	201	202	203	204	205	206	207	208	209	210	211	212	213	214	215	216	217	218	219	220	221	222	223	224	225	226	227	228	229	230	231	232	233	234	235	236	237	238	239	240	241	242	243	244	245	246	247	248	249	250	251	252	253	254	255	256	257	258	259	260	261	262	263	264	265	266	267	268	269	270	271	272	273	274	275	276	277	278	279	280	281	282	283	284	285	286	287	288	289	290	291	292	293	294	295	296	297	298	299	300	301	302	303	304	305	306	307	308	309	310	311	312	313	314	315	316	317	318	319	320	321	322	323	324	325	326	327	328	329	330	331	332	333	334	335	336	337	338	339	340	341	342	343	344	345	346	347	348	349	350	351	352	353	354	355	356	357	358	359	360	361	362	363	364	365	366	367	368	369	370	371	372	373	374	375	376	377	378	379	380	381	382	383	384	385	386	387	388	389	390	391	392	393	394	395	396	397	398	399	400	401	402	403	404	405	406	407	408	409	410	411	412	413	414	415	416	417	418	419	420	421	422	423	424	425	426	427	428	429	430	431	432	433	434	435	436	437	438	439	440	441	442	443	444	445	446	447	448	449	450	451	452	453	454	455	456	457	458	459	460	461	462	463	464	465	466	467	468	469	470	471	472	473	474	475	476	477	478	479	480	481	482	483	484	485	486	487	488	489	490	491	492	493	494	495	496	497	498	499	500	501	502	503	504	505	506	507	508	509	510	511	512	513	514	515	516	517	518	519	520	521	522	523	524	525	526	527	528	529	530	531	532	533	534	535	536	537	538	539	540	541	542	543	544	545	546	547	548	549	550	551	552	553	554	555	556	557	558	559	560	561	562	563	564	565	566	567	568	569	570	571	572	573	574	575	576	577	578	579	580	581	582	583	584	585	586	587	588	589	590	591	592	593	594	595	596	597	598	599	600	601	602	603	604	605	606	607	608	609	610	611	612	613	614	615	616	617	618	619	620	621	622	623	624	625	626	627	628	629	630	631	632	633	634	635	636	637	638	639	640	641	642	643	644	645	646	647	648	649	650	651	652	653	654	655	656	657	658	659	660	661	662	663	664	665	666	667	668	669	670	671	672	673	674	675	676	677	678	679	680	681	682	683	684	685	686	687	688	689	690	691	692	693	694	695	696	697	698	699	700	701	702	703	704	705	706	707	708	709	710	711	712	713	714	715	716	717	718	719	720	721	722	723	724	725	726	727	728	729	730	731	732	733	734	735	736	737	738	739	740	741	742	743	744	745	746	747	748	749	750	751	752	753	754	755	756	757	758	759	760	761	762	763	764	765	766	767	768	769	770	771	772	773	774	775	776	777	778	779	780	781	782	783	784	785	786	787	788	789	790	791	792	793	794	795	796	797	798	799	800	801	802	803	804	805	806	807	808	809	810	811	812	813	814	815	816	817	818	819	820	821	822	823	824	825	826	827	828	829	830	831	832	833	834	835	836	837	838	839	840	841	842	843	844	845	846	847	848	849	850	851	852	853	854	855	856	857	858	859	860	861	862	863	864	865	866	867	868	869	870	871	872	873	874	875	876	877	878	879	880	881	882	883	884	885	886	887	888	889	890	891	892	893	894	895	896	897	898	899	900	901	902	903	904	905	906	907	908	909	910	911	912	913	914	915	916	917	918	919	920	921	922	923	924	925	926	927	928	929	930	931	932	933	934	935	936	937	938	939	940	941	942	943	944	945	946	947	948	949	950	951	952	953	954	955	956	957	958	959	960	961	962	963	964	965	966	967	968	969	970	971	972	973	974	975	976	977	978	979	980	981	982	983	984	985	986	987	988	989	990	991	992	993	994	995	996	997	998	999	1000
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100	101	102	103	104	105	106	107	108	109	110	111	112	113	114	115	116	117	118	119	120	121	122	123	124	125	126	127	128	129	130	131	132	133	134	135	136	137	138	139	140	141	142	143	144	145	146	147	148	149	150	151	152	153	154	155	156	157	158	159	160	161	162	163	164	165	166	167	168	169	170	171	172	173	174	175	176	177	178	179	180	181	182	183	184	185	186	187	188	189	190	191	192	193	194	195	196	197	198	199	200	201	202	203	204	205	206	207	208	209	210	211	212	213	214	215	216	217	218	219	220	221	222	223	224	225	226	227	228	229	230	231	232	233	234	235	236	237	238	239	240	241	242	243	244	245	246	247	248	249	250	251	252	253	254	255	256	257	258	259	260	261	262	263	264	265	266	267	268	269	270	271	272	273	274	275	276	277	278	279	280	281	282	283	284	285	286	287	288	289	290	291	292	293	294	295	296	297	298	299	300	301	302	303	304	305	306	307	308	309	310	311	312	313	314	315	316	317	318	319	320	321	322	323	324	325	326	327	328	329	330	331	332	333	334	335	336	337	338	339	340	341	342	343	344	345	346	347	348	349	350	351	352	353	354	355	356	357	358	359	360	361	362	363	364	365	366	367	368	369	370	371	372	373	374	375	376	377	378	379	380	381	382	383	384	385	386	387	388	389	390	391	392	393	394	395	396	397	398	399	400	401	402	403	404	405	406	407	408	409	410	411	412	413	414	415	416	417	418	419	420	421	422	423	424	425	426	427	428	429	430	431	432	433	434	435	436	437	438	439	440	441	442	443	444	445	446	447	448	449	450	451	452	453	454	455	456	457	458	459	460	461	462	463	464	465	466	467	468	469	470	471	472	473	474	475	476	477	478	479	480	481	482	483	484	485	486	487	488	489	490	491	492	493	494	495	496	497	498	499	500	501	502	503	504	505	506	507	508	509	510	511	512	513	514	515	516	517	518	519	520	521	522	523	524	525	526	527	528	529	530	531	532	533	534	535	536	537	538	539	540	541	542	543	544	545	546	547	548	549	550	551	552	553	554	555	556	557	558	559	560	561	562	563	564	565	566	567	568	569	570	571	572	573	574																																																																																																																																																																																																																																																																																																																																																																																																																																										



TITLE: WAGENINCE 0-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:  
 CALCULATION CONTROL NALC = 2  
 NUMBER OF GLOBAL DESIGN VARIABLES, NDV = 3  
 NUMBER OF SENSITIVITY VARIABLES, NSV = 0  
 NUMBER OF FUNCTIONS IN TWO-SPACE, NZVAR = 0  
 NUMBER OF LOCAL OPTIMIZING VARIABLES, NLVAR = 0  
 NUMBER OF CONSTRAINTS, NCONS = 0  
 DEBUG PRINT CODE, IPDBG = 0

CALCULATION CONTROL, NALC  
 VALUE MEANING  
 1 ANALYSIS  
 2 OPTIMIZATION  
 3 SENSITIVITY  
 4 TSC-VARIABLE FUNCTION SPACE  
 5 CFTIMOM SENSITIVITY  
 6 APPROXIMATE OPTIMIZATION

# \* \* OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE  
 MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = -0.1000E+01  
 CONFIN PARAMETERS (IF ZERO, CONFIN DEFAULT WILL OVER-RIDE)

IPRINT ITHXA 0 ICHLCK NSCAL IIRM LINOBJ NALMAL NFOG  
 1 1000 0 0 30 0 15 0

FUCH 0.1000E-02 FUCHM 0.1000E-02 CT CTMIN  
 0.0 CILMIN 0.0 THETA PHI  
 0.0 DLFUN 0.0 OAFUN 0.0 ALPAX ABUJ1  
 0.0 0.0 0.0 0.0

DESIGN VARIABLE INFORMATION  
 NON-ZERO INITIAL VALUE WILL OVER-RIDE MODULE INPUT  
 O. V. LCMER UPPER INITIAL SCALE  
 NO. 1 C-2000E+00 0.1100E+01 0.1000E+01 0.1000E+01  
 2 C-3000E+00 0.1100E+01 0.1000E+01 0.1000E+01  
 3 C-5000E+00 0.5000E+00 0.1000E+01 0.1000E+00

DESIGN VARIABLES  
 O. V. GLOBAL MULTIPLYING  
 NO. AC. VAR. NO. FACTOR  
 1 1 2 0.1000E+01  
 2 3 0.1000E+01  
 3 9 0.1000E+01

# CONSTRAINT INFORMATION

THERE ARE 14 CONSTRAINT SETS  
 GLOBAL GLOBAL LINEAR

NO	VAR	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100	101	102	103	104	105	106	107	108	109	110	111	112	113	114	115	116	117	118	119	120	121	122	123	124	125	126	127	128	129	130	131	132	133	134	135	136	137	138	139	140	141	142	143	144	145	146	147	148	149	150	151	152	153	154	155	156	157	158	159	160	161	162	163	164	165	166	167	168	169	170	171	172	173	174	175	176	177	178	179	180	181	182	183	184	185	186	187	188	189	190	191	192	193	194	195	196	197	198	199	200	201	202	203	204	205	206	207	208	209	210	211	212	213	214	215	216	217	218	219	220	221	222	223	224	225	226	227	228	229	230	231	232	233	234	235	236	237	238	239	240	241	242	243	244	245	246	247	248	249	250	251	252	253	254	255	256	257	258	259	260	261	262	263	264	265	266	267	268	269	270	271	272	273	274	275	276	277	278	279	280	281	282	283	284	285	286	287	288	289	290	291	292	293	294	295	296	297	298	299	300	301	302	303	304	305	306	307	308	309	310	311	312	313	314	315	316	317	318	319	320	321	322	323	324	325	326	327	328	329	330	331	332	333	334	335	336	337	338	339	340	341	342	343	344	345	346	347	348	349	350	351	352	353	354	355	356	357	358	359	360	361	362	363	364	365	366	367	368	369	370	371	372	373	374	375	376	377	378	379	380	381	382	383	384	385	386	387	388	389	390	391	392	393	394	395	396	397	398	399	400	401	402	403	404	405	406	407	408	409	410	411	412	413	414	415	416	417	418	419	420	421	422	423	424	425	426	427	428	429	430	431	432	433	434	435	436	437	438	439	440	441	442	443	444	445	446	447	448	449	450	451	452	453	454	455	456	457	458	459	460	461	462	463	464	465	466	467	468	469	470	471	472	473	474	475	476	477	478	479	480	481	482	483	484	485	486	487	488	489	490	491	492	493	494	495	496	497	498	499	500	501	502	503	504	505	506	507	508	509	510	511	512	513	514	515	516	517	518	519	520	521	522	523	524	525	526	527	528	529	530	531	532	533	534	535	536	537	538	539	540	541	542	543	544	545	546	547	548	549	550	551	552	553	554	555	556	557	558	559	560	561	562	563	564	565	566	567	568	569	570	571	572	573	574	575	576	577	578	579	580	581	582	583	584	585	586	587	588	589	590	591	592	593	594	595	596	597	598	599	600	601	602	603	604	605	606	607	608	609	610	611	612	613	614	615	616	617	618	619	620	621	622	623	624	625	626	627	628	629	630	631	632	633	634	635	636	637	638	639	640	641	642	643	644	645	646	647	648	649	650	651	652	653	654	655	656	657	658	659	660	661	662	663	664	665	666	667	668	669	670	671	672	673	674	675	676	677	678	679	680	681	682	683	684	685	686	687	688	689	690	691	692	693	694	695	696	697	698	699	700	701	702	703	704	705	706	707	708	709	710	711	712	713	714	715	716	717	718	719	720	721	722	723	724	725	726	727	728	729	730	731	732	733	734	735	736	737	738	739	740	741	742	743	744	745	746	747	748	749	750	751	752	753	754	755	756	757	758	759	760	761	762	763	764	765	766	767	768	769	770	771	772	773	774	775	776	777	778	779	780	781	782	783	784	785	786	787	788	789	790	791	792	793	794	795	796	797	798	799	800	801	802	803	804	805	806	807	808	809	810	811	812	813	814	815	816	817	818	819	820	821	822	823	824	825	826	827	828	829	830	831	832	833	834	835	836	837	838	839	840	841	842	843	844	845	846	847	848	849	850	851	852	853	854	855	856	857	858	859	860	861	862	863	864	865	866	867	868	869	870	871	872	873	874	875	876	877	878	879	880	881	882	883	884	885	886	887	888	889	890	891	892	893	894	895	896	897	898	899	900	901	902	903	904	905	906	907	908	909	910	911	912	913	914	915	916	917	918	919	920	921	922	923	924	925	926	927	928	929	930	931	932	933	934	935	936	937	938	939	940	941	942	943	944	945	946	947	948	949	950	951	952	953	954	955	956	957	958	959	960	961	962	963	964	965	966	967	968	969	970	971	972	973	974	975	976	977	978	979	980	981	982	983	984	985	986	987	988	989	990	991	992	993	994	995	996	997	998	999	1000
1	11	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100	101	102	103	104	105	106	107	108	109	110	111	112	113	114	115	116	117	118	119	120	121	122	123	124	125	126	127	128	129	130	131	132	133	134	135	136	137	138	139	140	141	142	143	144	145	146	147	148	149	150	151	152	153	154	155	156	157	158	159	160	161	162	163	164	165	166	167	168	169	170	171	172	173	174	175	176	177	178	179	180	181	182	183	184	185	186	187	188	189	190	191	192	193	194	195	196	197	198	199	200	201	202	203	204	205	206	207	208	209	210	211	212	213	214	215	216	217	218	219	220	221	222	223	224	225	226	227	228	229	230	231	232	233	234	235	236	237	238	239	240	241	242	243	244	245	246	247	248	249	250	251	252	253	254	255	256	257	258	259	260	261	262	263	264	265	266	267	268	269	270	271	272	273	274																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																



\* \* ESTIMATED DATA STORAGE REQUIREMENTS

REAL	INTEGER
INPUT EXECUTION	INPUT EXECUTION
75	112
AVAILABLE	AVAILABLE
10000	1000





OPTIMIZATION RESULTS ----- DESIGN CASE NO. 3  
SUBROUTINE "STRUNG"

DESIGN VARIABLES SPECIFIED:

ENVIRONMENTAL PARAMETERS: PE,V,QS,N,DIA

TEMP (OLE, F) = 55.0000  
DESIGN LIFE (SEC) = 19.0000  
DESIGN LIFE (FT) = 18.7000  
ATMOSPHERIC PRESSURE (PSIA) = 0.147000E-04  
WATER VAPORIZATION PRESSURE (PSIA) = 14.7000  
WATER VAPORIZATION PRESSURE (PSIA) = 14.7000

FULL PARAMETERS:

WAKE ENACTICH = 0.2200  
THROUSTECTION FRACTION = 0.1750  
RELATIVE ROTATIVE EFFICIENCY = 1.0220  
NUMBER OF PROPELLERS = 1.0  
DEPTH TO SHAFT CENTERLINE (FT) = 15.0000  
DIAPETER LIMIT (FT) = 0.3

PROPELLER PARAMETERS:

NUMBER OF BLADES = 0.0  
MATERIAL TYPE = STAINLESS STEEL  
ALLOWABLE STRESS (PSI) = 5400.0

SELECTION VALUES:

PE (DPI/SEC) = 21492.0  
V (FPM) = 6.9111  
QS (FT-LBF) = 1120067.0  
PO (HP) = 30600.05

ETAL SPECIFIED

J = 0.4250  
KI = 0.2500  
KAC = 0.6597  
REV 5R = 0.6595  
OTA (FT) = 0.8E+08  
P/O = 21.0000  
AE/AG 75S = 1.0000  
T/C 75S = 1.0000  
BLADE HEIGHT (LBF) = 2553.1

CONSTRAINT VALUES:

MAX LIA (FT) = 0.0  
MIN AE/AG = 0.10258  
MIN T/C 75R

PROPELLER "FCUNTS":

PE DEVELOPED (HP) = 24856.6  
V (KNOTS) = 24.2400  
QS REQUIRED (FT-LBF) = 1275936.0  
N (RPM) = 1920000  
PU REQUIRED (HP) = 37203.82



```

      C L A M I N
      FORTAN PROGRAM FOR
      CUSTAFANEL FUNCTION MINIMIZATION

```

## INITIAL FUNCTION INFORMATION

$$CBJ = C.256313E+04$$

DECISION VARIABLES (X-VECTOR)

1	10+36001.3	10+32001.0	0.1000E-01
1	C.1000E+01	0.1200E+01	0.1000E-01

CONSTRAINT VALUES (C-VECTOR)

1	-0.5181E+00	-0.4818E+00	-0.3615E+02	-0.9622E+03	-0.59000E+00	0.20000E+00
2	-0.9598E-02	-0.2447E+00	-0.2613E+00	0.2512E+00	0.2856E-01	0.4483E+01



```

FINAL OPTIMIZATION INFORMATION
OBJ = 6.128426E+05
DECISION VARIABLES (X-VECTOR)
1) 6.75444E+00 0.11813E+01 0.79385E-01
CONSTRAINT VALUES (C-VECTOR)
1) -0.2801E+00 -0.7919E+00 -0.13450E+02 -0.24525E+02 -0.25444E+00 -0.25616E-06
11 -0.55491E-01 -0.17531E+00 0.5847E-02 -0.61655E-01 6.71621E-01 -0.12991E+00
THERE ARE 1 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
6
THERE ARE 2 VIOLATED CONSTRAINTS
CONSTRAINT NUMBERS ARE
9 11
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
176 CONSECUTIVE ITERATIONS FAILED TO PRODUCE A FEASIBLE DESIGN
NUMBER OF ITERATIONS = 10
OBJECTIVE FUNCTION WAS EVALUATED 37 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 37 TIMES

```



# OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 2 FUNCTION VALUE 0.12843E+05  
GLOBAL LOCATION

## DESIGN VARIABLES

IO	D. V. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	3	0.2000E+00	0.7944E+00	0.1000E+01
2	2	7	0.4000E+00	0.1181E+01	0.1400E+01
3	3	9	0.3000E-02	0.7938E-01	0.5000E+00

## DESIGN CONSTRAINTS

IO	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
10	1	-0.11000E+10	-0.51845E+00	0.0
11	2	-0.11000E+10	-0.14555E+03	0.0
12	3	-0.11000E+10	-0.35555E+02	0.0
13	4	-0.11000E+10	-0.25555E+02	0.0
14	5	-0.11000E+10	-0.55555E-01	0.0
15	6	-0.11000E+10	-0.55555E-01	0.0
16	7	-0.11000E+10	-0.55555E-01	0.0
17	8	-0.11000E+10	-0.55555E-01	0.0
18	9	-0.11000E+10	-0.55555E-01	0.0
19	10	-0.11000E+10	-0.55555E-01	0.0
20	11	-0.11000E+10	-0.55555E-01	0.0
21	12	-0.11000E+10	-0.55555E-01	0.0
22	13	-0.11000E+10	-0.55555E-01	0.0





DESIGN VARIABLES SPECIFIED:  
ENVIRONMENTAL PARAMETERS:

TEMP LOG FILE - SEC 27 F1  
OENSUS (FILE) 12/SEAL PSIAI  
WATER VAPORIZATION PRESURE  
PSIAI

5-0000  
.9903  
1.6100  
C.2470

4 3 2 1 0 4

WAKE FRACTION  
THRUST COEFFICIENT FRACTION  
RELATIVE ROTATIVE EFFICIENCY  
NUMBER OF PROPELLERS  
DEPTH TO SHAFT CENTERLINE (FT)  
DIAPETER LIMIT IF

WAKE FRACTION	0.2200
THRUST COEFFICIENT ON FRACTION	0.1725
RELATIVE ROTATIVE EFFICIENCY	1.025
NUMBER OF PROPELLERS	10
DEPTH OF SHAFT CENTERLINE (FT)	15.0000
OBSTACLE LIMIT (FT)	0.0

NUMBER OF BLADES	MATERIAL TYPE	ALLOWABLE STRESS (PSI)
6.0	STAINLESS STEEL	5400.0

NUMBER OF BLADES	6:0
MATERIAL TYPE	STAINLESS
ALLOWABLE STRESS (PSI)	5000.0

PE (HPI	21292.6
V (FT/SEC	46.9171
(RPM)	105.0000
US (FT-HPI	150060.0
US (HPI)	3500.05

PE (HPI	21292.6
V (FT/SEC	46.9171
(RPM)	105.0000
US (FT-HPI	150060.0
US (HPI)	3500.05

## ETAC SPICOFLEO = C.6527

ETAC SPICOFLEO = C.6527

106291

106291

KI  
KI  
● ●  
C.2349  
C-0248

KI  
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C.2349  
C-0248

0.6915

0.6915

22.000

22.000

AE/AC

AE/AC

172.136	BLADE WEIGHT 1 LB 11 OZ
12642.6	
5.0134	

172.136	BLADE WEIGHT 1 LB 11 OZ
12642.6	
5.0134	

## MAX CIA (FI) = 6.0

MAX CIA (FI) = 6.0

DEVELOPED 14P1 = 2116.1

DEVELOPED 14P1 = 2116.1



PROGRAM CALLS TO ANALYZE  
 ICALC CALLS  
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 2  
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CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC
C      C      C      C      C      C      C      C      C      C
C      C      C      C      C      C      C      C      C      C
C      C      C      C      C      C      C      C      C      C
C      C      C      C      C      C      C      C      C      C
CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC  CCCCC

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C E N T R A L   P R O G R A M  
 F O R  
 E N G I N E E R I N G   S Y N T H E S I S

T I T L E  
 WAGENINGEN B-SERIES PROPELLER OPTIMIZATION









TITLE:  
WAGENINGEN U-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:  
CALCULATION CONTROL 1  
NUMBER OF GLOBAL DESIGN VARIABLES, NCALC = 3  
NUMBER OF DESIGN VARIABLE GROUPS, NGV = 3  
NUMBER OF CONSTRAINT SETS, NCS = 0  
NUMBER OF APPROXIMATING VARIABLES, NAPPV = 0  
NUMBER OF APPROXIMATION PRINT CODES, NAPPD = 0  
INPUT PRINT CODE, IPDB = 0  
DEBUG PRINT CODE, IPDBG = 0

CALCULATION CONTROL, NCALC  
VALUE  
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\* \* OPTIMIZATION INFORMATION \* \*  
GLOBAL VARIABLE NUMBER OF OBJECTIVE  
MULTIPLIER NEGATIVE INDICATES MINIMIZATION = -0.1000E+01  
CONTROL PARAMETERS (IF ZERO, COMMON DEFAULT WILL OVER-RIDE)  
INPUT ILMX 1000 ICLIN 30 ILMN 0 ILMN 15  
FUCH 0.1000E-02 FUCH 0.1000E-02 C1 0.1000E-01 CMIN 0.0  
C1 0.0 C1MIN 0.0 IMETA 0.0 PHI 0.0  
DELFOA 3.0 DAFUN 0.0 ALPHAX 3.0 ABODJ 3.0  
DESIGN VARIABLE INFORMATION  
NON-ZERO INITIAL VALUE WILL OVER-RIDE MODULE INPUT  
D.V. NO. LOWER BOUND UPPER BOUND INITIAL VALUE SCALE  
1 0.2000E+00 0.1000E+01 0.1000E+01 0.1000E+01  
2 0.4000E+00 0.1000E+01 0.1000E+01 0.1000E+01  
3 0.3000E+00 0.1000E+01 0.1000E+01 0.1000E+01

DESIGN VARIABLES  
D.V. NO. GLOBAL MULTIPLYING  
1 1 1 1  
2 1 1 1  
3 1 1 1  
4 1 1 1  
5 1 1 1  
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99 1 1 1  
100 1 1 1

TOTAL NUMBER OF CONSTRAINED PARAMETERS = 12



• • ESTIMATED DATA STORAGE REQUIREMENTS			
INPUT	REAL	INTEGER	
75	EXECUTION	EXECUTION	
	504	112	
	AVAILABLE	AVAILABLE	
	10000	1000	



OPTIMIZATION RESULTS ----- DESIGN CASE NO. 3  
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:

ENVIRONMENTAL PARAMETERS: PE,V,CS,N,D,JA = 55.0000  
TEMP IDUC FT = 0.19905  
DESIGNITY (LBF-SEC2/FT4) = 0.128170004E-3  
VISCOSITY (FT2/SEC) = 12.0000  
ATMOSPHERIC PRESSURE (PSIA) = 14.7000  
WATER VAPORIZATION PRESSURE (PSIA) = 2.2470

NULL PARAMETERS:

WAKE FRACTION = 0.1700  
THRUST COEFFICIENT FRACTION = 0.1700  
RELATIVE ROTATIVE EFFICIENCY = 1.0250  
NUMBER OF STAGE ELEMENTS = 10  
DIAPHRAGM SPACING (FT) = 15.0000  
DIAPHRAGM LIMIT (FT) = 0.0

PROPELLER PARAMETERS:

NUMBER OF BLADES = 6  
MATERIAL TYPE = STAINLESS STEEL  
ALLOWABLE STRESS (PSI) = 240000

SELECTION VALUES:

PZ (HP) = 17630.0  
N (RPM) = 36.8240  
QS (FT-LBF) = 100.0000  
PO (HP) = 30000.00  
ETAC SPECIFIED = 0.5354

JT = 0.7800  
KT = 0.3212  
KQ = 0.0638  
ETA = 0.6302  
ETA50 = 0.8500  
Q/A (FT) = 2.0000  
P/C = 1.3000  
AE/AG = 1.0000  
T/C = 0.0100  
BLADE WEIGHT (LBF) = 2563.1

CONSTRAINT VALUES:

MAX CIA (FT) = 0.0  
MIN AE/AG = 1.0000  
MIN T/C = 0.06838

PROPELLER "POINTS":

PE DEVELOPED (HP) = 27514.5  
V (KNOTS) = 22.0000  
QS REQUIRED (FT-LBF) = 2605644.0  
N (RPM) = 100.0000  
PJ REQUIRED (HP) = 40076.84



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* * * * *
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* * * * * F O R T R A N   P R O G R A M   F O R
* * * * *
* * * * * C O N S T R A I N E C   F U N C T I O N   M I N I M I Z A T I O N
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INITIAL FUNCTION INFORMATION

```

OBJ = C.256313E+04
DECISION VARIABLES (X-VECTOR)
1) C.10000E+01 0.13000E+01 0.10000E-01
CONSTRAINT VALUES (C-VECTOR)
1) -C.45100E+00 -0.50840E+00 -0.36545E+02 -0.96245E+03 -C.50400E+00 0.
7) C.96987E-02 -0.24410E+00 -0.56066E+00 0.33589E+00 C.90555E-01 0.

```





# FINAL OPTIMIZATION INFORMATION

```

CBJ = C.1C6647E+05
DECISION VARIABLES (X-VECTOR)
1) C.77419E+00 0.10902E+01 0.66118E-01
CONSTRAINT VALUES (C-VECTOR)
1) -C.45160E+00 -C.5084C+00 -0.15063E+02 -0.98354E+03 -C.27419E+00 -0.
   -C.46220E-01 -C.18658E+00 -0.24834E-02 -0.22189E+00 -0.25204E-02 0.
THERE ARE 3 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
$ 11
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(CBJ11-OBJ11) LESS THAN DAEFUN FOR 30 ITERATIONS
ABS(CBJ11-OBJ11) LESS THAN DAEFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 46
OBJECTIVE FUNCTION WAS EVALUATED 102 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 102 TIMES

```



# OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 2 FUNCTION VALUE 0.10465E+05  
GLOBAL LOCATION

DESIGN VARIABLES		LOWER BOUND		VALUE		UPPER BOUND	
G. V. NO.	GLOBAL VAR. NO.	LOWER BOUND	UPPER BOUND	VALUE	LOWER BOUND	UPPER BOUND	
10	1	0.2000E+00	0.2000E+00	0.1741E+00	0.0	0.1000E+01	
11	2	0.4000E+00	0.4000E+00	0.1050E+01	0.0	0.1000E+01	
12	3	0.3000E+02	0.3000E+02	0.6811E-01	0.0	0.5000E+00	

## DESIGN CONSTRAINTS

DESIGN CONSTRAINTS		LOWER BOUND		VALUE		UPPER BOUND	
GLOBAL VAR. NO.	GLOBAL VAR. NO.	LOWER BOUND	UPPER BOUND	VALUE	LOWER BOUND	UPPER BOUND	
10	1	0.1000E+10	0.1000E+10	0.100E+00	0.0	0.0	
11	2	0.1000E+10	0.1000E+10	0.0849E+00	0.0	0.0	
12	3	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
13	4	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
14	5	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
15	6	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
16	7	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
17	8	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
18	9	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
19	10	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
20	11	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
21	12	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	
22	13	0.1000E+10	0.1000E+10	0.000E+00	0.0	0.0	



OPTIMIZATION RESULTS ----- DESIGN CASE NO. 3  
SUBROUTINE "STRCKM"

DESIGN VARIABLES SPECIFIED:

ENVIRONMENTAL PARAMETERS: PE,V,US,N,D,IA = 55.0000  
TIME DUE FI = 1.9905  
DENSITY (LBF-SEC/FT<sup>3</sup>) = 0.18170004E-0  
VISCOSITY (FT<sup>2</sup>/SEC) = 15.1000  
ATMOSPHERIC PRESSURE (PSIA) = 15.1000  
WATER VAPORIZATION PRESSURE (PSIA) = 0.2470

HULL PARAMETERS:

WAKE FRACTION = 0.4200  
THRUST REDUCTION FRACTION = 0.4710  
RELATIVE ROTATIVE EFFICIENCY = 1.0250  
NUMBER OF PROPELLERS = 1.0  
DIAPYCE (SPAN) CENTERLINE (FT) = 15.0000  
DIAPYCE (LIFT) (FT) = 0.0

PROPELLER PARAMETERS:

NUMBER OF BLADES = 6.0  
MATERIAL TYPE = STAINLESS STE  
ALLOWABLE STRESS (PSI) = 5400.0

SELECTION VALUES:

PE (HP) = 17030.0  
V (FT/SEC) = 56.8240  
N (RPM) = 105.0000  
QS (FT-LBF) = 150607.0  
PO (HP) = 35000.05

ETAC SPECIFIED

ETAC = 0.5394  
J = 0.7868  
KT = 0.3063  
KQ = 0.0372  
ETACAL = 0.8950  
REYNOLDS = 9356708  
DIA (FT) = 2.0000  
P/D = 1.0000  
A/EAC = 1.7742  
T/C = 0.158  
BLADE WEIGHT (LBF) = 10464.7

CONSTRAINT VALUES:

MAX DIA (FT) = 0.0  
MIN A/EAC = 0.268124  
MIN T/C = 0.158

PROPELLER "FCINTS":

PE DEVELOPED (HP) = 17030.0  
V (KNOTS) = 23.0000  
QS REQUIRED (FT-LBF) = 1167440.0  
N (RPM) = 105.0000  
PO REQUIRED (HP) = 23343.40



PROGRAM CALLS TO ANALYZE  
 ICALC CALLS  
 101  
 2  
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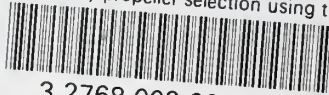
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